

Abstract
Cryogenic Optomechanics with a Silicon Nitride Membrane
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The field of optomechanics involves the study of the interaction between light and matter via the radiation pressure force. Though the radiation pressure force is quite weak compared with forces we normally experience in the macroscopic world, modern optical and microwave resonators are able to enhance the radiation pressure force so that it can be used to both measure and control the motion of macroscopic mechanical oscillators. Recently, optomechanical systems have reached a regime where the sensitivity to mechanical motion is limited only by quantum effects. Together with optical cooling techniques such as sideband cooling, this sensitivity has allowed experiments to probe the quantum behaviors of macroscopic objects, and also the quantum limits of measurement itself. In this dissertation I describe the physics underlying the modern field of optomechanics and provide an overview of experimental accomplishments of the field such as ground state cooling of mechanical oscillators, detection of radiation pressure shot noise, and preparation, storage, and transfer of quantum states between macroscopic objects and the electromagnetic field. I then describe the specific experimental work done in pursuit of my degree involving the ground state cooling of a silicon nitride membrane in a high finesse Fabry-Perot cavity, and a systematic characterization of the dynamics that occur when the membrane is coupled to two nearly degenerate cavity modes at an avoided crossing in the cavity spectrum. In the section on ground state cooling, particular attention is given to the influence of classical laser noise on the measurement of the membrane's motion at low phonon occupancies, and techniques for laser noise measurement and reduction are discussed.

Cryogenic Optomechanics with a Silicon Nitride Membrane

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by
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Table of Symbols

$A_1 \dots A_4, A_{\text{in}}, A_{\text{refl}}, A_{\text{trans}}$	Electric field amplitudes, used when solving for the solution of the 1D cavity with membrane-in-the middle
$A^{(n,m)}$	Amplitude of the mode n, m
\vec{A}	A vector consisting of $A_1 \dots A_4$
A_{rr}, A_{bb}	Shot noise-normalized anti-Lorentzian component of the power spectral density of the membrane's motion for the red and blue sidebands respectively (has units of area)
\hat{a}, \hat{a}_j	Photon annihilation operator
\bar{a}, \bar{a}_j	Average photon occupancy of the optical mode
$\vec{\hat{a}}$	Vector of the field operators of the modes at an avoided crossing
$\hat{a}_{\text{in}}, \hat{a}_{j,\text{in}}$	The cavity input field operator, or the input field of the delay line interferometer
\hat{a}_L, \hat{a}_R	Photon annihilation operators for the "left" and "right" modes at an avoided crossing
\bar{a}_{out}	Average cavity output field
a_{rr}, a_{bb}	Amplitude of the anti-Lorentzian component of the power spectral density of the membrane's motion for the red and blue sidebands respectively, in units of V^2/Hz (used as fitting parameters)
\hat{a}_1, \hat{a}_2	Output field operator of the non-delayed and delayed arms, respectively, of the delay line interferometer
α	Coefficient that relates the thermal temperature of the membrane to the temperature of the ^3He pot and the stage
$\vec{\alpha}$	Vector of the cavity enhanced optomechanical couplings for modes at an avoided crossing to oscillatory motion of the membrane
α_j	The cavity enhanced optomechanical coupling of mode j
α_L	Linear thermal expansion coefficient
α_x, α_y	Coefficients describing the tilt of the membrane in the cavity
\vec{B}	A term which describes the effects of classical laser noise on the measured heterodyne spectra
$B_{\pm}, B_{j,\pm}$	A placeholder term used in \vec{B} and elsewhere
b_{rr}, b_{bb}	The noise floor of the power spectral density of the membrane's motion for the red and blue sidebands respectively, in units of V^2/Hz (used as a fitting parameter)
β^2	Power percentage of promptly reflected light that recouples into the optical fiber inside the fridge
$C_{xx}, C_{xy}, C_{yy}, C_{j,xx}, C_{j,xy}, C_{j,yy}$	The magnitude of the classical laser noise relative to shot noise; xx=amplitude noise, yy=phase noise, xy=correlations between the two
$C_{xx0}, C_{xy0}, C_{yy0}$	The magnitude of the classical laser noise relative to shot noise per μW of laser power (has units of μW^{-1})
$c_{(n,m)}$	Coefficients describing the amount of each unperturbed eigenmode of the optical cavity that contributes to the new eigenmodes due to the perturbation caused by the membrane
\hat{c}	Phonon annihilation operator
c	The speed of light in vacuum
$\chi_c, \chi_{j,c}$	The susceptibility of the optical mode resonance

$\bar{\chi}_c$	Matrix of cavity susceptibilities for modes at an avoided crossing
χ_{FC}	The optical susceptibility of the filter cavity
χ_m	The susceptibility of the mechanical mode
D	Thermal diffusivity
\hat{d}, \hat{d}_j	Photon fluctuations annihilation operator
\hat{d}_{out}	Photon fluctuations operator of the cavity output field
$\Delta/2\pi, \Delta_j/2\pi$	Laser detuning from the optical mode
$\Delta_{noise}/2\pi$	The detuning of a region of laser noise relative to the filter cavity resonance
$\Delta\phi$	Phase difference, i.e. between the two paths in the delay line interferometer
δ	The difference between the phase accumulated during a round trip on one side of the cavity, vs. the other side of the cavity (with the membrane separating the two sides), or the Dirac delta function
$\delta\omega, \delta_j$	The optical spring shift (optionally, caused by optical mode j)
$\delta\gamma$	The optical damping
$\delta_{n,m}$	The Kronecker delta function
$\delta\phi$	Phase fluctuations, i.e. due to acoustic noise pickup in the delay line interferometer
$\delta x_j, \delta y_j$	Terms representing classical amplitude and phase noise, respectively
δ_T	The phase accumulated in one round trip of an optical cavity
e	The number e , except where indicated as the electron charge
$\hat{\eta}$	Thermal noise operator
ϵ	An error function used in least squares optimization
F_{rr}, F_{bb}	Shot noise-normalized background of the power spectral density of the mechanical motion around the red and blue sidebands
FSR	Free spectral range, i.e. $c/2L$
f	Frequency
f_c	Frequency of the red or blue mechanical sidebands relative to the center frequency of the recorded power spectral density (used as a fitting parameter)
f_h	Half linewidth of the membrane's mechanical Lorentzian in Hz (used as a fitting parameter)
G	Gain of the detection photodiode
G_r, G_b	Gain of the detection photodiode at the frequency of the red and blue sidebands respectively
$g/2\pi$	The linear optomechanical coupling per photon
$g_2/2\pi$	The quadratic optomechanical coupling per photon
$g_{0,L}/2\pi$ $g_{0,R}/2\pi$	In the quadratic optomechanics experiment, the <i>linear</i> optomechanical coupling per photon of each of the modes near an avoided crossing <i>to static displacements</i> of the membrane within the cavity
$g_{m,L}/2\pi$ $g_{m,R}/2\pi$	In the quadratic optomechanics experiment, the <i>linear</i> optomechanical coupling per photon of each of the modes near an avoided crossing <i>to oscillatory motion</i> of the membrane within the cavity
\bar{g}_m	In the quadratic optomechanics experiment, matrix of the <i>linear</i> optomechanical coupling per photon of each of the modes near an avoided crossing <i>to oscillatory motion</i> of the membrane mode
$\gamma/2\pi$	Coupling rate of the mechanical mode to the thermal bath
$\tilde{\gamma}/2\pi$	The total damping rate of the mechanical mode (the full linewidth of the mechanical mode)

$\gamma_j/2\pi$	The optical damping of the mechanical mode due to optical mode j
H	Hamiltonian
H_{int}	Interaction Hamiltonian
$H_{\text{int}}^{\text{lin}}$	Linearized interaction Hamiltonian
$H_{\text{diss}}, H_{\text{drive}}$	Hamiltonian components representing dissipation and the laser drive
H_{env}	Hamiltonian component representing the interaction with the environment
\hbar	Reduced Planck constant
\hat{f}	Photon flux operator, i.e. the number operator of the cavity output field
\bar{I}	Identity matrix
i	Imaginary unit, or photocurrent
j	Subscript placeholder to indicate either the measurement beam mode s or the cooling beam mode p
K^2, K_j^2	Optical power in units of photons per second
k	Wave vector
k_B	Boltzmann's constant
k_{min}	For an optical fiber, 2.405 divided by the radius of the fiber cladding; this is approximately the minimum wave vector at which the <i>cladding</i> supports at least one optical mode
k_{max}	For an optical fiber, 2 divided by the mode field radius; this is approximately the maximum wave vector at which the core supports single-mode propagation
$k_{(n,m)}^{(0)}$	The wave vector of the mode n, m for the unperturbed cavity, used when calculating the perturbation due to the membrane
$\kappa/2\pi, \kappa_j/2\pi$	Total decay rate of the optical mode (the full linewidth of the optical resonance)
$\bar{\kappa}$	Matrix containing the κ 's of modes at an avoided crossing
$\kappa_{\text{in}}/2\pi$ $\kappa_{j,\text{in}}/2\pi$	Decay rate of the optical mode through the cavity input mirror
$\sqrt{\kappa_{\text{in}}}$	Vector containing the square root of the κ_{in} 's of modes at an avoided crossing
$\kappa_{\text{int}}/2\pi$ $\kappa_{j,\text{int}}/2\pi$	Decay rate of the optical mode due to losses within the cavity
L	Length of the optical cavity, or length of one side of the membrane, or in the quadratic optomechanics experiment, a subscript indicating the "left" mode at an avoided crossing (generally the TEM _{0,0})
L_d	Thickness of the membrane
L_{rr}, L_{bb}	Shot noise-normalized area of the Lorentzian component of the power spectral density of the membrane's motion for the red and blue sidebands respectively
L_1	Length from cavity input mirror to the membrane
L_2	Length from membrane to cavity output mirror
l	Length of the fiber optic delay line in the delay line interferometer
Λ	A diagonal matrix consisting of the eigenvalues of another matrix
λ	Wavelength of light, generally 1064 nm unless otherwise specified
M	A generic symbol representing a matrix
m	Mass, or a generic symbol for eigenvalues of a matrix, or arbitrary integer such as a mode index
N	A placeholder in the cavity equations of motion
n	Photon number, or index of refraction, or an arbitrary integer such as a mode index
n_{int}	An arbitrary integer, for when m and n aren't enough

n_j	Effective <i>phonon</i> number of the optical mode j , in the sense of the phonon number the mechanical mode would have if in thermal equilibrium with the optical drive
n_m	Phonon number
$n_m^{(n,m)}$	Phonon number of mode n, m
$n_m^{(\gamma)}$	Phonon number calculated from the linewidth of the mechanical sidebands
$n_m^{(\xi')}$	Phonon number calculated from the sideband asymmetry
$n_m^{(rr)}, n_m^{(bb)}$	Phonon number calculated from the displacement-calibrated area under the red or blue sidebands respectively
n_{SiN}	Index of refraction of silicon nitride
n_{th}	Thermal phonon number of the mechanical mode
∇^2	Laplacian operator
ν	Optical frequency
P	Optical power, or a parameter in the theory of Gaussian laser beams related to the phase of the beam
P_{circ}	The circulating power in the optical cavity
P_j, P_{LO}	Optical power in the beam driving mode j , or in the local oscillator beam, respectively
p	Subscript indicating the cooling beam mode
ϕ	A general symbol for phase, sometimes used in more specific contexts
$\phi^{(n,m)}$	The phase of mode n, m
Φ	The Guoy phase
π	Pi
ψ	A component of the solution to the scalar wave equation
Q	Mechanical quality factor, or a matrix consisting of the eigenvalues of another matrix
q	Parameter in the theory of Gaussian laser beams related to the cross sectional profile of the beam
q_0	q evaluated at the beam waist
R	Radius of curvature of the phase front of a Gaussian beam, or power reflectivity of a cavity mirror, or responsivity of the measurement photodiode, or in the quadratic optomechanics experiment, a subscript indicating the “right” mode at an avoided crossing
$R1, R2$	In the quadratic optomechanics experiment, subscripts used to differentiate between additional modes at an avoided crossing
R_{mirror}	Radius of curvature of the cavity mirrors
r	Ratio of the local oscillator power to the measurement beam power
\tilde{r}_d	Complex field reflectivity of the membrane
r_d	Magnitude of \tilde{r}_d
r_1, r_2	Field reflectivities of the cavity input and output mirrors, respectively
ρ, ρ_p	Normalized amplitude of the reflected measurement beam and cooling beam respectively after interacting with the cavity
S	Power spectral density
$S^{(n)}$	The power spectral density of either sideband of the membrane’s motion computed from the n th recorded time trace
$S_{\text{avg}}^{(n)}$	The average power spectral density of the either sideband of the membrane’s motion computed after recording n time traces

S_{rr}, S_{bb}	Power spectral density of the membrane's motion for the red and blue sidebands respectively
$S_{rr,p}, S_{bb,p}$	Contribution of the cooling beam (shot noise) to S_{rr} and S_{bb}
$S_{\phi\phi}$	Power spectral density of phase noise due to thermal fluctuations in the delay line interferometer
s	Subscript indicating the measurement beam mode
S_{rr}, S_{bb}	Amplitude of the Lorentzian component of the red and blue mechanical sidebands respectively in the power spectral density of the membrane's motion, in units of V^2/Hz (used as a fitting parameter)
Σ	The optomechanical self energy
σ	The detection efficiency of the measurement photodiode
σ_{QE}	The quantum efficiency of the measurement photodiode
T	Absolute temperature
$T_{\text{tot}}, T_{\text{ff}}, T_{99}$ $T_{99 \rightarrow \text{cir}}, T_{\text{cir}}$	Respective power transmissivities of: all optical components after the circulator, the splice to the fridge fiber, the 99% output port of the 99:1 beamsplitter, the splice between the 99:1 beamsplitter and the circulator, and the circulator itself. See Table 1 on page 129.
$T_{\text{stage}}, T_{\text{pot}}$	Absolute temperature of the stage and the ^3He pot respectively
t	Time, or the thickness of the membrane (equivalent to L_d), or the coupling between two modes at an avoided crossing
\tilde{t}_d	Complex field transmissivity of the membrane
t_d	Magnitude of \tilde{t}_d
t_1, t_2	Field transmissivities of the cavity input and output mirrors, respectively
τ	A delay time, or a ringdown time
τ_{meas}	The length of time over which a measurement was taken
Θ	The Heaviside step function
θ	Phase of the local oscillator beam relative to the measurement beam
u	Solution to the scalar wave equation
$u_{(n,m)}^{(0)}$	The unperturbed eigenmodes of the Fabry-Perot cavity, used when calculating the effects of the perturbation due to the membrane
V	A perturbation to the free-space scalar wave equation
$V_{(n,m)}$	The expectation value of the perturbation V for the modes n, m , i.e.: $\langle u_{n,m}^{(0)} V u_{n,m}^{(0)} \rangle$
W_{rr}, W_{bb}	The red and blue sideband "weights", which are normalized versions of L_{rr} and L_{bb}
w	Radius of a Gaussian laser beam
w_0	Radius of Gaussian laser beam at the beam waist, or if in a fiber, the mode field radius
Ω_L, Ω_j	Drive laser angular frequency
ω	Angular frequency
ω_c, ω_j	Angular frequency of the optical cavity mode
$\bar{\bar{\omega}}_c$	Matrix describing the resonant angular frequencies of, and couplings between modes at an avoided crossing
ω'_c, ω''_c	First and second derivatives of the cavity's resonant frequency as a function of membrane position
ω_{if}	Frequency of the heterodyne beat note

ω_m	Angular frequency of the mechanical mode
$\tilde{\omega}_m$	Angular frequency of the mechanical mode as perturbed by the lasers
$\omega_m^{(n,m)}$	Angular frequency of the n, m mechanical mode
ω_r, ω_b	Angular frequency of the red and blue mechanical sidebands in the power spectral density of the membrane's motion
ω_0	Angular frequency at which there is an avoided crossing in the optical cavity spectrum
\hat{x}	Except where defined differently, the dimensionless position operator of the mechanical mode
x_{zpf}	Amplitude of the mechanical mode's zero-point fluctuations
\hat{X}_ϕ	Amplitude of one quadrature of the oscillator's position operator
ξ, ξ_j	Vacuum noise of the optical mode, or the sideband asymmetry (red/blue)
ξ'	Time derivative of the vacuum noise of the optical mode, or the gain and detuning-corrected sideband asymmetry
$\xi_{j,\text{in}}$	Vacuum noise of the laser driving mode j
ξ_1, ξ_2	Vacuum noise of the fields in the non-delayed and delayed arms, respectively, of the delay line interferometer
\hat{Y}_ϕ	Amplitude of another quadrature of the oscillator's position operator
\hat{z}	Except where defined differently, the dimensionless position operator of the mechanical mode (\hat{z} is used in place of \hat{x} for the chapter on Quadratic Optomechanics to be consistent with our prior publication on the subject)
z_{zpf}	Amplitude of the mechanical mode's zero-point fluctuations
z_c	An equation describing the orientation of the membrane within the cavity
z_0	The position of the membrane in the cavity
ζ_j	A placeholder in the cavity equations of motion

I. Introduction

1. An Historical Anecdote

In 1880, Alexander Graham Bell received a patent¹ on a device which he called the photophone. Little known today, the photophone was a device which enabled wireless voice communication by encoding sound onto a beam of light. The device consisted of a thin and flexible silvered plane mirror which would oscillate in response to sound from a person's voice. As it oscillated, it would alternately become slightly convex or slightly concave. Angled appropriately with respect to the sun, the changing concavity of the mirror would cause reflected sunlight to alternately diverge or converge. By placing a parabolic collector some distance away, light reflected from the mirror could be focused onto a glass receiver cell filled with lampblack or onto a photosensitive piece of selenium. The changing divergence of the incoming light would cause the intensity of the light landing on the receiver cell to fluctuate, creating a mechanical vibration (for lampblack), or a variation in electrical resistance (for selenium). For the lampblack receiver, variations in air pressure caused by photothermal heating of the sample (sometimes referred to as the photoacoustic effect) could be large enough to directly reproduce the sound that originally drove the flexible mirror. For the selenium version, an electrical current passing through the sample would experience fluctuations which could be used to drive a standard telephone earpiece to reproduce the sound. As novel as the photophone was at the time, its relatively short range and its susceptibility to interference from atmospheric conditions (e.g. rain, fog, or snow) along with exciting progress in radio technology caused the photophone to become little more than an historical footnote. Though the photophone never made it to the mainstream, it was nonetheless a revolutionary device in that it was one of the first devices ever to directly use a beam of light to detect the motion of a mechanical oscillator and reproduce that motion elsewhere.

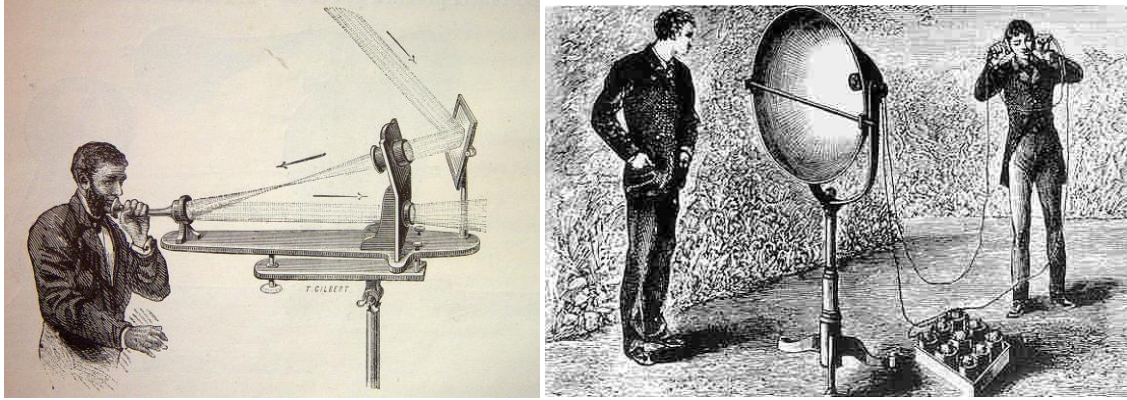


Figure 1: Photophone transmitter (left)², and receiver (right)³

While the scientific principles upon which Bell's photophone was based differ from the principles upon which modern optomechanics is based, many of themes are the same. Modern optomechanical experiments involve a mechanical oscillator which modulates the phase of a beam of light as it oscillates, just as the photophone modulated the convergence of a beam of light, and hence the intensity landing on a photoreceiver. Many experiments today also focus on coupling oscillators together using light as a coupling mechanism, just as the photophone coupled the vibrations of granular lampblack to the vibrations of the flexible mirror. Electro-opto-mechanical devices today attempt to transfer electrical (microwave) signals in a quantum coherent manner using mechanical oscillators as up/down-converters to and from optical frequencies, just as the photophone paired with a speaker at the transmitter could have conceivably upconverted audio frequency electrical signals to the optical regime and back again with a selenium receiver on the other end.

2. Modern Optomechanics

The modern field of optomechanics seeks to harness mechanical interactions with light to both control and measure the motion of mechanical oscillators with extremely high precision. Some goals of modern optomechanics include measurement of extremely small forces, displacements, and accelerations, and detection of small masses.⁴ The incredible sensitivity to these quantities obtained in modern optomechanical systems allows for exploration of the quantum mechanical behaviors of

mechanical oscillators using light to both control and measure oscillators' behavior. Modern optomechanical systems can use the radiation pressure force to cool mechanical oscillators to their vibrational ground states,⁵⁻¹¹ prepare mechanical oscillators in non-classical states,^{5,12,13} and coherently transfer states between mechanical oscillators and the light field.^{14,15} The long lifetimes of mechanical states make them well suited for use as quantum memory elements, and the ability to transfer states from mechanical oscillators to the light field and back brings about potential uses for future quantum information processors.¹⁶ The broadband nature of the coupling between mechanical oscillators and electromagnetic fields also allows for coherent quantum state transfer between electromagnetic modes of vastly different frequencies. On a more basic level, modern optomechanical systems provide the tools needed to test predictions of quantum mechanics in macroscopic objects, and even to explore the fundamental limits of measurement itself.

While Bell's photophone was a novel invention for its time, the goals of modern optomechanics require a more advanced system than a membrane mirror and some lampblack or a selenium photoresistor. Unlike the photophone, the interaction between light and mechanical oscillators in modern optomechanical systems is driven by radiation pressure. Radiation pressure is the force exerted on any surface which absorbs or reflects light due to conservation of the momentum carried by the electromagnetic fields that make up the light. For the purposes of building an optomechanical system, radiation pressure is superior to the photoacoustic and photoresistive effects used in the photophone for several reasons. First, the fact that radiation pressure can exert a force without being absorbed means that in an appropriate system the same light can interact with a mechanical oscillator multiple times, increasing the strength of the radiation pressure force. During each of these multiple interactions, the light acquires information about the state of the oscillator, allowing for detection of the oscillator's state. Additionally, since slow thermal excitations or high latency photoresistive responses are not

required for the mechanical oscillator to respond to the light, the bandwidth of the interaction is much higher than can be achieved via the photoacoustic effect.

3. Basic Optomechanical Effects

To understand the types of effects that can be expected to appear in a modern optomechanical system, it is useful to consider a simple model called the “canonical optomechanical system”. The canonical system consists of a high finesse Fabry-Perot cavity with a moveable end mirror, as shown in Figure 2. The cavity is optically driven by an electromagnetic field input from the left side. If the frequency of the driving field is near the resonant frequency of the Fabry-Perot cavity, light from the optical drive leaks inside the cavity and builds up a standing wave within it. This standing wave exerts a force on the moveable end mirror via radiation pressure, causing the mirror to move. As the mirror moves, it changes the length of the cavity and hence the cavity’s resonant frequency, which in turn modulates both the amount of light in the cavity and the phase of that light.



Figure 2: Schematic of the canonical optomechanical system.

The coupling between the electromagnetic field and the mechanical oscillator as described above produces several effects. First, the radiation pressure force acting on the moveable mirror modifies the mirror’s mechanical susceptibility, changing both its mechanical resonance frequency and its damping rate. These effects are known as the optical spring and optical damping effects, respectively. The magnitude of the optical spring and damping effects depends on the detuning of the optical drive from the cavity’s resonant frequency, as well as the optical drive power. Thus, the electromagnetic field

can be used to control the motion of the mechanical oscillator. A second consequence of the optomechanical coupling is that the changing cavity length caused by the motion of the mirror produces modulation in the optical field. This modulation carries away information about the mirror's motion. In this way, the electromagnetic field can also be used to measure the oscillator's motion.

An example of the optical spectrum of light exiting the cavity after interacting with the moveable mirror is shown in Figure 3 below. The cavity's resonance is shown in gray, and the on-resonance optical drive tone in green. Phase modulation sidebands acquired by the optical field's interaction with the moveable mirror are shown in red and blue.

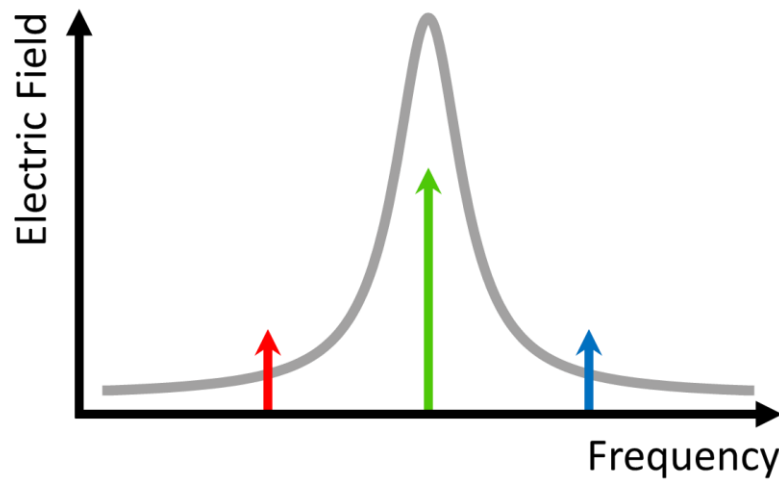


Figure 3: Spectrum of light exiting the optical cavity after the interaction of an on-resonance optical drive with the moveable mirror.

The upper and lower phase modulation sidebands in Figure 3 are respectively made up of light that has carried energy away from, or given energy up into the movable mirror. If the optical drive is precisely on resonance with the cavity as it is in Figure 3, both processes are equally likely, so the sidebands are symmetric and there is no net effect on the moveable mirror. The sidebands each contain a record of the membrane's motion, and therefore are useful for measurement of the membrane's position.

If the optical drive is detuned from resonance however, it is possible to enhance one of the sidebands while suppressing the other, as shown in Figure 4. If the optical drive is red detuned from the cavity's resonance such as in the left side of Figure 4, the upper phase modulation sideband is enhanced by the cavity susceptibility, while the lower sideband is suppressed. In this case, the optical field carries energy away from the moveable mirror, damping its motion and reducing the effective temperature of the mechanical mode. When blue detuned, as in the right side of Figure 4, the opposite process occurs, producing anti-damping of the mirror's motion, and increasing the effective temperature of the mode. Thus, the effective temperature of the mechanical mode can be controlled by the detuning of the optical drive.

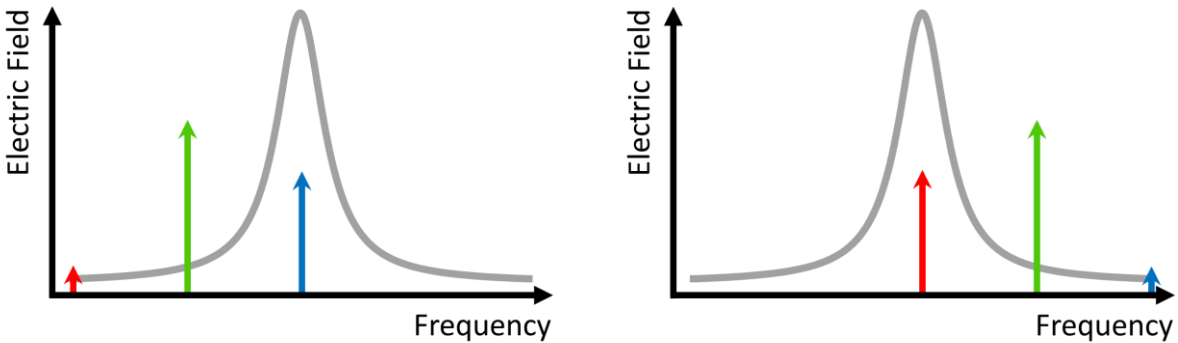


Figure 4: Spectrum of light exiting the optical cavity after the interaction of a red detuned (left) or blue detuned (right) optical drive with the moveable mirror.

Naively, it would seem that arbitrarily strong measurements of the mirror's motion can be made, and arbitrarily low effective temperatures can be reached, by simply turning up the optical drive power. Einstein's photon description of light, however, suggests that the optical drive has inherent fluctuations due to the uneven distribution of photons within the light beam.¹⁷ These fluctuations are called shot noise, and they set an ultimate limit to the sensitivity of an optical position measurement. The contribution to a measurement's noise background due to shot noise is referred to as the

measurement imprecision. Measurement imprecision scales inversely with the total power of the optical beam used to perform a measurement. However, as the power of the measurement beam is turned up, the amount of radiation pressure the beam exerts on the mirror increases as well. Fluctuations in the radiation pressure caused by the shot noise fluctuations in the light beam can then drive the mechanical oscillator, creating fluctuations in its position. These fluctuations are referred to generically as the measurement back-action, or more specifically in this case, as measurement back-action due to radiation pressure shot noise. Radiation pressure shot noise sets another limit to the sensitivity of an optical position measurement, and it increases proportionally with the total power of the optical beam used to perform the measurement. Thus, in any optomechanical position measurement there is a compromise between measurement imprecision and measurement back-action due to radiation pressure shot noise of the beam used to perform the measurement. There exists an optimal point at which the sum of the measurement imprecision and the back-action is minimized. This point is called the standard quantum limit, and represents the maximally sensitive position measurement possible in an optomechanical system without employing special techniques to evade the back-action of the measurement.^{4, 18, 19} In addition to setting limits on measurement sensitivity, radiation pressure shot noise also limits the amount of cooling that can be done using sideband cooling techniques.²⁰

Many modern optomechanical systems operate in a regime where the effects of radiation pressure shot noise are significant. For example, the phase modulation sidebands produced by the interaction between an on-resonance optical drive and a mechanical oscillator that has been cooled close to its quantum vibrational ground state are expected to show asymmetry which can be attributed in part to the fact that the radiation pressure shot noise from the optical field used for measurement is correlated with the shot noise of the light exiting the optical cavity.²¹ This sideband asymmetry is significant because it provides evidence of the quantized nature of the electromagnetic field in the cavity, and can be used as a probe of the effective temperature of the mechanical mode. Other

experiments have reached the regime where the measurement's sensitivity to the position of the mechanical oscillator is dominated by back-action from radiation pressure shot noise rather than the measurement imprecision,²² which similarly provides evidence of the quantization of the electromagnetic field within the cavity and confirms predictions¹⁹ that radiation pressure shot noise can affect the motion of mechanical oscillators.

While radiation pressure limits the precision to which it is possible to measure a mechanical oscillator's position, this is not necessarily true for measurements of other characteristics of an oscillator's motion. Measurements which measure such characteristics are called quantum non-demolition, or back-action evading measurements. These measurements involve measuring an observable that commutes with the full Hamiltonian of the system. With such an observable, the back-action caused by the measurement affects a quadrature of the motion that does not perturb the one being measured. Thus, unlike position, it is possible to measure such a quantum non-demolition observable to arbitrary precision. One example of this is a measurement of only one quadrature of the oscillator's position. For example, the position operator of a mechanical oscillator can be decomposed into quadratures as follows:

$$\hat{x}(t) = \hat{X}_\phi \cos(\omega_m t + \phi) + \hat{Y}_\phi \sin(\omega_m t + \phi) \quad (1)$$

In Equation (1), $\omega_m/2\pi$ is the mechanical frequency of the oscillator, t is time, ϕ is an arbitrary phase, and \hat{X}_ϕ and \hat{Y}_ϕ are the quadrature amplitudes. It can be shown that measurements of either only \hat{X}_ϕ or only \hat{Y}_ϕ can be made to arbitrary precision, and that back-action from measurement of either affects only the other quadrature.²³ Such a measurement can be implemented, for example, by measuring the oscillator's position stroboscopically at times when one quadrature is maximized and the other is minimized,²³ or through a reservoir engineering scheme in which two measurement tones are used to achieve the same result in the steady state.¹³ When a quantum non-demolition measurement is

performed in such a way as to measure one quadrature of motion to an uncertainty less than the fluctuations caused by the oscillator's zero-point motion, the mechanical oscillator is said to be in a squeezed state.²³

From this brief description of optomechanical interactions, it is clear that optomechanical systems are excellent tools for probing quantum behavior of mechanical oscillators, and also for characterizing properties of quantum measurements. However, optomechanical systems also have many practical uses beyond just testing the predictions of quantum mechanics itself. In the next section, I will discuss some of these applications.

4. Applications of Optomechanics

The excellent displacement sensitivity of optomechanical systems makes them useful for construction of precision accelerometers, where it is desirable to measure the motion of a test mass due to weak forces or accelerations. Perhaps the largest examples of this are the Laser Interferometric Gravitational Wave (LIGO) observatories, which use optomechanical systems consisting of Fabry-Perot cavities with kilogram scale moveable mirrors, very similar to the canonical system, to detect tiny displacements caused by passing gravitational waves.²⁴

In sensitive accelerometers such as LIGO, the standard quantum limit sets a bound on the weakest forces and accelerations that can be detected. One way to improve sensitivity beyond the standard quantum limit is by using squeezed light to perform the measurement. In the last section I described how a quantum non-demolition measurement can reduce the fluctuations in one quadrature of an oscillator's motion to below the level of the oscillator's zero-point fluctuations at the expense of larger fluctuations in the other quadrature. Analogously, it is possible to reduce the fluctuations in one quadrature of a light field to below the shot noise level at the expense of fluctuations in the other quadrature. By reducing fluctuations in the amplitude quadrature of a light field below the shot noise

level, back-action due to the radiation pressure shot noise of that light acting on a mechanical oscillator can be reduced. Optomechanical systems themselves are one way in which such squeezed light can be produced. An optomechanical system whose position fluctuations are driven predominantly by the radiation pressure shot noise of an on-resonance optical drive will produce phase modulation of the intracavity optical field that is correlated with the shot noise in the amplitude quadrature. These correlations can destructively interfere along the axis of some particular quadrature, reducing fluctuations in such a quadrature to below the shot noise level.²⁵

Beyond sensitive displacement measurements, optomechanical systems are useful because they can be used to store and transduce quantum information. I have already described how an optomechanical system can prepare a mechanical oscillator in its vibrational ground state. From the ground state, an oscillator can be excited into other states by the electromagnetic field. Under the right conditions, it is possible to transfer the quantum state of the electromagnetic field into the mechanical oscillator which can store it, acting like a quantum memory element.¹⁶ The state that is stored in the mechanical oscillator can then be transferred back into the electromagnetic field at a later time. The ability to store, manipulate, and retrieve information is fundamental to the operation of a computer; the ability of optomechanical systems to do these things with quantum states therefore makes them promising candidates for future uses in quantum computing.

Another aspect of optomechanical systems which makes them particularly attractive for quantum computing is the fact that mechanical oscillators can couple to electromagnetic radiation of many different frequencies. Thus, for example, a quantum state can be transmitted on a microwave signal within a quantum computer, stored in a mechanical oscillator via optomechanical interactions, and then retrieved by an optical light pulse interacting with the same mechanical oscillator.^{15, 16} The optical light can then be transmitted a long distance via a low-loss optical fiber and stored into another mechanical element at a remote location. Such transduction of quantum states between

electromagnetic modes of different frequencies, and also between mechanical memory elements which can be physically separated by long distances, holds promise for future quantum networking.

Optomechanical systems which are specifically designed to couple to more than one type of electromagnetic radiation are referred to as “hybrid” optomechanical systems.

It is clear that there are many practical applications of optomechanical systems, ranging from putting their precision displacement sensitivity to use, to building fundamental components of quantum computers. In the next section I will describe some of technical properties of practical optomechanical systems, before giving a review of some of the experiments in the field.

5. Practical Optomechanical Systems

In the last several years, experiments characterizing the fundamental effects of optomechanics, utilizing optomechanical systems to probe predictions of quantum mechanics, and demonstrating many of the practical applications just mentioned, have been developed. The systems used in these experiments operate using the same physical principles as the canonical system, but their designs often differ in order to accommodate various practical requirements.

One common practical requirement is cryogenic compatibility. According to the correspondence principle, a typical meso- or macro-scale mechanical oscillator at room temperature behaves classically. In a quantum mechanical sense, this means that the oscillator exists in a mixed thermal state where it is strongly entangled with and influenced by the thermal bath. Measurements of the instantaneous phonon occupancy of the mechanical oscillator in a thermal state should recover values consistent with the Maxwell-Boltzmann distribution. Due to the large spread of this distribution for room temperature oscillators relative to the quantum of mechanical motion $\hbar\omega_m$ and the frequent fluctuations in phonon occupancy due to oscillator’s interaction with the thermal bath, it is difficult to make a measurement strong enough to observe the quantization of the mechanical oscillator’s phonon number. Observation

of quantization in a mechanical oscillator therefore requires cooling the mechanical oscillator to the point where the width of the Boltzmann distribution is closer to the size of the quantum of mechanical motion. Since the standard deviation of the Maxwell-Boltzmann distribution for the energy of an oscillator in a thermal state scales $\propto k_B T$, this condition is approximately expressed as $k_B T \approx \hbar \omega_m$. Thus, the temperature to which a mechanical oscillator must be cooled to observe quantum effects scales with the frequency of the mechanical oscillator. For oscillators in the GHz regime, the necessary temperatures can be achieved using conventional cryogenic techniques. For example, a 1 GHz oscillator would satisfy $k_B T \approx \hbar \omega_m$ at a temperature of 50 mK, well within range of temperatures achievable with a modern helium dilution refrigerator. For lower frequency oscillators, however, this criterion is much more difficult to satisfy. A 1 MHz oscillator would have to be cooled to 50 μ K, well beyond the reach of conventional cryogenic refrigeration.

Optomechanical systems therefore typically must be designed with components that can tolerate extremely cold temperatures. Both the mechanical oscillator itself and the system around it such as the oscillator's mount and the optics used to couple the electromagnetic field into the system must be thermally stable and remain aligned over the range of temperatures from room temperature down to the cryogenic realm. Additionally, it is desirable to select a mechanical oscillator with a high intrinsic quality factor (as low a thermal damping rate as possible), so that the oscillator's motion will be dominated by its coupling to the optical drive, rather than the coupling to the thermal bath. Finally, the oscillator must be well suited for control and measurement with an electromagnetic field. For this, the mechanical oscillator must have low optical absorption at the frequencies chosen for the optical drive, geometric compatibility with the optical cavity, and a low enough mass for the radiation pressure force to have an appreciable effect.

While particular experiments may have other requirements, these are the general properties that are desirable in most optomechanical systems. Simultaneously realizing all of these characteristics

in a system like the canonical system can be difficult, and therefore optomechanical systems come in a diverse array of designs. Each system is tailored to the scientific goals of the experiment for which the system is built. In the next section I will describe some of these systems, as well as the scientific goals they have achieved.

6. Review of the Field

The field of optomechanics has truly blossomed within the last ten years, with optomechanical systems of many different shapes and sizes achieving significant results. Systems that achieve ground state cooling of macroscopic oscillators have been developed, as have systems that use mechanical oscillators to detect radiation pressure shot noise. Yet other systems have prepared oscillators into non-classical states of motion, and demonstrated coherent transfer of quantum states between mechanical oscillators and the light field. In this section I will give a review of the first modern optomechanical device, as well as some of these more recent papers, and discuss how each is relevant to the goals of the modern field of optomechanics so that the reader might appreciate the progress the field has made, and gain an understanding of how this dissertation sits in relation to the rest of the field.

The first modern optomechanical device was Braginskii, Manukin, and Tikhonov's 1969 microwave cavity with a moveable wall.²⁶ A schematic illustration of the system, taken from the original publication²⁶ is shown in Figure 5 below.

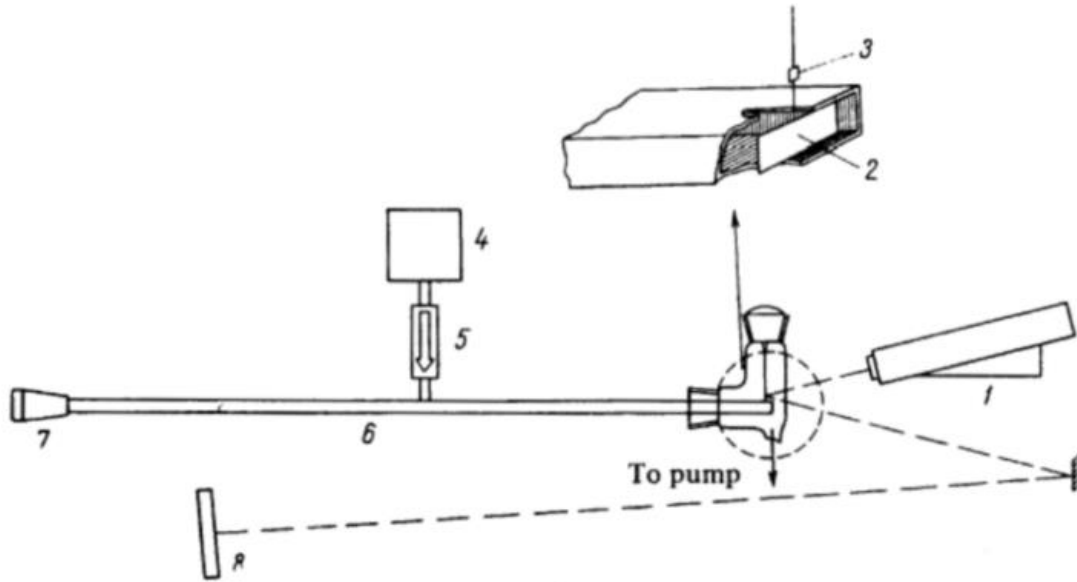


Figure 5: Illustration of Branginskii, Manukin, and Tikhonov's microwave cavity with moveable wall. Numbered components are described in the text.

The microwave cavity was constructed of a rectangular 3 cm waveguide with a length of 10 m (labelled “6” in Figure 5). At the end of the waveguide was a 1 micron thick sheet of aluminum foil stiffened by a frame made of quartz fiber (“2”). The framed foil was suspended on one end by an additional thin quartz fiber (“3”) to make a torsional pendulum. The system was placed in vacuum to reduce the effects of air resistance. The natural resonant frequency of the pendulum was 3 Hz, and in the absence of any microwaves in the cavity, the mechanical oscillations in the system would take about 80 minutes to damp out. The microwave cavity was coupled to a microwave source (“4”), and the resonance frequency of the cavity could be tuned by moving a piston (“7”) at the end of the waveguide opposite the torsional pendulum end wall. It was therefore possible to change the resonance frequency of the cavity such that the microwave generator would have arbitrary detuning from the cavity resonance.

A laser (“1”) pointed at the torsional pendulum end wall reflected off of it, and propagated in free-space to a photographic film (“8”). The long free-space propagation of the laser beam translated

the small angular oscillations of the torsional pendulum into large sinusoidal oscillations on the photographic film. The amplitude of the oscillations of the pendulum could then be extracted by measuring the size of the exposed area on the photographic film. By changing the film at fixed time intervals, it was therefore possible to measure the oscillation amplitude vs. time in order to characterize the damping rate of the torsional pendulum. Braginskiĭ, et al found that the damping rate of the torsional pendulum was directly related to the power of the microwaves in the cavity as well as the detuning of the microwave source from the cavity resonance. Positive detunings would decrease the damping time, and negative detunings would increase it. By carefully characterizing the damping time and resonant frequency of the pendulum at different detunings, they were able to compute the damping force applied to the oscillator by the microwave field. They also noted a corresponding change in the resonant frequency of pendulum, which they attributed to an extra spring force attributable to the microwave field. This was the first experimental observation of the optical damping and optical spring effects.

As described in the section “Practical Optomechanical Systems” above, it is very difficult to observe quantum mechanical effects in mechanical oscillators at low frequencies and high temperatures. It is perhaps no surprise then that the first experiment to observe quantum effects in a mesoscopic oscillator used a dilatational mode of a cantilever with a mechanical frequency of $\omega_m/2\pi = 6$ GHz that was cooled in a dilution refrigerator.⁵ This system was published in 2010 by the Cleland and Martinis group. The cantilever was constructed of a layer of aluminum nitride sandwiched between layers of aluminum. As a piezoelectric material, the aluminum nitride at the center of the cantilever allowed for direct coupling of the mechanical mode to electrical signals. These electrical signals were then capacitively coupled to a qubit, i.e. a Josephson junction in parallel with a capacitor and an inductor, which behaves like a tunable quantum two-level system. When the qubit was tuned such that the energy difference between its two levels corresponded to the frequency of the mechanical mode,

the coupling between the two systems caused a periodic exchange of energy between the two: Rabi oscillations. As the state of the qubit could be easily readout using a magnetic flux bias pulse technique, it was possible to characterize the state of the mechanical oscillator by measuring its effects on the qubit. This technique was used to demonstrate unambiguously that the cantilever was cooled to its vibrational ground state. Once in the ground state, a microwave drive pulse was applied to produce single quantum excitations in the cantilever, which were similarly verified via their effect on the qubit.

Mechanical oscillators with lower resonant frequencies cannot be cooled into the quantum regime with cryogenic cooling alone. Typically, some form of optical cooling such as the sideband cooling technique discussed briefly earlier in this dissertation is used. This technique was originally described in two pioneering theory papers from 2007,^{27, 20} which suggested that a mechanical oscillator can be cooled to its motional ground state in the resolved sideband limit, i.e. when the lifetime of the electromagnetic mode being driven is significantly longer than the mechanical oscillation period.

One of the first optomechanical devices to achieve ground state cooling using this technique was the Painter group's photonic crystal "zipper" cavity in 2011.⁶ This device consisted of an integrated optical and mechanical oscillator fabricated in the surface layer of a silicon-on-insulator chip. The oscillator was a silicon bridge which had co-localized optical and mechanical resonances thanks to periodic patterning of the bridge which confined both the optical and mechanical mode to the same region of the structure. Light was coupled into the oscillator evanescently from a nearby tapered optical fiber. The oscillator had a mechanical frequency of 3.68 GHz, and was cryogenically cooled to 20 K, significantly higher than the dilatational cantilever device discussed earlier. At 20 K, the phonon occupancy of the photonic crystal oscillator was estimated to be around 100 phonons, but a red-detuned optical drive tone was then used to cool the mechanical mode to ~ 0.86 phonons. In 2012 and 2015, variants of the photonic crystal zipper cavity device demonstrated the sideband asymmetry expected to occur at low phonon numbers,^{28, 29} and in 2013 another version of this system

demonstrated the production of light with non-classical statistics.³⁰ A modified version consisting of a photonic crystal nanobeam oscillator suspended between two photonic crystal slab oscillators also demonstrated strong non-linear coupling between the optical mode frequencies and the positions of the mechanical oscillators in 2015.³¹

Another device to achieve ground state cooling in 2011 was the aluminum capacitor drumhead oscillator pioneered by the Simmonds group.⁷ The oscillator was the fundamental drumhead mode of a circularly shaped parallel plate capacitor, integrated into a superconducting microwave LC circuit. The mechanical frequency of the oscillator was 10 MHz, significantly lower than the Cleland and Martinis and Painter group devices, and the resonant frequency of the LC circuit was 7.54 GHz. The device was cryogenically cooled to 15 mK in a cryostat, giving it a starting phonon occupancy of just 30 phonons. A microwave tone detuned to the red side of the LC circuit resonance was then used to cool the oscillator to a phonon occupancy of less than one. In 2013, the bidirectional transfer of a coherent quantum state between itinerant microwave fields and a mechanical oscillator was demonstrated.¹⁴ Later that year, a similar system was used to demonstrate entanglement between the mechanical oscillator and a propagating microwave field.³² In 2015, a system consisting of an aluminum capacitor oscillator coupled to a qubit was used to demonstrate quantum coherent amplification of the zero-point fluctuations of the mechanical motion of the oscillator.³³ Most recently, such a system was used to prepare a squeezed mechanical state which was then monitored continuously via a two-tone measurement technique to perform a quantum non-demolition measurement of one mechanical quadrature of the capacitor's motion.¹²

A similar device was introduced by the Schwab group. This device consisted of a rectangular aluminum capacitor with fundamental drumhead frequency of 4 MHz, shunted by an inductor to produce an LC electrical resonance at a frequency of 5.4 GHz. In 2014, this device was cryogenically cooled in a dilution refrigerator to low phonon number, and further cooling was performed using a

detuned microwave tone to a phonon occupancy of less than one.⁸ Using a back-action evading two-tone measurement technique originally published by the Schwab group in work on an earlier silicon nitride/aluminum nanostring oscillator³⁴, the group performed a back-action evading measurement of their aluminum capacitor system, demonstrating 8.5 dB suppression of quantum back-action.³⁵ A final paper from 2015 demonstrates mechanical squeezing of the same oscillator, with one quadrature of the oscillator's motion exhibiting fluctuations a factor of 0.8 times the ordinary magnitude of the oscillator's zero-point fluctuations.¹³

In 2013, the Regal group detected evidence of radiation pressure shot noise on a silicon nitride membrane oscillator.²² Later that year, they demonstrated optical squeezing by 1.7 dB below the shot noise level using a system with a similar membrane.²⁵ In 2014, the Regal and Lehnert groups demonstrated bidirectional quantum coherent state transfer between microwave and optical light using a partially metal-coated membrane as an intermediary.¹⁵ Finally, in 2015, the Regal group published a paper demonstrating cooling of a 1.5 MHz membrane mechanical mode to a phonon occupancy of about 2, using sideband asymmetry as a means of extracting the effective temperature of the mechanical mode.⁹ Around the same time as the Regal group, the Steele group at Delft University used an aluminum coated silicon nitride membrane to demonstrate cooling of a 123 kHz mechanical mode to a phonon occupancy of about 5.¹⁰ As discussed in this dissertation, the Harris group at Yale also achieved near ground state cooling of a membrane around this time, cooling a 705 kHz mechanical mode to a phonon occupancy of ~ 0.84 .¹¹

While the optomechanical devices discussed here all sound very different, they are all fundamentally similar in that they consist of one (or two) electromagnetic modes coupled to one mechanical mode, and all can ultimately be described as more advanced variations of the canonical optomechanical system. Each system has its particular merits, as demonstrated by the range of physical

phenomena that each has been used to explore. In the next section, I will describe the topic of my research, as well as an overview of the history of this research in the Harris group at Yale.

7. History of the Membrane-in-the-Middle Experiment in the Harris Group

This dissertation presents results of two optomechanical experiments with a silicon nitride membrane mechanical oscillator placed in the middle of a free-space Fabry-Perot cavity. The focus of the first experiment is on ground state cooling, and the second is on characterizing the classical dynamics of the system when the coupling between the mechanical oscillator and the optical cavity is non-linear. The results presented in this dissertation were the product of many years of effort which began before my time in the Harris group. In this section I will review previous publications about the membrane-in-the-middle experiment in the Harris group.

The first, a senior essay by Thompson,³⁶ derived the behavior of the cavity eigenmodes as a function of membrane position for the one dimensional case and described construction of the first experimental membrane-in-the-middle setup. Cavity alignment and locking techniques were introduced in this essay, and a basic method of membrane displacement measurement was discussed. The experiment was conducted in vacuum at room temperature, and demonstrated laser cooling of the membrane's effective temperature to 6.82 ± 0.61 mK.³⁷ In this experiment there was only a single laser used for locking to cavity, measuring the membrane's motion, and cooling it through the sideband cooling technique. Unfortunately, the optimal laser detuning for each of these processes is different: Pound-Drever-Hall locking³⁸ and measurement requires the laser detuning be close to resonance, while sideband cooling requires the laser detuning to be to the red by the mechanical frequency. Furthermore, a large amount of classical laser noise on the cooling beam in this experiment would cause heating of the membrane at high cooling powers, limiting the lowest phonon number that could be reached. Thus, while the results were an excellent proof of concept that inspired future research into

the membrane-in-the-middle design, the experiment was not able to cool the membrane to the ground state. Additionally, the Pound-Drever-Hall technique used for locking and measurement does not permit measurement of the red and the blue phase modulation sidebands individually, which would have precluded detection of sideband asymmetry, even if ground state cooling were possible.

The Ph.D. dissertation by Zwickl³⁹ introduced a new heterodyne detection scheme, used in this dissertation as well. Using a heterodyne detection scheme permits independent measurement of the red and the blue sidebands of mechanical motion. The main focus of the Zwickl dissertation was efforts to detect radiation pressure shot noise using the membrane-in-the-middle setup. As already briefly discussed, in the simplest case, detection of radiation pressure shot noise requires high enough laser power such that the noise due to back-action from radiation pressure shot noise exceeds the measurement imprecision noise due to shot noise in the optical field. In practical efforts to measure radiation pressure shot noise however, the measurement imprecision is generally larger than just the imprecision due to shot noise because of technical, or “classical” laser noise, and noise added by the photodetector and its amplifier. Additionally, the thermal motion of the mechanical oscillator also contributes to the measured spectrum, and is often much larger than the driven motion due to radiation pressure shot noise. In Zwickl’s dissertation it was determined that the ratio of the radiation pressure shot noise fluctuations to the thermal fluctuations alone, not even considering classical laser noise, would be on the order of $10^{-5} - 10^{-4}$. The heterodyne detection scheme used in Zwickl’s dissertation could have allowed for detection of the relatively weak contribution of radiation pressure shot noise to the spectrum of the mechanical oscillator’s motion by examining correlations between the red and the blue sidebands, as described in a theory paper by Børkje et al.⁴⁰ Unfortunately, even with this novel detection scheme, the additional noise sources proved too large for detection of radiation pressure shot noise in a reasonable measurement time.

The dissertations by Yang⁴¹ and Jayich⁴² focused on attempts to cool a membrane-in-the-middle to its vibrational ground state. Though the primary scientific goal in these dissertations was not the detection of radiation pressure shot noise, many improvements were made to overcome the technical challenges encountered in Zwickl's dissertation. First, the membrane-in-the-middle system was put in a cryostat cooled to ~ 300 mK to reduce the membrane's thermal motion. Second, means were investigated to reduce classical laser noise. A narrow-linewidth Fabry-Perot cavity was constructed and used to passively filter classical laser noise on the laser used for sideband cooling of the membrane. The use of a cryogenic system in the Yang and Jayich dissertations also introduced new challenges in regard to alignment of the optical system. Thermal contraction as the system was cooled from room temperature to 300 mK necessitated the introduction of an in situ positioning and tip/tilt adjustment system for the membrane, so that it could be aligned within the cavity after cooling down. This positioning and tip/tilt adjustment system also opened the possibility of exploring points of non-linear coupling in the cavity spectrum due to avoided crossings between cavity modes, and even tuning the properties of these crossings in situ.⁴³ Ultimately, Yang and Jayich demonstrated sideband cooling of the membrane's fundamental mechanical mode at 261 kHz from a starting phonon occupancy of $\sim 30,000$ to a final occupancy of about 60. Further cooling of the membrane was precluded by difficulties in resolving the membrane's motion in the measured heterodyne spectrum at lower phonon occupancies. The signal-to-noise ratio of the measurement was low due to limited measurement beam power. The power in the measurement beam could not be increased due to the additional classical laser noise this would introduce. Unwanted mechanical vibrations within the cryostat added spurious noise spikes to the spectrum of the membrane's motion, and a lack of adequate vibration isolation made it difficult to keep the lasers locked to the cavity long enough to achieve adequately long measurement averaging times.

This dissertation presents a new version of the cryogenic membrane-in-the-middle setup which was redesigned from the ground up to alleviate the problems discovered by Yang and Jayich. A new design of the cryogenic portion of the experiment provides excellent vibration isolation within the cryostat, and the cryostat itself is suspended on pneumatic air legs for additional vibration reduction. A new filter cavity provides classical laser noise reduction on the measurement beam (which was unfiltered previously), and this together with a novel locking setup allows for higher signal-to-noise ratio detection of membrane's motion. Additionally, a smaller membrane, and higher frequency mechanical mode pushes the system into a regime where laser noise and spurious mechanical vibrations are smaller to begin with. An improved kinematic computer-controlled positioning system within the cryostat allows for in situ adjustment of not only the membrane's position and tip/tilt, but also allows for in situ realignment of the optics to the Fabry-Perot cavity. All of these improvements together allowed for the sideband cooling of the membrane to phonon occupancies less than one and the observation of sideband asymmetry.¹¹ Additionally, they provided a very stable platform for exploration of the avoided crossings between cavity modes.⁴⁴

As of the writing of this dissertation, the membrane-in-the-middle experiment in the Harris lab is nearly a decade in the making. Significant progress has been made over the years, and the experiment is at a point where some of the scientific goals originally proposed theoretically can be achieved experimentally. In the following chapters, I will derive the optical modes in a Fabry-Perot cavity and discuss how the presence of a membrane can affect these modes. I will then give details of the current cryogenic membrane-in-the-middle optomechanical system, and present the methods and results of efforts to cool the membrane to its vibrational ground state, and explore the dynamics of avoided crossings between cavity modes.

II. The Membrane-in-the-Middle Geometry

The type of mechanical oscillator described in this dissertation, a silicon nitride membrane, has been extensively studied both by the Harris group and others. In silicon nitride membrane optomechanical systems, the mechanical mode of the oscillator has a typical frequency ranging from several hundred kilohertz up to several megahertz, and the electromagnetic mode is typically in the optical regime. Coupling between the two is achieved by placing the membrane inside a Fabry-Perot optical cavity. The presence of the dielectric membrane within the cavity changes the optical path length within the cavity, thus detuning the cavity's resonant frequency. The amount to which the cavity is detuned by the membrane depends on the overlap between the intracavity standing wave and the membrane's thickness. The thickness is usually much less than the wavelength of the electromagnetic field, and therefore the overlap can be adjusted by moving the membrane within the intracavity standing wave. This is illustrated in Figure 6, which shows a membrane within a Fabry-Perot cavity. The intracavity field is plotted in red, and the cavity detuning (i.e. the perturbation of the cavity's resonant frequency by the membrane) corresponding to the overlap between the membrane and the intracavity field, is plotted in blue.

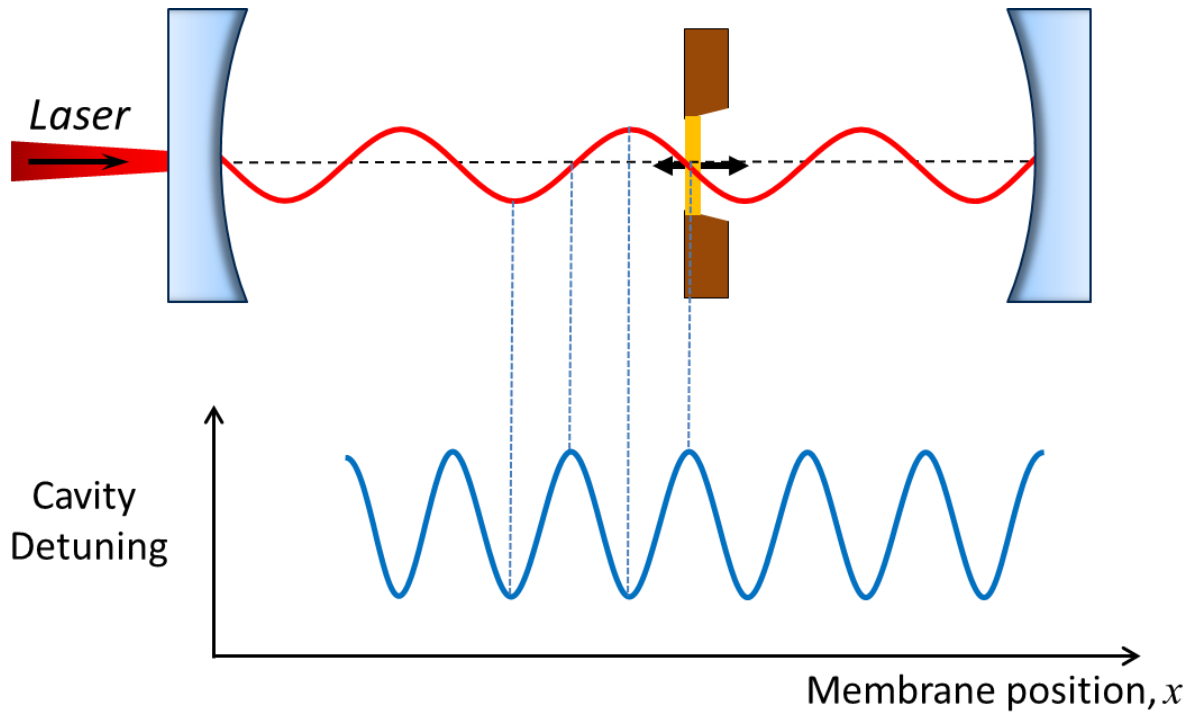


Figure 6: Top, illustration of a membrane in the middle of a Fabry-Perot optical cavity, with the intracavity field plotted in red. Bottom, plot of the cavity detuning as a function of membrane position.

The membrane-in-the-middle geometry has several advantages as an optomechanical system. First, since the mechanical oscillator is separate from the cavity mirrors, there is no need for mirrors that are simultaneously highly reflective, mechanically compliant, and of high mechanical quality factor as would be the case in the canonical system. Instead, rigidly fixed high reflectivity mirrors define the optical cavity, and the system can exploit the excellent properties of the silicon nitride membrane as the mechanical element. The high stress silicon nitride membranes used in these systems have high mechanical quality factors ($\sim 10^6 - 10^7$) and low optical absorption ($< 1 \times 10^{-4}$) in the near infrared regime. Additionally, they are commercially available and inexpensive. Though not necessarily an advantage, silicon nitride membranes tend to have large masses ($\sim 10 - 100$ ng), and low mechanical frequencies (~ 1 MHz) compared with the mechanical oscillators used in other systems.

Since the membrane is placed within a free-space optical cavity, its position within the cavity, as well as its angular orientation relative to the cavity axis can be manipulated. This gives the membrane-

in-the-middle system a large degree of in situ tunability, an advantage over many other optical systems which consist of rigidly mounted oscillators, or hybrid mechanical and electromagnetic resonators. At the same time, however, this tunability also introduces new challenges like the need for precise alignment and levelling of the membrane within the cavity. These challenges will be discussed later in this dissertation when I describe the experimental setup. In the meantime, however, I will discuss the mathematical framework of the membrane-in-the-middle system, starting with the Fabry-Perot optical cavity.

1. Derivation of the Fields in a Fabry-Perot Cavity

To derive the electromagnetic field inside a Fabry-Perot cavity it is useful to begin with the time-independent scalar wave equation, which arises from Maxwell's equations in free-space.⁴⁵ The scalar wave equation is:

$$\nabla^2 u + k^2 u = 0 \quad (2)$$

Here u generically represents either the electric or magnetic field of the electromagnetic wave, and $k = 2\pi/\lambda$ is the wavenumber of the light. For light traveling along an axis which we define as z , a reasonable place to begin is the general solution:

$$u = \psi(x, y, z)e^{-ikz} \quad (3)$$

In this expression, e^{-ikz} is the expression of a plane wave, and $\psi(x, y, z)$ represents the difference between this general solution and a plane wave. Substituting Equation (3) into the wave equation gives

$$\left(\frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} \right) e^{-ikz} + \frac{\partial^2}{\partial z^2} [\psi e^{-ikz}] + k^2 \psi e^{-ikz} = 0 \quad (4)$$

Expanding the z -derivative produces several terms:

$$\frac{\partial^2}{\partial z^2} [\psi e^{-ikz}] = \frac{\partial^2 \psi}{\partial z^2} e^{-ikz} - 2ik \left(\frac{\partial \psi}{\partial z} \right) e^{-ikz} - k^2 \psi e^{-ikz} \quad (5)$$

In most Fabry-Perot cavities, the beam has relatively small divergence and remains close to the central axis of the cavity. These are the conditions necessary for the “paraxial approximation” which assumes that ψ changes slowly along the z -axis. In the paraxial approximation $\frac{\partial^2 \psi}{\partial z^2} \approx 0$, eliminating the first term in Equation (5). Equation (4) now becomes

$$\frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} - 2ik \left(\frac{\partial \psi}{\partial z} \right) = 0 \quad (6)$$

The following expression satisfies Equation (6):

$$\psi = \exp \left(-i \left(P + \frac{k}{2q} r^2 \right) \right) \quad (7)$$

In Equation (7), $r^2 = x^2 + y^2$ is the distance from the z -axis, and $P(z)$ and $q(z)$ are complex parameters which describe the phase and cross sectional profile of the beam. By substituting Equation (7) back into Equation (6) and exploiting the linear independence of polynomials in r to compare coefficients on terms with equal powers of r , it is clear that

$$\begin{aligned} \frac{\partial q}{\partial z} &= 1 \Rightarrow q = z + q_0 \\ \frac{\partial P}{\partial z} &= -i/q \end{aligned} \quad (8)$$

Since q is complex, it is possible to break it into real and imaginary parts, which with some foresight, can be written as

$$\frac{1}{q} = \frac{1}{R} - i \frac{\lambda}{\pi w^2} \quad (9)$$

Plugging Equation (9) into ψ gives insight into the meaning of R and w :

$$\psi = \exp \left(-i \left(P + \frac{kr^2}{2R} - \frac{ik\lambda r^2}{2\pi w^2} \right) \right) = \exp \left(-i \left(P + \frac{kr^2}{2R} \right) - \frac{r^2}{w^2} \right) \quad (10)$$

From Equation (10) it becomes clear that $R(z)$ represents the radius of curvature of the wavefront at any point along z , and $w(z)$ describes the radius of the beam in horizontal cross section. For any beam, there will be a value of z at which w is minimized. It is convenient to define this point as $z = 0$ and to define $w_0 \equiv w|_{z=0}$. By symmetry, it can be predicted that the radius of curvature $R = \infty$ at this point. Then, $q_0 \equiv q|_{z=0} = i\pi w_0^2/\lambda$. A general expression for q can be obtained by substituting this into the expression for q in Equation (8):

$$q(z) = z + i \frac{\pi w_0^2}{\lambda} \quad (11)$$

Substituting Equation (11) into the original expression for $1/q$, Equation (9), leads to explicit formulas for $w^2(z)$ and $R(z)$:

$$\begin{aligned} w^2(z) &= w_0^2 \left[1 + \left(\frac{\lambda z}{\pi w_0^2} \right)^2 \right] \\ R(z) &= z \left[1 + \left(\frac{\pi w_0^2}{\lambda z} \right)^2 \right] \end{aligned} \quad (12)$$

To obtain an expression for the phase parameter P , it is useful to substitute Equation (11) into the expression for $\frac{\partial P}{\partial z}$ in Equation (8) and integrate to get

$$P(z) = -i \ln \left[1 - i \left(\frac{\lambda z}{\pi w_0^2} \right) \right] = -i \ln \sqrt{1 + \left(\frac{\lambda z}{\pi w_0^2} \right)^2} - \tan^{-1} \left(\frac{\lambda z}{\pi w_0^2} \right) \quad (13)$$

From Equation (12), the radical within the natural logarithm simplifies to just w/w_0 . Since $P(z)$ is multiplied by i in Equation (10), this represents the expected decay of field amplitude as the beam spreads out in the radial direction. The $\tan^{-1} \left(\frac{\lambda z}{\pi w_0^2} \right)$ term represents the additional phase accumulated by the beam as a result of its Gaussian nature, compared with a plane wave. This phase is referred to as the Guoy phase, and is often represented by the symbol Φ .

The final expression for the field is therefore:

$$u(r, z) = \frac{w_0}{w} \exp \left[-i(kz - \Phi) - r^2 \left(\frac{1}{w^2} + \frac{ik}{2R} \right) \right] \quad (14)$$

Careful observation of the scalar wave equation and the solution found above reveals that Equation (14) is actually a special case of a more general form. Starting with Equation (7), one might postulate that a more general solution would be of the form:

$$\psi = g\left(\frac{x}{w}\right) h\left(\frac{y}{w}\right) \exp \left(-i \left(P + \frac{k}{2q} r^2 \right) \right) \quad (15)$$

By substituting this trial solution into the wave equation, it becomes clear that the wave equation is satisfied when $g\left(\frac{x}{w}\right)$ and $h\left(\frac{y}{w}\right)$ are Hermite polynomials. Equation (7) is simply the special case of this expression for which $g\left(\frac{x}{w}\right) = h\left(\frac{y}{w}\right) = 1$, which is the zeroth order Hermite polynomial. Higher-order Hermite polynomials produce solutions with different transverse spatial profiles. Mode numbers n and m specify the order of the Hermite polynomial used to describe the field in the x and y directions respectively. The Guoy phase, is also modified for the higher-order modes:

$$\Phi = (1 + m + n) \arctan \left(\frac{\lambda z}{\pi w_0^2} \right) \quad (16)$$

The final expression for all of the modes (within the paraxial approximation) is therefore:

$$u(x, y, z, m, n) = H_n \left(\frac{\sqrt{2}x}{w} \right) H_m \left(\frac{\sqrt{2}y}{w} \right) \frac{w_0}{w} \exp \left[-i(kz - \Phi) - r^2 \left(\frac{1}{w^2} + \frac{ik}{2R} \right) \right] \quad (17)$$

The transverse spatial profiles of several of these modes are shown in Figure 7 below.

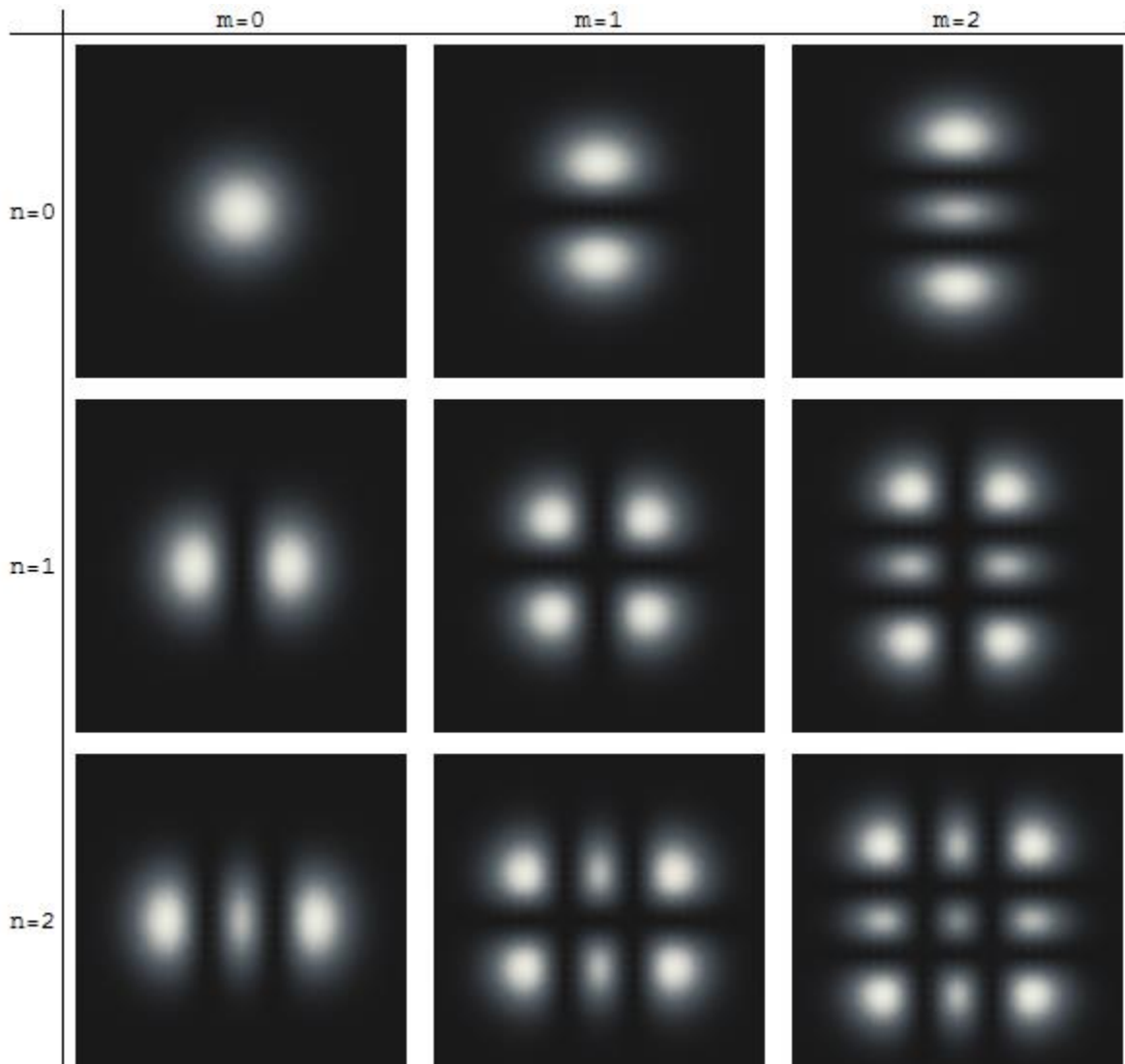


Figure 7: Transverse spatial mode profiles of the first 9 Hermite Gauss optical cavity modes.

2. Hermite Gauss Modes in a Fabry-Perot Cavity

The derivation of the Hermite Gauss modes so far has assumed free-space boundary conditions. For a mode to be stable within a Fabry-Perot cavity, it is required that the mode remain self-consistent after one complete round trip through the cavity. This has a few logical implications: (1) the phase fronts of the beam must be parallel to the surfaces of the mirror, such that light incident on the mirrors retraces its path upon reflection, and (2) the total phase change between one mirror and the next must be an integer multiple of π so that destructive interference does not occur when beams reflect back and

forth many times. For a perfectly symmetrical system of two concave mirrors with radius of curvature R_{mirror} a distance L apart, the first of these can be expressed mathematically $R\left(\frac{L}{2}\right) = R_{\text{mirror}}$.

Substituting this into Equation (12) yields $R_{\text{mirror}} = \left(\frac{L}{2}\right) \left[1 + \left(\frac{\pi w_0^2}{\lambda(L/2)}\right)^2\right]$, which can be solved for the beam waist w_0 .

The second implication can be expressed as $kz - 2\Phi = n_{\text{int}}\pi$, where n_{int} represents an arbitrary integer. From the second implication it is possible to derive a formula for the resonant frequency of the optical modes:

$$\frac{\nu}{\text{FSR}} = (n_{\text{int}} + 1) + \frac{1}{\pi} (m + n + 1) \arccos\left(1 - \frac{L}{R_{\text{mirror}}}\right) \quad (18)$$

In Equation (18), ν is the frequency of the optical mode and $\text{FSR} \equiv c/2L$ is the free spectral range of the optical cavity. From Equation (18), it is clear that except for special cases where the second term produces integers, higher-order modes of different total mode number $m + n$ are in general non-degenerate in frequency with each other. Therefore, to couple light into an optical cavity, the light must match both the spatial profile and frequency of a mode as defined by the cavity mirrors. In the experiments described in the forthcoming chapters, since light from the lasers is typically in a free-space fundamental Gaussian mode ($n = m = 0$), the light typically couples most strongly to the corresponding mode in our optical cavities. However, precisely matching the beam shape and convergence angle to the fundamental Gaussian mode defined by the cavity mirrors is technically challenging, and therefore the spatial profile of the incident beam always has some overlap with the higher-order modes. Thus, changing laser frequency alone is generally sufficient to drive higher-order Hermite-Gauss modes, albeit at reduced efficiency.

3. A Membrane near the Cavity Waist

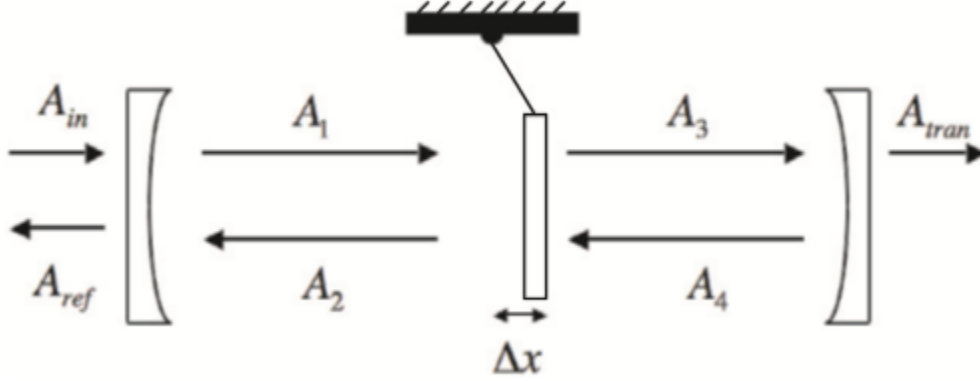


Figure 8: Schematic of a partially reflective membrane inside an optical cavity. ⁴⁶

In the experiments presented in this thesis, a silicon nitride membrane is placed in the middle of a high finesse optical cavity made up of two highly reflective concave mirrors. To determine the effects of placing such a membrane in the cavity, it is useful to start with the simple case where the membrane is located close to the beam waist and is oriented perpendicularly to the optical mode's direction of propagation. In this position and orientation, it is possible to approximate the fields inside the cavity as plane waves at normal incidence to a dielectric slab, which simplifies the math.

To begin, I model the membrane as a dielectric slab of thickness L_d and index of refraction n . For light with wavenumber k , the field reflectivity and transmissivity coefficients for light at normal incidence in vacuum are given by: ⁴⁷

$$\tilde{r}_d = \frac{(n^2 - 1) \sin knL_d}{2in \cos knL_d + (n^2 + 1) \sin knL_d} \quad (19)$$

$$i\tilde{t}_d = \frac{2in}{2in \cos knL_d + (n^2 + 1) \sin knL_d} \quad (20)$$

As expected due to energy conservation, it can be shown that $|\tilde{r}_d|^2 + |i\tilde{t}_d|^2 = 1$, and $\arg(\tilde{r}_d/i\tilde{t}_d) = \pm \pi/2$. This second condition implies that $\arg(\tilde{r}_d) = \arg(\tilde{t}_d) + n_{\text{int}}\pi$, where n_{int} is either -1, 0 or 1

depending on the values of k , L_d , and n . In the limit $L_d \ll \lambda$ with realistic index of refraction $n \approx 2$, it can be shown that the complex phases of \tilde{r}_d and \tilde{t}_d are equal, which is important later.

With r_1 (it_1) and r_2 (it_2) representing the field reflectivity (transmissivity) of mirrors 1 and 2 respectively, the system of equations describing the field amplitudes within the system is:

$$\begin{aligned}
A_1 &= it_1 A_{\text{in}} + r_1 A_2 e^{ikL_1} \\
A_2 &= \tilde{r}_d A_1 e^{ikL_1} + i\tilde{t}_d A_4 e^{ikL_2} \\
A_3 &= i\tilde{t}_d A_1 e^{ikL_1} + \tilde{r}_d A_4 e^{ikL_2} \\
A_4 &= r_2 A_3 e^{ikL_2} \\
A_{\text{refl}} &= it_1 A_2 e^{ikL_1} + r_1 A_{\text{in}} \\
A_{\text{tran}} &= it_2 A_3 e^{ikL_2}
\end{aligned} \tag{21}$$

As a system of six equations with six unknowns, it is possible to obtain analytic expressions for each field component. However, this is unnecessarily complicated and yields unwieldy equations. In practice, what we care about is just the frequency of the cavity eigenmodes as a function of membrane position, i.e. we want to find combinations of k , L_1 , and L_2 that provide non-trivial solutions to the system of equations. Since this depends only on the fields and boundary conditions within the cavity and is independent of the presence of any laser drive, we can neglect the equations for A_{refl} and A_{tran} and set $A_{\text{in}} \rightarrow 0$. We also, for clarity, shall write out explicitly the complex phases of \tilde{r}_d and \tilde{t}_d as $\tilde{r}_d = r_d e^{i\phi}$ and $\tilde{t}_d = t_d e^{i\phi}$ so that it is easier to follow them through the computation. We can then write the system of equations in matrix form as follows:

$$\begin{pmatrix} A_1 \\ A_2 \\ A_3 \\ A_4 \end{pmatrix} = \begin{bmatrix} 0 & r_1 e^{ikL_1} & 0 & 0 \\ r_d e^{ikL_1+i\phi} & 0 & 0 & it_d e^{ikL_2+i\phi} \\ it_d e^{ikL_1+i\phi} & 0 & 0 & r_d e^{ikL_2+i\phi} \\ 0 & 0 & r_2 e^{ikL_2} & 0 \end{bmatrix} \begin{pmatrix} A_1 \\ A_2 \\ A_3 \\ A_4 \end{pmatrix} \tag{22}$$

The four eigenvalues of this matrix are:

$$\begin{aligned}
m = \pm \frac{1}{\sqrt{2}} & \left[e^{2ikL_1+i\phi} r_1 r_d + e^{2ikL_2+i\phi} r_2 r_d \right. \\
& \pm \left((e^{2ikL_1+i\phi} r_1 r_d + e^{2ikL_2+i\phi} r_2 r_d)^2 \right. \\
& \left. \left. - 4(e^{2ik(L_1+L_2)+2i\phi} r_1 r_2 r_d^2 + e^{2ik(L_1+L_2)+2i\phi} r_1 r_2 t_d^2) \right)^{\frac{1}{2}} \right]^{\frac{1}{2}}
\end{aligned} \tag{23}$$

There are several ways to simplify this expression. First, typical silicon nitride membranes are much less reflective than the end mirrors of the cavity, so $(r_1, r_2) \gg r_d$ and $(r_1, r_2) \gg t_d$. Since r_1 and r_2 only appear in conjunction with r_d and t_d and we can approximate $r_1 \approx r_2 \approx 1$. Second, we can define $\delta_T = 2k(L_1 + L_2)$ to be equal to the phase accumulated during one round trip of the cavity, and $\delta = 2k(L_1 - L_2) = 2kx$ to be equal to the differential round trip phase between the two sides of the cavity. In this notation:

$$\begin{aligned}
m &= \pm \frac{1}{\sqrt{2}} \sqrt{e^{\frac{i\delta_T}{2}+i\phi} (e^{ik(L_1-L_2)} + e^{-ik(L_1-L_2)}) r_d \pm e^{i\phi} \sqrt{(e^{i(\delta_T+2\delta)} + e^{i(\delta_T-2\delta)}) r_d^2 - 2e^{i\delta_T} (r_d^2 + 2t_d^2)}} \\
&= \pm \frac{1}{\sqrt{2}} \sqrt{e^{\frac{i\delta_T}{2}+i\phi} \cos(\delta) r_d \pm e^{\frac{i\delta_T}{2}+i\phi} \sqrt{r_d^2 \cos^2(\delta) - 1}}
\end{aligned} \tag{24}$$

Setting $m = 1$ is the condition for Equation (22) to have a non-trivial solution.^a Squaring both sides, the condition can be written as:

$$1 = \frac{1}{2} e^{\frac{i\delta_T}{2}+i\phi} \left(r_d \cos(\delta) \pm \sqrt{r_d^2 \cos^2(\delta) - 1} \right) \tag{25}$$

Simple algebra then yields:

^a This condition stems from the eigendecomposition of the matrix in Equation (22). To show this, I will rewrite Equation (22) in the more concise notation $\vec{A} = M\vec{A}$ where M is the matrix and \vec{A} is the vector of field amplitudes. As a diagonalizable matrix, M can be written $M = Q\Lambda Q^{-1}$, where Λ is a matrix with the eigenvalues of M on the diagonal, and Q is a matrix of the eigenvectors of M . This allows us to write the field amplitudes vector as $\vec{A} = Q\Lambda Q^{-1}\vec{A}$. Left multiplying both sides by Q^{-1} gives $Q^{-1}\vec{A} = \Lambda Q^{-1}\vec{A}$ which is true only if Λ is equal to the identity, i.e. all the eigenvalues are 1.

$$\frac{1}{2} \left(\cos\left(\frac{\delta_T}{2} + \phi\right) - i \sin\left(\frac{\delta_T}{2} + \phi\right) \right) = \frac{1}{2} \left(r_d \cos(\delta) \pm i \sqrt{1 - r_d^2 \cos^2(\delta)} \right) \quad (26)$$

Finally, considering only the equality of the real parts:

$$\cos\left(\frac{\delta_T}{2} + \phi\right) = r_d \cos(\delta) \quad (27)$$

We can then solve this expression for δ_T :^b

$$\delta_T + 2\phi = 2 \cos^{-1}(r_d \cos(\delta)) \quad (28)$$

In terms of frequency and membrane position, this can be expressed as follows, where the free spectral range of the cavity $\text{FSR} \equiv c/2L$ describes the frequency spacing between longitudinal cavity modes:

$$\nu = \frac{\text{FSR}}{2\pi} (-2\phi + 2 \cos^{-1}(r_d \cos(2kx))) \quad (29)$$

The inverse cosine function in Equation (29) is multivalued, with each value corresponding to a different longitudinal cavity mode.

With this expression in hand, it is now possible to plot the resonant frequencies of the cavity as a function of the membrane position. This plot is shown in Figure 9 below. Several things can be seen from this figure. First, it is clear that for a membrane reflectivity of zero (100% transmissive), the position of the membrane has no effect on the cavity's resonant frequency. As the membrane's reflectivity increases, the position of the membrane starts to more strongly affect the cavity's resonant frequency. When the membrane is at an anti-node of the electric field within the cavity (e.g., at $x = 0$, $x = \lambda/2$, or $x = \lambda$ in Figure 9) the membrane's overlap with the electric field is large and the cavity's resonance frequency is strongly perturbed. At nodes of the field (e.g., at $x = \lambda/4$ or $x = 3\lambda/4$ in Figure 9), since the field strength is small, the interaction is weak and the cavity's resonant frequency nearly returns to the unperturbed frequency. As membrane reflectivity is increased further, the perturbation to

^b This result differs from the one given in Ref. [46] by the sign of the 2ϕ term. I believe the result in Ref. [46] is incorrect, in spite of the same result being mentioned Ref. [39].

the cavity's resonant frequency becomes stronger and stronger, until eventually (in the 100% reflectivity case), the cavity can be modeled as two independent cavities – one between the input mirror and the membrane, and the other between the membrane and the output mirror. The resonant frequency of both of these cavities scale linearly, but they have opposite slopes, since as one cavity gets longer, the other gets shorter.

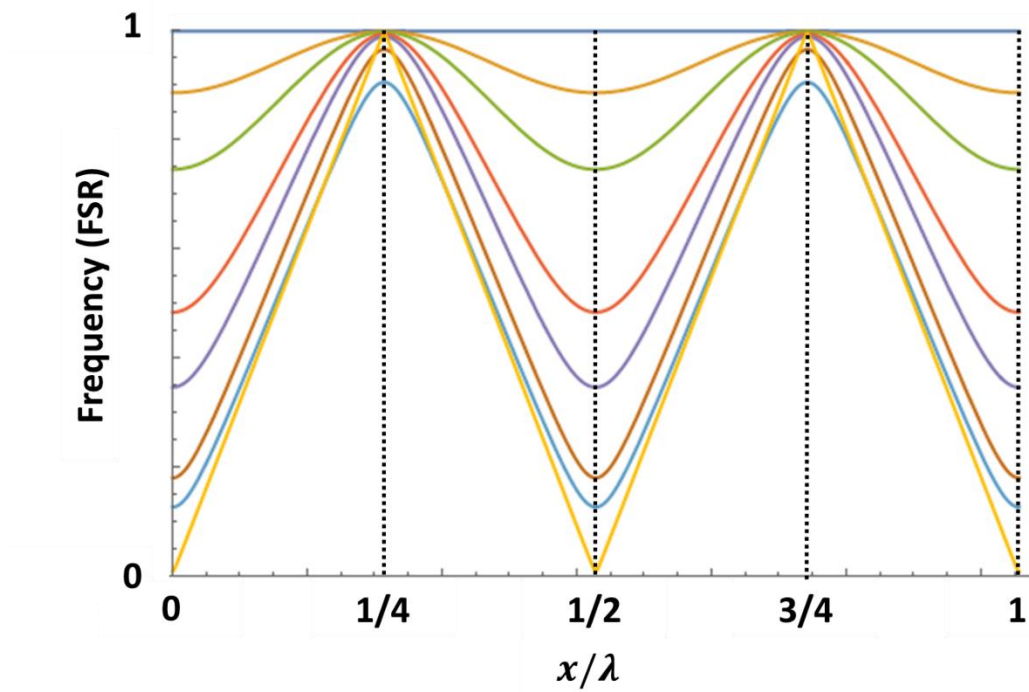


Figure 9: Numerically generated plot of cavity resonant frequency in units of free spectral range vs. membrane position normalized to wavelength for membrane reflectivities 0.00, 0.03, 0.15, 0.51, 0.72, 0.89 and 0.98 (respectively blue, orange, green, red, purple, brown, cyan, yellow). Changes in membrane reflectivity are simulated by changing the index of refraction of the membrane.

4. General Treatment of a Membrane in a Cavity by Perturbation Theory

The assumptions made in the last section are convenient for a basic derivation of cavity properties with a membrane-in-the-middle, but are not always realistic. In particular they do not account for the higher-order modes of the cavity, and even for the $m = n = 0$ mode, give only a plane wave approximation. Additionally, they do not account for the fact that the membrane may be tilted, and not perfectly orthogonal to the optical mode's propagation axis. Solving for the optical modes of the

cavity analytically in this general case is incredibly complicated. However, since the membrane is a thin slab ($L_d \ll \lambda$) and is generally weakly reflective ($r_d \ll 1$), it should only weakly perturb the eigenmodes of the cavity. Therefore, perturbation theory can be used to derive a more accurate theory for the general case of the behavior of a cavity with a membrane-in-the-middle.

Light that transmits through the membrane has a different wavevector while inside the membrane due to the membrane's index of refraction. To account for this, the scalar wave equation needs to be modified from Equation (2):

$$\nabla^2 u + k^2(1 + V(z - z_c))u = 0 \quad (30)$$

In Equation (30), z_c is the equation for the plane of the membrane, and can optionally depend on x and y if the membrane is tilted, e.g. $z_c = z_0 + \alpha_x x + \alpha_y y$. The perturbation term $V(z - z_c) = (n_{\text{SiN}}^2 - 1) \left(\Theta \left[z - \left(z_c - \frac{t}{2} \right) \right] \Theta \left[\left(z_c + \frac{t}{2} \right) - z \right] \right)$, where Θ is the Heaviside function, t is the thickness of the membrane, and n_{SiN} is the index of refraction of the membrane, describes the modified wavevector within the membrane. The perturbed modes of the cavity can now be expressed as a sum over the unperturbed eigenmodes, which will be represented as $u_{(n,m)}^{(0)}$. The unperturbed eigenmodes are orthogonal and normalized such that $\langle u_{n',m'}^{(0)} | u_{n,m}^{(0)} \rangle = \delta_{n,n'} \delta_{m,m'}$. Therefore:

$$u = \sum_{n=0}^{\infty} \sum_{m=0}^{\infty} c_{(n,m)} u_{(n,m)}^{(0)} \quad (31)$$

This solution can now be substituted back into Equation (30).

$$(\nabla^2 + k^2 + k^2 V) \sum_{n=0}^{\infty} \sum_{m=0}^{\infty} c_{(n,m)} u_{(n,m)}^{(0)} = 0 \quad (32)$$

Taking an inner product of this expression with each of the unperturbed modes,

$$\sum_{n,m,n',m'} \left\langle u_{(n',m')}^{(0)} \left| (\nabla^2 + k^2 + k^2 V) c_{(n,m)} \right| u_{(n,m)}^{(0)} \right\rangle = 0 \quad (33)$$

Defining $V_{(n,m)} \equiv \left\langle u_{(n,m)}^{(0)} \left| V \right| u_{(n,m)}^{(0)} \right\rangle$, Equation (33) becomes:

$$\sum_{n,m} \left(\left(k_{(n,m)}^{(0)} \right)^2 c_{(n,m)} \delta_{n,m} + k^2 c_{(n,m)} V_{(n,m)} \right) = 0 \quad (34)$$

This expression becomes somewhat more practical when it is further simplified by the assumption that, to first order, most of the $c_{(n,m)}$ are vanishingly small. This is a valid assumption when calculating the effect of the membrane on a single mode that is well separated from other modes. For example, if we just want to reproduce the quasi-sinusoidal behavior seen in Figure 9, we might assume that $c_{(0,0)}$ is the only non-vanishing $c_{(n,m)}$, and therefore by normalization $c_{(0,0)} \approx 1$. Then, Equation (34) becomes

$$\left(k_{(0,0)}^{(0)} \right)^2 + k^2 V_{(0,0)} = 0 \quad (35)$$

The value of $V_{(0,0)}$ can be expressed as an integral of the mode over the region inside the membrane.

Explicitly this is:

$$\begin{aligned} V_{(0,0)} &= \iiint \frac{w_0}{w} \exp \left[i(kz - \Phi) - r^2 \left(\frac{1}{w^2} - \frac{ik}{2R} \right) \right] (n_{\text{SiN}}^2 \\ &\quad - 1) \left(\Theta \left[z - \left(z_c - \frac{t}{2} \right) \right] \Theta \left[\left(z_c + \frac{t}{2} \right) - z \right] \right) \frac{w_0}{w} \exp \left[-i(kz - \Phi) \right. \\ &\quad \left. - r^2 \left(\frac{1}{w^2} + \frac{ik}{2R} \right) \right] r dr dz d\theta \\ &= 2\pi (n_{\text{SiN}}^2 - 1) \int_0^\infty \int_{(z_c - \frac{t}{2})}^{(z_c + \frac{t}{2})} \left(\frac{w_0}{w} \right)^2 \exp \left[-2r^2 \left(\frac{1}{w^2} \right) \right] dz r dr \end{aligned} \quad (36)$$

While this integral looks simple, the dependence of w on z makes it difficult to compute analytically.

However, it is possible to numerically integrate Equation (36). The result of the integral can then be substituted into Equation (35) which can then be solved to find the new k^2 . Performing this procedure with $\lambda = 1064$ nm, $t = 50$ nm, and $n_{\text{SiN}} = 2$ (appropriate values for the experiments described in this

dissertation) and plotting the resulting resonance frequency in units of free spectral range reproduces the expected quasi-sinusoidal behavior that was derived in the original 1D solution. Figure 10 shows a plot of this numerically computed resonance frequency, which agrees well with the 1D case in Figure 9. For the sake of comparing the two figures, it should be noted that with the parameters $\lambda = 1064$ nm, $t = 50$ nm, and $n_{\text{SiN}} = 2$, the membrane reflectivity is $r_d^2 = 0.15$. Therefore, this curve should lie between the pink and orange curves in Figure 9, which it does.

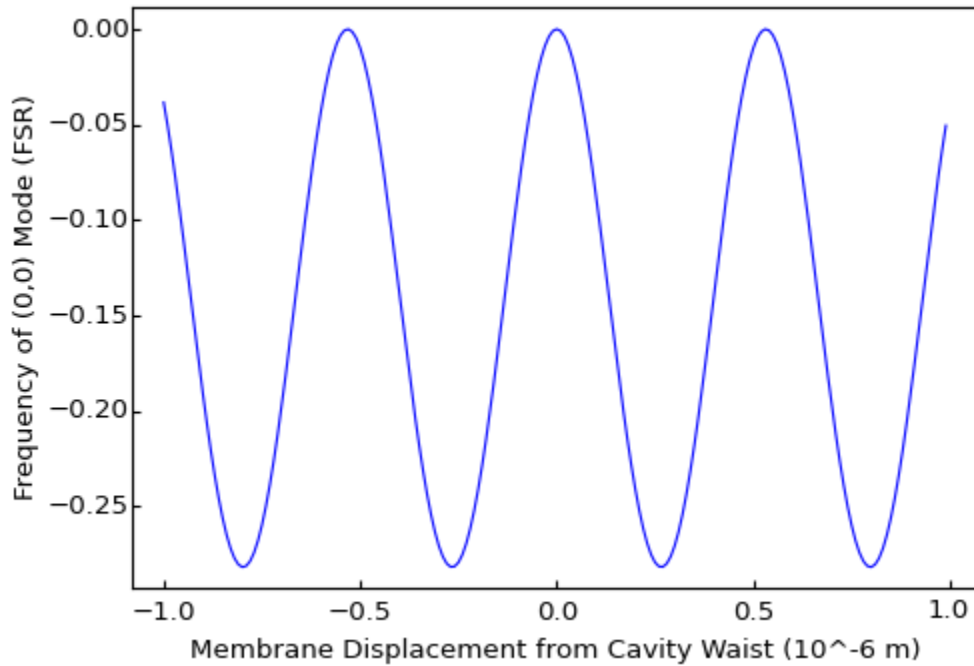


Figure 10: A plot of the result of numerical integration of Equation (36) with parameters selected to match those used in the experiment.

The true power of the perturbation theory approach is not so much its ability to reproduce the quasi-sinusoidal result from the 1D case, but rather its ability to account for tip and tilt of the membrane, as well as predict how the cavity spectrum behaves when the membrane is positioned such that two normally non-degenerate modes cross. Tip and tilt are easily accounted for with nonzero α_x and α_y in the expression $z_c = z_0 + \alpha_x x + \alpha_y y$. The effects of crossing modes can be accounted for by including additional modes in the summation in Equation (34). Equation (35) can then be written as a

matrix equation, and the integrals for each $V_{(n,m)}$ computed numerically. Solving for the eigenvalues of the matrix yields the perturbed cavity resonance frequencies. Figure 11 shows an example of the interaction between the three triplet modes ($TEM_{0,2}$, $TEM_{1,1}$, and $TEM_{2,0}$) of one longitudinal mode and the fundamental transverse mode of the next longitudinal mode.

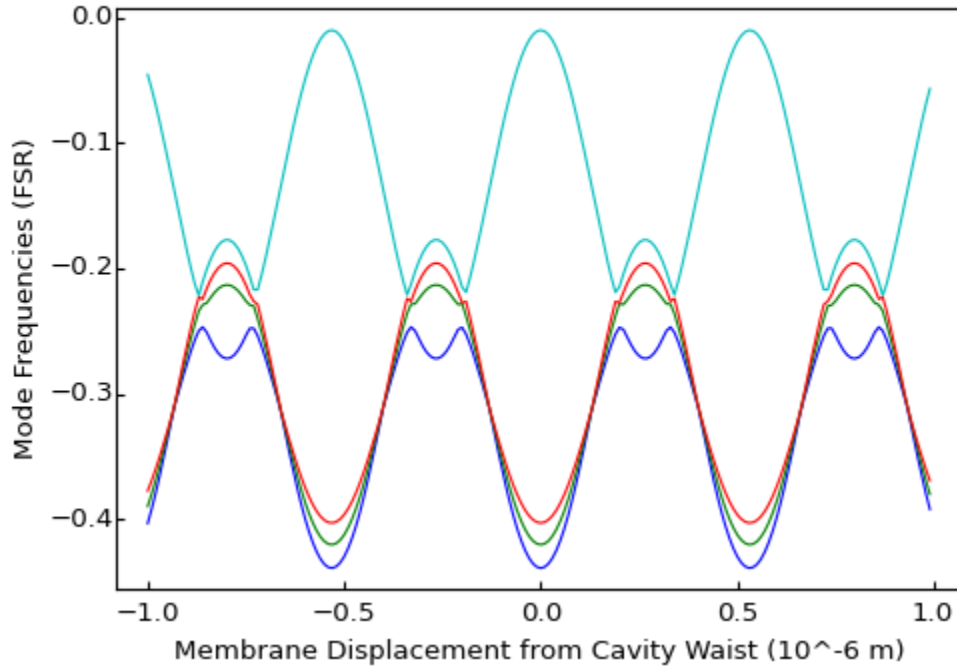


Figure 11: A plot of the frequencies of the (0,0), (0,2), (1,1), and (2,0) transverse modes of the optical cavity for a membrane of thickness $t = 50$ nm and a membrane tilt of $\sim 0.29^\circ$ along both the x and y axes. Plot is based on numerical computation of the $V_{(n,m)}$ integrals as described in the text.

Qualitatively, it is apparent that as the modes cross each other, they actually avoid becoming fully degenerate. This is due to the reduced symmetry of the cavity in the presence of membrane tilt. As a consequence of the modes avoiding each other, there are points in the cavity spectrum for which a Taylor expansion of the mode frequency as a function of membrane position has no first-order term. At these points, the cavity responds quadratically to membrane position, which can yield interesting

physics not accessible with strictly linear coupling. These points of quadratic coupling form the basis of the experiment described in Chapter IV.

While the numerical integration approach is adequate for getting a feel of the interaction between cavity modes at avoided crossings, it is interesting to note that under certain assumptions, it is possible to obtain analytic expressions for the cavity modes. The assumptions necessary to obtain analytic solutions are: small membrane displacement from the cavity waist, small membrane tip/tilt, and membrane dimensions much greater than the laser mode waist. Detailed discussion of the necessary assumptions and full analytic expressions for arbitrary transverse cavity modes can be found in Ref.[48]

5. Mechanical Modes of a Silicon Nitride Membrane

The square silicon nitride membrane-in-the-middle oscillators discussed in this dissertation support multiple mechanical modes. These modes are described by the solution of the two-dimensional time-dependent wave equation with fixed boundary conditions, and have the form

$$z(x, y, t) = A^{(n,m)} \sin\left(\frac{n\pi x}{L}\right) \sin\left(\frac{m\pi y}{L}\right) \sin(\omega_m^{(n,m)} t + \phi^{(n,m)}) \quad (37)$$

In Equation (37), $z(x, y, t)$ is the displacement of the membrane away from its equilibrium position, $A^{(n,m)}$ is the mode amplitude, n and m are integers, x and y are the axes that define the plane of the square membrane (parallel to the edges of the square), L is the length of one side of the membrane, $\omega_m^{(n,m)}$ is the mode's resonant frequency, and $\phi^{(n,m)}$ is an arbitrary phase. The amplitude of each mode depends on the applied forces. In the absence of deliberately applied forces, coupling of the membrane's motion to the thermal bath gives rise to a fluctuating Langevin force, resulting in a mean-squared displacement proportional to the temperature of the thermal bath. The frequencies of the modes are determined by the material properties of the silicon nitride membrane, and the amount that the membrane is stressed. If the frequency of the fundamental mode is known, the frequencies of other modes can be predicted by the following expression:

$$\omega_m^{(n,m)} = \omega_m^{(1,1)} \sqrt{\frac{n^2 + m^2}{2}} \quad (38)$$

This expression suggests that modes of the same $n^2 + m^2$ are degenerate. In practice, however, the membrane is never perfectly symmetrical, and the degeneracy is only approximate.

A scanning electron micrograph of a silicon nitride membrane similar to the ones used in the experiments discussed in this dissertation is shown in Figure 12. The membrane is the $1 \text{ mm} \times 1 \text{ mm}$ square at the center, and is made of high stress stoichiometric Si_3N_4 with a thickness of 50 nm. The frame around this region is called the membrane “chip” and is made of silicon. The chip is what provides the fixed boundary conditions necessary for the modes described above.

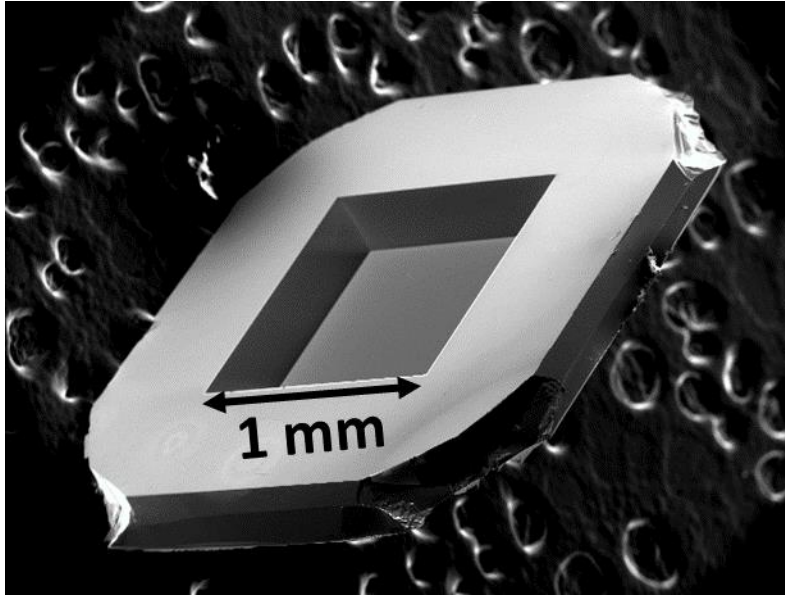


Figure 12: Scanning electron micrograph of a membrane similar to the ones used in the experiments.

Three dimensional plots of the first nine mechanical modes of such a membrane are shown in Figure 13 below.

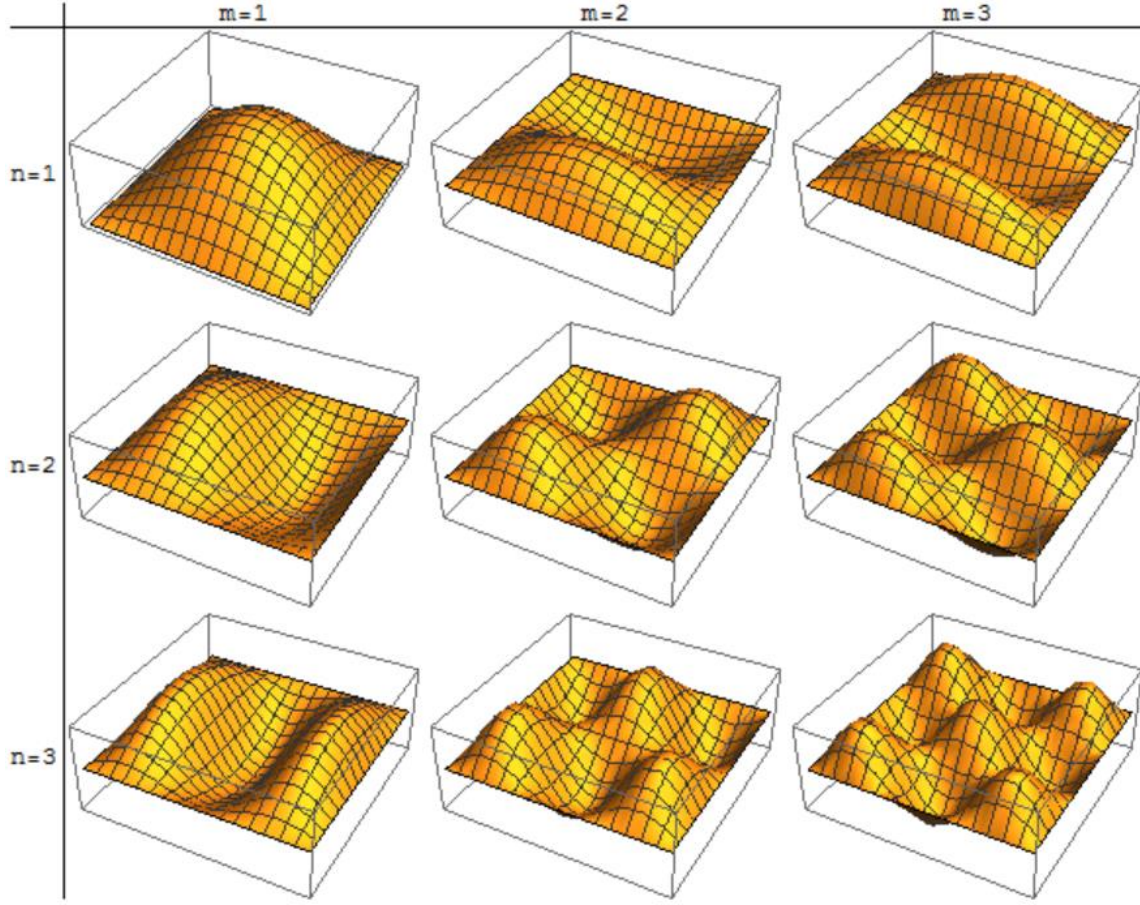


Figure 13: 3D plots of the first nine mechanical modes of a square membrane.

6. The Optomechanical Hamiltonian

The optomechanical interaction between a single membrane mechanical mode and a single cavity optical mode is governed by a Hamiltonian that includes terms for total energy of the mechanical and optical modes. In the simplest case, this is:

$$H = \hbar\omega_c(\hat{x})\hat{a}^\dagger\hat{a} + \hbar\omega_m\hat{c}^\dagger\hat{c} \quad (39)$$

In this expression, \hat{a} is the optical mode photon annihilation operator, \hat{c} is the mechanical mode phonon annihilation operator, and ω_m is the mechanical mode frequency. $\omega_c(\hat{x})$ is the resonant frequency of the optical cavity, which as shown in Figure 6, is a function of the membrane position. The dimensionless membrane position $\hat{x} = (\hat{c}^\dagger + \hat{c})$. To effectively describe the coupling between the

mechanical and optical modes, it is useful to expand $\omega_c(\hat{x})$ in a Taylor series. For a membrane near an arbitrary position within the cavity:

$$\omega_c(\hat{x}) \approx \omega_c + \omega'_c x_{\text{zpf}} \hat{x} + \frac{1}{2} \omega''_c x_{\text{zpf}}^2 \hat{x}^2 + \dots \quad (40)$$

where ω_c , ω'_c , and ω''_c represent derivatives of $\omega_c(\hat{x})$ evaluated at the position of the membrane, and

$x_{\text{zpf}} \equiv \sqrt{\frac{\hbar}{2m\omega_m}}$ is the magnitude of the zero-point fluctuations of the mechanical mode.

Depending on the position of the membrane within the cavity, certain terms of this Taylor expansion may be more or less important. For example, as seen in Figure 6, in between nodes and antinodes of the intracavity field, the cavity detuning as a function of membrane position is predominantly linear, and $\omega_c(\hat{x}) \approx \omega_c + \omega'_c x_{\text{zpf}} \hat{x}$. At points such as these, the coupling between the mechanical and optical modes is said to be “linear,” and its strength is characterized by the linear optomechanical coupling parameter: $g \equiv \omega'_c x_{\text{zpf}}$. At the node or antinode however, the linear term of the Taylor expansion vanishes, and the second-order term of the Taylor expansion becomes significant, $\omega_c(\hat{x}) \approx \omega_c + \frac{1}{2} \omega''_c x_{\text{zpf}}^2 \hat{x}^2$. In this case, the optomechanical coupling is said to be quadratic, and the strength of the coupling is characterized by the quadratic optomechanical coupling parameter $g_2 \equiv \omega''_c x_{\text{zpf}}^2$.

Therefore, changing the position of the membrane within the cavity changes both the strength and functional form of the coupling between the mechanical oscillator and the cavity’s resonant frequency.³⁷ Avoided crossings between cavity modes, as described above, can produce points that exhibit much stronger quadratic coupling than can be obtained at nodes and antinodes of the intracavity field, permitting exploration of a range of new physical effects.^{43, 49}

In this dissertation I will describe two experiments. The first involves the use of a point of linear coupling to demonstrate near ground state cooling of one mechanical mode of the membrane. The

second involves the use of a point of quadratic coupling formed by the avoided crossing between two cavity modes to explore the classical dynamics of the quadratic optomechanical interaction. I shall discuss the details of how the optomechanical Hamiltonian leads to the observed effects in each case in the Theory section of each of following chapters.

III. Ground State Cooling

As discussed in the introduction, cooling a mechanical oscillator to its vibrational ground state is a useful first step in preparing the oscillator to store an arbitrary quantum state. Additionally, cooling an oscillator to the ground state reduces the thermal contribution to the spectrum of the oscillator's motion, which can enhance the visibility of radiation pressure shot noise. Furthermore, the smaller thermal contribution to the oscillator's motion may, with an appropriate measurement, allow for observation of quantum jumps between phonon levels.

The term “ground state cooling” refers to cooling one or more of the membrane's mechanical modes until the phonon occupancy of the mode or modes is close to zero, i.e. the amplitude of motion is close to the zero-point fluctuations of the mode as predicted by quantum mechanics: $A^{(n,m)} \approx x_{\text{zpf}} \equiv \sqrt{\frac{\hbar}{2m\omega_m}}$. If cooling is done by lowering the temperature of the thermal bath (such as by placing the system inside a cryostat), then by the classical equipartition theorem, all of the modes will be cooled to a phonon occupancy of $n_m^{(n,m)} = k_B T / \hbar \omega_m^{(n,m)}$. However, as discussed in the introduction, cryogenic cooling is generally insufficient to cool low-frequency modes of massive mechanical oscillators to their ground states. Sideband cooling of the sort discussed in the introduction does not generally cool all modes by the same amount. Therefore, the term “ground state cooling” in reference to systems cooled by sideband cooling generally refers to efforts targeted at cooling only one mechanical mode of the oscillator.

Though ground state cooling has previously been achieved in many optomechanical systems,^{5,6,7,8,9,10} it is still a goal worth pursuing in the membrane-in-the-middle system. First, the membrane cooled in the experiment described in this dissertation has an effective mass on the order of tens of nanograms, and dimensions on the order of 1 mm^2 , making it visible to the naked eye. The mechanical

frequency of the mode selected for ground state cooling is less than 1 MHz.¹¹ These parameters put this system among the heaviest, lowest frequency mechanical oscillators cooled to the ground state. Cooling such a massive and low-frequency oscillator to the ground state is incredibly challenging because of the low effective temperature that must be achieved. As discussed in the introduction, a \sim MHz frequency oscillator in its ground state has an effective temperature of approximately 50 μ K, necessitating very careful experimental design to isolate the oscillator from the comparatively “hot” cryogenic bath. Also, classical laser noise is generally worse at \sim MHz and lower frequencies, and this further confounds efforts to cool such low-frequency oscillators.

Notwithstanding the technical accomplishments required to cool a membrane oscillator to its ground state, the effort to achieve ground state cooling in such a system is worthwhile because it allows for tests of the predictions of quantum mechanics in a truly macroscopic object. Additionally, a membrane-in-the-middle system has particular advantages over many other optomechanical systems in terms of tunability. As previously discussed, the membrane can be translated, tipped, and tilted within the optical cavity to change the functional form of the coupling between the cavity’s resonant frequency and the mechanical mode. By selecting appropriate points within the cavity spectrum, it is possible to obtain linear coupling to one optical mode, and quadratic coupling to another mode, allowing for ground state cooling while simultaneously exploring the dynamics of non-linear coupling. This flexibility is not found in other systems, and is a major reason why ground state cooling of a membrane-in-the-middle is valuable.

1. Theory

In this section I seek to derive the how sideband cooling of a mechanical oscillator works using a Hamiltonian formalism. While I shall refer to the mechanical oscillator as the “membrane”, this formalism is actually completely general, and could apply to any optomechanics experiment that uses

one laser to measure the motion of the mechanical oscillator, and a second laser to perform sideband cooling. I will begin with the Hamiltonian of the membrane/cavity system, which can be written as:

$$H = \sum_{j=s,p} \hbar(\omega_j + g_j \hat{x}) \hat{a}_j^\dagger \hat{a}_j + \hbar\omega_m \hat{c}^\dagger \hat{c} + H_{\text{drive}} + H_{\text{diss}} \quad (41)$$

In the summation over j , s represents the optical cavity mode addressed by the measurement laser, and p represents the mode addressed by the cooling laser. ω_j represents the frequency of each mode, $g_j = \omega'_j x_{\text{zpf}}$ (where ω'_j is the slope of the cavity's resonant frequency as a function of membrane position, and $x_{\text{zpf}} = \sqrt{\frac{\hbar}{2m\omega_m}}$ is the amplitude of the zero-point fluctuations of the membrane's position) is the optomechanical coupling of membrane to each optical mode, and \hat{a}_j is the photon annihilation operator for each mode. ω_m is the frequency of the membrane's mechanical mode of interest, and \hat{c} is the phonon annihilation operator for that mode. The membrane's position is therefore given by $\hat{x} = x_0 + (\hat{c} + \hat{c}^\dagger)$, where $x_0 = \langle \hat{x} \rangle$ is the membrane's time-averaged position (in units of the zero-point fluctuations) which we may define to be 0. H_{drive} and H_{diss} respectively represent the laser drive, and the coupling rate of both the cavity and the membrane to the outside environment.

The interaction term, $\hbar g_j \hat{x} \hat{a}_j^\dagger \hat{a}_j$ describes how the membrane's motion interacts with the electromagnetic field inside the cavity, as well as how the electromagnetic field interacts with the membrane. In the limit of a strong laser drive and weak coupling such that the membrane induces only small fluctuations in the amplitude of the electromagnetic field in the cavity, the interaction term can be linearized. In this approximation, it is simple to predict the types of interactions that can occur between the mechanical mode and the optical cavity mode(s) by inspection of the Hamiltonian. In the linearized approximation, the cavity field is expressed in terms of an average coherent amplitude \bar{a}_j and fluctuations \hat{d}_j :

$$\hat{a}_j = \bar{a}_j + \hat{d}_j \quad (42)$$

Substituting this into the interaction Hamiltonian gives

$$H_{\text{int}} = \hbar g_j (\bar{a}_j + \hat{d}_j)^\dagger (\bar{a}_j + \hat{d}_j) (\hat{c} + \hat{c}^\dagger) \quad (43)$$

In this expression, there are several terms. The $|\bar{a}_j|^2$ term indicates that there is a static force acting on the membrane due to the average field in the cavity, which results in a constant shift in membrane's position. This term can therefore be removed from the expression by simply redefining x_0 . Terms that are quadratic in \hat{d}_j can be neglected since \hat{d}_j is already small to first order. The remaining terms describe the linear interaction between the mechanical and optical modes:

$$H_{\text{int}}^{\text{lin}} = \hbar g_j (\bar{a}_j^* \hat{d}_j + \bar{a}_j \hat{d}_j^\dagger) (\hat{c} + \hat{c}^\dagger) = \hbar g_j \bar{a}_j (\hat{d}_j \hat{c} + \hat{d}_j \hat{c}^\dagger + \hat{d}_j^\dagger \hat{c} + \hat{d}_j^\dagger \hat{c}^\dagger) \quad (44)$$

The terms $\hat{d}_j \hat{c}^\dagger + \hat{d}_j^\dagger \hat{c}$ represent an exchange of energy between optical mode photons and mechanical mode phonons, while the $\hat{d}_j \hat{c} + \hat{d}_j^\dagger \hat{c}^\dagger$ terms represent a two-mode squeezing interaction in which the creation/annihilation of photons in the optical mode is correlated with creation/annihilation (respectively) of phonons in the mechanical mode. In the photon/phonon exchange processes, the $\hat{d}_j^\dagger \hat{c}$ term is referred to as the anti-Stokes process, and represents the annihilation of phonons in the membrane in exchange for the creation of blue detuned photons. This term is the origin of optomechanical cooling and the blue sideband in the optical spectrum. The $\hat{d}_j \hat{c}^\dagger$ process is referred to as the Stokes process, and represents the creation of phonons in the membrane in exchange for a photon from the optical beam. This term is the origin of red sideband in the optical spectrum.

The rotating wave approximation provides a means of determining which of these four processes is expected to occur as a function of drive laser detuning from the optical cavity mode. The rotating wave approximation is a common method used in optical physics, in which the Hamiltonian is rewritten in a reference frame that is oscillating at the frequency of the laser drive. To apply the rotating

wave approximation it is first necessary to explicitly express the phase (frequency) of each operator in the Hamiltonian, which can be accomplished by making the following substitutions: the photon annihilation operator can be rewritten as $\hat{d}_j \rightarrow \hat{d}_j e^{i\omega_j t}$, and the phonon annihilation operator can be rewritten as $\hat{c} \rightarrow \hat{c} e^{i\omega_m t}$. Next, to move into a rotating frame, the laser drive frequency Ω_j gets factored out of the photon operators, i.e.: $\hat{d}_j \rightarrow \hat{d}_j e^{i(\omega_j - \Omega_j)t} = \hat{d}_j e^{-i\Delta_j t}$, where Δ_j is the laser's detuning from the cavity mode. Therefore, in the rotating wave approximation, the linearized Hamiltonian takes the following form:

$$H_{\text{int}}^{\text{lin}} = \hbar g_j \bar{a}_j \left(\hat{d}_j \hat{c}^\dagger e^{i(-\Delta_j - \omega_m)t} + \hat{d}_j^\dagger \hat{c} e^{i(\Delta_j + \omega_m)t} + \hat{d}_j \hat{c} e^{i(-\Delta_j + \omega_m)t} + \hat{d}_j^\dagger \hat{c}^\dagger e^{i(\Delta_j - \omega_m)t} \right) \quad (45)$$

In the rotating wave approximation, terms with rapidly oscillating phase are assumed to average out to zero. For example, if the laser detuning $\Delta_j = -\omega_m$, the terms $\hat{d}_j \hat{c}^\dagger e^{i(-\Delta_j - \omega_m)t} + \hat{d}_j^\dagger \hat{c} e^{i(\Delta_j + \omega_m)t}$ have a fixed phase, while the terms $\hat{d}_j \hat{c} e^{i(-\Delta_j + \omega_m)t} + \hat{d}_j^\dagger \hat{c}^\dagger e^{i(\Delta_j - \omega_m)t}$ are rapidly oscillating with a phase of $e^{\pm 2i\omega_m t}$. Therefore, the rotating wave approximation predicts that the photon-phonon exchange terms are enhanced, while the two-mode squeezing terms are suppressed. Conversely, when blue detuned, the two-mode squeezing terms are enhanced, and the photon/phonon exchange terms are rapidly oscillating and suppressed.⁵⁰

While the Stokes ($\hat{d}_j \hat{c}^\dagger$) and anti-Stokes ($\hat{d}_j^\dagger \hat{c}$) processes are equally weighted in the linearized Hamiltonian and are both in phase with the rotating frame, the fact that the laser is red detuned relative to the cavity mode breaks the symmetry of the two processes, making the anti-Stokes process more likely. Fermi's golden rule dictates that the transition probability between two energy eigenstates of a system is proportional to the density of states in the final configuration (i.e., after the transition). For the anti-Stokes process, a photon absorbs a phonon and becomes blue-shifted. For the Stokes process, a photon gives up energy to produce a phonon and becomes red-shifted. As shown in the left side of Figure 4 in the introduction, at a laser detuning of $\Delta_j = -\omega_m$ for example, the blue-shifted photon is on-

resonance with the cavity, while the red-shifted photon is detuned by $-2\omega_m$. Due to the cavity susceptibility, the density of states is enhanced on resonance, and suppressed at large detuning. Therefore, the anti-Stokes process is preferred over the Stokes process, leading to net laser cooling.⁵⁰

The general equations of motion for the system can be found using input-output theory, described in detail in Refs.[39, 50]. The equations of motion are:⁵¹

$$\dot{\hat{a}}_j = -\left(\frac{\kappa_j}{2} + i\omega_j\right)\hat{a}_j - ig_j\hat{x}\hat{a}_j + \sqrt{\kappa_{j,\text{in}}}\hat{a}_{j,\text{in}} + \sqrt{\kappa_{j,\text{int}}}\hat{\xi}_j \quad (46)$$

$$\dot{\hat{c}} = -\left(\frac{\gamma}{2} + i\omega_m\right)\hat{c} - i \sum_j g_j \hat{a}_j^\dagger \hat{a}_j + \sqrt{\gamma}\hat{\eta} \quad (47)$$

In Equation (46), $\kappa_{j,\text{int}}$ represents the decay rate of each cavity mode due to absorption or scattering within the cavity, and $\kappa_{j,\text{in}}$ represents the decay rate of those modes through the cavity input mirror. $\kappa_j = \kappa_{j,\text{int}} + \kappa_{j,\text{in}}$ is defined as the total optical decay rate. $\hat{\xi}_j$ describes vacuum noise within each cavity mode and obeys the relations

$$\begin{aligned} \langle \hat{\xi}_j^\dagger(t) \hat{\xi}_j(t') \rangle &= 0 \\ \langle \hat{\xi}_j(t) \hat{\xi}_j^\dagger(t') \rangle &= \delta(t - t') \delta_{j,j'} \end{aligned} \quad (48)$$

The laser drives can be expressed as

$$\hat{a}_{j,\text{in}}(t) = e^{-i\Omega_j t} \left[K_j + \frac{1}{2} (\delta x_j(t) + i\delta y_j(t)) \right] + \hat{\xi}_{j,\text{in}} \quad (49)$$

Where $K_j^2 = P_j/\hbar\Omega_j$ is the incident photon flux in the measurement or the cooling laser drive in units of photons per second. The terms $\delta x_j(t)$ and $\delta y_j(t)$ represent the classical amplitude and phase noise of each laser drive, respectively. As we are only concerned with these noises in a relatively narrow bandwidth, we treat them as white noise, obeying the following criteria:

$$\begin{aligned}
\langle \delta x_j(t) \delta x_{j'}(t') \rangle &= C_{j,xx} \delta(t - t') \delta_{j,j'} \\
\langle \delta x_j(t) \delta y_{j'}(t') \rangle &= C_{j,xy} \delta(t - t') \delta_{j,j'} \\
\langle \delta y_j(t) \delta y_{j'}(t') \rangle &= C_{j,yy} \delta(t - t') \delta_{j,j'}
\end{aligned} \tag{50}$$

The real coefficients $C_{j,xx}$, $C_{j,xy}$, and $C_{j,yy}$ describe the magnitude of the laser noise relative to vacuum noise (e.g., a coefficient of 1 implies noise magnitude equal to vacuum noise), and scale linearly with laser power. By the Cauchy-Bunyakovsky-Schwartz inequality, $C_{j,xy}^2 \leq C_{j,xx} C_{j,yy}$.

The $\hat{\xi}_{j,\text{in}}$ term in Equation (49) represents the optical vacuum noise in the laser drive and like $\hat{\xi}_j$, has the following properties:

$$\begin{aligned}
\langle \hat{\xi}_{j,\text{in}}^\dagger(t) \hat{\xi}_{j,\text{in}}(t') \rangle &= 0 \\
\langle \hat{\xi}_{j,\text{in}}(t) \hat{\xi}_{j,\text{in}}^\dagger(t') \rangle &= \delta(t - t') \delta_{j,j'}
\end{aligned} \tag{51}$$

In Equation (47), γ represents the coupling of the mechanical mode of the membrane to the thermal bath. $\hat{\eta}$ represents thermal noise from the bath and obeys the relations

$$\langle \hat{\eta}(t) \hat{\eta}^\dagger(t') \rangle \approx \langle \hat{\eta}^\dagger(t) \hat{\eta}(t') \rangle = n_{\text{th}} \delta(t - t') \tag{52}$$

In Equation (52), $n_{\text{th}} \approx k_B T / \hbar \omega_m$ is the phonon number of the mechanical mode in the absence of any laser drive.

In the limit of a strong laser drive and weak optomechanical coupling, the equations of motion can be linearized by making the same substitution used to linearize the Hamiltonian (i.e. Equation (42)):

$$\hat{a}_j(t) = e^{-i\Omega_j t} (\bar{a}_j + \hat{d}_j(t)) \tag{53}$$

Here, $\bar{a}_j = \frac{\sqrt{\kappa_{j,\text{in}} K_j}}{\kappa_j/2 - i\Delta_j}$, where $\Delta_j = \Omega_j - \omega_j - g_j x_0$ represents the detuning of the laser from the cavity mode after the mode has been shifted due to the presence of the membrane.

The optical and mechanical resonances are characterized by their susceptibilities, which have the following forms:

$$\chi_{j,c}[\omega] = \frac{1}{\frac{\kappa_j}{2} - i(\omega + \Delta_j)} \quad (54)$$

$$\chi_m[\omega] = \frac{1}{\frac{\gamma}{2} - i(\omega - \omega_m)} \quad (55)$$

The linearized equations of motion can then be written as:

$$\hat{d}_j[\omega] = \chi_{j,c}[\omega](\zeta_j[\omega] - i\alpha_j\hat{z}[\omega]) \quad (56)$$

$$\begin{aligned} \hat{z}[\omega] = \frac{1}{N[\omega]} & \left[\sqrt{\gamma}(\chi_m^{-1*}[-\omega]\eta[\omega] + \chi_m^{-1}[\omega]\eta^\dagger[\omega]) \right. \\ & \left. - 2\omega_m \sum_j (\alpha_j^* \chi_{j,c}[\omega]\zeta_j[\omega] + \alpha_j \chi_{j,c}^*[-\omega]\zeta_j^\dagger[\omega]) \right] \end{aligned} \quad (57)$$

In these expressions,

$$\alpha_j = g_j \bar{a}_j \quad (58)$$

$$\zeta_j[\omega] = \sqrt{\kappa_{j,\text{in}}} \left[\frac{1}{2} (\delta x_j[\omega] + i\delta y_j[\omega]) + \hat{\xi}_{j,\text{in}}[\omega] \right] + \sqrt{\kappa_{j,\text{int}}} \hat{\xi}_j[\omega] \quad (59)$$

$$N[\omega] = \chi_m^{-1}[\omega]\chi_m^{-1*}[-\omega] - 2i\omega_m \sum_j |\alpha_j|^2 (\chi_{j,c}[\omega] - \chi_{j,c}^*[-\omega]) \quad (60)$$

The expression for $N[\omega]$ can be rewritten as

$$N[\omega] = \chi_m^{-1}[\omega]\chi_m^{-1*}[-\omega] - 2\omega_m \Sigma[\omega] \quad (61)$$

where $\Sigma[\omega] = -i|\alpha_j|^2 (\chi_{j,c}[\omega] - \chi_{j,c}^*[-\omega])$.

To grasp the expected behavior of the mechanical mode from the equations of motion, it is helpful to make some approximations. First, it is useful to note that for frequencies $\omega \approx \omega_m$ and high quality factor $Q = \omega_m/\gamma \gg 1$, the mechanical susceptibility $\chi_m^{-1}[-\omega] = \gamma/2 + i(\omega + \omega_m) \approx 2i\omega_m \gg$

$\chi_m^{-1}[\omega]$. Therefore, the first term in Equation (57) can be simplified to $N^{-1}[\omega]\sqrt{\gamma}(\chi_m^{-1*}[-\omega]\eta[\omega])$. The denominator of this expression is given by

$$N^{-1}[\omega]\chi_m^{-1*}[-\omega] = \gamma/2 - i(\omega - \omega_m) + i\Sigma[\omega] \quad (62)$$

Thus, the mechanical damping rate γ and the mechanical resonant frequency ω_m are modified by the $\Sigma[\omega]$ term, which we will call the optomechanical self-energy. In the limit that $\gamma \ll \kappa$, we can approximate that for frequencies at which the membrane can respond ($\omega \approx \omega_m$) the self energy $\Sigma[\omega]$ is nearly constant, and therefore $\Sigma[\omega] \approx \Sigma[\omega_m]$. The change in resonant frequency is then given by:

$$\delta_j = \text{Re}[\Sigma[\omega_m]] = 2|\chi_{j,c}[\omega_m]|^2 |\chi_{j,c}[-\omega_m]|^2 \Delta_j |\alpha_j|^2 \left[\left(\frac{\kappa_j}{2} \right)^2 - \omega_m^2 + \Delta_j^2 \right] \quad (63)$$

This change in resonant frequency is referred to as the “optical spring.” The extra damping provided by the optomechanical interaction is given by:

$$\gamma_j = -2\text{Im}[\Sigma[\omega_m]] = -4|\chi_{j,c}[\omega_m]|^2 |\chi_{j,c}[-\omega_m]|^2 \Delta_j |\alpha_j|^2 \kappa_j \omega_m \quad (64)$$

This effect is referred to as the “optical damping.”

The phonon number of the membrane is given by the expectation value of the number operator $\hat{c}^\dagger \hat{c}$. In the limit of weak coupling such that $|\bar{\alpha}_j| \ll \kappa_j$, it can be shown⁴¹ that the phonon number is

$$n_m \equiv \langle \hat{c}^\dagger \hat{c} \rangle = \frac{\gamma n_{\text{th}} + \sum_j \gamma_j n_j}{\tilde{\gamma}} \quad (65)$$

where $\tilde{\gamma} = \gamma + \sum_j \gamma_j$ is the total damping of the mechanical oscillator, and n_j is an effective temperature of each optical mode, determined by the amount of classical laser noise and radiation pressure shot noise present. Defining $B_{j,\pm}[\omega] = e^{-i\phi_j} \chi_{j,c}[\omega] \pm e^{i\phi_j} \chi_{j,c}^*[-\omega]$ and $e^{i\phi_j} = \alpha_j / |\alpha_j|$, n_j can be expressed as:⁵¹

$$n_j = \frac{1}{4} \left\{ \kappa_{j,\text{in}} \left[|B_{j,+}[\omega_m]|^2 C_{j,xx} + |B_{j,-}[\omega_m]|^2 C_{j,yy} + 2 \text{Im}(B_{j,+}[\omega_m] B_{j,-}^*[\omega_m]) C_{j,yy} \right] + \kappa_j |\chi_{j,c}[-\omega_m]|^2 \right\} \left(-4 |\chi_{j,c}[\omega_m]|^2 |\chi_{j,c}[-\omega_m]|^2 \Delta_j \kappa_j \omega_m \right)^{-1} \quad (66)$$

2. Heterodyne Detection

With these expressions for the behavior of the membrane and the electromagnetic field, it is now useful to consider how we actually measure the membrane's motion. In this experiment, we use a heterodyne detection scheme which consists of a weak measurement beam that is locked on resonance with the cavity, and a strong local oscillator beam detuned by frequency ω_{if} . Therefore, we modify Equation (49) to include a term for the local oscillator. In this discussion, except where it may introduce ambiguity, I shall drop the j subscript since only the measurement beam is used for heterodyne detection of the membrane's motion.

$$\hat{a}_{\text{in}}(t) = e^{-i\Omega t} \left[K + \frac{1}{2} (\delta x(t) + i\delta y(t)) \right] (1 + \sqrt{r} e^{i(\omega_{\text{if}} t + \theta)}) + \xi_{\text{in}} \quad (67)$$

The term $\sqrt{r} e^{i(\omega_{\text{if}} t + \theta)}$ represents the local oscillator beam. Since the local oscillator is much higher power than the measurement beam, $r = \left(\frac{P_{\text{lo}}}{P_s} \right) \times \frac{\omega_s}{\omega_s + \omega_{\text{if}}} \approx \frac{P_{\text{lo}}}{P_s} \gg 1$ describes the relative amount of power in the local oscillator. The term θ in the exponential describes the phase of the local oscillator beam relative to the measurement beam, though it is a quantity which generally has no physical effect on the measurement. It is assumed that the classical laser noise, represented by $\delta x(t)$ and $i\delta y(t)$, is common-mode to both beams. (This is a reasonable assumption because the two beams originate from the same laser, in practice.)

Since the local oscillator beam is far detuned from the cavity resonance, it is promptly reflected and does not significantly leak into the cavity or interact with the membrane. Therefore, the local oscillator can simply be added as is to the cavity output field:

$$\bar{a}_{\text{out}}(t) = -K(\rho + \beta\sqrt{r}e^{i(\omega_{\text{if}} + \theta)}) \quad (68)$$

Here, the term $\rho = \beta - \frac{\kappa_{\text{in}}}{\kappa/2 + i\Delta}$ describes the amplitude of the reflected measurement beam after interacting with the cavity, with β as a parameter to account for the fact that promptly reflected light will not necessarily be mode-matched with light that coupled into the cavity mode, and therefore may experience a different amount of loss before detection than light that reflected from within the cavity. (In the context of the experiment, β describes the efficiency with which light promptly reflected from the cavity recouples into the single mode optical fiber that brought the light to the cavity originally.) The second term, of course represents the fully reflected local oscillator beam. By expressing the output field as a sum of an average field amplitude plus fluctuations around that amplitude, using the same notation as in Equation (53):

$$\hat{d}_{\text{out}} = \sqrt{\kappa_{\text{in}}}\hat{d}(t) - \frac{1}{2}(\delta x(t) + i\delta y(t))(1 + \beta\sqrt{r}e^{i(\omega_{\text{if}} + \theta)}) - \hat{\xi}_{\text{in}}(t) \quad (69)$$

In this equation, $\hat{d}(t)$ is the linearized solution to the cavity equations of motion as given by Equation (56).

The measured heterodyne spectrum will be proportional to the spectrum of photocurrent produced at the photodiode used to measure the light reflected from the optical cavity. The photocurrent spectrum is defined as:

$$S[\omega] = \lim_{T \rightarrow \infty} \frac{1}{T} \int_{-T/2}^{T/2} dt \int_{-\infty}^{\infty} d\tau e^{i\omega\tau} \overline{i(t)i(t+\tau)} \quad (70)$$

where $i(t)$ is the photocurrent, and the current autocorrelation function is:

$$\overline{i(t)i(t+\tau)} = G^2 \left(\sigma^2 \langle \hat{I}(t)\hat{I}(t+\tau) \rangle + \sigma \langle \hat{I}(t) \rangle \delta(t) \right) \quad (71)$$

In Equation (71), $\hat{I}(t) = \hat{a}_{\text{out}}^\dagger(t)\hat{a}_{\text{out}}(t)$ is the photon flux, and the colons represent normal and time ordering. The parameter σ represents the dimensionless detection efficiency, i.e. the percentage of

photons in \hat{a}_{out} which reach the photodiode and are detected. G is the gain of the photodiode in units of current produced per number of incident photons. The final term $\sigma\langle\hat{I}(t)\rangle\delta(t)$ is due to the self-correlation of photoelectric pulses.

Since we are generally only interested in the portion of the spectrum containing information about the membrane's oscillations, we can compute the spectrum only in the vicinity of the beat note between the local oscillator and the measurement beam. Therefore, we consider only terms of \hat{I} that oscillate around the frequency ω_{if} . We find that the detected spectrum, for the red sideband, takes the form:

$$S_{rr} = G_r^2 \sigma \beta^2 r K^2 \left[F_{rr} + \frac{\tilde{\gamma} L_{rr} + (\omega - \omega_r) A_{rr}}{(\tilde{\gamma}/2)^2 + (\omega - \omega_r)^2} \right] \quad (72)$$

In this expression, G_r is the photodetector gain at frequency $\omega_r = \omega_{if} - \tilde{\omega}_m$, where $\tilde{\omega}_m = \omega_m - \sum_j \delta_j$ is the frequency of the mechanical mode as perturbed by the interaction with the lasers. As defined in the last section, $\tilde{\gamma} = \gamma + \sum_j \gamma_j$ is the effective linewidth of the mechanical mode, i.e. the linewidth as modified by interaction with the lasers. The parameters F_{rr} , L_{rr} , and A_{rr} respectively represent the white noise background consisting of classical laser noise and shot noise, the area of a Lorentzian corresponding to the motion of the membrane, and the amplitude of an antisymmetric component which arises due to correlations between the membrane's motion and classical noise on the measurement beam. In the convention of Ref. [41], I shall refer to the antisymmetric component as an "anti-Lorentzian."

The noise floor F_{rr} takes the following form:

$$F_{rr} = 1 + \frac{\sigma}{4} \left[(|\rho|^2 + |\kappa_{in} \chi_c[-\omega_m] - \beta|^2) (C_{xx} + C_{yy}) - 2 \text{Re}[\rho^* (\kappa_{in} \chi_c[-\omega_m] - \beta) (C_{xx} + 2iC_{xy} - C_{yy})] \right] \quad (73)$$

In this expression, all noises have been normalized to shot noise – therefore the 1 at the beginning of the expression is shot noise itself. The other terms represent contributions from classical laser noise, and scale linearly in power.

The Lorentzian component of Equation (72) is

$$L_{rr} = \sigma \kappa_{\text{in}} |\alpha|^2 |\chi_c[-\omega_m]|^2 (n_m + 1) + \text{Re} \tilde{B}[\omega_m] \quad (74)$$

\tilde{B} represents the effects of classical noise on L_{rr} (and A_{rr}). Using the definition of $B_{\pm}[\omega]$ from the last section, \tilde{B} takes the form:

$$\begin{aligned} \tilde{B}[\omega] = & \frac{\kappa_{\text{in}}}{4} |\chi_c[-\omega]|^2 e^{-i\phi} \left[(C_{xx} + iC_{xy})B_+[\omega] + (iC_{xy} - C_{yy})B_-[\omega] \right] \\ & - \frac{\beta}{4} \chi_c^*[-\omega] e^{-i\phi} \left[(C_{xx}B_+[\omega] + iC_{xy}B_-[\omega])(1 + \rho) \right. \\ & \left. + (iC_{xy}B_+[\omega] - C_{yy}B_-[\omega])(1 - \rho) \right] \end{aligned} \quad (75)$$

The anti-Lorentzian component of Equation (72) is:

$$A_{rr} = 2\sigma \kappa_{\text{in}} |\alpha|^2 \text{Im} \tilde{B}[\omega_m] \quad (76)$$

The heterodyne spectrum of the blue sideband has a form very similar to that of the red sideband, except for a few subtle differences: namely, the sign of ω_m is switched since this sideband occurs at a frequency higher than ω_{if} , \tilde{B} is subtracted instead of added, and $(n_m + 1) \rightarrow n_m$ in the expression for L_{bb} . Explicitly, these expressions are:

$$S_{bb} = G_b^2 \sigma \beta^2 r K^2 \left[F_{rr} + \frac{\tilde{\gamma} L_{rr} + (\omega - \omega_b) A_{rr}}{(\tilde{\gamma}/2)^2 + (\omega - \omega_b)^2} \right] \quad (77)$$

In Equation (77), analogous to ω_r , $\omega_b = \omega_{\text{if}} + \tilde{\omega}_m$.

$$\begin{aligned} F_{bb} = & 1 + \frac{\sigma}{4} \left[(|\rho|^2 + |\kappa_{\text{in}} \chi_c[\omega_m] - \beta|^2) (C_{xx} + C_{yy}) \right. \\ & \left. - 2\text{Re}[\rho^* (\kappa_{\text{in}} \chi_c[\omega_m] - \beta) (C_{xx} + 2iC_{xy} - C_{yy})] \right] \end{aligned} \quad (78)$$

$$L_{bb} = \sigma\kappa_{\text{in}}|\alpha|^2|\chi_c[\omega_m]|^2(n_m) - \text{Re}\tilde{B}[-\omega_m] \quad (79)$$

$$A_{bb} = -2\sigma\kappa_{\text{in}}|\alpha|^2 \text{Im}\tilde{B}[-\omega_m] \quad (80)$$

So far in this discussion I have neglected the cooling beam since it does not contribute any information about the membrane's motion to the photocurrent spectrum around ω_{if} . However, it does still make a white contribution to the spectrum from shot noise. This contribution takes the form

$$\begin{aligned} S_{rr,p} &= G_r^2 \sigma \beta^2 K_p^2 |\rho_p|^2 \\ S_{bb,p} &= G_b^2 \sigma \beta^2 K_p^2 |\rho_p|^2 \end{aligned} \quad (81)$$

for the red and the blue sidebands respectively. These expressions may be added to Equations (72) and (77) to produce the full expressions for the heterodyne spectrum in cases where absolute calibration the white background is important.

The origin and interpretation of the asymmetry in the dependence on n_m in the expressions for L_{rr} and L_{bb} are non-trivial, and are treated in Ref.[8]. Summarized briefly, for a linear power detector like the photodiode used in this experiment, there are four physical contributions to S_{rr} and S_{bb} :

1. Quantum fluctuations of the electromagnetic field, which manifest as shot noise.
2. Thermal motion of the mechanical oscillator, which produces sidebands of equal area, each proportional to the mean phonon number of the mechanical oscillator $n_m \approx k_B T / \hbar \omega_m$.
3. Zero-point motion of the oscillator, which contributes equally to each of the sidebands by an amount equivalent to increasing the oscillator's mean phonon number by $\frac{1}{2}$.
4. The response of the oscillator to the quantum fluctuations of the electromagnetic field, which contributes to the red sideband by an amount equivalent to *increasing* the mean phonon number by $\frac{1}{2}$, and to the blue sideband by an amount equivalent to *decreasing* the mean phonon number by $\frac{1}{2}$.

Contributions 2-4 together indicate that L_{rr} should have an area that is equal to the area that would be expected if the oscillator had one more phonon than it really does, while L_{bb} should scale linearly with phonon number all the way down to zero. These contributions are summarized schematically in Figure 14.

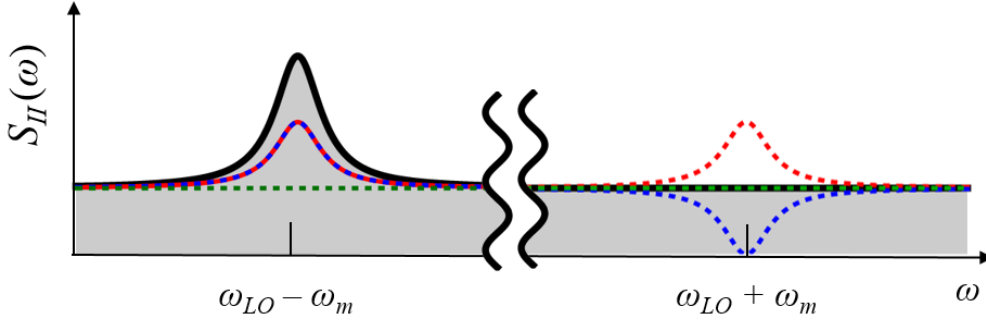


Figure 14: Schematic representation of contributions to the red (left) and blue (right) sidebands for a membrane with phonon number $n_m = 0$. The dashed green line represents the contribution due to the quantum fluctuations of the electromagnetic field (shot noise), the red dashed line represents the contribution due to the oscillator's zero-point motion, and the blue represents the contribution due to the response of the oscillator to the quantum fluctuations of the electromagnetic field. Since this diagram is for $n_m = 0$, there is no contribution from the oscillator's thermal motion.

Using this description of the asymmetry of the sidebands, it is possible to compute the mean phonon number by comparing the ratio of the sideband areas. This can be done by subtracting the F_{rr} and F_{bb} background terms from Equations (72) and (77), normalizing the gain of one sideband to the gain of the other, then integrate the remaining terms over all frequency space. The result of this integral is referred to as the “sideband weight”, which is essentially just L_{rr} and L_{bb} normalized so that they may be directly compared.⁵¹ Taking the ratio of the red sideband weight to the blue sideband weight gives the sideband ratio, which we define as ξ :

$$\xi = \frac{W_{rr}}{W_{bb}} = \frac{|\chi_c[-\omega_m]|^2 (n_m + 1) + \text{Re } \tilde{B}[\omega_m]}{|\chi_c[\omega_m]|^2 n_m - \text{Re } \tilde{B}[-\omega_m]} \quad (82)$$

In the absence of classical laser noise, $\tilde{B} \equiv 0$. If the measurement laser is exactly on resonance with the optical cavity (measurement beam detuning $\Delta_s = 0$), then $|\chi_c[\omega]|^2$ is an even function and the sideband ratio ξ depends only on the mean phonon number:

$$\xi = \frac{W_{rr}}{W_{bb}} \Big|_{\substack{\tilde{B} \equiv 0 \\ \Delta_s = 0}} = \frac{n_m + 1}{n_m} \quad (83)$$

Equations (82) and (83) can be used to determine the mean phonon number of the mechanical mode. However, it is important to note that using the simplified definition of ξ in Equation (83) can lead to incorrect results if the assumptions of $\tilde{B} \equiv 0$ and $\Delta_s = 0$ are wrong. A non-zero \tilde{B} makes W_{bb} smaller and W_{rr} bigger, yielding a larger asymmetry than would otherwise be expected. Conversely, when $\Delta_s < 0$, as it usually is in real measurements for stability purposes, W_{bb} becomes slightly larger, and W_{rr} becomes slightly smaller, reducing the ratio. To extract the mean phonon number from the sideband asymmetry, then, requires careful calibration of classical laser noise, the detuning of the measurement beam, and the gains of the red and blue sidebands at the photodiode and the ensuing electronics.

Another way in which the mean phonon number of the mechanical mode can be estimated from the heterodyne spectrum is from the linewidth of the mechanical Lorentzian. As already stated, the linewidth represents the total damping of the mechanical mode: $\tilde{\gamma} = \gamma + \sum_j \gamma_j$. Equation (65) from the Theory section connects this to the mean phonon number. The values of γ_j and n_j in Equation (65) can be computed based on estimates of laser power and detuning in order to compute a phonon number based on linewidth.

3. Experimental Setup

A. Mechanical Mode

The membrane used in this experiment is a commercial high-stress stoichiometric Si_3N_4 membrane from Norcada⁵² with dimensions of 1 mm x 1 mm x 50 nm, model NX5100AS. The fundamental mechanical frequency of the membrane is approximately 354 kHz. However, the mechanical mode used in the experiment was the membrane's (2, 2) vibrational mode at $\omega_m/2\pi = 705$ kHz. This mode was selected for several reasons: the mode had (a) higher mechanical quality factor

than the fundamental mode, (b) better optomechanical coupling than the fundamental mode, and (c) higher frequency than the fundamental mode, which puts the system more in the resolved sideband regime (enhancing laser cooling), and reduces problems due to classical laser noise since laser noise is typically worse at low frequencies. The shape of this mode is shown in Figure 15.

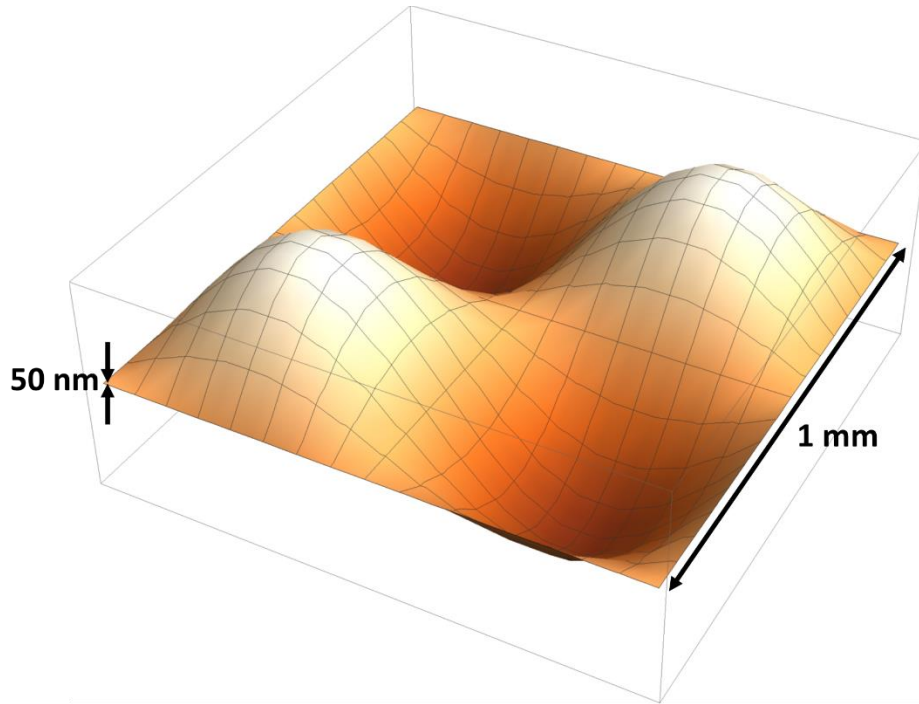


Figure 15: 3D plot of the (2, 2) vibrational mode of the membrane, with phase selected such that antinodes are at their maximum deviations from equilibrium. The vertical axis has been scaled to make the oscillation easier to see – in reality at 4 K the amplitude of the oscillation is only about 10 fm.

B. Optical Setup

Two lasers were used in the experiment. Both were free-space continuous wave 1064 nm Innolight Prometheus Nd:YAG lasers. These lasers were selected for their excellent phase and amplitude noise performance. A summary schematic of the optical setup is shown below.

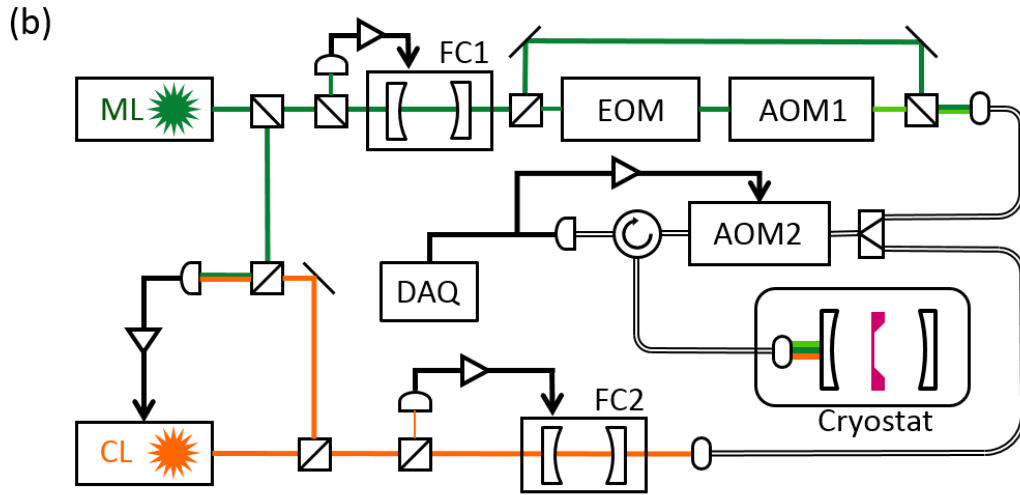


Figure 16: Schematic of the optical setup for the ground state cooling experiment.

The measurement laser, labelled “ML” in Figure 16, first passes through a beamsplitter which picks off a small percentage of the beam to use as a reference for measuring the frequency difference between the measurement laser and the cooling laser (labelled “CL” in Figure 16). The rest of the measurement laser passes through an optical filter cavity labelled “FC1”. (I will explain more about how the filter cavity works in the next section.) The filter cavity is locked to the measurement laser using a Pound-Drever-Hall locking technique,³⁸ with feedback going to a piezo actuator that changes the length of the optical cavity to keep it on resonance with the laser. Light that transmits through the filter cavity goes through a beamsplitter which breaks it into two paths. One path goes through an electrooptic modulator (EOM) which produces phase modulation sidebands for Pound-Drever-Hall locking to the cryogenic optical cavity. It also passes through an acoustooptic modulator (AOM1), which shifts its frequency by 80 MHz so that it can be used for heterodyne detection. This beam is referred to as the “measurement beam” or the “probe beam”. The other path from the beamsplitter doesn’t pass through any modulators, and is used as and referred to as the “local oscillator” beam. The two beams are recombined in another beam splitter, before being coupled into an optical fiber.

Once in the optical fiber, the measurement laser passes through a wide bandwidth fiber acoustooptic modulator (AOM2). The servo used to keep the measurement beam locked to the cryogenic optical cavity works by changing the drive frequency of AOM2. After passing through AOM2, the measurement beam goes through a fiber optic circulator which sends the beam into the cryostat. Once in the cryostat, the measurement beam enters the optical cavity and interacts with the membrane, acquiring phase modulation sidebands containing information about the membrane's motion at $\pm\omega_m$. The local oscillator beam goes into the cryostat, but is 80 MHz away from the cavity resonance, and promptly reflects from the input mirror. Both beams recouple into the same optical fiber that took them into the cryostat originally, and are diverted by the fiber optic circulator to a Thorlabs PDA10CF photodiode for measurement. In Figure 16, the measurement apparatus is labelled "DAQ".

The cooling laser, like the measurement laser, has a small percentage picked off to combine with the light that was picked off from the measurement laser. The two beams combine using a beamsplitter and land on a fast photodiode which can detect the several GHz beat note between the two lasers. The signal from this photodiode goes through a mixing circuit and then to a servo which feeds back to the laser piezo in the cooling laser to lock it a fixed frequency difference away from the measurement laser (more information about this in the Laser Locking Setup section). The rest of the cooling laser passes through another filter cavity, labelled "FC2", which is locked to the cooling laser using a similar technique to FC1. After passing through the filter cavity, the cooling laser goes through some waveplates and a polarizer (not shown in Figure 16), which are used to adjust the optical power of the cooling beam. The cooling beam then couples into an optical fiber and is combined with the measurement laser using a polarizing beam combiner, following the same path into the cryostat and back. Since the cooling beam passes through AOM2 just like the measurement beam, it acquires the same frequency shifts as the measurement beam, allowing it to track the fluctuations of the cold cavity.

The combined spectrum of all the lasers relative to the cavity modes are shown in Figure 17. The measurement (probe) beam is locked on resonance with one optical mode of the cavity, and acquires sidebands at the frequency of the mechanical mode (705 kHz). The local oscillator beam is detuned by 80 MHz, as already discussed. The cooling beam is detuned by approximately the mechanical mode frequency from a different optical cavity resonance that is two free spectral ranges away from the measurement beam (about 8 GHz). The reason for using a different cavity mode for cooling is to prevent creation of a ~ 705 kHz beat note between the probe and cooling lasers within the cavity, which would drive the mechanical mode. It is also worth noting that the cooling beam acquires sidebands from its interaction with the mechanical mode just like the measurement beam does – in fact, the presence of these sidebands is essential for laser cooling. Our heterodyne measurement scheme does not detect them, however, since the local oscillator beam is $8\text{ GHz} + 80\text{ MHz}$ away.

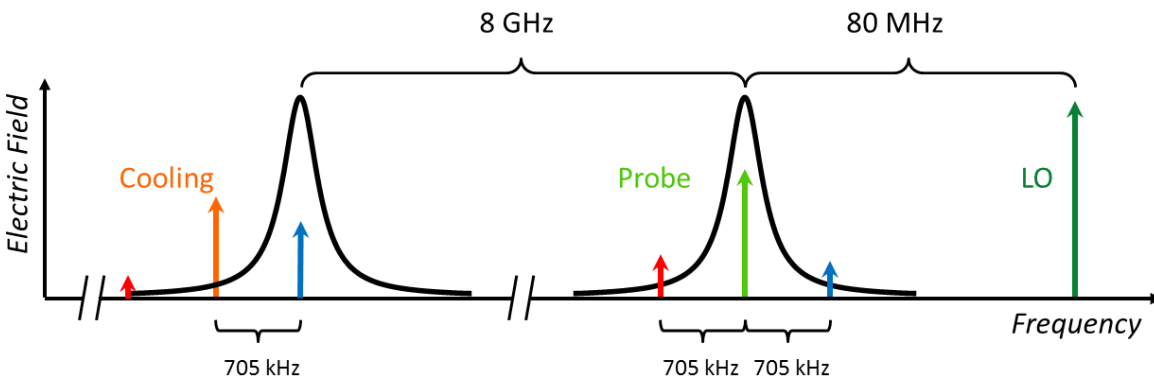


Figure 17: Combined spectrum of the lasers showing their detunings relative to the cavity resonances.

When the lasers fall on the measurement photodiode, they produce the photocurrent spectrum shown in Figure 18. The DC level of the photocurrent scales proportionally to the total incident power. The shot noise background (not shown in Figure 18) gets contributions from all of the beams, and its power spectral density also scales proportionally to the total power. There is a beat note at 80 MHz caused by interference of the probe and local oscillator beams at 80 MHz, and this beat note has sidebands at ± 705 kHz, corresponding to interference between the probe beam sidebands and the local

oscillator. There are also beat notes corresponding to interference between the cooling beam and the other beams, but at ~ 8 GHz they are outside the 150 MHz bandwidth of the Thorlabs PDA10CF photodiode used for detection. Finally, there are beat notes at 705 kHz corresponding to the beating of the mechanical sidebands with the measurement and cooling beams.

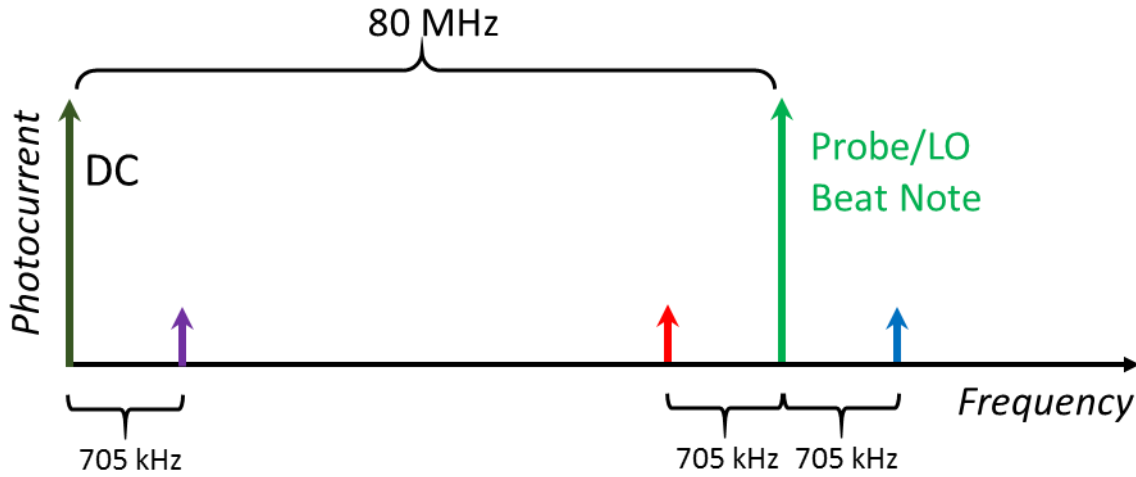


Figure 18: Spectrum of the photocurrent produced when the lasers land on the PDA10CF photodiode.

C. Filter Cavities

The purpose of FC1 and FC2 described above is to reduce classical laser noise on the measurement and cooling beams respectively. Though FC1 and FC2 have some minor design differences, they have nominally similar performance in terms of noise filtration. FC1, used on the measurement beam, was originally constructed by Yang and Jayich and is described in detail described in Ref. [41]. FC2 was constructed later, and will be described here.

Filter cavity FC2 is a high finesse Fabry-Perot cavity constructed from two mirrors with nominal power reflectivity $R = 0.99997$ mounted on an invar spacer, just like FC1. The primary difference between FC1 and FC2 is that the cavity spacer in FC2 is shorter and fatter. The mirrors in FC2 are approximately $L = 4.1$ cm apart from each other. At this length, the free spectral range of the cavity is approximately $\text{FSR} = 3.7$ GHz. The shorter cavity length was selected with the hope of making the

cavity more stable against vibrations. This shorter cavity length, together with mirrors of nominally similar reflectivity to FC1 should result in a cavity linewidth of approximately $\kappa = 2\pi \times 37$ kHz, slightly larger than the $\kappa = 2\pi \times 22$ kHz of FC1. Nonetheless, this still provides power filtration of classical laser noise at the mechanical frequency of 705 kHz by a factor of approximately 1.5×10^3 . Another difference between FC2 and FC1 is that the mirrors in FC2 have a diameter of 0.635 cm, compared with the 1.27 cm mirrors in FC1. These smaller diameter mirrors were selected in the hope that they might reduce stress-induced birefringence in the mirror coatings.

A photograph of filter cavity FC2 is shown in Figure 19. The mirrors on both sides of the cavity are glued to a piezoelectric actuator to allow for fast sweeping of the optical cavity length by a servo. The piezoelectric actuator is clamped between two metal plates. The larger of these plates, labelled “rotatable mirror mounting plate” in the photo, is rigidly bolted onto the invar spacer. The smaller plate is held in place by screws which are threaded into the larger plate. Spring washers are used to provide a counterforce to the piezo actuator and ensure that these two plates provide an even clamping force on the piezo actuator, while still allowing it to expand and contract in response to the servo.

The purpose of making the larger metal plate rotatable was to allow for easy rotation of the mirror so as to find the orientation with the minimum amount of birefringence. In practice, we found that birefringence was negligible in all orientations. The loose tolerance of the slots in the rotatable plate meant that the transverse positioning of one mirror relative to the other could be adjusted by moving the entire mirror assembly until the position with highest cavity finesse was found.

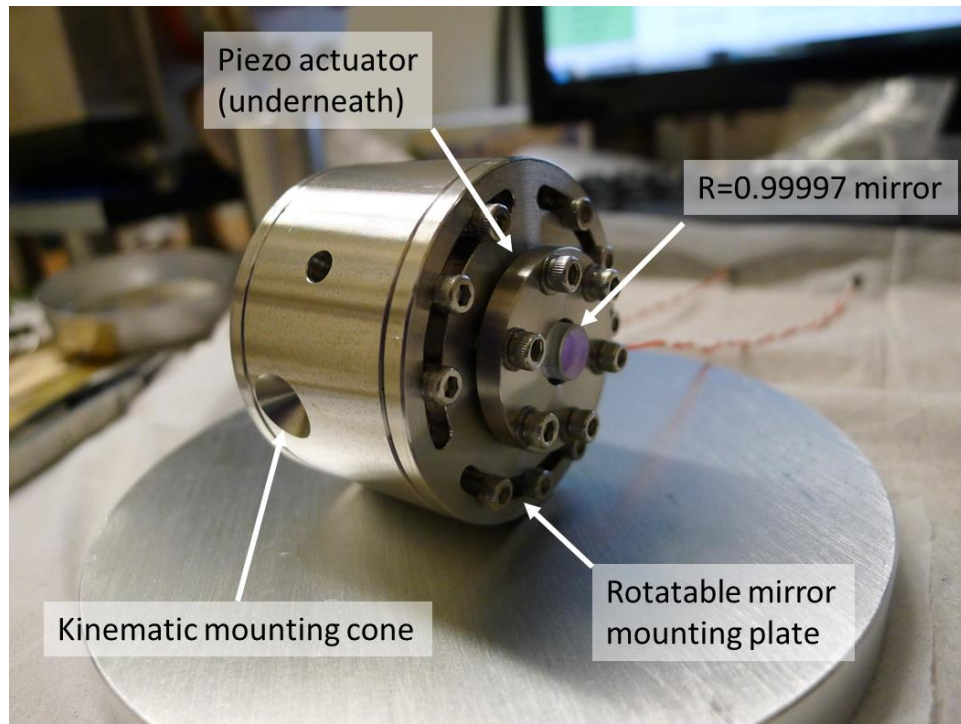


Figure 19: Photograph of filter cavity FC2.

A kinematic mount inside a vacuum can provides a reliable mount for the filter cavity within the optical beam path. Sapphire balls sit between the cavity spacer and the mount. One large ball mates with the cone which can be seen in photograph above, and the other balls touch the outer cylindrical surface of the invar spacer. The combination of all these contact points ensures repeatable positioning of the cavity within the optical beam path so that the cavity can be removed, adjusted, and reseated without requiring extensive optical realignment. The CAD drawing in Figure 20 shows how the filter cavity sits on this mount.

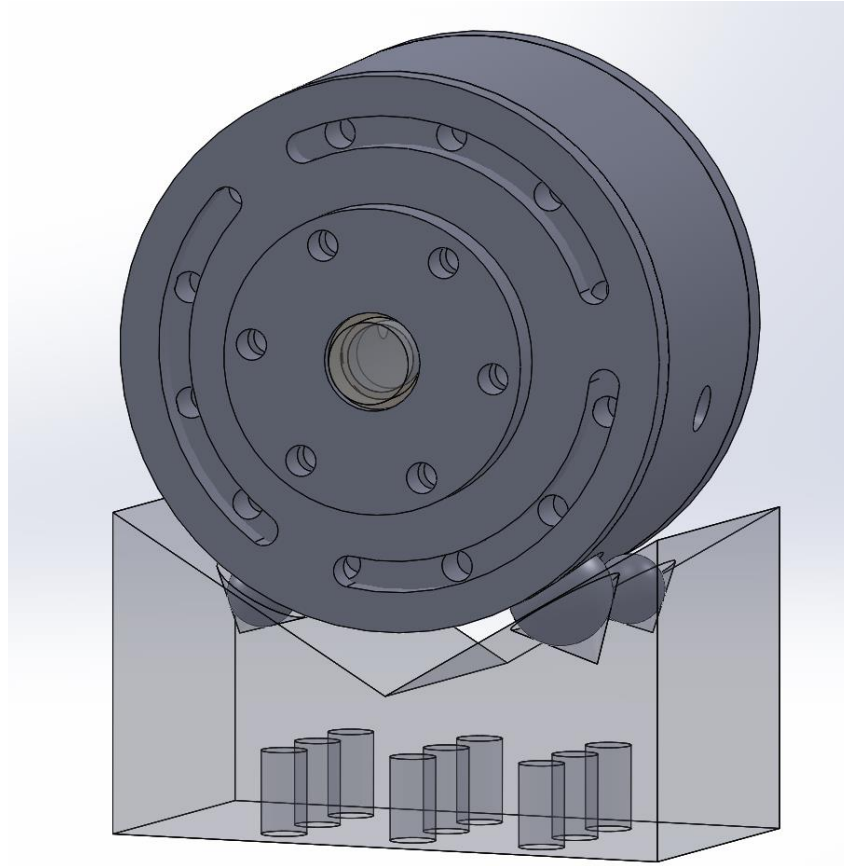


Figure 20: CAD drawing showing the filter cavity sitting on its kinematic mount.

The mount is bolted to the bottom of a vacuum can, which is clamped onto the optical table. After initially being pumped down with a turbomolecular pump, both filter cavity vacuum cans are kept in high vacuum at all times by ion pumps. Light enters the vacuum can through a vacuum window on one side of the can, passes through the filter cavity, and exits through a window on the other side. Lenses and mirrors on the optical table are used to shape and align the beam with the $TEM_{0,0}$ mode of the cavity before it enters the vacuum can. A servo is then used to adjust the cavity length so that the $TEM_{0,0}$ mode frequency is matched to the frequency of the laser. The details of this servo mechanism are discussed in the Laser Locking Setup section. Light exiting the vacuum can passes through further optics which pick off a small amount of transmitted beam for visualization on a camera, and couple most of the beam into an optical fiber.

With perfect mode matching to a lossless, symmetrical optical cavity, it should be possible to obtain 100% transmission of a shot noise limited laser that is on resonance with the cavity. In practice, primarily due to imperfect mode matching, we typically obtain at best 50% transmission. In spite of this power loss, the light exiting the cavity has significantly less classical laser noise than the light incident on the cavity due to filtration by the cavity. Filtration of classical laser noise occurs passively due to the narrow linewidth of the cavity. The noise power filtration factor is given by the ratio of the squared cavity susceptibility measured on resonance to the squared cavity susceptibility at the frequency of the noise, i.e. $|\chi_{FC}[0]|^2 / |\chi_{FC}[\Delta_{\text{noise}}]|^2$, where $\chi_{FC}[\omega]$ is the cavity susceptibility and Δ_{noise} is the detuning of the noise relative to the cavity resonance.

While this reduction in noise is desirable during attempts to cool the membrane to the ground state, the extra steps involved to lock the filter cavities can be burdensome for miscellaneous measurements and general experimental tasks that don't require shot noise limited performance. Therefore the optics surrounding the filter cavities have been laid out such that either filter cavity may be bypassed on demand via removable mirrors in the beam path. Engaging the bypass of either FC1 or FC2 has the net effect of completely removing the filter cavity, its preceding beamsplitter, and its servo loop from the summary schematic in Figure 16.

A photograph of the complete FC2 optical setup is shown in Figure 21 below.

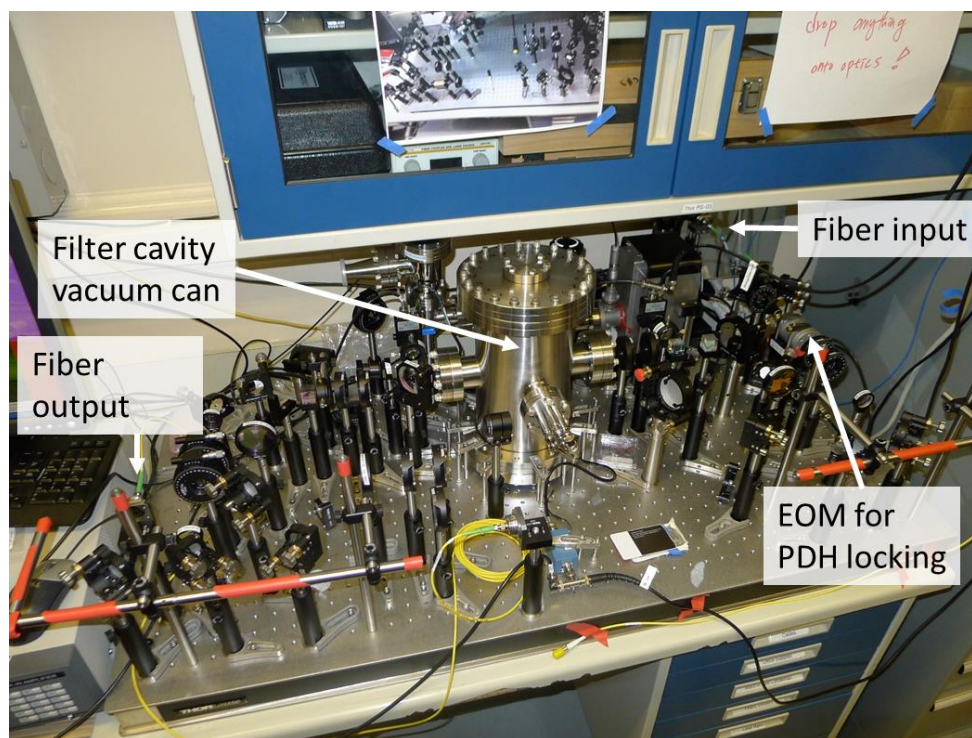


Figure 21: Photograph of the complete optical setup of FC2.

D. Cryogenic Setup

The cryogenics system used in this experiment is a Janis ^3He refrigerator. It consists of an internal vacuum chamber (IVC) which is submerged in a bath of liquid ^4He at a temperature of ~ 4 K. This bath has to be replenished approximately once per week by transferring liquid helium from a storage dewar provided by a supplier. Within the IVC is a series of cryogenic stages that achieve temperatures as low as 300 mK by evaporative cooling of liquid ^3He in a sealed chamber. A charcoal “sorb” adsorbs evaporated ^3He atoms to prevent ^3He vapor pressure from rising over the liquid and slowing the evaporative cooling process. Eventually, all the ^3He in the system evaporates and is adsorbed in sorb, at which point the temperature of the system rises to the bath temperature of 4 K. To cool the system down again, it is necessary to run a refrigeration cycle which consists of heating the sorb to release the adsorbed ^3He , and pumping on a small chamber of ^4He with a mechanical pump to achieve temperatures low enough to condense the ^3He back into a liquid. Once most of the ^3He has been recondensed, the sorb heater and mechanical pump are switched off, allowing for evaporative cooling of the liquid ^3He to resume. This recondensation cycle needs to be performed approximately once per week.

The use of a sorb-pumped ^3He refrigerator is nice because during normal operation, there are no mechanical pumps running, keeping vibrations to a minimum. However, the tradeoff for this normally quiet operation is an unavoidable weekly warm up of the experimental apparatus in the IVC to 4 K during the recondensation process. While the experimental apparatus about to be discussed generally tolerates this brief thermal excursion quite well, it sometimes can have effects on the system, as will be discussed.

The cryogenic “stage” of the membrane-in-the-middle experiment is stored entirely within the IVC, and is coupled to the coldest part of the ^3He refrigerator. Due to thermal conductivity through

electrical wiring and the optical fiber, the temperature of the cryogenic stage generally rests around 450 mK when no lasers are shining into the cryostat rather than the 300 mK base temperature that the fridge can attain when the IVC is empty. When the lasers are turned on and shining into the cryostat, the temperature of the stage rises to as high as 1 K, which must be accounted for when analyzing results from the experiment. The implications of this are discussed later in the Data Analysis section.

A CAD drawing of the cryogenic stage is shown in Figure 22 below.

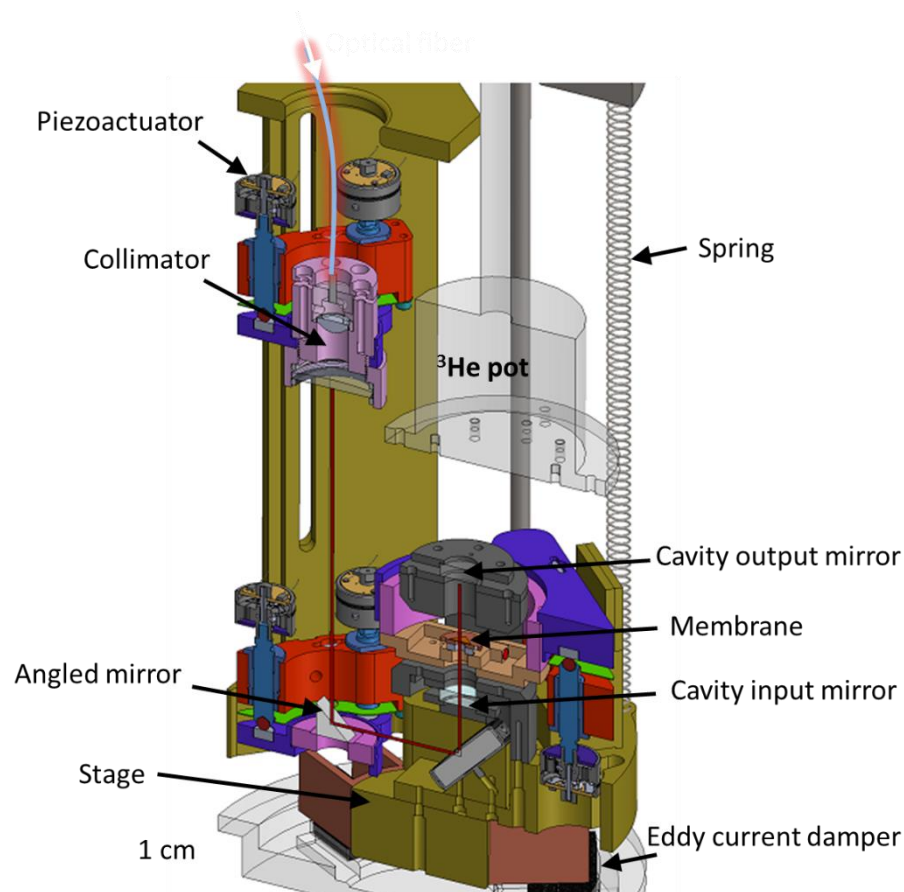


Figure 22: CAD drawing of the cryogenic stage. See main text for a thorough description.

After passing through the optical circulator on the optics table, all of the laser beams are directed into the cryostat in a single mode optical fiber. Once inside the cryostat, the fiber terminates in a collimator, as seen in Figure 22, which sends the beam into free-space. The collimator is mounted within a specially designed mirror mount which allows for adjustment of its alignment in situ via a set of

three piezo-actuated mirror knobs. The knobs, appropriately called PiezoKnobs, are computer-controlled actuators that can adjust all the degrees of freedom of the mirror mount even at cryogenic temperatures.

The free-space beam coming out of the collimator travels downward and hits an adjustable 45 degree mirror, which is mounted in a similar piezo-actuated mirror mount. Since both the collimator and this mirror can be adjusted on multiple axes, it is possible to use two-knob-adjustment techniques to align the beams with the cryogenic cavity for optimal mode matching.

The light from the adjustable 45 degree mirror bounces off another non-adjustable 45 degree mirror which directs it toward the cavity input mirror. If well coupled and tuned to the right frequency, light enters the cavity and interacts with the membrane. The membrane itself is mounted on a somewhat larger, but otherwise similar piezo-actuated mirror mount. This allows for adjustment of the membrane's tip, tilt, and translation along the cavity axis.

The cavity output mirror ($R = 0.99997$) was selected to be significantly more reflective than the input mirror ($R = 0.9998$), so that most of the light within the cavity reflects and traces the same path back to collimator and optical fiber. Light that manages to recouple into the fiber returns to the optical table for measurement.

Light that transmits through the cavity's output mirror is collected on a germanium photodiode (not shown in Figure 22). The photodiode is not particularly useful for measurement of the mechanical mode due to the poor responsivity of germanium photodiodes at 1064 nm at cryogenic temperatures, and also the high RC time constant of that photodiode and its readout electronics system. Nonetheless, the transmission photodiode is useful for initial alignment of the system at cryogenic temperatures using 935 nm light, as will be discussed in the next section.

In addition to providing exceptional in situ alignment capabilities, the cryogenic stage was designed to provide a high degree of vibration isolation from external seismic and acoustic noise. Though perhaps obvious since it is a cryogenic system, the first line of defense against acoustic noise is simply the fact that the stage is in a cryogenic vacuum chamber. The lack of air or other gasses around the system limits the ways in which acoustic noise can couple in. To mitigate the effects of noise that does manage to couple into the system, the cryogenic optical setup is mounted on a heavy, ~1 kg piece of titanium which forms the base of the cryogenic stage. Titanium was selected for its high thermal conductivity at low temperatures.⁵³

As shown in Figure 22, the titanium base is suspended on springs, and has copper fins mounted on its underside. In between the copper fins are rare earth magnets which are attached to a fixed plate underneath the stage. As the suspended stage oscillates, the change in magnetic flux through the fins induces eddy currents, which dissipate mechanical vibrations as heat. At room temperature, the stage/spring system is approximately critically damped, and returns to equilibrium when perturbed without overshooting. The resonance frequency of the system is about 2 Hz.

Figure 23 below shows a photograph of the cryogenic system, in which some features omitted from the CAD drawing are visible. Most notably, a series of electrical connections, as well as myriad thermal heat sinking wire can be seen. To reduce the coupling of external vibrations into the stage through these wires or the optical fiber, both are kept as loose as possible. For the heat sinking wires, many hundreds of thin wires are used rather than fewer thicker wires. The number of heat sinking wires was kept to a minimum by using gold plated oxygen-free high conductivity copper wires which offer high thermal conductance with minimal cross sectional area.

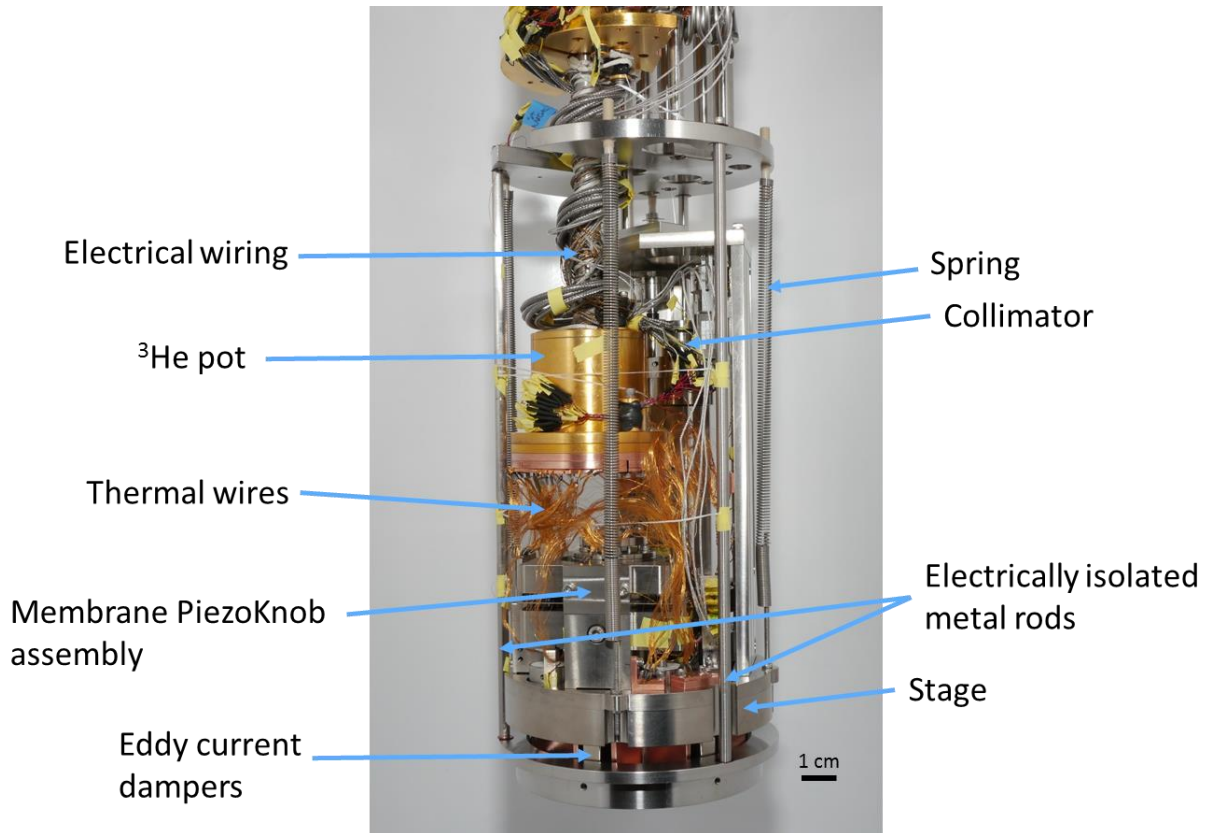


Figure 23: Photograph of the cryogenic stage.

Another mechanism through which outside vibrations can couple into the system is accidental mechanical contact between the suspended stage and parts of the cryogenic system that are not vibrationally isolated. While there is no way to correct this if it occurs when the stage is cooled to base temperature, it is at least useful to know if contact is being made so that it can be corrected when the fridge is warmed up next. Therefore, all of the metal parts surrounding the stage including the base plate holding the rare earth magnets for the eddy current dampers, and the three metal rods that support it (two of the rods are shown in Figure 23) are electrically isolated from each other and are connected to wires which are accessible outside the fridge. The stage itself is electrically grounded. By checking for continuity between the isolated parts and ground, we can tell whether there is accidental contact between the stage and any of the surrounding parts.

For further isolation, the cryostat itself sits on four pneumatic vibration isolators. These isolators, Newport Model S-2000A-128, are typically used to support and isolate optical tables. As can be seen in Figure 24, a heavy duty aluminum frame made of 8020 brand extrusions rests on top of the vibration isolators. The frame supports a thick aluminum plate in the middle to which the dewar is bolted and suspended (not shown in Figure 24). Surrounding the dewar setup are removable walls and a ceiling made of plexi-glass that are covered in soundproofing foam to reduce the influence of acoustic noise. When fully enclosed within this acoustic isolation cube, the ambient sound levels around the dewar are reduced by 13 dB.

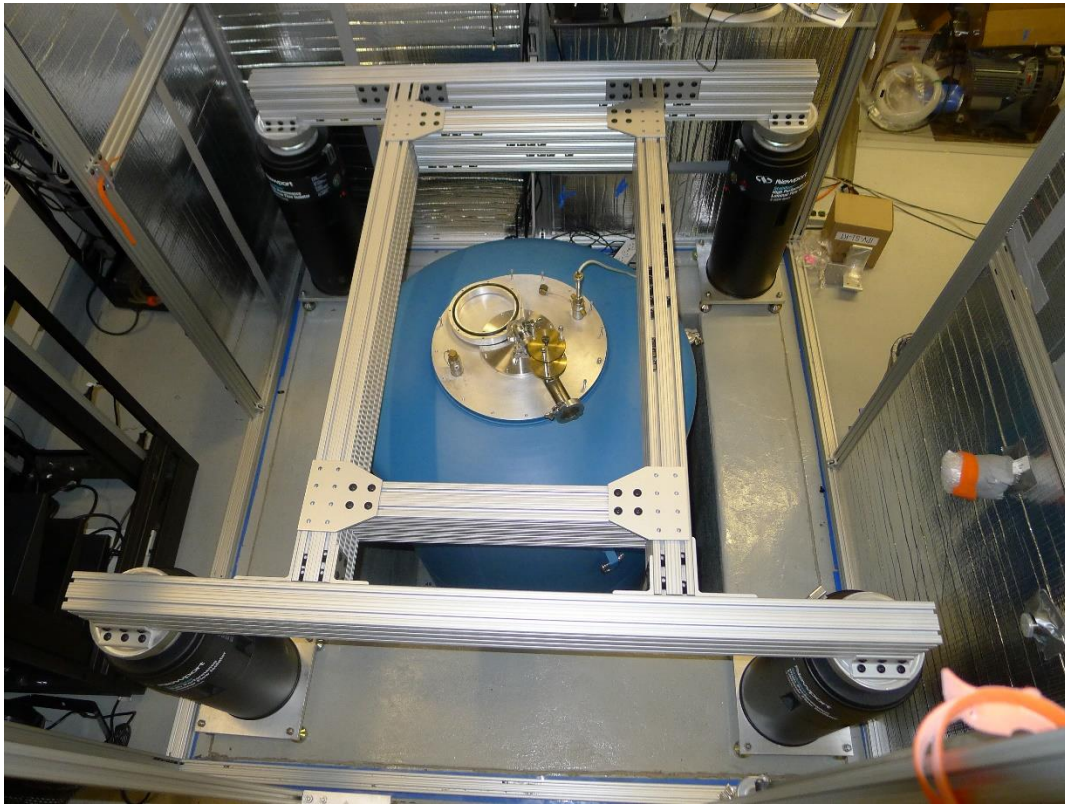


Figure 24: A top down photograph of the cryostat during assembly of the vibration isolation system.

While I have discussed details of the refrigeration system, the cryogenic stage, and vibration isolation, I have thus far neglected to discuss the membrane itself. The membrane is mounted on a winged circular piece of copper that is a little smaller than a penny called the “membrane plate”, as

shown in Figure 25. The membrane plate has a circle of tapped holes which can optionally be used to attach a leaf spring to hold the membrane in place, or the membrane can be glued. Figure 25 shows the membrane secured with a leaf spring, though in the final configuration used for this experiment, the membrane was secured with small dabs of Stycast 2850 epoxy on 3 corners. Though we never performed a systematic analysis of membrane mounting methods, we found through trial and error that the excellent mechanical properties of silicon nitride membranes at cryogenic temperatures⁵⁴ were most reliably achieved with membranes that were glued as opposed to membranes that we clamped with a leaf spring.

The membrane plate is secured to a rectangular titanium block via screws through its two wings. Between the membrane plate and the titanium block is a thin sheet of oxygen free high conductivity (OFHC) copper, to create a good thermal link to the heat sinking wires which are pressed onto the copper sheet at the corners of the titanium block. Underneath the titanium block is a ring piezo actuator (only the wires for the ring piezo actuator are visible in Figure 25) which allows for angstrom scale adjustments to the position of the membrane within the optical cavity. The ring piezo sits on the large titanium “bridge”, which is used to support the entire assembly within the optical cavity and to attach it to the PiezoKnob-based mirror mount for in situ adjustment.

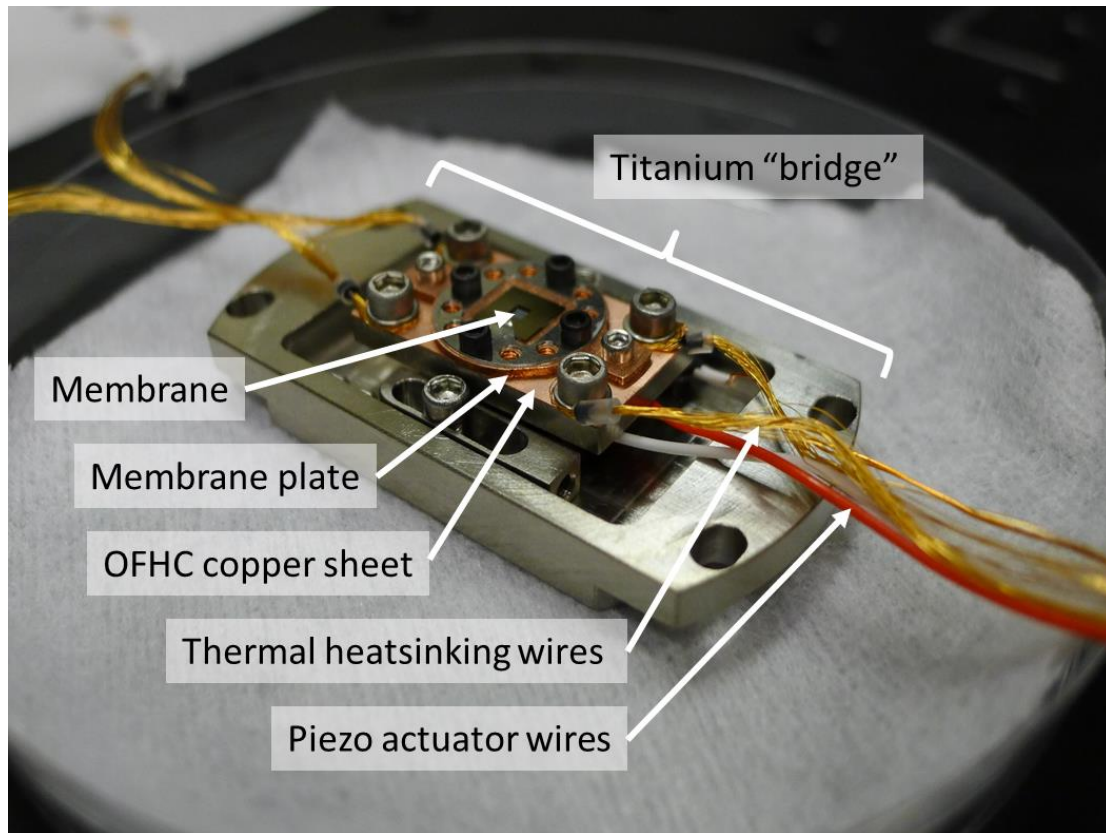


Figure 25: Photograph of a prototype of the membrane mounting assembly. The final assembly was identical to this prototype, except the membrane was secured with dabs of Stycast epoxy on 3 corners instead of with a leaf spring.

Figure 26 shows the membrane assembly sitting within the optical cavity before installation of the piezo-actuated mirror mount. The cavity spacer is made of titanium and is designed to ensure the appropriate separation between the two mirrors (about 3.7 cm). The spacer is rigidly mounted to the titanium stage. The mirrors themselves are clamped between two plates which are held together by screws. Spring washers ensure that the screws provide an even clamping force on the mirror during the cooldown process, when differential thermal contraction between the mirror substrate and the screws would otherwise cause the clamping force to change dramatically. The upper mirror, visible in Figure 26 is the output mirror. The input mirror lies under the cavity and is not visible in Figure 26. The piezo-actuated mirror mount attaches to the stage by way of the chairs shown in Figure 26. A hole in the titanium stage can be seen at the bottom of Figure 26 to accommodate one of the three PiezoKnob actuators used to adjust the membrane's tip, tilt, and translation within the optical cavity.

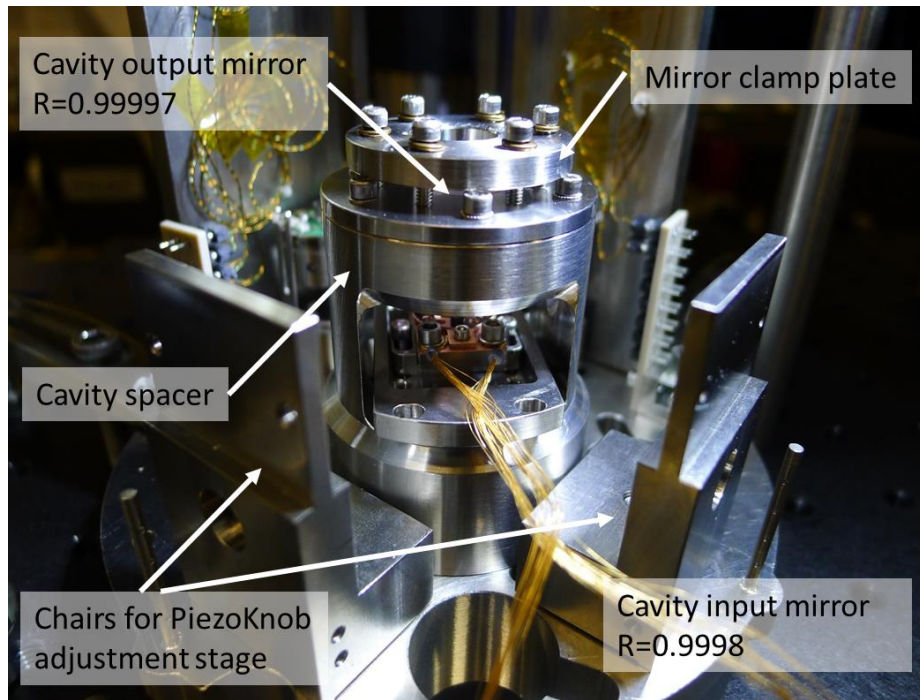


Figure 26: Photograph of the membrane assembly resting inside the cryogenic optical cavity before assembly of the piezo-actuated mirror mount used to hold and position the membrane assembly.

E. Initial Preparation

Before installing the membrane assembly into the cryogenic cavity, it was necessary to align the light exiting the fiber collimator with the $TEM_{0,0}$ mode of the cryogenic cavity. This initial alignment was performed at room temperature on the optical table. A photograph of the alignment setup is shown in Figure 27 below.

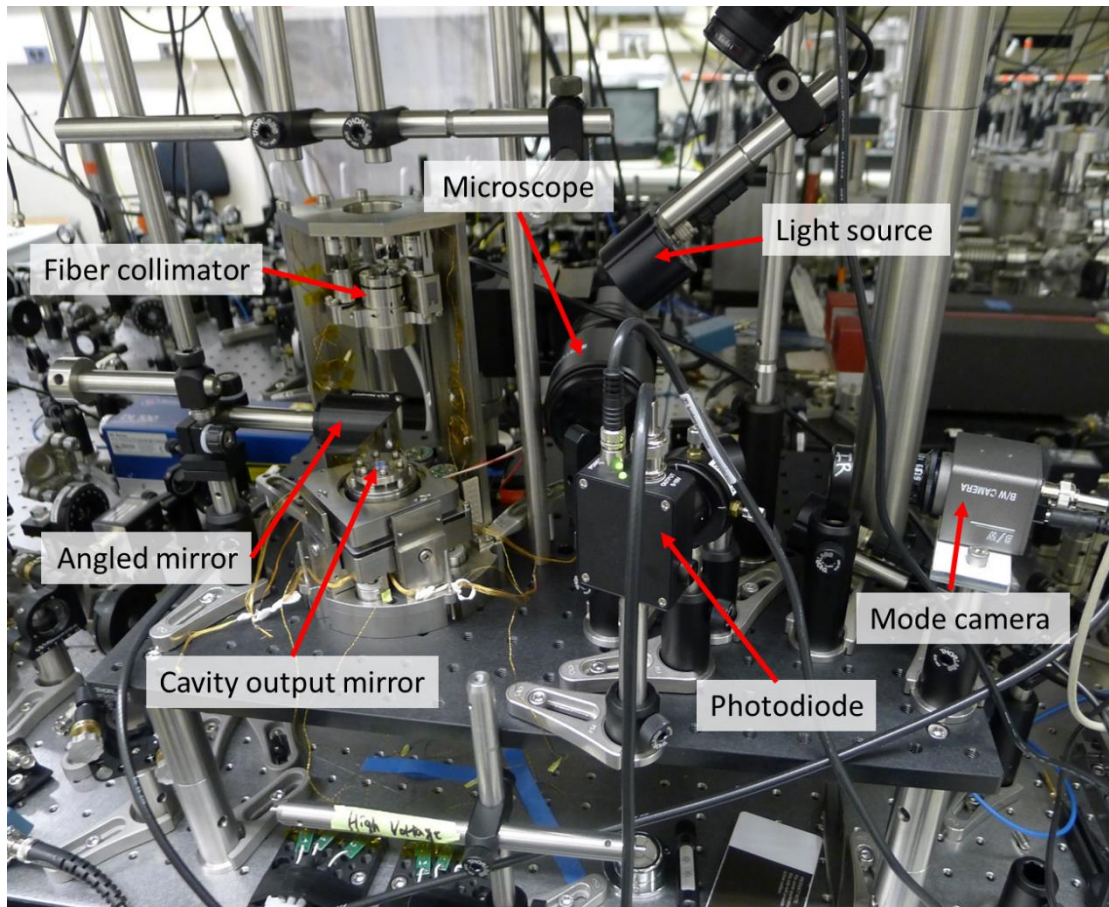


Figure 27: Photograph of the alignment setup.

The alignment setup consisted of the cryogenic stage clamped onto an optical breadboard, along with several pieces of equipment to assist in aligning both the collimator with the cavity mode, and the membrane within the cavity. A 45° angled mirror was placed directly above the cavity's output mirror to direct light exiting the cavity (both laser light, and ambient light) in a direction parallel to the surface of the breadboard. A removable mirror (not labelled in Figure 27) could then be used to direct the light toward a microscope for visual inspection of the cavity output mirror and the space within the cavity (the cavity mirrors are transparent at visible wavelengths). Without the removable mirror, the light from the angled mirror would go through a beam sampler, picking off a portion of the light to go to a photodiode for quantitative transmission measurements, and passing the rest to an infrared camera

for visualizing the shape of the optical mode exiting the cavity. Schematically, this is summarized in Figure 28 below.

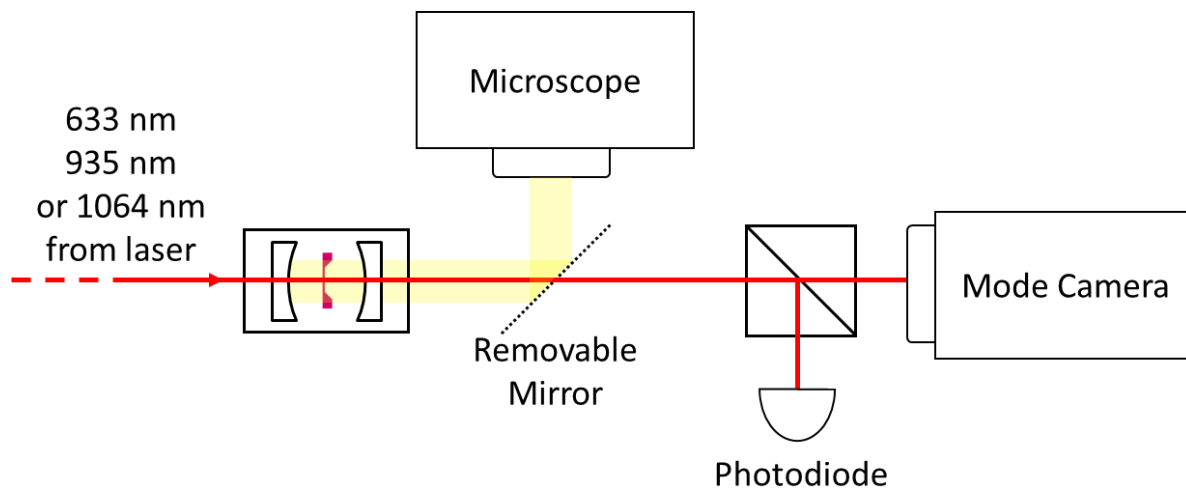


Figure 28: Schematic of the initial alignment setup.

The initial alignment procedure was first performed without a membrane inside the cavity. The first step in the alignment process was to inject 633 nm visible light into the optical fiber that feeds light to the cavity. Visible light allowed for easy inspection of coarse alignment since it can be seen with the naked eye. We adjusted the tip and tilt of the collimator's mirror mount, and also of the angled mirror at the bottom of the stage (see Figure 22) by a combination of hand-turning the PiezoKnob actuators and by computer control. The primary goal of aligning with 633 nm light was just to make sure that the light was transmitting through the optical cavity and that it appeared to be centered on the cavity mirrors.

The optical fiber used to feed light to the cavity is SM980 fiber, which is single mode only for wavelengths longer than 980 nm. This means that at 633 nm, the fiber is actually multimode. Together with chromatic dispersion in the collimator optics, this meant that we could only make very coarse estimates about the alignment of the collimator with the optical cavity using 633 nm light. Therefore, the second step of the alignment process was performed using 935 nm light. Though the fiber is technically still multimode at this wavelength, there are fewer modes supported and the chromatic

dispersion in the collimator optics is less significant (since their design wavelength is 1064 nm), allowing for a “better” rough alignment to be performed. 935 nm light can’t be seen with the naked eye, but using a detector card, we estimated the position of the light exiting the cavity and adjusted the collimator and angled mirror mounts until the light was centered on the output mirror.

The final step in alignment to the cavity was to switch to 1064 nm light. Since the cavity is high finesse at 1064 nm, light will only transmit through if it is mode-matched and at the frequency of an optical mode as defined by the mirrors (see the Hermite Gauss Modes in a Fabry-Perot Cavity section). Since the frequency of the desired $TEM_{0,0}$ mode was unknown a priori, we swept the laser frequency over a large frequency range and used both the mode camera and the photodiode to look for signs of transmission through the cavity. We then used two-knob adjustment techniques to optimize mode-matching to the $TEM_{0,0}$ mode, which could be identified using the mode camera. We considered the mode-matching to be optimized when the height of the peak detected by the photodiode as the laser swept over the $TEM_{0,0}$ mode was as large as possible, and peaks corresponding to other modes were minimized.

Once this alignment procedure was complete, the next step was to insert the membrane. The titanium bridge supporting the membrane was slid inside the optical cavity, and the piezo-actuated mirror mount assembled around it. The removable mirror was put in place so that we could inspect the membrane within the optical cavity using the microscope, and adjust its position until it was visually centered. Next, 633 nm light was used to check coarse centering of the membrane, followed by 935 nm light and a detector card to ensure the laser was not clipping on the side of the membrane. Finally, we switched back to 1064 nm light with a swept laser frequency, as in the empty cavity case.

Initially, with 1064 nm light there was no transmission through the cavity. This was due to the fact that the membrane was angled significantly, such that the plane of the membrane was not

perpendicular to the optical axis through the cavity. Though we tried to use calipers to measure the angle of mirror mount that holds the titanium bridge of the membrane assembly relative to the base of the stage, finding the initial alignment of the membrane to allow any transmission through the cavity at 1064 nm was largely a trial and error process. Eventually, we did notice some transmission using the photodiode and the mode camera, and were able further adjust the membrane angle to recover coupling to the TEM_{0,0} mode.

The initial cooldown of the system in the cryostat was to a temperature of 77 K using liquid nitrogen in place of the liquid helium bath. Since liquid nitrogen is significantly less expensive than liquid helium and also has a significantly larger heat of vaporization, it is an excellent cryogen for initial cooling. Upon cooling the system down in the cryostat, due to uneven thermal contraction of various parts of the stage assembly, alignment of the optics was invariably lost. We were generally able to recover it using the same techniques described above, with slight modifications due to the fact that once the system was inside the cryostat, we no longer had the luxury of a microscope, a photodiode sensitive at 1064 nm, and a mode camera to help with alignment. Instead, we only had the germanium transmission photodiode, which as discussed earlier, was not sensitive to 1064 nm light. The first step to realigning after cooling was therefore to use the 935 nm laser to ensure that the laser was not clipping on the frame of the membrane. To do this, we systematically stepped one of the collimator's PiezoKnob motors while recording transmission of 935 nm light using the transmission diode at each step. We swept over the full range within which there was detectable transmission and eventually stopped near the center of the transmission plateau. We then repeated this with the PiezoKnob that controlled the orthogonal degree of freedom. The transmission as a function of the PiezoKnob's linear displacement (calibrated by the PiezoKnob's response to various drive signals at room temperature) for one of these knobs resembled the following:

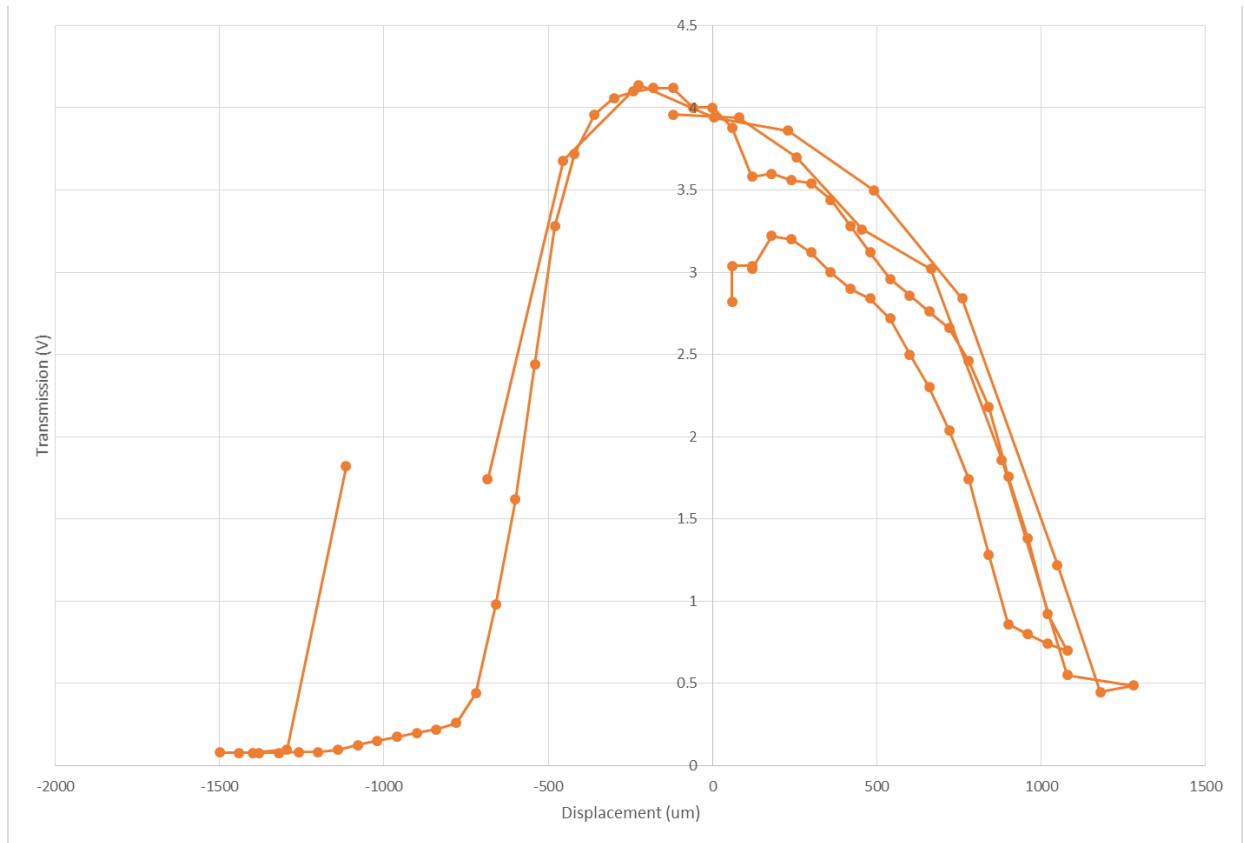


Figure 29: Plot of 935 nm light transmission measured by the germanium photodiode as a function of linear displacement of one of the collimator PiezoKnobs.

In Figure 29, some displacements were measured more than once, and did not yield the same amount of transmission with each measurement. From this we inferred that there is some non-reproducibility in the motion of the PiezoKnobs, and therefore the “Displacement” may have some random errors in addition to systematic errors caused by the change in PiezoKnob response from room temperature to 77 K. Nonetheless, this measurement allowed us to maneuver the collimator such that the 935 nm light was roughly centered on the membrane.

Once we were confident that the angle of the light exiting from the collimator was approximately correct, we switched back to 1064 nm light, sweeping the laser over a large frequency range, just as we did at room temperature. Since the transmission diode was not sensitive to this 1064 nm light at cryogenic temperatures, we used light reflected from the cavity to look for cavity modes,

which manifested as “dips” in the DC level coming from the reflection photodiode as the laser frequency swept over a cavity mode. We then used two-knob adjustment techniques with the collimator PiezoKnobs and the 45 degree angled mirror PiezoKnobs to improve coupling to whatever modes we could find. The identity of these modes was not known a priori.

Identifying cavity modes without using a mode camera can be quite challenging, since there is no simple way identify a mode based exclusively on the reflection dip it produces as the laser sweeps over the mode’s resonant frequency. However, several factors helped us to identify the unknown modes we found after performing a cursory alignment of the optics. First, there is the fact that the $TEM_{0,0}$ mode is the only non-degenerate Hermite Gauss optical cavity mode. Though normally degenerate modes can become non-degenerate due to symmetry breaking by the membrane, these modes will still show roughly the same behavior as a function of membrane position, making it easier to identify them as nearly degenerate groups. Second, since the free-space beam leaving the collimator has a Gaussian beam shape, it is well matched with the shape of $TEM_{0,0}$ mode, and will therefore couple more strongly to the $TEM_{0,0}$ (assuming good alignment) than it will to any other modes.

The process of aligning the beam with the $TEM_{0,0}$ mode of the cavity was therefore an iterative process in which we measured the behavior of the modes as a function of membrane position in between adjustments to the piezo-actuated mirror mounts within the cryostat. During this process, we also tried to level the membrane within the optical cavity, with a goal of making it as perpendicular as possible to the cavity’s optical axis. We could identify how level the membrane was based on the degree of degeneracy observed in normally degenerate modes such as the $TEM_{0,1}$ and $TEM_{1,0}$ modes. A perfectly level membrane should not lift the degeneracy of such modes.

We refer to each measurement of the behavior of the cavity modes as a function of membrane position as a “bandstructure”. This term has no physical relation to the term bandstructure used in solid

state physics, and instead just refers to a two-dimensional plot of cavity reflection vs. membrane position and laser frequency. To record a bandstructure, we control the laser frequency using the laser piezo which allows for several hundred MHz of tuning range, and we control the membrane position using a ring piezo under the membrane which can translate the membrane a few hundred nanometers. The procedure is to slowly step the membrane in small increments across its full range, pausing at each step to sweep the laser frequency over the full tuning range of the laser piezo. As the sweep occurs, a DAQ device records the cavity reflection. By plotting this, we can observe the structure of the cavity modes as a function of membrane position. An example of a typical bandstructure measured after several days of optimizing membrane position and optical alignment is shown in Figure 30. The $TEM_{0,0}$ mode can be identified in the bandstructure by the fact that it is the only mode that is non-degenerate. Higher-order modes all have nearly degenerate cousins, and can be identified based on the number of nearly degenerate modes seen in the bandstructure.

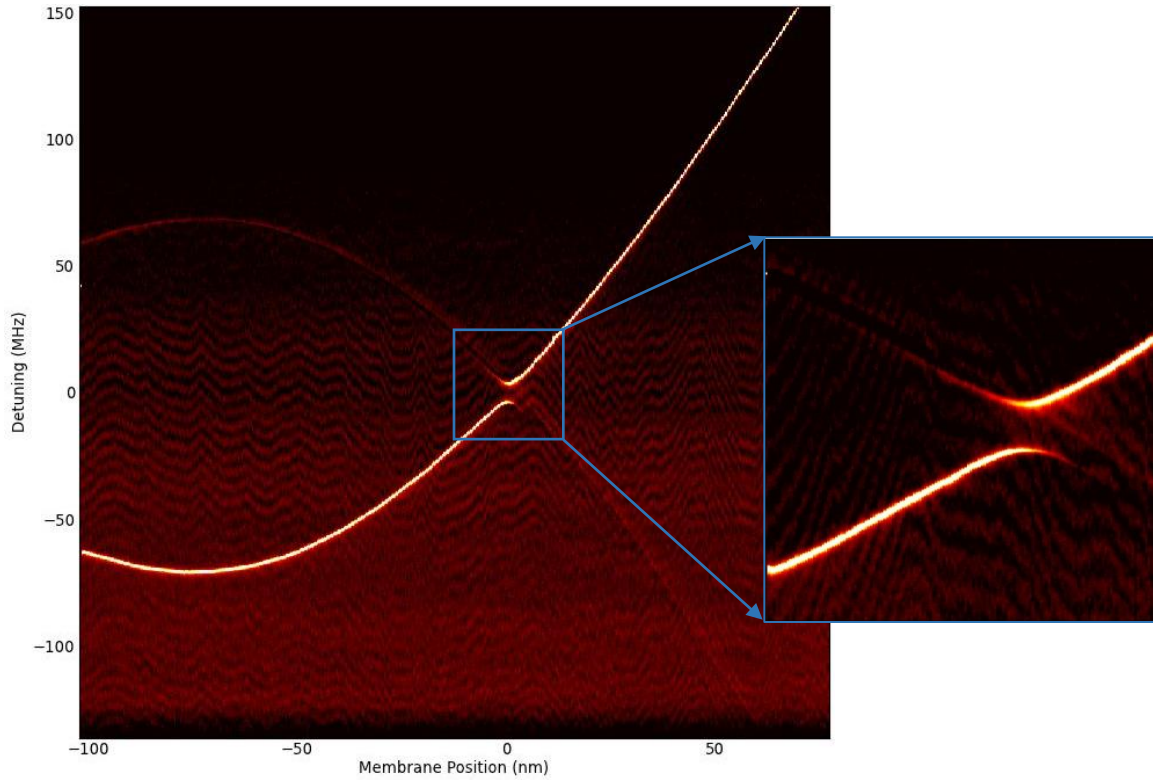


Figure 30: A plot of the cavity reflection as a function of membrane position and laser frequency, informally called a “bandstructure”. The “brighter” mode is the $TEM_{0,0}$ mode, which can be determined by the fact that it is non-degenerate. The other mode is actually made of 3 nearly degenerate modes: $TEM_{0,2}$, $TEM_{1,1}$, and $TEM_{2,0}$. The three nearly degenerate modes can be seen in the inset, which is a higher resolution zoom-in of the bandstructure around where the modes cross.

Once the optical alignment and membrane levelling was complete, we would remove all of the liquid nitrogen from the bath space inside the cryogenic dewar and cool the system to 4K using liquid helium. After cooling to 4 K, we would repeat the optical alignment and membrane levelling procedure. Once at 4 K, further cooling to the base temperature of the cryostat generally did not necessitate additional adjustments to the optical alignment or membrane levelling. However, the weekly recondensation procedure was generally found to cause small sub-wavelength translational changes of the membrane position within the optical cavity. These small displacements were generally correctable using the ring piezo actuator underneath the membrane, as described in the next section on the Laser Locking Setup.

The final step in the preparing the system for operation was to survey several mechanical modes of the membrane to determine which would be most suitable for ground state cooling. We did this by measuring the mechanical quality factor and optomechanical coupling of each mode. The mechanical quality factor was measured by mechanical ringdown, and the optomechanical coupling was estimated by fitting measurements of the optical spring and damping using Equations (63) and (64). Unfortunately, between the time this initial survey of modes was taken and the time the results of the experiment were published¹¹, the trustworthiness of the results from this initial survey was called into question due to a combination of technical problems with the 935 nm laser used for mechanical ringdowns, unusual results when trying to reproduce these ringdowns, and questions over exactly how much laser power was used in measurements of optomechanical coupling. I therefore will not discuss at length how these measurements were performed. In spite of the questions about these results, they indicated that the (2,2) mechanical mode at 705 kHz was the most promising for ground state cooling due to a high mechanical quality factor and strong optomechanical coupling relative to other modes. A much more careful measurement of mechanical quality factor and optomechanical coupling of this mode is presented in the Data Analysis->Characterizing Parameters section.

F. Laser Locking Setup

There are many servo feedback loops used to keep the lasers and cavities on resonance with each other or when appropriate, detuned from each other. In this section I will describe these locking setups.

The first step in any measurement we make is to find where the membrane is within the cavity. For this process, the classical noise performance of the lasers is not critical, so we bypass both of the filter cavities for convenience. For ground state cooling, we ideally want the membrane positioned at a point where the optomechanical coupling is maximized, i.e. at a point where the slope of the cavity's

resonant frequency as a function of membrane position is maximized. We find the coarse position of the membrane relative to this point by measuring the cavity bandstructure. We then identify the $TEM_{0,0}$ mode and note where the node and/or anti-node of the mode is in the bandstructure. By positioning the membrane halfway between the node and anti-node, we know that the membrane is close to the point of maximum slope. For example, in Figure 30 the antinode of the $TEM_{0,0}$ mode occurs at a membrane position of about -75 nm (the offset of the x-axis in this figure is arbitrary). Therefore, we expect the point of maximum linearity of the $TEM_{0,0}$ mode to occur at a membrane position of +58 nm ($\lambda/8$ away from the antinode).

Once the approximate point of maximum slope has been found, we move the membrane to this point and lock the measurement laser to the cavity resonance using a Pound-Drever-Hall technique. Optical phase modulation sidebands from the EOM (see Figure 16) are detected by the measurement photodiode and go through a mixing circuit to produce a Pound-Drever-Hall error signal. The mixing circuit is shown in Figure 31.

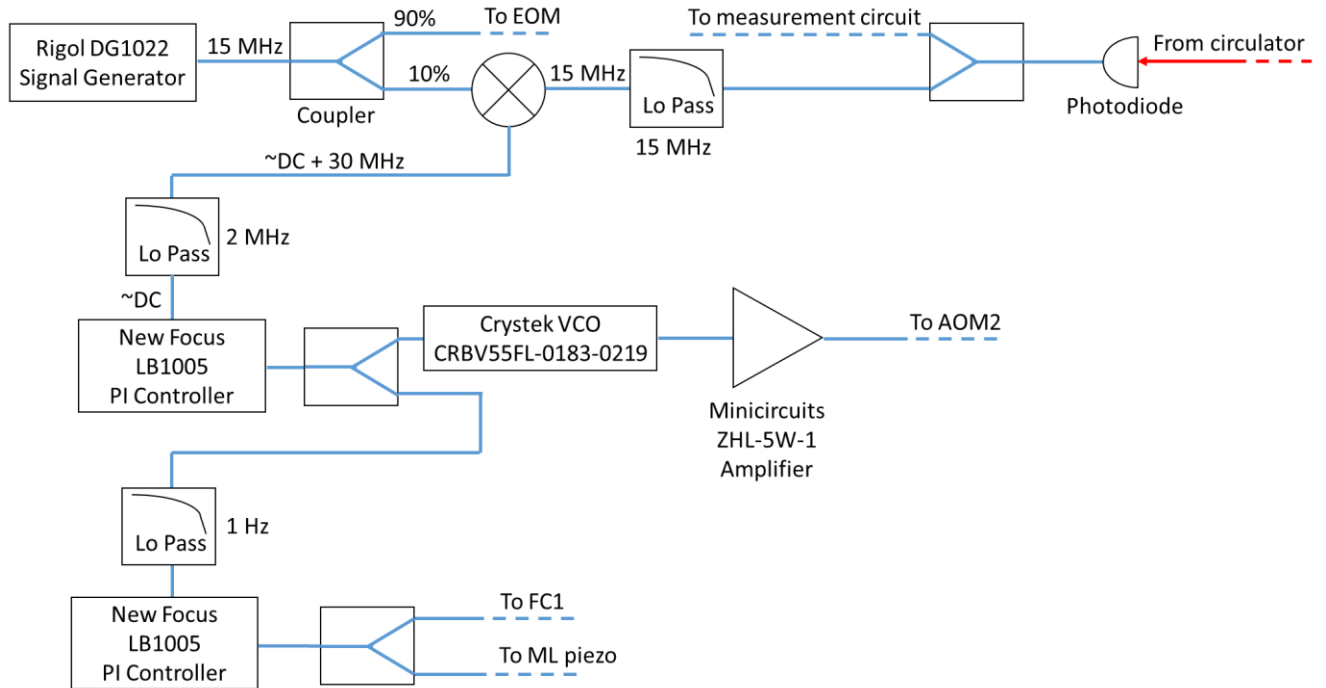


Figure 31: Block diagram of electrical circuit for locking the measurement laser to the optical cavity.

The error signal is fed to a PI controller (New Focus LB1005) which adjusts the tuning voltage of a voltage controlled oscillator (Crystek CRBV55FL-0183-0219) which, after amplification through a high power amplifier (Minicircuits ZHL-5W-1), drives AOM2. In this way, the frequency of the measurement laser can be kept on resonance with the optical cavity. The frequency shifting range of AOM2 is approximately $200 \text{ MHz} \pm \sim 100 \text{ MHz}$. However, the diffraction efficiency of AOM2 drops as the deviation from 200 MHz gets larger. Therefore a second PI controller takes the output of the first PI controller as its error signal and sends feedback to the laser piezo of the measurement laser. This feedback is low-passed at several Hz, and is primarily intended to keep the drive frequency of AOM2 centered near 200 MHz. A portion of this signal also gets split off to feed forward to FC1, which will be discussed in a bit.

Once the probe beam has been locked to the cavity, the next step is to lock the cooling laser to the measurement laser with an offset equal to approximately two free spectral ranges of the cold cavity. This is done for several reasons: (1) since the cooling laser is typically detuned from the cavity resonance by an amount comparable to ω_m , coupling the laser to a different longitudinal mode several GHz away prevents the formation of a beat note at ω_m inside the cavity, and (2) the slope of the linear region of the cavity resonance as a function of membrane position has a sign which alternates with each free spectral range – therefore to keep both lasers in a region with the same sign, it is necessary to detune the cooling laser by two free spectral ranges. It is important that the two modes have the same sign because the measurement and cooling beams both receive the same frequency shifts from AOM2. If the beams are coupling to modes which have different slopes as a function of membrane position, then only one of the two beams will receive the necessary frequency shifts from AOM2 to track the cavity's resonance – the other will receive frequency shifts that are the opposite of the actual fluctuations of the mode it is coupled to.

To actually perform the offset lock, a scheme similar to the one described in Ref. [55] is employed. As shown in Figure 16, a portion of both the measurement and cooling lasers are picked off

right after light exits the lasers themselves and combined on a photodiode. This photodiode is a Newport Model 1554-B, which has a 12 GHz bandwidth and is therefore able to directly detect the ~8 GHz optical beat note between the two lasers. The output of this photodiode is sent into a high frequency mixer, where it is combined with a reference tone from a Rohde & Schwartz signal generator. When the frequency of the optical beat note and the reference tone are near each other, the output of the high frequency mixer has frequency components near DC and near 16 GHz. The 16 GHz component is suppressed due to the bandwidth of the mixer. The component near DC has a frequency which represents the offset in frequency between the optical beat note and the reference tone from the signal generator. To produce an error signal for servo feedback to keep the frequency offset locked, it is necessary to convert the frequency of this signal into a voltage. This is easily done by mixing it with a low-passed version of itself, where the low-pass corner frequency is selected so that there is a steep linear roll-off of phase around the desired frequency offset. We do this using a Minicircuits BLP-1.9+ low pass filter which has a 3 dB corner frequency of 2.5MHz. (The frequency dependent phase shift from this filter replaces the phase shift introduced by the coaxial delay line in Ref. [55].) The phase shift introduced by this filter makes it so that after mixing the low passed and non-low passed signals, there is a component at DC corresponding to the error in frequency between the optical beat note and the reference tone. The other component at a few MHz is filtered out with a second BLP-1.9+ filter. The DC component is fed into a New Focus LB1005 servo controller, and the output of that controller is sent to the laser piezo of the cooling laser to keep it locked with the appropriate offset. A schematic of this setup is shown in Figure 32.

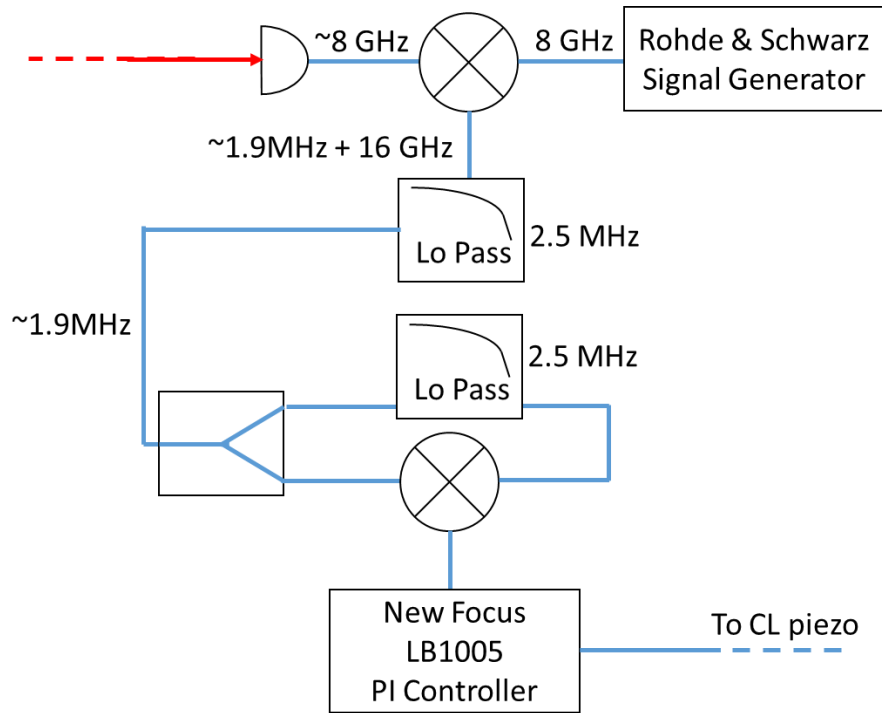


Figure 32: Schematic of the electrical circuit used to lock the cooling laser to the measurement laser with an adjustable frequency offset.

We call this lock the “slock”, as a portmanteau of “slave laser lock”. Once it is locked, both the membrane position, and the frequency of the lock need to be optimized. With regard to the membrane position, the goal is to find a spot within the cavity where the slope of the cavity resonance as a function of the membrane position is the same for both the measurement and cooling lasers. There are a few methods by which this can be done. One method is to step the membrane through a series of positions and measure the separation in frequency between the measurement beam optical mode and the cooling beam optical mode. We measure the frequency separation between the modes by sweeping the frequency of the Rohde & Schwarz signal generator with a deviation of several MHz at a frequency of 1-2 Hz, which the slock then tracks, sweeping the frequency of the cooling laser. We then look for dips in the reflected light from the cavity as the cooling beam passes over the optical mode and adjust the center frequency of the sweep until the dips are observed at the center of the sweep range. The sweep center frequency is then a good estimate of the frequency difference between the two modes.

When the membrane is near a point of identical slopes for the two modes, the frequency difference between the two modes should minimally depend on membrane position. A typical plot of frequency difference vs. membrane position is shown below:

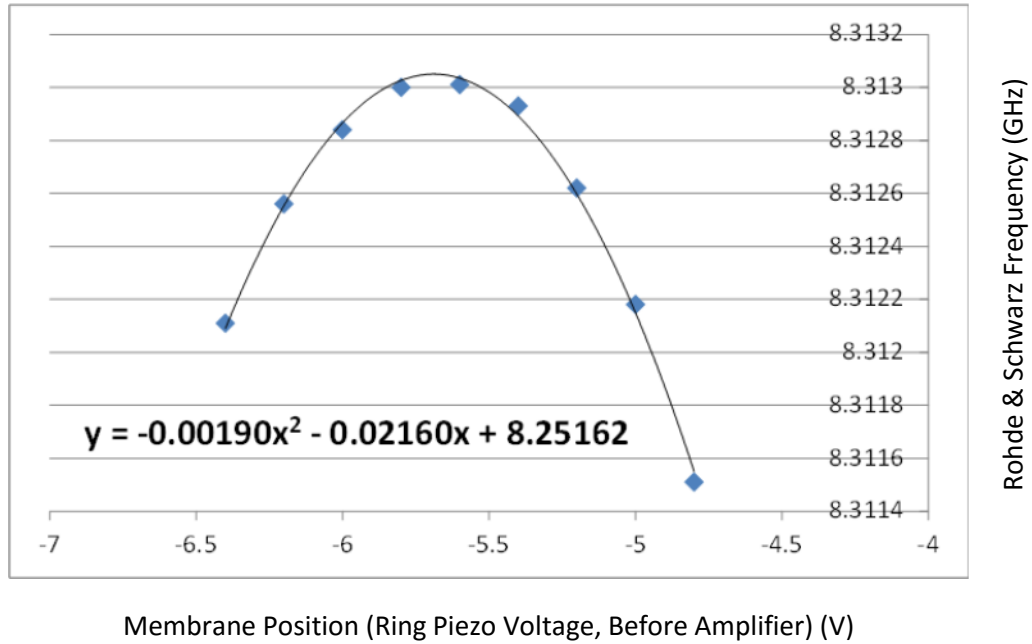


Figure 33: Plot of the frequency difference between the measurement and cooling laser optical modes as a function of membrane position.

By fitting the plot with a quadratic function, we find both the optimal membrane position and the frequency difference between the two modes at the point of maximally identical slopes in the cavity bandstructure. This point is called the “sweet spot”. The membrane position is then fixed at the sweet spot using many batteries connected in series to provide a stable DC voltage to the ring piezo actuator beneath the membrane. Since the piezo acts like a capacitor and has infinite impedance at DC, batteries connected to it will not drain, and will provide a much more stable voltage (and hence much more stable membrane position) than any high voltage amplifier can.

Once the membrane position is fixed, we set the cooling laser frequency to be red-detuned from the cavity mode and gradually step its frequency closer to resonance by hand until we see a dip in the DC reflection of light from the cavity, indicating that the beam is on resonance. We record the frequency

offset of the cooling laser at this point, and then detune it by $\Delta_p = -\omega_m$ to achieve the best laser cooling.

At this point, the lasers are both tuned to the appropriate optical modes with the correct detunings for a laser cooling experiment, however, both filter cavities are still bypassed, meaning that the lasers still have lots of classical noise. The final step in preparing for a measurement is therefore to lock the filter cavities. This is done by first breaking all the locks that were used to find the sweet spot. Next, filter cavity FC1 is locked to the free running measurement laser using a Pound-Drever-Hall technique. The circuit used to lock FC1 is shown schematically in Figure 34 below. Light reflected off FC1 is directed to a photodiode which connects to a mixing circuit to produce a Pound-Drever-Hall error signal. The error signal goes through a New Focus LB1005 servo, which drives a high voltage amplifier to feedback to a piezo within the filter cavity to change the cavity length, keeping it on resonance with the laser. The lock is initiated by sweeping the cavity length around resonance using a DC output from a Rigol signal generator until eventually the servo latches onto the resonance peak.

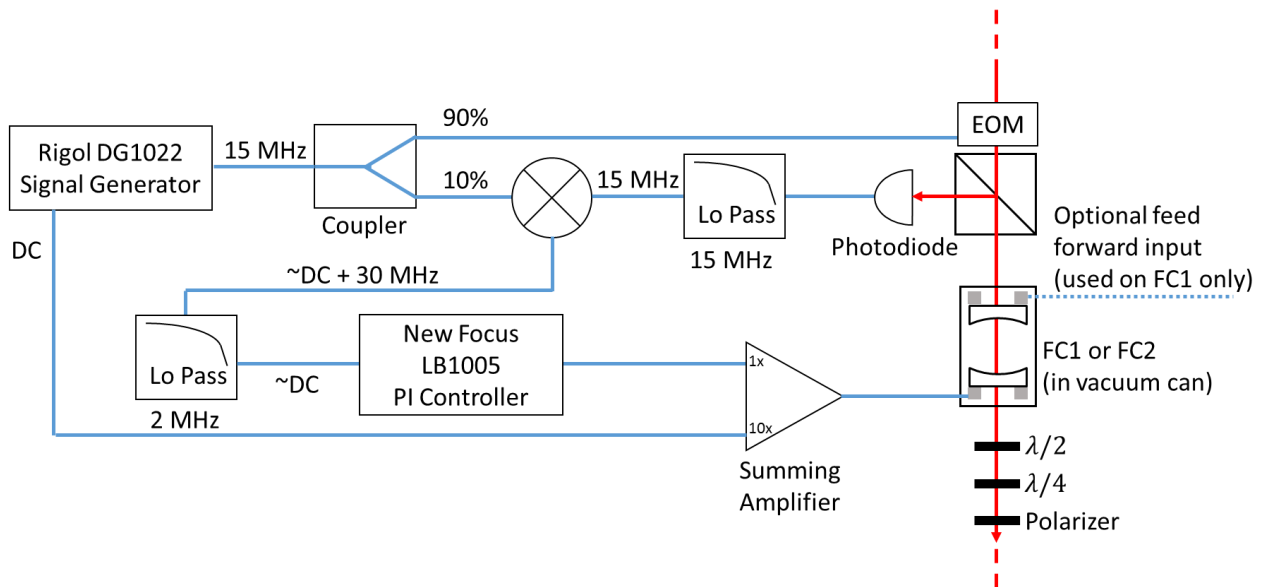


Figure 34: Schematic of the filter cavity locking circuits.

Once FC1 is locked, it is necessary to lock to the cold cavity again. Since the laser was previously locked to the optical mode of the cryogenic cavity before locking FC1, it is typically still close to resonance, and can be locked just by sweeping the drive frequency of AOM2. On occasion, if things have drifted, it is necessary to make slow adjustments to the measurement laser frequency using the measurement laser's piezo. These adjustments must be made very slowly so that FC1 can track the changes in frequency without unlocking. This is where feeding forward a portion of the ML piezo drive signal to FC1 is helpful, as it reduces the work that must be done by the PDH lock to track changes to the measurement laser frequency. Details of how the feed forward scheme works and the requirements necessary to implement it are described in Ref. [41].

Once the measurement laser has been locked to the cold cavity, the slock is relocked as already described at the optimal frequency for laser cooling, and then filter cavity FC2 is locked to the cooling laser. Now, since all the laser beams pass through AOM2 before reaching the cold cavity, they all track fluctuations of the cold cavity and stay appropriately on resonance with, or detuned from, the cavity modes. At this point it is possible to start measuring the behavior of the membrane as it is cooled by the laser.

4. Measuring the Membrane's Motion

As shown in Figure 17, the light reflected from the cryogenic cavity has several different frequency components. When these land on the measurement photodiode, interference between these components produces beat notes in the photocurrent at the sum and difference frequencies between all the components, as shown in Figure 18. The beat note between the local oscillator and the measurement beam and its mechanical sidebands produces components in the photocurrent at $80 \text{ MHz} \pm 705 \text{ kHz}$. The circuit we use to isolate and record these components works as follows. The signal from the photodiode first goes through a bias tee to separate the DC level from the RF

components. The DC level is sent to an oscilloscope for monitoring, but is not generally recorded. The RF components go through a power splitter, and a portion is picked off to generate the PDH error signal for locking. The remaining signal goes through an 80 MHz bandpass filter (Microwave Filter Company 3303 B50-5) with ~ 4 MHz bandwidth such that the RF beat note between the local oscillator and the measurement beam/sidebands can pass, while residual PDH locking signals and other noise is rejected. These ~ 80 MHz signals then mix with a 101.3985 MHz reference tone from an HP 8648A signal generator to produce a signal at 21.3985 MHz for digitization. The rather peculiarly precise frequency of 21.3985 MHz was selected due to the availability of extremely high quality factor crystal filters at this frequency, though those filters are no longer used in the experiment. At 21.3985 MHz, the photocurrent signal is now at low enough frequency for digitization by the Zurich Instruments HF2-LI lock-in amplifier.

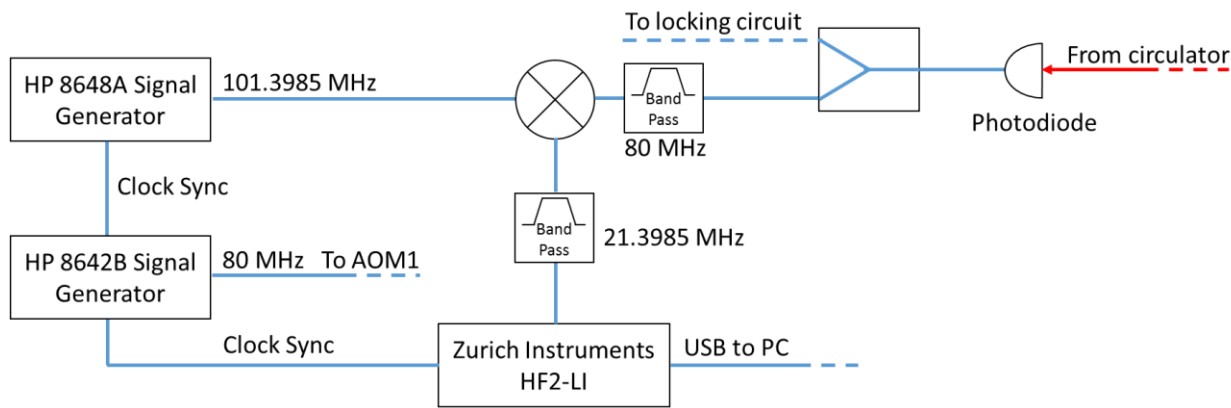


Figure 35: Electrical schematic of the circuit used to isolate and digitize the heterodyne signal from the photocurrent.

The Zurich Instruments HF2-LI lock-in amplifier is a fast lock-in amplifier with multiple demodulators that can simultaneously digitize up to six narrow bandwidth windows from up to two analog inputs that have a total bandwidth of up to 50 MHz each. During a typical measurement of the membrane, we use the HF2-LI to demodulate the mixed down photocurrent at three frequencies: the carrier frequency (21.3985 MHz), the red sideband (22.1035 MHz), and the blue sideband (21.6935 MHz). (The fact that the red sideband is at higher frequency than the blue sideband is due to the mixing of the 80 MHz photocurrent beat note with the 101.3985 MHz reference tone.) We set the bandwidth of

each demodulator to 115 kHz, and apply a multipole analog bandpass filter with a bandwidth of 8 kHz to the demodulated time traces. This filter helps to mitigate the effects of Shannon-Nyquist noise folding by attenuating folded-in noise below the noise floor of the demodulator. We then apply the inverse filter in post-processing so that the filter does not affect the portion of the photocurrent that includes the membrane's Brownian motion.

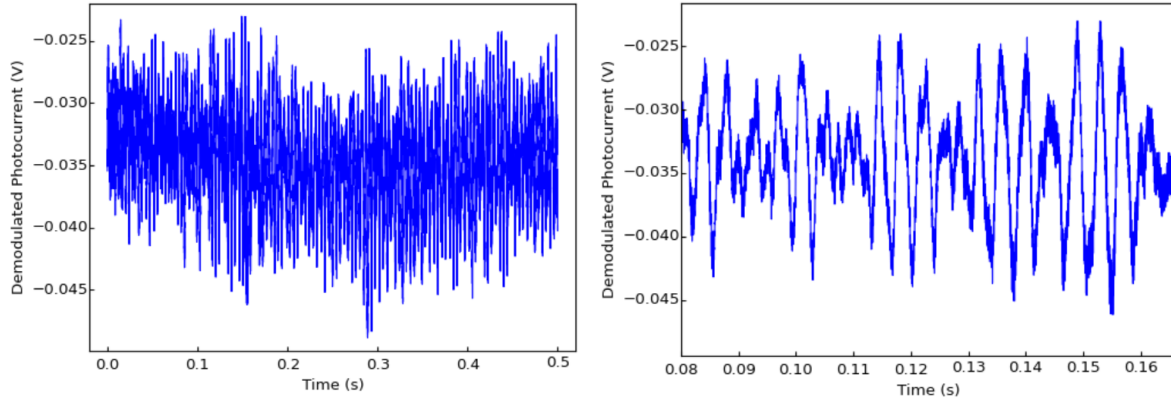


Figure 36: Left: A demodulated time trace of the red sideband of the membrane's motion at 10 μ W cooling power. Right: A zoom-in of the time trace. The membrane's motion appears as a low-frequency (several hundred hertz) oscillation in the demodulated time trace, due to a slight mismatch between the HF2-LI's demodulator frequency and the actual mechanical mode frequency.

The raw data that are recorded during a measurement are the demodulated time traces from the HF2-LI as shown in Figure 36 above. However, it is much more informative to look at the power spectral density of these time traces so that they can be analyzed using the heterodyne detection theory discussed in the Heterodyne Detection section – in particular, by comparison to Equations (72) and (77). To accomplish this, we use an iterative algorithm which begins by computing a power spectral density for a single time trace. Next, it computes a power spectral density for a second time trace, and averages it with the power spectral density of the first. This power spectral density is referred to as the average power spectral density. The algorithm then computes a power spectral density for a third time trace, computes a weighted average of the third power spectral density with the average power spectral density. If $S^{(n+1)}$ represents the power spectral density of the $(n + 1)$ th time trace, and $S_{\text{avg}}^{(n)}$ represents

the average power spectral density after n time traces, the following recursion relation describes how the average power spectral density is computed:

$$S_{\text{avg}}^{(n+1)} = \frac{nS_{\text{avg}}^{(n)} + S^{(n+1)}}{n + 1} \quad (84)$$

Using a recursive process to combine separate power spectral densities into an average power spectral density is convenient for several reasons. First, it allows for real time display of the power spectral density while collecting data. Second, it allows for computation of a power spectral density representing, potentially, hours of data, without the need to Fourier transform an hours-long time trace. Since the length of the time trace used to compute any individual Fourier transform or power spectral density determines the spectral resolution, we set the length of our time traces to 0.5 seconds, which corresponds to a resolution of 2 Hz. An hours-long time trace would produce a spectrum with millihertz resolution which could then be binned to 2 Hz to reduce noise, but would be more computationally complex to achieve, and would not allow for real-time viewing of the power spectral density.

For a laser cooling measurement, a typical measurement involves collecting sufficiently many time traces at each cooling laser power to clearly resolve the shape of the Brownian motion above the fuzz in the noise floor in the real time power spectral density. At low cooling powers, the Brownian motion is very large, and can be resolved after only a few dozen 0.5 second time traces. In practice, for the best data, we typically collect far more time traces than are actually needed to resolve the motion above the fuzz, so a low cooling power measurement might have an average PSD computed from on the order of 100 time traces. With a time trace length of 0.5 seconds, this represents a total data length of under a minute, which can be collected (after accounting for overhead in data acquisition) in 1-2 minutes. Figure 37 shows a typical low cooling power power spectral density.

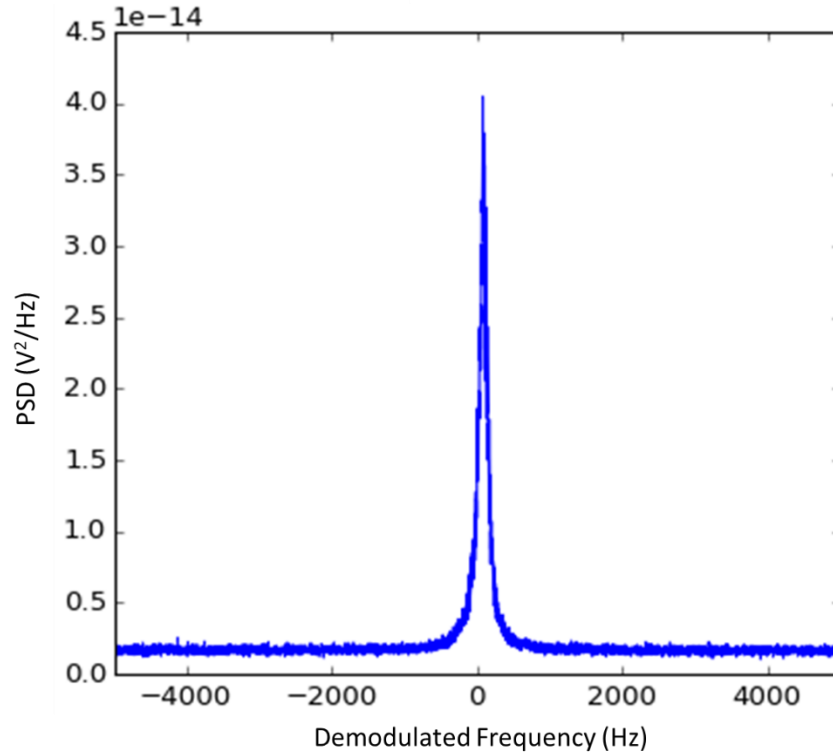


Figure 37: Typical average power spectral density of one sideband of the membrane's motion at a cooling power of 10 μ W. This power spectral density was produced using 102 time traces of 0.5 seconds each.

At the highest cooling powers, the Brownian motion is heavily damped and has a much smaller amplitude that is on the order of the size of the fuzz in Figure 37. In order to resolve the Brownian motion in the power spectral density when it is so heavily damped, it is necessary to collect drastically more time traces than at low cooling power. Since the fuzz in the power spectral density is assumed to be Gaussian distributed, its standard deviation should become smaller like the square root of the number of time traces that were used to compute the average power spectrum. Therefore, after averaging ten times more time traces, the fuzz gets smaller only by a factor of about 3.2. The longest practical measurement we can make (due to stability concerns, and the fact that humans eventually get tired), is on the order of several hours, or tens of thousands of seconds, and therefore achieving even an order of magnitude reduction in the size of the fuzz by collecting more data is difficult. However, since the heavily damped motion of the oscillator at high cooling powers can have linewidths of several

kilohertz, we are able to trade some spectral resolution in exchange for reduced fuzz by binning together adjacent points in the average power spectral density. Since our spectral resolution is 2 Hz and the linewidth of the Brownian motion is several kilohertz, selecting a bin size of 20 Hz (10x the original spectral resolution), for example, has no negative effect on the Brownian motion in the power spectrum, but reduces the fuzz in a way equivalent to if we measured 10x longer than we actually did. Therefore, a combination of long measurements and appropriate binning are used to produce good power spectral densities of data taken at high cooling powers. Figure 38 shows an example of the reduction in fuzz that can be achieved by binning from a spectral resolution of 2 Hz to a spectral resolution of 20 Hz as described above. (The ostensibly larger-than-3.2x reduction in the fuzz is an artifact of trying to display such a high spectral resolution in the left plot – relatively infrequent large deviations from the mean are smooshed together in the plot due to the limited resolution of the screen or printer from which you are reading this, making the fuzz seem larger. Nonetheless, the reduction in fuzz is still dramatic.) By looking at the real-time power spectral density as we collect the data and adjusting the bin size as needed, we are able to estimate when we have collected enough time traces for a given measurement.

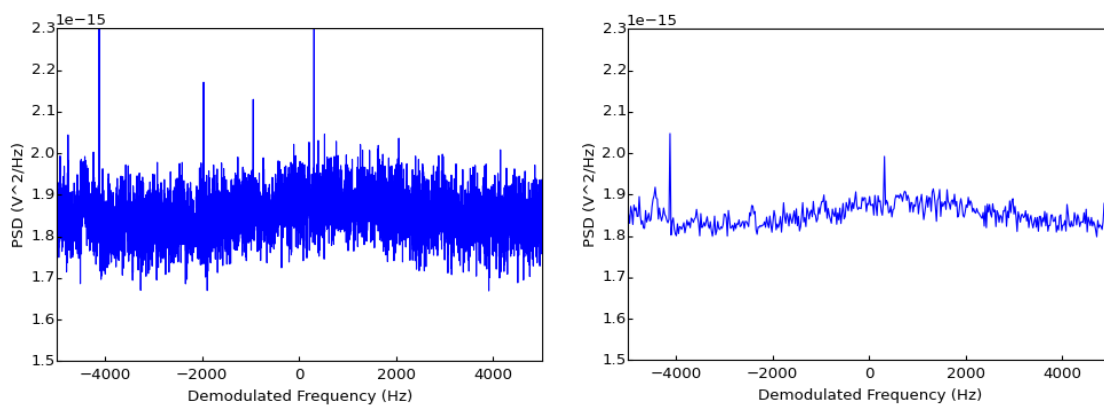


Figure 38: Left, an average power spectral density of the membrane's motion at 450 uW cooling power computed from 1,302 time traces (651 seconds of data). Right, the same average power spectral density after binning to reduce the spectral resolution by a factor of 10, in exchange for reducing the fuzz as if the power spectral density was computed from 13,020 time traces (6510 seconds of data).

5. Data Analysis

A. First Steps

After collecting an adequate number of time traces at each cooling laser power we begin data analysis of the average power spectral densities. While the power spectral densities are completely described by Equations (72), (77), and (81), we typically begin analysis by fitting the power spectral densities to a generic Lorentzian form. Using a generic Lorentzian instead of the full equations for the power spectral densities is a good first step in analysis because it is possible to extract the basic behavior of the mechanical mode without the need to carefully calibrate each parameter in Equations (72), (77), and (81). Fitting to just a Lorentzian rather than a Fano lineshape assumes that A_{rr} and A_{bb} are negligibly small compared with, L_{rr} and L_{bb} , and F_{rr} and F_{bb} respectively. This is a valid assumption in the limit of small classical laser noise on the measurement beam – an assumption which I will revisit later in the section Characterizing Parameters->Classical Laser Noise.

Before fitting the data, we first subtract the heterodyne carrier frequency from the x-axis such that the red sideband occurs at +705 kHz, and the blue sideband at -705 kHz. We then reflect the blue sideband about zero frequency for easier visual comparison with the red sideband, and scale the x-axis of both sidebands into units of kilohertz. The Lorentzian fits to the data are then simultaneously performed for the red and the blue sidebands. The form of the fits are:

$$\begin{cases} S_{rr} = b_{rr} + \frac{s_{rr}}{1 + \left(\frac{f - f_c}{f_h}\right)^2} \\ S_{bb} = b_{bb} + \frac{s_{bb}}{1 + \left(\frac{f - f_c}{f_h}\right)^2} \end{cases} \quad (85)$$

The red and the blue sidebands are each allowed to have different noise backgrounds b_{rr} and b_{bb} . This is because dispersion in the dark noise and gain of the photodiode/mixing circuit can cause the noise backgrounds of each sideband to differ. The amplitudes of the Lorentzian peaks, s_{rr} and s_{bb} , are

also allowed to differ between the red and the blue sidebands, as asymmetry in the height of the sidebands is one of the fundamental predictions of the behavior of a mechanical oscillator cooled to the ground state. As long as the heterodyne carrier beat note is centered precisely at the demodulation frequency of the HF2's first demodulator, both sidebands should be displaced from the carrier by the same frequency f_c . Also, both sidebands are always expected to have the same half linewidth f_h . Figure 39 show a typical fit using the expressions in Equation (85).

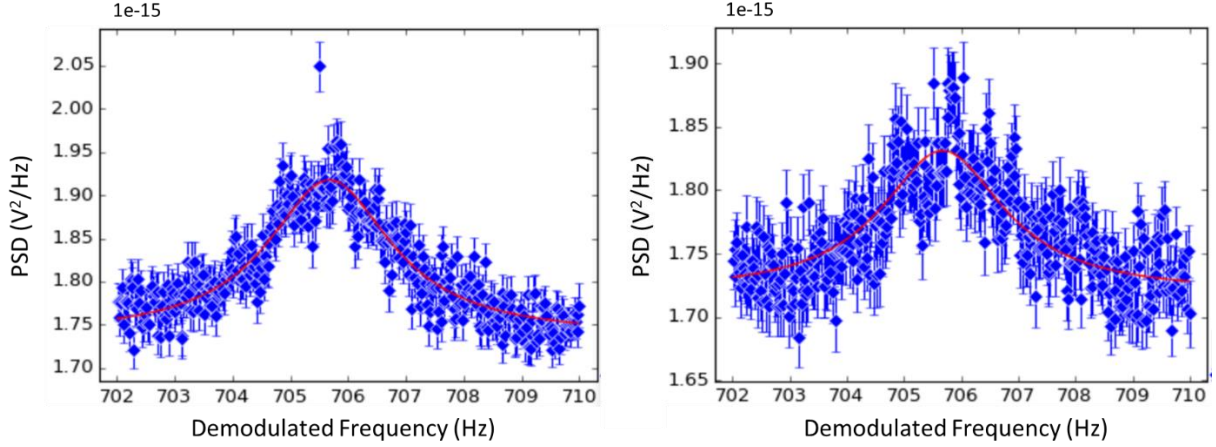


Figure 39: Example of a simultaneous fit to the red (left) and blue (right) sidebands of the membrane's Brownian motion at a medium cooling power of 250 μ W.

Once fits have been performed for all of the average power spectral densities from a given measurement, plots are produced to look at the membrane's behavior. The first plot to look at is a plot of the inverse area of the Lorentzian peak (L_{rr}^{-1} or L_{bb}^{-1}) vs. cooling power. Equations (74) and (79) show that in the absence of classical laser noise $L_{rr} \propto (n_m + 1)$ and $L_{bb} \propto n_m$. In the limit that the thermal bath temperature is fixed, the bare damping rate γ is fixed, the noise characteristics of the laser are fixed, and the power of the measurement and LO beams are fixed, Equation (65) predicts that $n_m \propto \gamma_p^{-1}$. Under the same assumptions, $\tilde{\gamma} \propto \gamma_p \propto P_p$, where P_p is the cooling beam power. Therefore, when plotting L_{rr}^{-1} and L_{bb}^{-1} as a function of P_p , L_{rr}^{-1} should be proportional to $\left(\frac{1}{P_p} + 1\right)^{-1}$, and L_{bb}^{-1} should be linear in P_p . In terms of the fitting parameters in Equation (85), the area can be represented, up to a

constant multiplicative factor, by $2000s_{rr}f_h$ for the red sideband or $2000s_{bb}f_h$ for the blue, where the coefficient of 2000 both converts the half linewidth f_h to full linewidth, and converts the units of the linewidth back to hertz (instead of kHz). Substituting this quantity for the actual L_{rr} and L_{bb} , the behavior of L_{bb}^{-1} vs. P_p is easily confirmed visually from a plot, and serves as a first sanity check on the data. Figure 40 shows a typical plot of inverse area vs. power, where the inverse area of the blue sideband is seen to be clearly linear.

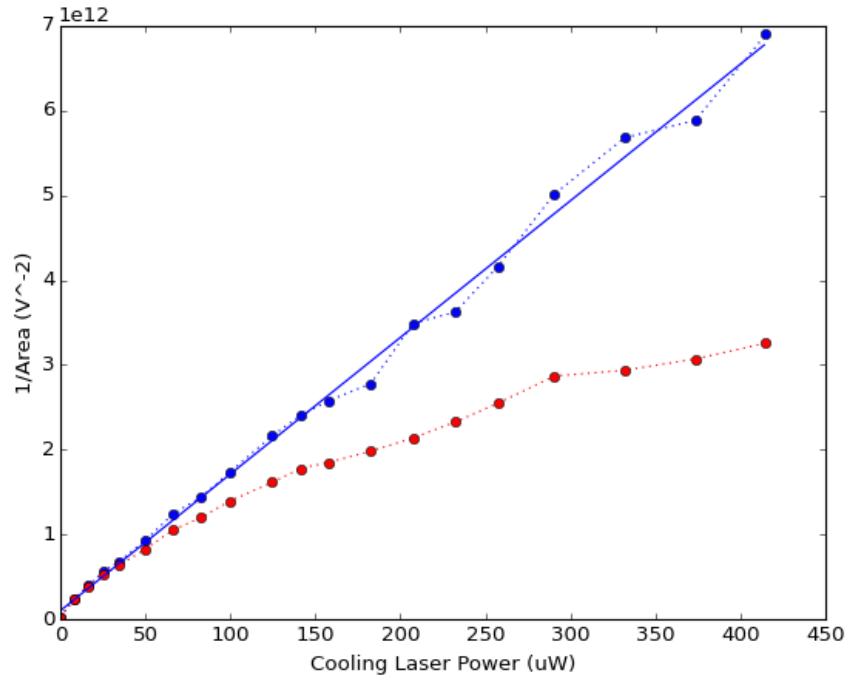


Figure 40: A plot of the inverse of the fitted Lorentzian area vs. power for the red and blue sidebands.

Seeing that the plot of inverse area vs. power resembles what we expect, we can conclude that the membrane's interaction with the cooling laser was strictly an optomechanical cooling interaction and that thermal heating (due to absorption of power from the cooling laser) is insignificant. If there had been significant thermal heating, n_{th} in Equation (65) would have increased with laser power, which would make the simple proportionality $n_m \propto \gamma_p^{-1}$ incorrect. This would cause the plot of the blue sideband's inverse area vs. power to be sub-linear, since the extra thermal contribution to the membrane's motion would increase the Lorentzian area.

A second check on the behavior of the membrane is to plot the linewidth $\tilde{\gamma} = \gamma + \sum_j \gamma_j$ of the sidebands vs. cooling power. The bare damping rate γ of the membrane is very small, and acts as a nearly negligible offset to this plot. Assuming the measurement beam power and detuning is constant, γ_s is constant as well, while γ_p scales linearly with cooling power (see Equation (64)). Therefore, $\tilde{\gamma}$ should be linear in cooling power with some small offset from the bare linewidth and damping due to the measurement beam. Figure 41 shows linewidth vs. power for a typical data set. Within the error bars, the linewidth does indeed appear to be linear in cooling power.

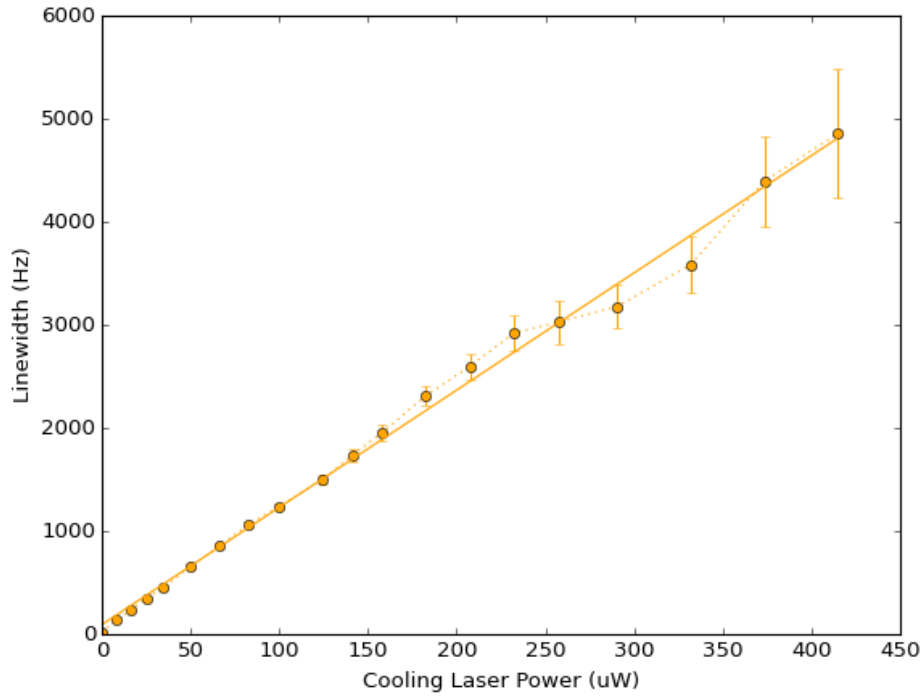


Figure 41: A plot of the linewidth of the sidebands (the linewidth of the red and blue sidebands are constrained by the fitting expressions to be identical) vs. cooling laser power.

The sideband asymmetry, which is the ratio of the area of the red sideband to the area of the blue sideband, provides another sanity check on the data. In the absence of classical laser noise, Equation (83) says that it should scale like $(n_m + 1)/n_m$. Since n_m should scale inversely with cooling power, the sideband asymmetry should therefore scale linearly with cooling power. Additionally, for large phonon numbers $n_m \gg 1$, the sideband asymmetry should equal 1. Figure 42 shows the sideband

asymmetry for a typical data set. It does appear to scale linearly in cooling power and has a y-intercept consistent with 1, as expected.

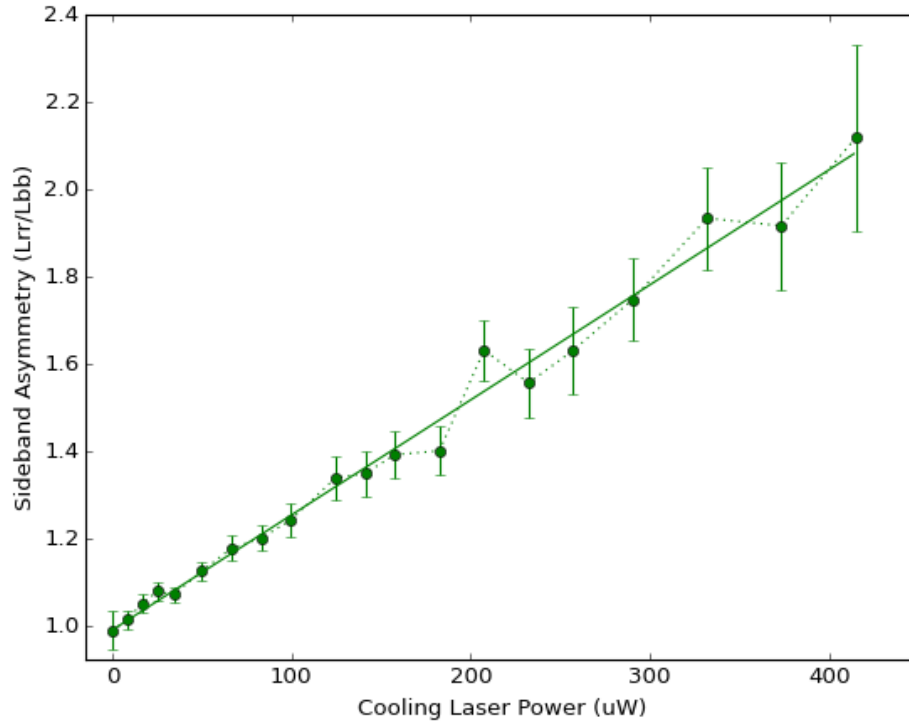


Figure 42: A plot of the sideband asymmetry for a typical data set. At low cooling powers the sidebands are symmetric, and they become less symmetric as the cooling power increases.

While the y-intercept of the linear fit to the sideband asymmetry data is consistent with 1 as expected, the data could still be influenced by asymmetric filtering of the mechanical sidebands due to the cavity susceptibility. This would occur if the measurement beam was locked slightly off resonance from the cavity. In practice, the Pound-Drever-Hall locking scheme used with the measurement laser generally tends to lock slightly to the red side of the cavity resonance, which would enhance the blue sideband while suppressing the red. Figure 43 is an exaggerated cartoon plot showing how a detuned measurement beam can lead to asymmetric filtering of the sidebands. In Figure 42, this would manifest as a downward shift of all the data points.

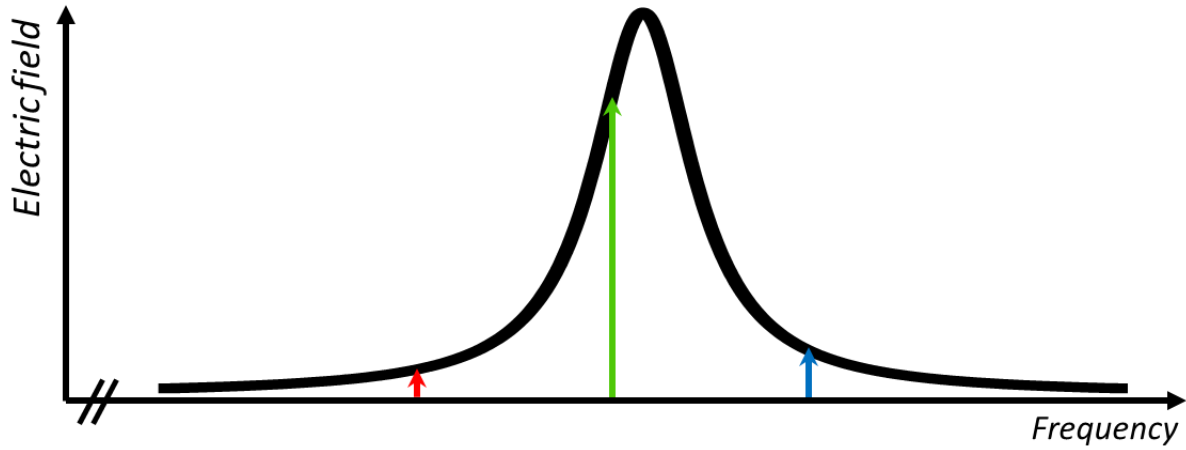


Figure 43: An exaggerated cartoon schematic showing how a red-detuned measurement beam leads to asymmetric filtering of the mechanical sidebands. The red sideband is suppressed more than the blue sideband, which can produce sideband asymmetries less than one at low cooling powers.

There are also other effects which can skew the sideband asymmetry data. First, recall from Equations (72) and (77) that S_{rr} and S_{bb} have separate gain terms G_r and G_b . If $G_r \neq G_b$ then the sideband asymmetry can be shifted larger or smaller than it really is. Second, if there is classical laser noise on the measurement beam, then Equation (82) predicts that that the red sideband will appear larger than it should and the blue sideband will appear smaller than it should compared with a noiseless measurement beam. (Noise on the cooling beam can increase the membrane's phonon number causing the sidebands to be more symmetric, but in this case the effect on the asymmetry is due to an actual change in n_m and not a direct effect of laser noise through \tilde{B} as is the case for noise on the measurement beam.) Therefore, in order to correct the sideband asymmetry plot for all of these effects, we need to independently measure the measurement beam detuning Δ_s , the photodiode amplifier's gain at $\pm\omega_m$ (G_r and G_b), and ensure that there is no classical laser noise.

B. Characterizing Parameters

Cavity Parameters

While the finesse of the optical cavity can be predicted theoretically for mirrors that perform exactly to the manufacturer's specifications, in practice, mirrors sometimes have higher reflectivity than specified, or can become dirty, which reduces reflectivity below the manufacturer's specifications. Additionally, the presence of the membrane inside the cavity can result in extra scattering or absorption, reducing the finesse of the cavity from the theoretical maximum.

For a first estimate of the optical linewidth κ , we simply sweep the frequency of the laser over the cavity resonance and record the DC reflection spectrum and the PDH error signal on an oscilloscope. Since the laser has PDH sidebands at a known frequency, we can use these sidebands to calibrate the time axis of the sweep into units of laser frequency. The PDH sidebands are generally very small, and so can be hard to see in the DC reflection, but they are very visible in the recorded error signal. Since both the error signal and DC reflection are recorded simultaneously, we can use the error signal time trace to calibrate the time axis of both plots into units of laser frequency, and then fit the optical resonance in the DC reflection to a Lorentzian to determine its linewidth. κ is this linewidth converted into radians per second. Figure 44 shows the recorded error signal and DC reflection used to make this measurement.

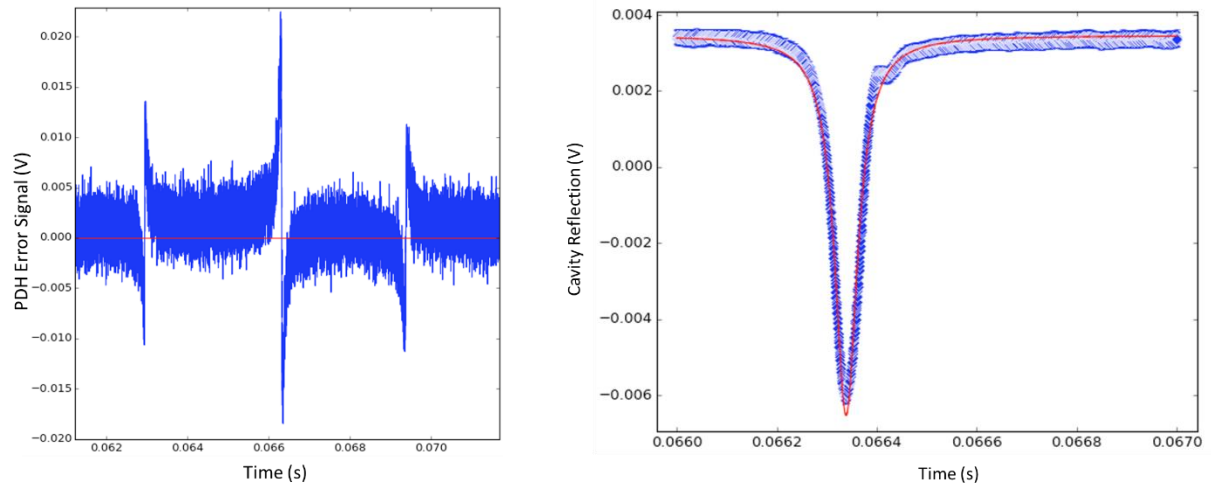


Figure 44: Left, the PDH error signal recorded as the laser was swept over the cavity resonance. The sidebands were applied to the laser using an EOM and are at a frequency of 10.8 MHz away from the carrier, meaning the time axis calibration factor is approximately 3200 MHz/sec. Right, a Lorentzian fit to the optical resonance. The linewidth $\kappa/2\pi \approx 176$ kHz.

While this technique does provide a good estimate of the optical linewidth, it is susceptible to errors introduced by fluctuations in the cavity's resonance frequency. Since it takes finite time to sweep over the cavity resonance, a change in the resonant frequency of the cavity during the sweep can cause the Lorentzian to be stretched or squished in a way that would give an ostensibly larger or smaller linewidth. Therefore, we confirm this result by fitting measurements of the optical spring and damping at fixed laser power as a function of cooling beam detuning. Our best measurement of κ for the cavity used in the ground state cooling experiment was actually obtained by performing simultaneous fits to the optical spring and damping of the (1,3) and (3,1) mechanical modes versus laser detuning at several laser powers. The values of κ from each set of these fits were then averaged to obtain our best estimate of κ . (These fits were ancillary to the first steps of a new experiment studying interactions between nearly degenerate mechanical modes.⁵⁶) Below in Figure 45 is one of several fits from which a value of κ was extracted. The best estimate of κ obtained by averaging the results of several of these fits together is $\kappa/2\pi = 165$ kHz, which agrees with the value of κ determined by cavity sweeping within $\sim 6\%$.

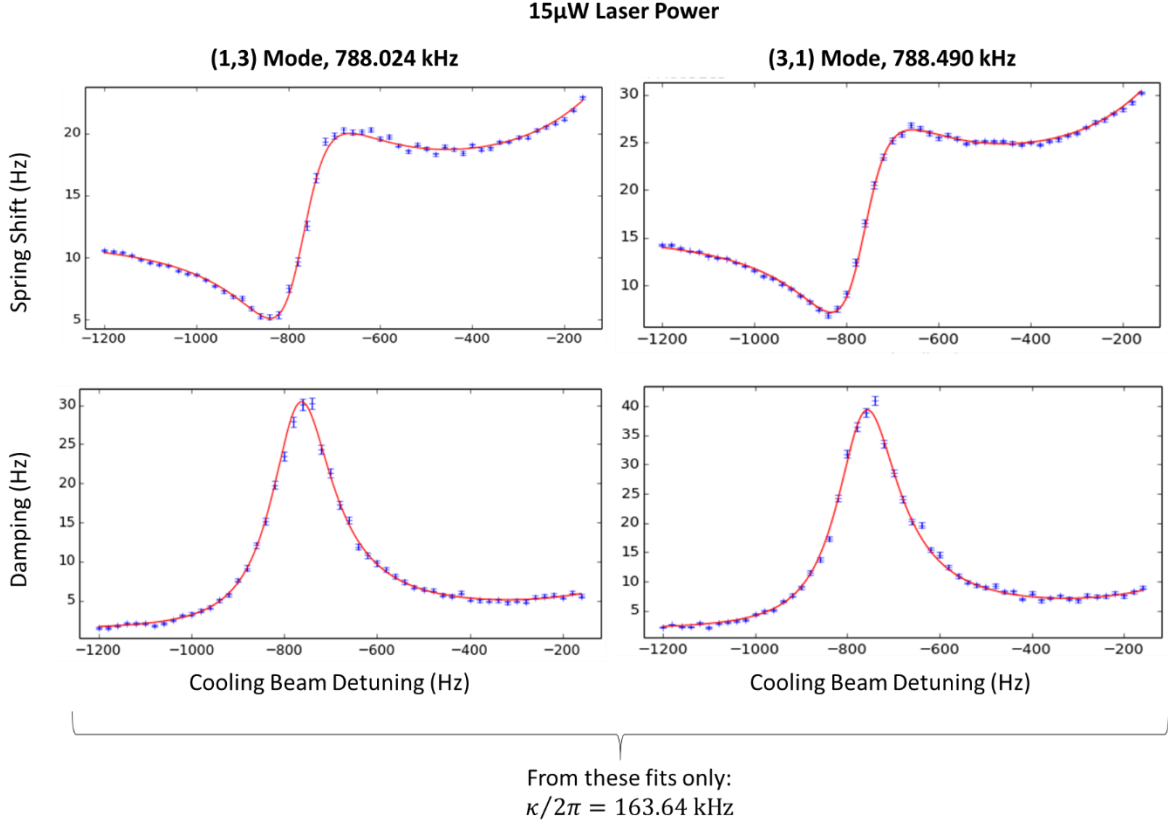


Figure 45: Simultaneous fits to the optical spring and damping of the (1,3) and (3,1) modes to obtain κ .

κ_{in} is the cavity parameter which characterizes the transmission through the mirrors of the optical cavity. Recalling from Equation (68) the expression for the normalized amplitude of the reflected field after the laser's interaction with the cavity, $\rho(\Delta) = \beta - \frac{\kappa_{\text{in}}}{\kappa/2 + i\Delta}$, it is possible to solve for κ_{ext} using the measured value of κ and the value of β which shall be discussed in the next section. Precisely on resonance, $\Delta = 0$, and $\rho(0) = \beta - 2\kappa_{\text{in}}/\kappa$. This can be solved for $\kappa_{\text{in}} = \frac{\kappa}{2}(\beta - \rho(0))$. By this method we find $\kappa_{\text{in}}/2\pi \approx 65$ kHz.

Optical Losses and Efficiencies

Since the cooling measurement of the mechanical mode measures the mode's behavior as a function of cooling laser power, it is essential to know exactly how much laser power reaches the optical cavity, how much of that power returns to the photodiode, and how much of the power reaching the

photodiode is actually detected. These efficiencies and losses are described by several parameters, most of which can be independently measured. Following the circulator in Figure 16, there is a 99:1 fiber beam splitter which is used to pick off 1% of the optical power to measure with a power meter. The power meter is calibrated so that from the 1% output of the beam splitter, it reads the amount of power that is actually coming out of the 99% port. The 99% port is fusion spliced onto the fiber that goes into the fridge. The loss in this splice is one of the few losses that cannot be directly measured, but it is estimated based on other losses as will be described.

Once in the fiber that goes into the fridge, light is assumed to pass losslessly out the fiber collimator and then in free-space to the optical cavity. Light that reflects back from the optical cavity recouples into the optical fiber with an efficiency which we can measure. This measurement is performed by sending light into the fridge from free-space directly into the fridge fiber (without an inline circulator or 99:1 beamsplitter) and measuring how much power reflects back. We then cut the fiber and measure how much light comes out to determine the free-space-to-fiber coupling. The ratio of the the power of the reflected light to the power of the light coming out of the cut fiber gives the fiber recoupling efficiency inside the fridge, $\beta^2 \approx 0.77$. It should be noted that this measurement is performed without locking the lasers to the cavity, and therefore this recoupling efficiency is only valid for light that promptly reflects off the cavity's input mirror. The equivalent parameter for light that couples into the cavity would be a coefficient inserted before κ_{in} everywhere that κ_{in} appears in the theory. We cannot independently measure this coefficient in our experiment, so we absorb it into κ_{in} . Therefore, the fiber recoupling efficiency of light that couples into the cavity is accounted for with our measured value of κ_{in} .

The detection efficiency σ represents the total efficiency in detection of light that is reflected from the optical cavity, excluding the fiber recoupling efficiency β . The detection efficiency is therefore the product of the power transmissivities of the splice between the fridge fiber and the 99:1 beam

splitter, the 99% output port of the beam splitter, the fiber splice between the 99:1 beam splitter and the circulator, and the circulator's output to the photodiode, along with the quantum detection efficiency of the photodiode itself. A schematic indicating the location of each of these points is shown in Figure 46 below.

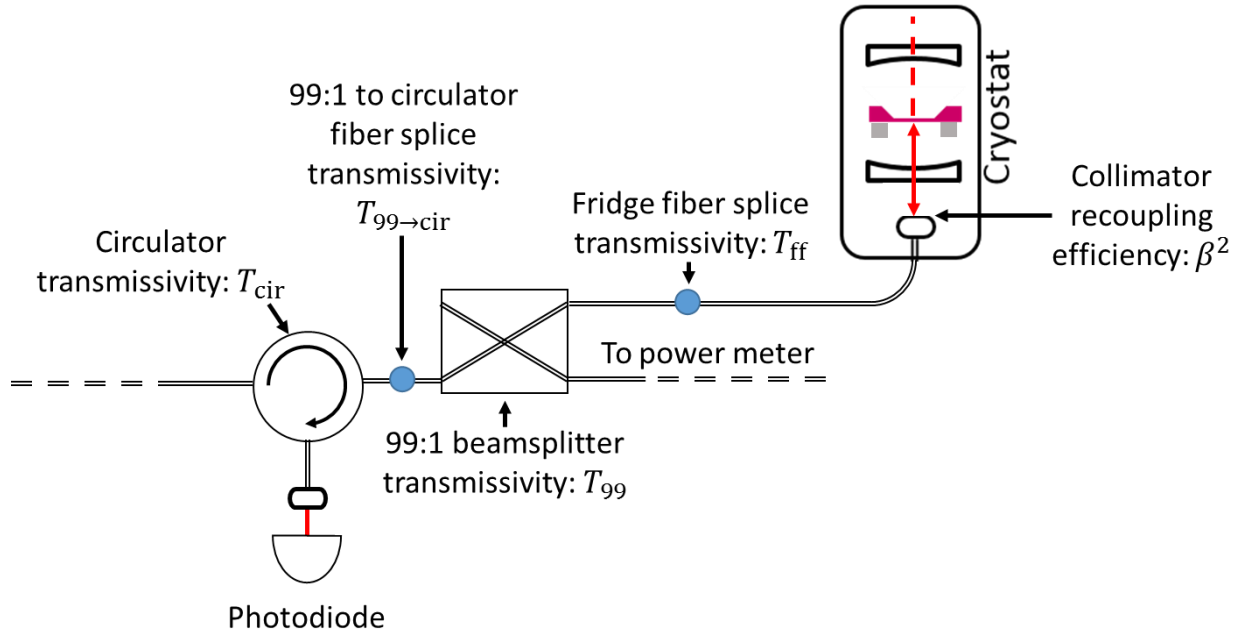


Figure 46: Schematic indicating location of components whose transmissivities contribute to the overall detection efficiency.

To calculate σ , we send power into the optical circulator so that it enters the fridge and reflects back to the photodiode. We measure the amount of power we are sending in using the calibrated power meter on the 1% port of the beam splitter, and also measure the power reflecting back just before the photodiode. We find that the round trip power transmissivity is $T_{\text{tot}} = 0.303$. A table of the measured transmissivities of individual components follows, in the order that light going into the fridge encounters them:

Table 1: Transmissivities of components that couple light into and out of the fridge.

Fridge fiber splice, T_{ff}	Unknown
Fiber recoupling ratio, β^2	0.77
Fridge fiber splice, T_{ff}	Unknown
99:1 beam splitter, 99% output port, T_{99}	0.98
99:1 to circulator fiber splice, $T_{99 \rightarrow \text{cir}}$	0.87
Circulator, T_{cir}	0.67

$$T_{\text{tot}} = T_{\text{ff}}\beta^2 T_{\text{ff}}T_{99}T_{99 \rightarrow \text{cir}}T_{\text{cir}}, \text{ so therefore } T_{\text{ff}} = \sqrt{\frac{T_{\text{tot}}}{\beta^2 T_{99}T_{99 \rightarrow \text{cir}}T_{\text{cir}}}}. \text{ Plugging in numbers gives}$$

$T_{\text{ff}} = 0.83$. The quantum efficiency of the photodiode is determined by measuring its responsivity in amps of photocurrent per watt of light. We do this by shining a known amount of light onto the photodiode and measuring the DC level of the photocurrent on an oscilloscope. We measure $R \cong 0.65 \text{ A/W}$ responsivity, which is consistent with the manufacturer's specifications and corresponds to a quantum efficiency of $\sigma_{\text{QE}} = R \hbar \omega_L / e = 0.757$ (here e is the magnitude of the electron charge). Therefore $\sigma = T_{\text{ff}}T_{99}T_{99 \rightarrow \text{cir}}T_{\text{cir}}\sigma_{\text{QE}} = 0.359$.

Amplifier Gains and Circuit Losses

We extract the total gain of the measurement circuit, which includes losses from mixers, filters, and other components in between the photodiode and the HF2-LI, by fitting the dependence of the photocurrent noise floor upon the incident cooling laser power. These fits are shown in Figure 47. The slopes of these fits give the prefactors in Equations (72), (77), and (81). Since β^2 , σ , r , and K^2 are already known, this allows us to solve for G_r and G_b . We find $G_r = 5.548 \times 10^{-16} \text{ V/Hz}$ and $G_b = 5.533 \times 10^{-16} \text{ V/Hz}$.

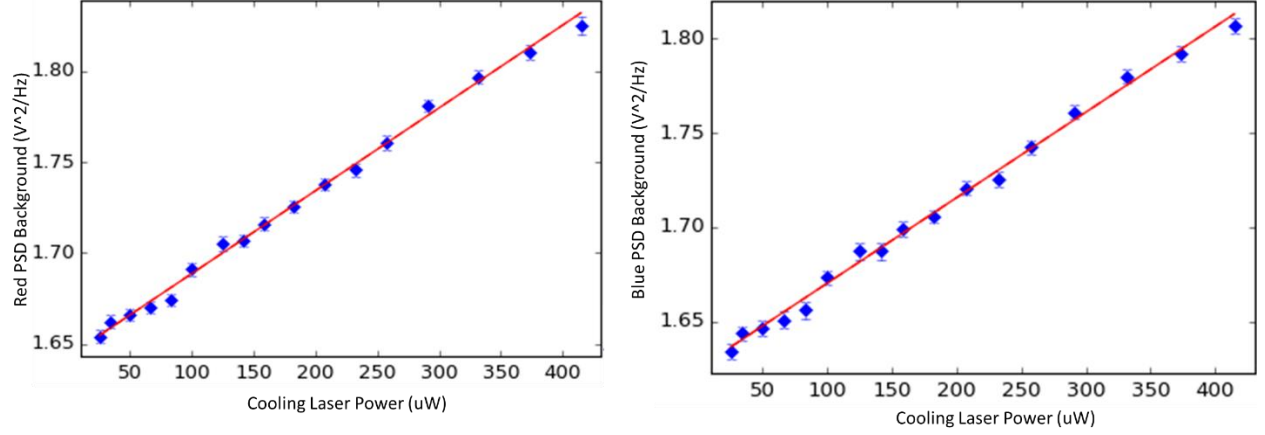


Figure 47: Fits to power spectral density background as a function of cooling power. Left, red sideband. Right, blue sideband. The slope of the linear fit represents the measurement circuit gain.

Optomechanical Coupling and Cooling Beam Detuning

By performing simultaneous fits to the mechanical linewidth and the mechanical frequency as a function of cooling power, we can extract the optomechanical coupling and cooling beam detuning. The expressions used to fit the mechanical linewidth and frequency are Equations (63) and (64) which give the optical spring and damping, where the cooling beam detuning Δ_p appears explicitly, and the dependence on optomechanical coupling is through the term $|\alpha_j|^2$. The linewidth and frequency of the mechanical mode at zero cooling power is added to the expressions in Equations (63) and (64) before performing the fit. These fits are shown in Figure 48. The optomechanical coupling is found to be 4.033 MHz/nm, and the cooling beam detuning is found to be $\Delta_p/2\pi = -664$ kHz.

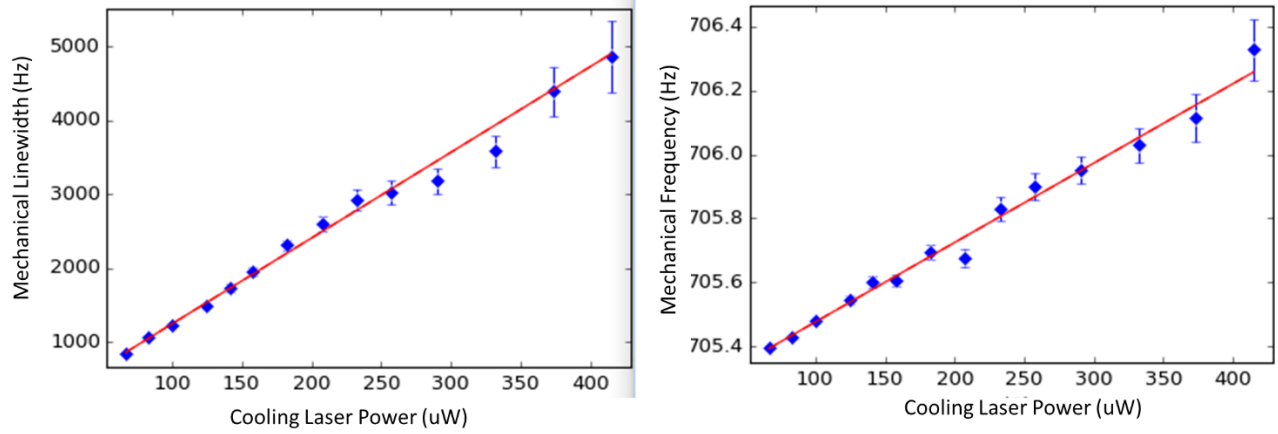


Figure 48: Simultaneous fits to the optical damping (left) and optical spring (right) as a function of cooling power.

Bare Mechanical Linewidth

We characterize the bare linewidth of the mechanical mode using a time-domain mechanical ringdown measurement. To perform this measurement, we use a laser that is outside of the cavity mirrors' high reflectivity bandwidth, a ThorLabs 1310 nm fiber laser. Use of this laser is necessary because locking the 1064 nm laser to the cavity when the membrane is oscillating with high amplitude is extremely difficult, and because the optical damping effect caused by 1064 nm laser perturbs the linewidth significantly even for a weak beam that is nearly on resonance. The 1310 nm laser can enter the cavity without any need for locking because the finesse of the cavity at 1310 nm is close to unity due to the lower reflectivity of the mirrors at this wavelength. Also, since the finesse is so low, the optomechanical interaction (and optical damping) is negligible.

Once the 1310 nm laser is on, we use the HF2-LI to drive the membrane at a frequency near ω_m . Light from the 1310 nm laser enters the cavity and reflects off of the driven membrane, carrying away information about the membrane's motion in the phase of the light. This light then combines with light that promptly reflected off the cavity input mirror inside the cryostat. When the two light beams eventually return to the optics table, they pass through a circulator which directs them to a photodiode.

The interference between the two beams then produces amplitude modulation of the photocurrent from the detector. This signal is then sent back to the HF2-LI, where it is demodulated to record the amplitude of the detected membrane motion. Figure 49 shows a schematic of the electrical and optical circuits used to make this mechanical ringdown measurement.

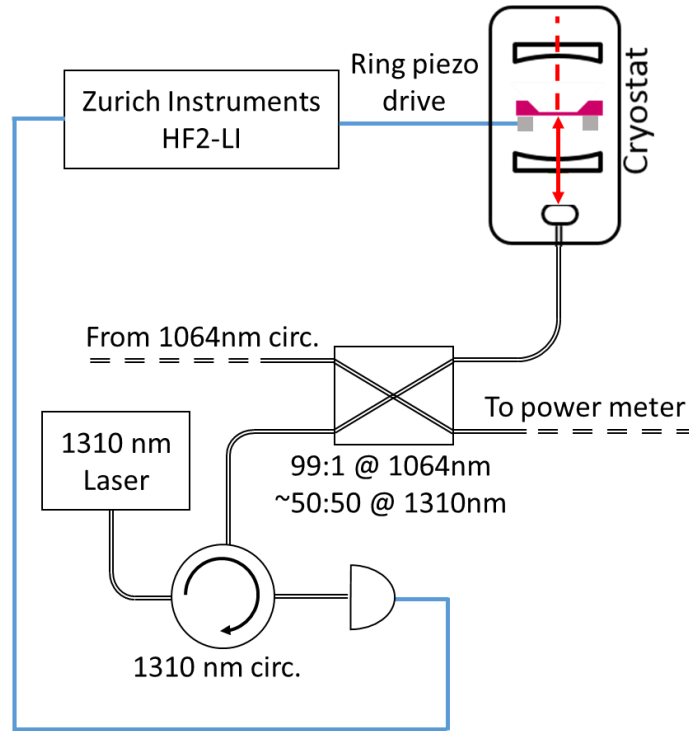


Figure 49: Schematic of the measurement circuit for making mechanical ringdown measurements.

The measurement is performed by adjusting the drive frequency as needed until the drive is precisely on resonance with the membrane, and then abruptly turning the drive off, recording the demodulated photocurrent in the process. A typical mechanical ringdown time trace is shown in Figure 50. This raw time trace contains three ringdowns (the regions of exponential decay) which can each be fit to determine an average ringdown time. The ringdown time $\tau = 2/\gamma$. From this, it is possible to compute the quality factor of the mode, $Q = \pi\tau\omega_m/2\pi = \omega_m/\gamma$.

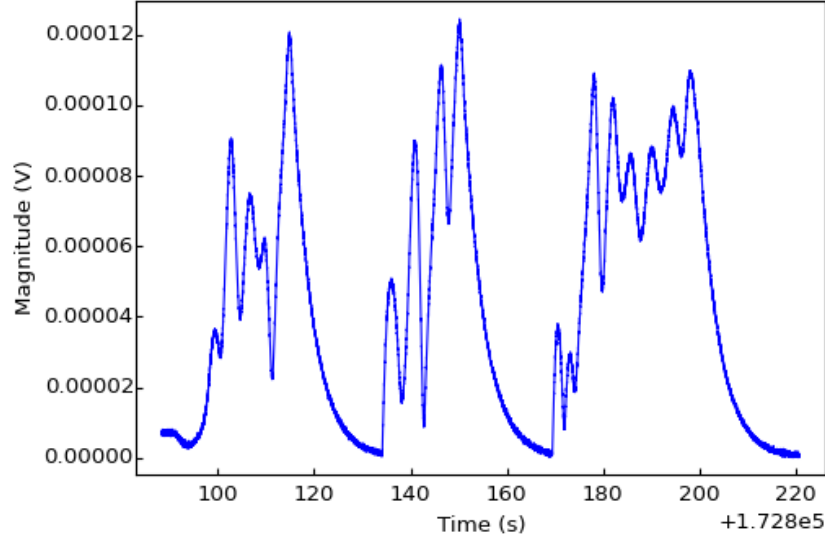


Figure 50: A plot of the demodulated time trace of a mechanical ringdown measurement. Shown are three ringdowns (areas where the magnitude undergoes an exponential decay). Oscillations between ringdowns occur as the drive frequency is tuned by hand to match the resonant frequency of the mechanical mode.

The magnitude of each ringdown should be given by:

$$f(t) = \left| a \exp\left(-\frac{t-t_0}{\tau}\right) \exp(i\Delta t) + b \exp(i\phi(t)) \right| \quad (86)$$

$$= \sqrt{\left(a \exp\left(-\frac{t-t_0}{\tau}\right)\right)^2 + b^2 + ab \exp\left(-\frac{t-t_0}{\tau}\right) \cos(\Delta t - \phi(t))}$$

In this expression, a is an amplitude determined by the strength of the piezo drive, the power of the 1310 nm laser, and other factors, t_0 is the time at which the ringdown began, τ is the ringdown time as described above, and b is the measurement background. Δ is the frequency difference between the mechanical oscillator and the HF2-LI's demodulator frequency and $\phi(t)$ is the random phase of the noise that makes up the measurement background. As we are not attempting to fit the time dependence of the noise term, we may average over all ϕ , eliminating the final term under the radical. Then, the fitting expression below may be used:

$$f(t) = \sqrt{a^2 \exp\left(-\frac{2(t-t_0)}{\tau}\right) + b^2} \quad (87)$$

An example of a fit to one of these ringdowns is shown in Figure 51.

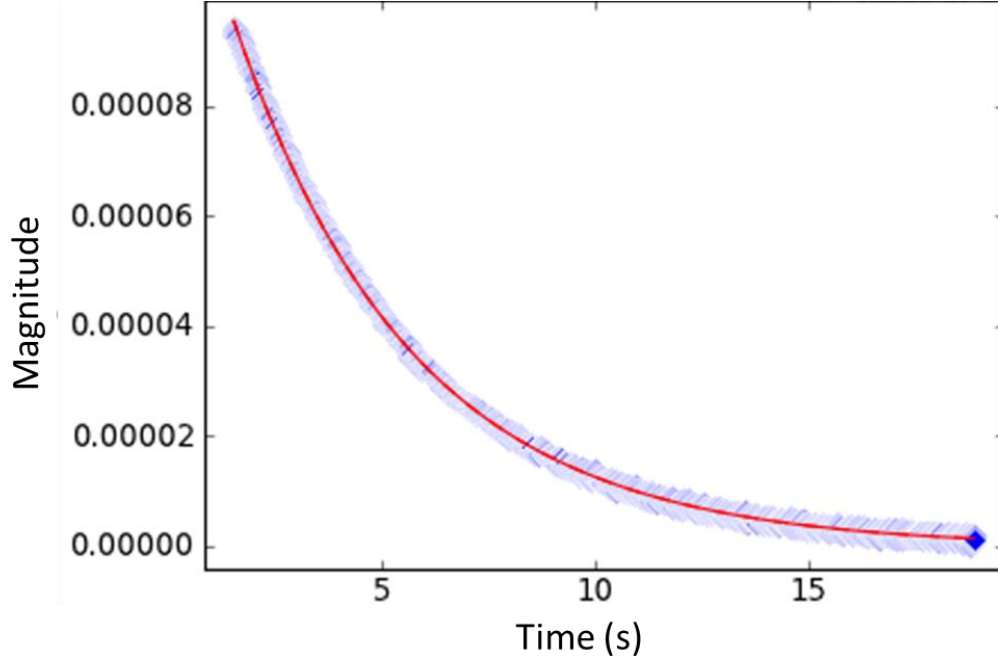


Figure 51: A plot showing a least squares fit to a mechanical ringdown. In this case, $\tau = 4.2$ s, so $\gamma = 2\pi \times .075$ Hz and $Q = 9.3 \times 10^6$.

We performed many measurements of the quality factor of the membrane's mechanical mode and found that it generally varied between 4 and 10 million. We were unable to directly tie this variation to changes in bath temperature or other factors, though we did notice that large temperature swings, such as the weekly warm up to 4 K when the ^3He in the fridge boiled off, would often cause Q to change. We also, on one occasion, warmed the stage to around 20 K while simultaneously pumping on the IVC with a turbomolecular pump to remove any helium that may have leaked in over time, and noticed a markedly higher Q afterward. In practice, there is no way to directly measure the membrane's Q during an actual optomechanics measurement, so we simply accept the range over which we saw Q vary and treat it as providing rough upper and lower bounds for the likely Q during measurement. In our final

data analysis, we treat Q as an empirically determined parameter, chosen from within the range of measured Q 's to maximize self-consistency of the data.

Measurement Beam Detuning

To compensate for asymmetric sideband filtering as shown in Figure 43, we estimate the measurement beam detuning from the linewidth of the Lorentzian in the zero cooling power power spectral density. The fit used to extract this linewidth is shown in Figure 52.

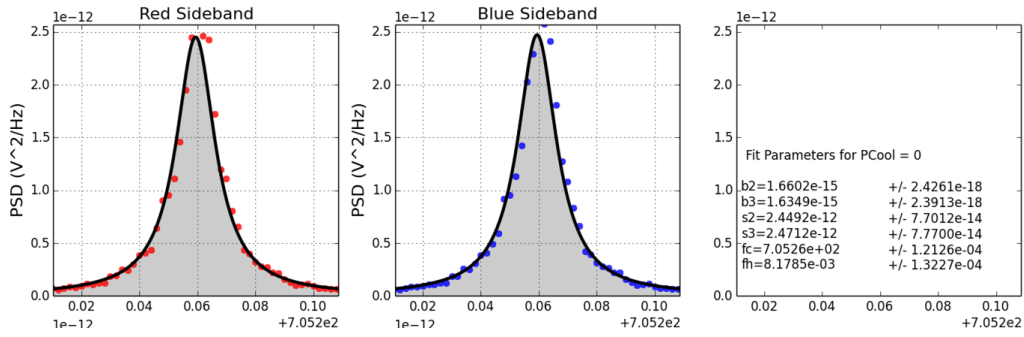


Figure 52: Plot of the power spectral density of the mechanical mode's motion with the cooling laser blocked. Damping in excess of the bare mechanical linewidth is due to non-zero detuning of the measurement beam.

γ_s in Equation (64) gives the damping of the mechanical mode due to the measurement beam as a function of detuning and power. Algebraically inverting this expression to solve for detuning as a function of damping and power is not feasible, but through numerical methods we find the measurement beam detuning to be $\Delta_s/2\pi \approx -6.5$ kHz.

Classical Laser Noise

Throughout this chapter on ground state cooling, I have given general expressions for the behavior of the membrane with classical laser noise, and then described what should happen in the absence of classical laser noise. In order to use these noiseless expressions to describe the measurement, it is necessary to confirm, as best as possible, that there is in fact negligible laser noise in

our experiment. We attempted to independently characterize the noise of both lasers using several techniques, the first of which is a self-homodyne measurement with a long fiber optic delay line.

The basic optical setup of the delayed self-homodyne measurement is shown in Figure 53. The laser light to be tested is split into two paths. One travels through an $l = 435$ m fiber optic delay line, which due to its length, takes the light $\tau \approx 2.1$ μ s to traverse, assuming the fiber's index of refraction is $n = 1.4585$. The other path is negligibly small, and light can be assumed to traverse it instantaneously. The light that took the longer path accumulates an extra phase $\Delta\phi = \Omega_L \tau$ at its nominal frequency Ω_L , or $\Delta\phi = (\Omega_L + \Delta_{\text{noise}})\tau$ at the frequency of laser noise that is $(\Delta_{\text{noise}}/2\pi)$ Hz away from the laser's nominal frequency.

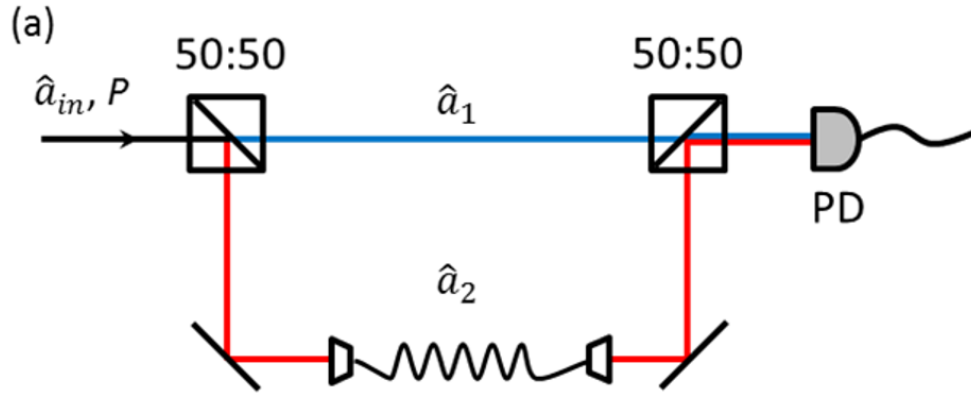


Figure 53: Schematic of the optical setup used for the delayed self-homodyne laser noise measurement.

We can formally derive the how this system allows for measurement of laser noise and extraction of parameters C_{xx} , C_{xy} , and C_{yy} . The fields \hat{a}_1 and \hat{a}_2 can be written as follows

$$\hat{a}_1(t) = \frac{\hat{a}_{in}(t)}{\sqrt{2}} + \frac{i\hat{\xi}'(t)}{\sqrt{2}} = \frac{1}{\sqrt{2}} \left[K + \frac{1}{2}(\delta x(t) + i\delta y(t)) \right] e^{i\Omega_L t} + \hat{\xi}_1(t) \quad (88)$$

$$\hat{a}_2(t) = \frac{\hat{a}_{in}(t - \tau)}{\sqrt{2}} + \frac{i\hat{\xi}'(t)}{\sqrt{2}} = \frac{1}{\sqrt{2}} \left[K + \frac{1}{2}(\delta x(t - \tau) + i\delta y(t - \tau)) \right] e^{i\phi(t)} e^{i\Omega_L t} + \hat{\xi}_2(t) \quad (89)$$

In Equation (89) $\phi(t) = -i\Omega_L \tau + \delta\phi(t)$ represents the extra phase accumulated by the delayed beam as it traverses the fiber optic delay line due to the non-zero traversal time, signified by $-i\Omega_L \tau$, and

due to large (but slow compared with τ) fluctuations in the length of the delay line due to vibrational, acoustic, or thermal effects, signified by $\delta\phi(t)$. When the beams recombine at a 50:50 beam splitter, the total optical field can be written as the sum of $\hat{a}_1(t)/\sqrt{2}$ and $\hat{a}_2(t)/\sqrt{2}$:

$$\hat{a}(t) = \frac{1}{2} \left\{ \left[K + \frac{1}{2} (\delta x(t) + i(\delta y(t))) \right] + \left[K + \frac{1}{2} (\delta x(t - \tau) + i\delta y(t - \tau)) \right] e^{i\phi(t)} \right\} e^{i\Omega t} + \hat{\xi}(t) \quad (90)$$

In this expression, the total vacuum noise has been rewritten as just $\hat{\xi}(t)$.

As in the heterodyne case, the power spectrum can be derived using the Weiner-Khinchin theorem from the current autocorrelation function given by Equation (71), except now $\hat{I}(t)$ arises from the fields of the homodyne measurement:

$$\hat{I}(t) = \frac{K}{2} \left\{ \left[K + \frac{1}{2} (\delta x(t) + \delta x(t - \tau)) \right] \left(1 + \cos(\phi(t)) \right) + \frac{1}{2} (\delta y(t) + \delta y(t - \tau)) \sin(\phi(t)) \right\} \quad (91)$$

Since laser noise is generally assumed to be small, terms quadratic in $\delta x(t)$ or $\delta y(t)$ can be neglected. Exploiting Equation (50) and also assuming that ϕ varies slowly compared to the fiber delay such that $\phi(t + \tau) \approx \phi(t)$, the first term of the current autocorrelation function can be written as

$$\begin{aligned} \langle \hat{I}(t) \hat{I}(t + \tau) \rangle &= \frac{K^2}{4} \left\{ 4 \cos^4 \left(\frac{\phi(t)}{2} \right) K^2 \right. \\ &\quad + \cos^4 \left(\frac{\phi(t)}{2} \right) C_{xx} [2\delta(t_c) + \delta(t_c - \tau) + \delta(t_c + \tau)] \\ &\quad \left. + \cos^2 \left(\frac{\phi(t)}{2} \right) \sin^2 \left(\frac{\phi(t)}{2} \right) C_{yy} [2\delta(t_c) - \delta(t_c - \tau) - \delta(t_c + \tau)] \right\} \quad (92) \end{aligned}$$

After averaging ϕ over the interval $(0, 2\pi)$ to account for the fluctuations in fiber delay line length that occur during the timescale of a measurement (measurement time $\tau_{\text{meas}} \gg \tau$), the power spectrum is

$$S[\omega] = G^2 \sigma \frac{K^2}{2} \left[1 + \sigma \left(\frac{3}{4} C_{xx} \cos^2 \frac{\omega\tau}{2} + \frac{1}{4} C_{yy} \sin^2 \frac{\omega\tau}{2} \right) \right] \quad (93)$$

From Equation (93) it is clear that the spectrum is everywhere sensitive to shot noise, represented by the 1 at the beginning of the expression. For classical noise, the spectrum is maximally sensitive to amplitude noise at maxima of $\cos^2 \frac{\omega\tau}{2}$, and maximally sensitive to phase noise at maxima of $\sin^2 \frac{\omega\tau}{2}$. For an approximate delay of $2.1 \mu\text{s}$, the first phase noise sensitivity maximum occurs near 238 kHz, and the second near 713 kHz. The precise positions of these sensitivity maxima depend on the precise delay time, which depends on the precise index of refraction of the fiber. However, due to the short delay time of our fiber and the ensuing slow periodicity of $\cos^2 \frac{\omega\tau}{2}$ and $\sin^2 \frac{\omega\tau}{2}$ the bandwidth of the region of nearly maximum sensitivity to phase noise is many tens of kilohertz wide. Since we care about being able to measure phase noise at $\omega_m/2\pi = 705 \text{ kHz}$ and there is a predicted maximum less than 10 kHz away, precisely measuring the fiber's index of refraction and delay time is not necessary.

The coefficients of C_{xx} and C_{yy} in of Equation (93) determine the interferometer's relative sensitivity to each type of noise. Figure 54 shows a plot of the relative sensitivity of the delay line interferometer to amplitude noise (blue) and phase noise (yellow) as a function of frequency after dividing out the common coefficients $G^2 \sigma^2 K^2 / 2$. As can be seen in Figure 54 the interferometer is maximally sensitive to phase noise in the region around 713 kHz.

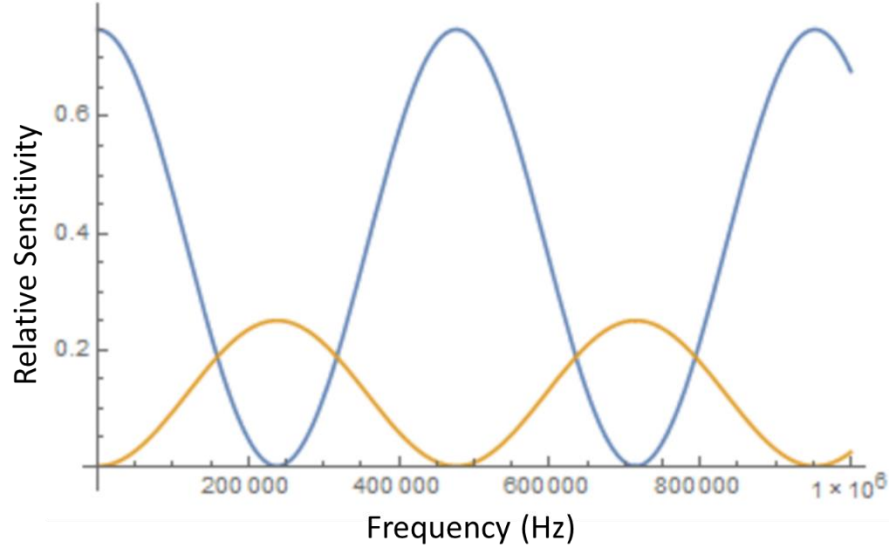


Figure 54: Plot of the relative sensitivity of delay line interferometer to classical amplitude noise (blue) and classical phase noise (yellow) as a function of noise frequency.

While the delay line measurement can be used to extract both C_{xx} and C_{yy} , in practice it is much easier to measure C_{xx} with just a single beam, because a photodiode is inherently sensitive to amplitude fluctuations. In this case we expect the power spectrum to take the form:

$$S[\omega] = G^2 \sigma K^2 (1 + \sigma C_{xx}) = G^2 \sigma K^2 (1 + \sigma P C_{xx0}) \quad (94)$$

In this expression, $C_{xx0} = C_{xx}/P$, where P is the power of the laser in microwatts: thus C_{xx0} is the amount of laser noise present at a power of 1 μ W.

Since we are examining the power spectrum over a narrow bandwidth around 705 kHz, we fit it to a constant, and then plot it as a function of power. We then fit the plot vs. power using Equation (94) to extract C_{xx0} . To calibrate the pre-factors in Equation (94), we exploit the fact that the amplitude noise measurement for light that has passed through a filter cavity appears to have a background linear in power, consistent with shot noise. This means that for the filtered light, $C_{xx0} \approx 0.00/\mu\text{W}$, and therefore a linear fit to the background vs. power gives just the prefactors of Equation (94) with the

power divided out. Figure 55 shows plots of the background vs. power for filtered (green) and unfiltered (blue) laser light from the measurement laser.

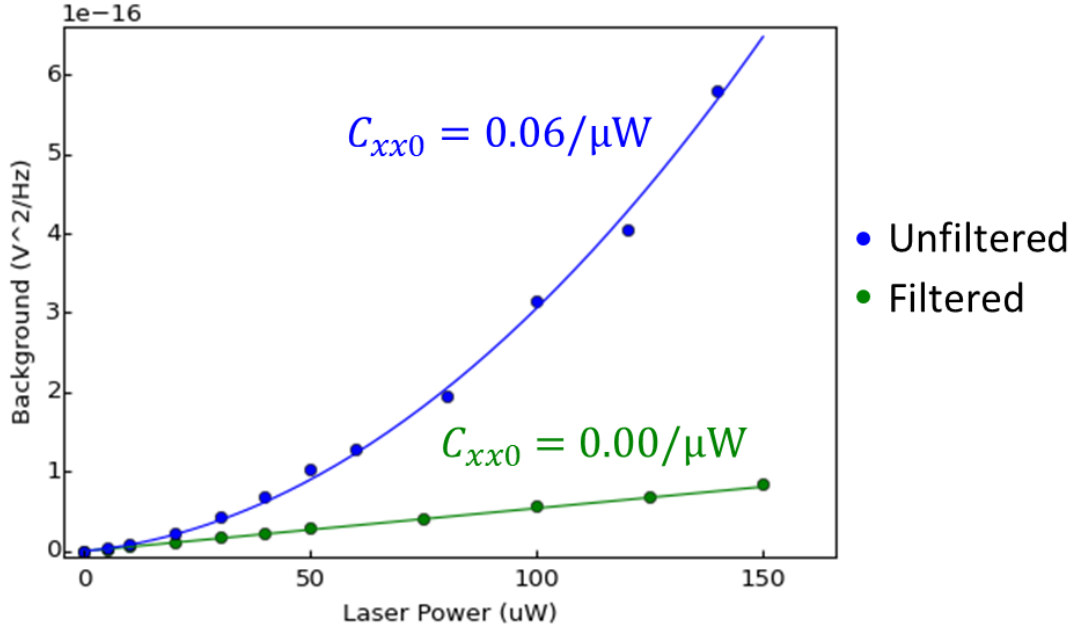


Figure 55: Plot of the background of the power spectrum from a classical amplitude noise measurement for the measurement laser. Blue = unfiltered, green = filtered.

Fits of the plots in Figure 55 to Equation (94) give $G^2\sigma K^2/P = 5.402 \times 10^{-19} \text{ V}^2/(\mu\text{W} \cdot \text{Hz})$ for the filtered beam and $C_{xx0} \approx .06/\mu\text{W}$ for the unfiltered beam.

To measure the phase noise, the background as a function of power around 705 kHz is recorded using the delay line setup. The function used to fit the background vs. laser power is an approximation of Equation (93) in the limit that 705 kHz is a point of near maximum sensitivity to phase noise, and very little sensitivity to amplitude noise such that $\sin^2\left(\frac{\omega\tau}{2}\right) \approx 1$ and $\cos^2\left(\frac{\omega\tau}{2}\right) \approx 0$. Note that K^2 for the delay line measurement is defined using the power at the input to the delay line interferometer, and thus is a factor of four larger than K^2 in the C_{xx} measurement:

$$S[\omega] \approx G^2\sigma \frac{K^2}{2} \left[1 + \sigma \left(\frac{1}{4} C_{yy} \right) \right] = G^2\sigma \frac{K^2}{2} \left[1 + \sigma \left(\frac{1}{4} P C_{yy0} \right) \right] \quad (95)$$

The prefactors are calibrated by the shot noise as measured from the filtered C_{xx} measurement. As a sanity check, it should be noted that in the absence of phase noise on the laser, Equation (95) is simply a factor of two times the prefactor values obtained in the C_{xx} measurement, which is expected since there is twice as much power hitting the photodiode as in the C_{xx} measurement (since each arm has equal power, and the power in Equation (95) represents the power in only one arm.)

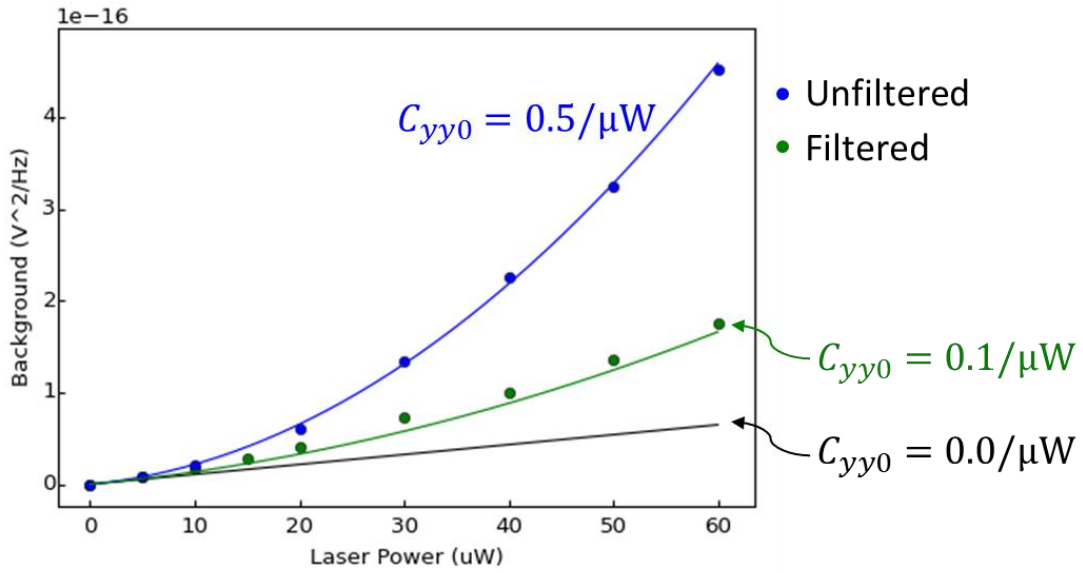


Figure 56: Plot of the background of the power spectrum from a classical phase noise measurement of the measurement beam. Blue = unfiltered, green = filtered, black=shot noise reference.

Fitting the curves in Figure 56, we find that for the measurement beam, the unfiltered $C_{yy0} \approx 0.5/\mu\text{W}$, and the filtered $C_{yy0} \approx 0.1/\mu\text{W}$. Similar fits for the filtered cooling beam give the same result of $C_{yy0} \approx 0.1/\mu\text{W}$. This suggests that classical phase noise is reduced by a factor of 5 by the filter cavities, which is a lot less than the amount of filtration expected (of order 10^3) based on the susceptibilities of the filter cavities at 705 kHz. However, we hesitate to accept these measurements at face value, because the delay line itself could be contributing the measured phase noise.

Theory work by Wanser⁵⁷ suggests that there is an intrinsic limit to the phase noise that can be measured by any fiber interferometer due to extra phase noise caused by thermal fluctuations in the

fiber. Since these thermal fluctuations change the effective length of the fiber, they can modulate the phase of the light passing through the fiber, adding noise that is indistinguishable from the classical phase noise that the fiber is being used to measure. Wanser's expression for the spectrum of phase noise introduced by thermal fluctuations in the fiber is

$$S_{\phi\phi}[l, \nu] = \frac{\pi k_B T^2 l}{\kappa \lambda^2} \left(\left(\frac{dn}{dT} \right) + n \alpha_L \right)^2 \ln \left\{ \frac{\left(\left[k_{\max}^2 + \left(\frac{2\pi\nu}{c/n} \right)^2 \right]^2 + \left(\frac{2\pi\nu}{D} \right)^2 \right)}{\left(\left[k_{\min}^2 + \left(\frac{2\pi\nu}{c/n} \right)^2 \right]^2 + \left(\frac{2\pi\nu}{D} \right)^2 \right)} \right\} \quad (96)$$

In this equation, k_B is Boltzmann's constant, T is the temperature, l is the length of the fiber, κ is the thermal conductivity of the fiber, D is the thermal diffusivity of the fiber, $\frac{dn}{dT}$ is the temperature derivative of the fiber's index of refraction n , α_L is the linear thermal expansion coefficient of the fiber, $k_{\min} = 2.405/a_f$ where a_f is the fiber cladding radius, $k_{\max} = 2/w_0$ where w_0 is the mode field radius, c is the speed of light in vacuum, ν is the frequency (in Hertz) at which the spectrum is being evaluated.

After plugging reasonable estimates of these parameters into Equation (96), $S_{\phi\phi}[435 \text{ m}, 705 \text{ kHz}] \approx 7.21 \times 10^{-16} \frac{\text{radians}^2}{\text{Hz}}$. Shot noise at one microwatt should be $S_{\phi\phi}^{\text{shot}} = \frac{\hbar\omega_L}{P} = 1.86 \times 10^{-13} \frac{\text{radians}^2}{\text{Hz}}$. The ratio of these two should be equivalent to the expected C_{yy0} value if the additional noise in our measurement were entirely from the intrinsic phase noise of the fiber, but it is actually somewhat lower at 0.0039. While this suggests that our measured noise cannot be entirely attributed to the intrinsic phase noise of the fiber, it is still an encouraging result because it gives some perspective to the susceptibility of the fiber delay line technique to fluctuations from both internal and external sources. Statistical fluctuations in temperature within the delay line are incredibly small, but yet theory says that they introduce extra phase noise of magnitude equal to a few percent of shot noise even at laser powers as low as a few microwatts.

Though we have no way of independently measuring the ambient acoustic noise at 705 kHz, it is reasonable that it may have added to the statistical thermal fluctuations in the fiber at 705 kHz, perhaps explaining the difference between the theory prediction of the intrinsic phase noise and the phase noise we measured. Indeed, experiments which have attempted to directly measure the intrinsic noise predicted by Wanser's theory have taken excruciating steps to eliminate the influence of external noise sources when performing measurements, such as using an interferometer with equal length arms co-wound around a common spool and acoustically isolated from the environment by alternating layers of lead and open cell foam inside a rigid aluminum box mounted on an air bladder on top of a vibrationally isolated optical table in an empty building with air handlers turned off and a spectrum analyzer isolated from sources of RF interference.⁵⁸ Thus we accept that we cannot obtain a reliable measurement of C_{yy0} using the delay line alone. However, we can say that since C_{xx0} is unambiguously zero, and C_{yy0} is close to the limit of detectable phase noise using the common delay line interferometer technique, our laser noise is likely very small.

Nonetheless, if appreciable classical laser noise is present in our setup, it should manifest in the optomechanical experiments in several distinct ways. Noise on the measurement beam appears in the heterodyne spectrum in three ways: (1) as an increased background, as can be seen from Equations (73) and (78); (2) as a change in the area of the Lorentzian component of the membrane's motion, as can be seen from Equation (74) and Equation (79); and (3) as a non-zero anti-Lorentzian component of the membrane's motion, as can be seen from Equations (76) and (80). Careful inspection of the equations reveals that the effect of laser noise on L_{rr} and L_{bb} is such that it leads to a larger asymmetry in the sideband areas than would be the case for a noiseless laser. If this effect is not accounted for, a naive analysis of the asymmetry between the sidebands would yield an incorrect phonon number that is too small.

Classical noise on the cooling laser, since it is detuned by two free spectral ranges from the local oscillator beam, will not appear in the heterodyne spectrum unless it is at extremely high frequencies. However, noise near the mechanical frequency will create a beat note in the intracavity power that can drive the membrane, causing heating. This will manifest in a way that is difficult to distinguish from thermal heating of the membrane due to absorption.

As a sanity check on the delay line measurement, it would seem useful to consider whether any of the above consequences of laser noise can be observed in our data. Unfortunately however, extracting values of C_{xx} , C_{xy} , and C_{yy} from the above effects is not an easy task. To extract C_{xx} , C_{xy} , and C_{yy} from the noise background in the heterodyne spectrum requires very careful measurement of additional background noise added by the mixing circuit. There are several sources of noise in the mixing circuit. The first is dark noise from the amplifier in the PDA10CF photodiode. The influence of this noise can be easily measured by simply turning off the lasers and measuring the output of the mixing circuit. A second source of noise, however, is noise from the 101 MHz signal generator used to mix down the heterodyne beat note to 21 MHz to be within the bandwidth of the HF2-LI. Approximating the output of the mixer as the product of the 101 MHz LO and the 80 MHz beat note, for a constant power at 101 MHz, this would scale with the amplitude of the 80 MHz beat note. Classical laser noise is also expected to scale with the size of the 80 MHz beat note, and therefore these two effects are difficult to distinguish. With enough time and appropriate test equipment, the influence of noise from the 101 MHz signal generator probably could be singled out, but the measurement introduces significant new sources of error.

The effect of laser noise on L_{rr} and L_{bb} also cannot be easily used to extract C_{xx} , C_{xy} , and C_{yy} , since the additional laser noise simply mimics lower phonon numbers. Without a fully independently characterized phonon number, there is nothing to compare L_{rr} and L_{bb} to in order to extract the laser

noise. The phonon number from linewidth is not suitable as an independently characterized phonon number in this case, since it is also affected by classical laser noise, as seen in Equations (65) and (66).

The final two effects of laser noise, however, do provide some opportunities to “sanity check” our estimate of very small laser noise. In the absence of laser noise, the anti-Lorentzian part of S_{rr} and S_{bb} should be zero. While fits to a generic Lorentzian lineshape qualitatively appear to represent the data well (indicating that the anti-Lorentzian part is close to zero), we also tried to fit the data with a full Fano lineshape to extract any anti-Lorentzian part that might be present. The fits with Fano lineshapes were performed using the following fit function, with the same fitting window as used for the Lorentzian fits. No particular steps were taken to encourage agreement between the fits other than to enforce the condition that f_h be positive.

$$\begin{cases} S_{rr} = b_{rr} + \frac{s_{rr} + \left(\frac{f-f_c}{f_h}\right) a_{rr}}{1 + \left(\frac{f-f_c}{f_h}\right)^2} \\ S_{bb} = b_{bb} + \frac{s_{bb} + \left(\frac{f-f_c}{f_h}\right) a_{bb}}{1 + \left(\frac{f-f_c}{f_h}\right)^2} \end{cases} \quad (97)$$

The results of the fits overall agree well with the results of the Lorentzian fits, both in linewidth, and the area of the Lorentzian part. The Lorentzian part area is given by $2000\pi s_{(rr/bb)} f_h$ for the (red/blue) sidebands respectively:

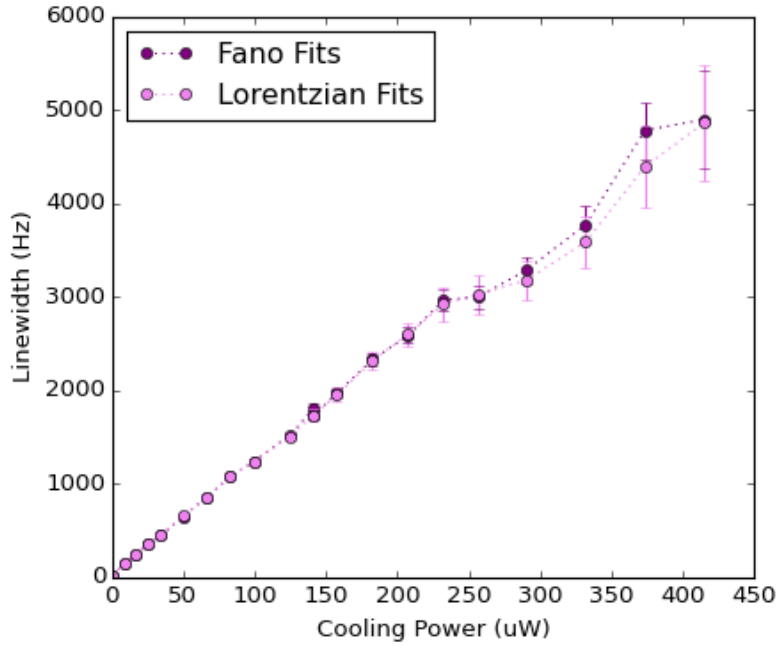


Figure 57: Plots of the linewidth of the Lorentzian component of the mechanical mode's Brownian motion taken from strictly Lorentzian fits (pink), and Fano fits (purple).

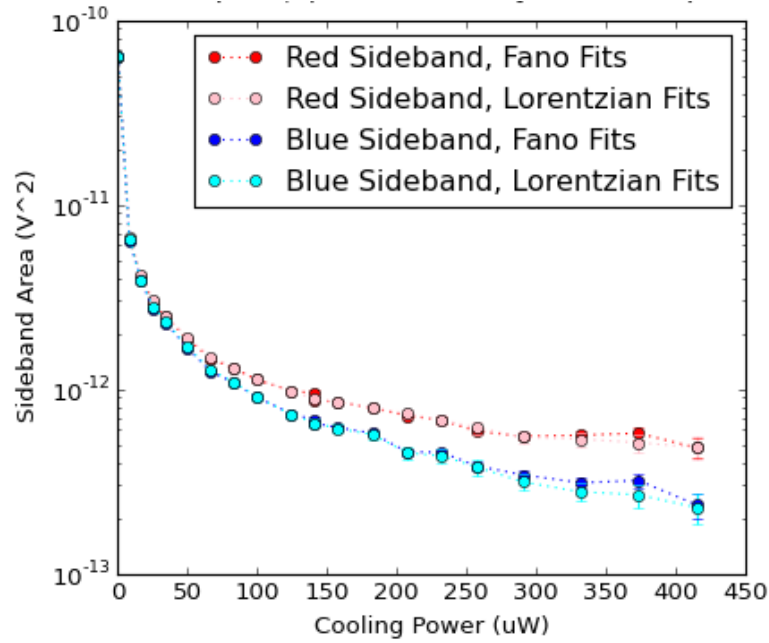


Figure 58: Plots of the area of the Lorentzian components of the mechanical mode's Brownian motion taken from strictly Lorentzian fits (pink/cyan), and Fano fits (red/blue).

While it is possible to characterize the Lorentzian part of the Fano fit results in terms of the area under the Lorentzian, the anti-Lorentzian part has odd symmetry and therefore always has an area of zero. However, it is still possible to characterize the size of the anti-Lorentzian by considering A_{rr} and A_{bb} (Equations (76) and (80)) which have units of area. In terms of the variables in the fitting expressions (Equation (97)) these are $A_{(rr/bb)} = \frac{2\pi \times 1000 f_h a_{(rr/bb)}}{G_r^2 \sigma r \beta^2 K^2}$. For easier comparison to Figure 58 however, the gain factors need not be divided out. Thus I refer to the “area” of the anti-Lorentzian component as the anti-Lorentzian “magnitude”, which I define as $2\pi \times 1000 f_h a_{(rr/bb)}$.

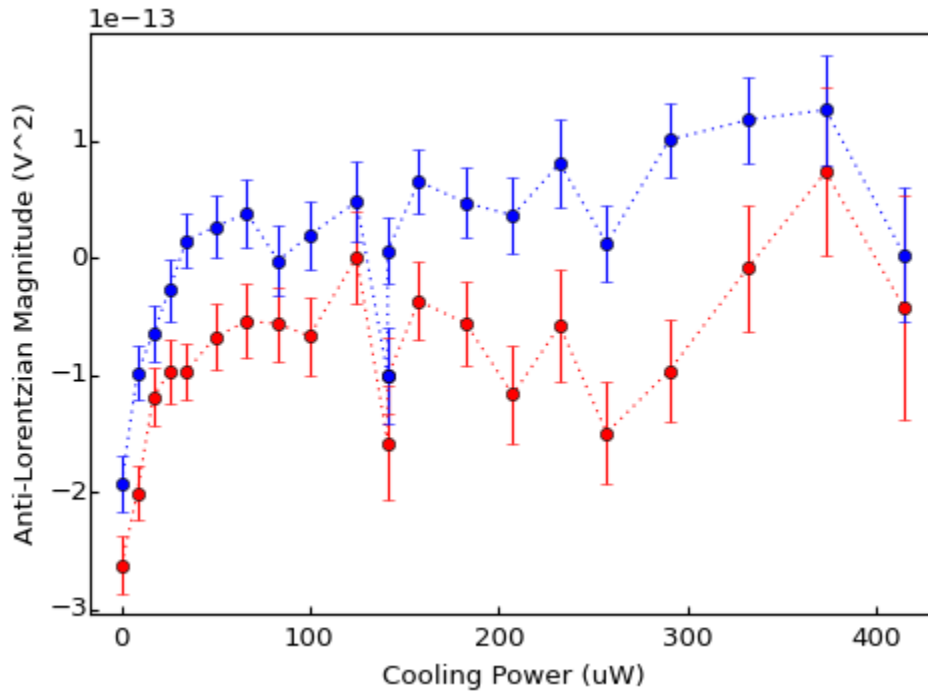


Figure 59: Anti-Lorentzian component of the Fano fits as a function of cooling power for the red and the blue sidebands.

Upon first glance, the anti-Lorentzian part of the fit results shown in Figure 59 looks scattered and does not seem to show a trend as consistent as the Lorentzian part in Figure 58. Theory predicts that the anti-Lorentzian component should depend only on the power of the measurement/LO beams and not depend on cooling power, so therefore the data should be flat, which they appear to be only

roughly. Additionally, to good approximation, for a small non-zero detuning of the measurement beam from the cavity resonance, $A_{rr} \approx -A_{bb}$. This antisymmetric behavior can be seen in Figure 60, which shows the expected magnitude of the antisymmetric part of the Fano lineshape from theory as a function of classical laser phase noise on the measurement beam (C_{yy0}). This plot assumes all of the parameter values that have already been determined through independent measurements, including the measurement beam detuning, the known measurement and local oscillator beam powers, and the fact that $C_{xx0} = 0$.

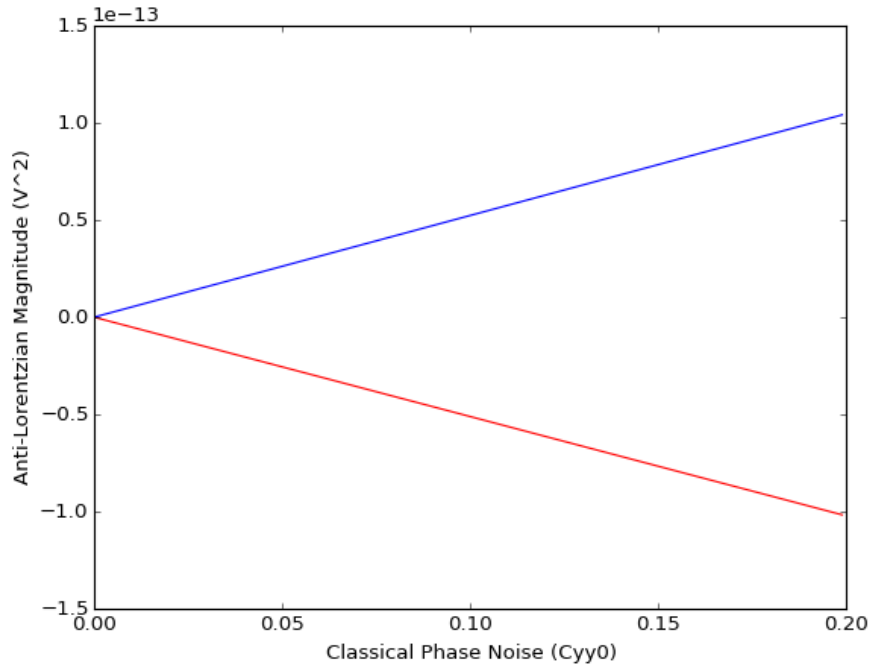


Figure 60: Theory plots of the expected anti-Lorentzian magnitude as a function of C_{yy0} for the red and the blue sidebands.

The expected antisymmetric behavior of the anti-Lorentzian components of the Fano fits does not seem to appear in the fit results, at least not with very high consistency on a data point by data point basis. Nonetheless, it is possible to fit the data in Figure 59, constraining the fit to the theory prediction that the red and the blue anti-Lorentzian components should have characteristic sizes that are equal in the magnitude but opposite in sign. Ignoring the three lowest power data points since they

seem to indicate some unexpected trend, the fit results shown in Figure 61 do indicate a non-zero average anti-Lorentzian component to the Fano fits, with characteristic size for the red and blue sidebands of $\mp(5.3 \pm 0.8) \times 10^{-14} \text{ V}^2$ respectively. Using the defined relationship between the plotted anti-Lorentzian magnitudes and A_{rr}/A_{bb} it is possible to solve for $\text{Im}(\tilde{B})$, and hence extract a value for C_{yy0} from these results.

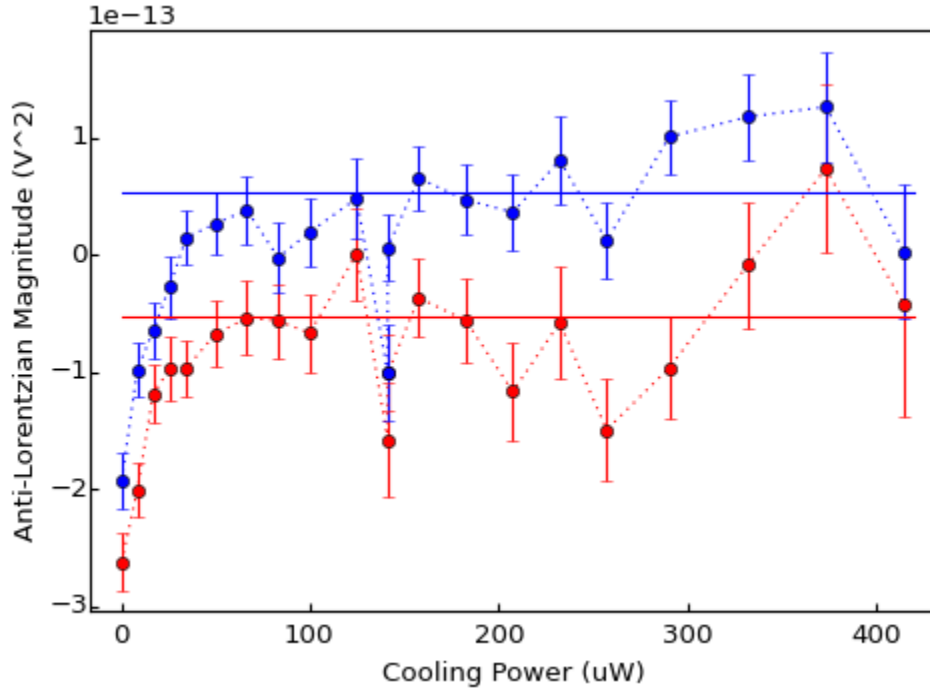


Figure 61: Fits to the anti-Lorentzian components of the red and blue sidebands.

The value of C_{yy0} extracted from the fitted magnitudes of the anti-Lorentzian components of the Fano fits is $C_{yy0} \approx 0.1$. This value is surprisingly large, and actually agrees well with the value obtained from the delay line measurement. However, there is still reason to question this result. First, though the constrained fit above produces a result with reasonable uncertainty, $\mp(5.3 \pm 0.8) \times 10^{-14} \text{ V}^2$, the red and the blue data sets independently have very large uncertainties. For example, excluding the first three data points, the average value and standard deviation of the red data set is $(-6.7 \pm 5.4) \times 10^{-14} \text{ V}^2$, which is an uncertainty of 81%. For the blue data set, the average value and

standard deviation is $(3.6 \pm 5.3) \times 10^{-14} \text{ V}^2$, which is an uncertainty of 150%, and is potentially consistent with zero. Additionally, it is odd that fluctuations in the anti-Lorentzian magnitude qualitatively seem to be common mode to both the red and the blue sidebands, rather than antisymmetric about zero as expected from theory. This indicates that the change in the anti-Lorentzian magnitudes from one cooling power to the next is probably not due to fluctuations in the amount of noise on the measurement laser, and calls into question how well the data actually represent the laser noise.

Ultimately, Fano lineshapes are caused by the presence of correlations between the motion of the driven mechanical mode and the laser light used to measure that mode. The only source of such correlations considered in our theory of heterodyne detection is classical laser noise. However, there may be other ways in which correlations between the mechanical mode and the laser light can occur. For example, if a different mechanical mode is present in the system, such as a mode of the membrane chip, the cavity mirrors, or the cavity body, that mode can mechanically drive the membrane. If this mode also couples to the laser light inside the cavity, this would introduce correlations between the membrane's motion and the measurement laser, resulting in a Fano lineshape.

Because our theory of heterodyne detection considers laser noise as the only source of correlations between the measurement beam and the mechanical mode, attempting to extract a value of C_{yy0} from a Fano lineshape whose source is not classical laser noise will give an incorrect result. While it is true that the coupling between this hypothetical extra mode and the measurement beam means that there is extra “noise” on the laser as far as the membrane is concerned, the heterodyne detection theory models noise with particular properties: (1) the noise is white within a bandwidth $\gg \tilde{\gamma}$ around ω_m , (2) the noise on the measurement beam is not correlated with the noise on the cooling beam, and (3) the coupling between the noise and the mechanical motion is entirely described by the

cavity enhanced optomechanical coupling α_j . Noise from an additional mechanical mode need not satisfy any of these properties and thus cannot be described by this model. Such noise could, for example, strongly drive the membrane mode mechanically but only weakly couple to the laser. This could result in a large anti-Lorentzian component in the Fano fits, but would not necessarily introduce much additional classical noise on the lasers – which is an important characteristic, since as will be discussed below, there is strong evidence that the noise on at least the cooling beam was very close to zero.

Whether or not this speculation about the causes of the Fano lineshapes holds any merit, there is compelling evidence that the laser noise on at least the cooling beam was close to zero. In the heterodyne detection theory, classical laser noise ultimately has two effects:

1. The equations for L_{rr} and L_{bb} indicate that the sideband asymmetry will be affected by the presence of laser noise on the measurement beam, making the sideband ratio larger.
2. The equation for n_m (Equation (65)) indicates that n_m itself is also affected by classical laser noise on both lasers, with excess laser noise causing n_m to become larger. Since $\xi \rightarrow 1$ for large n_m , this makes the sideband ratio smaller.

Theory plots of the sideband asymmetry as a function of cooling laser power which account for these two effects with the amount of laser noise on the measurement and the cooling lasers suggested by the delay line and Fano fit techniques do not agree well with the experimentally measured sideband asymmetry. The solid green line in Figure 62 shows the expected sideband asymmetry for noiseless lasers. The dashed line shows the expected sideband asymmetry if the measurement laser has $C_{yy0} = 0.1$. Admittedly, this is a small enough change in the expected sideband asymmetry that qualitatively, the data still agrees with the theory line. However, the dashed lime line in Figure 62 shows the expected

sideband asymmetry if both lasers have $C_{yy0} = 0.1$, which very clearly disagrees with what was measured.

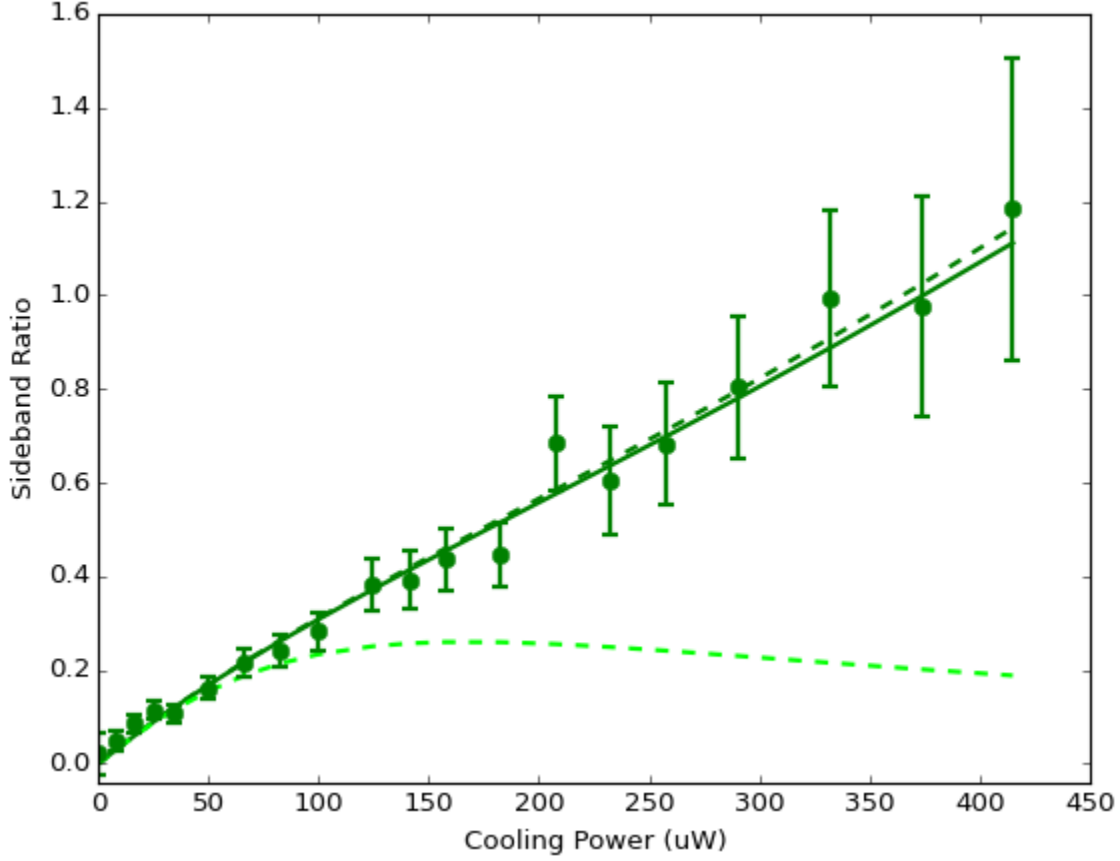


Figure 62: Plot showing the theoretically expected sideband asymmetry as a function of cooling laser power for noiseless lasers (solid green line), measurement laser $C_{yy0} = 0.1$ (dashed green line), and both lasers $C_{yy0} = 0.1$ (dashed lime line). The experimentally measured sideband asymmetry is overlaid as green data points.

With reasonable confidence, it is therefore possible to conclude that at least the cooling laser must have been nearly noiseless during the course of the measurement. Since the measurement beam is a nominally identical laser to the cooling laser and passes through a similar filter cavity, it is likely that it was nearly noiseless as well. In any case though, even if the measurement beam was not noiseless, a small amount of laser noise (of order $C_{yy0} \ll 1$) should have only a very small effect on the sideband asymmetry and would not dramatically alter our conclusions about phonon number. Thus, we choose to treat both lasers as fully noiseless.

C. Calculating the Mean Phonon Number

As previously mentioned, there are several methods by which to calculate the mean phonon number. One way is by analysis of the sideband asymmetry. As discussed previously, the sideband asymmetry can be skewed by measurement beam detuning and photodiode gain dispersion. Therefore, we correct the sideband asymmetry data for these effects using the following formula where ξ' represents the corrected asymmetry:

$$\xi' = \xi \frac{|\chi_c(-\omega_m)|^2}{|\chi_c(\omega_m)|^2} \left(\frac{G_b^2}{G_r^2} \right) \quad (98)$$

Substituting ξ' for ξ in Equation (83) and rearranging gives $n_m = 1/(\xi' - 1)$. The phonon number calculated in this way is shown in Figure 63. At maximum cooling power, the phonon number calculated from this method is $n_m = 0.84 \pm .23$.

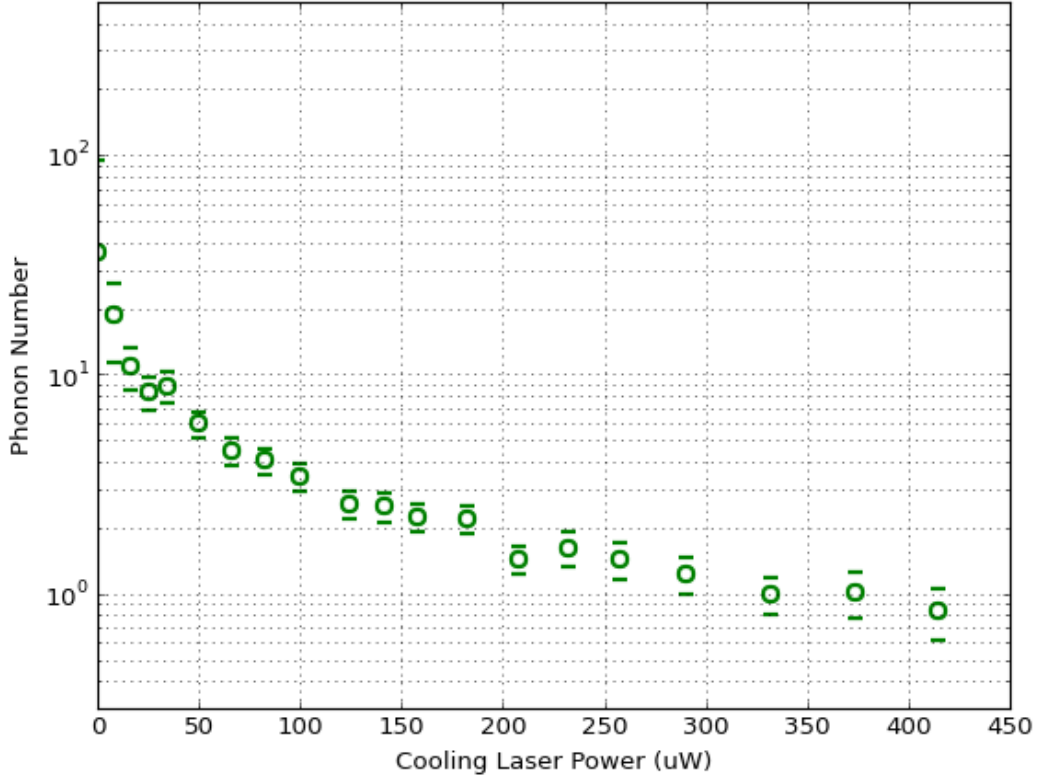


Figure 63: Plot of phonon number calculated from sideband asymmetry, as a function of cooling laser power.

A second way of calculating the phonon occupancy of the mechanical mode is from the linewidth of the mechanical Lorentzian. Recalling Equation (65), $n_m = \frac{n_{th}\gamma + \sum_j n_j \gamma_j}{\tilde{\gamma}}$. Everything that is needed to compute this expression has been discussed in the last section except the bath temperature, contained within $n_{th} = k_B T / \hbar \omega_m$. Since the membrane is thermally anchored to both the ^3He pot and the experimental stage, we expect that its temperature should be somewhere between the temperatures of these two things. Ruthenium dioxide thermometers present on both the stage and the ^3He pot therefore give bounds to the bath temperature of the membrane. We postulate that the membrane temperature should follow a simple linear model interpolating between the temperature of the stage and the temperature of the ^3He pot. This model takes the form:

$$T = T_{\text{pot}} + \alpha(T_{\text{stage}} - T_{\text{pot}}) \quad (99)$$

The value of α is unknown a priori. We determine α using a technique based on least squares optimization. The procedure used involves minimization of an error function which characterizes the difference between the phonon number calculated from sideband asymmetry and the phonon number from linewidth. The form of the error function is as follows:

$$\epsilon(\alpha) = \sum \left(\frac{1}{n_m^{(\gamma)}} - \frac{1}{n_m^{(\xi')}} \right)^2 \quad (100)$$

In this expression, $n_m^{(\gamma)}$ is the phonon number from the linewidth, and $n_m^{(\xi')}$ is the phonon number from sideband asymmetry. The summation is over all cooling powers at which data was collected during the measurement. The reason for using the difference between the inverse phonon numbers rather than the difference between the phonon numbers themselves is to minimize the impact that low cooling power (large phonon number) data points have on the error function. Since the ratio of the sidebands quickly approaches unity when $n_m > 1$, sideband asymmetry is a particularly poor metric for computing phonon number at low cooling powers. Therefore we seek to limit the influence that these low cooling power/high phonon number data points have in the least squares optimization.

Before performing the optimization technique, we need to select a membrane quality factor from within the range measured by the mechanical ringdown technique. Since $\gamma \propto 1/Q$ and T appear as a product in the equation for phonon number as a function of linewidth, they are not fully independent of each other. However, due to the constraints placed on T by the linear model used (Equation (99)), the optimization technique is not able to fully compensate for multiplicative errors in Q just by varying T . Therefore, we empirically selected a value of $Q = 4.5 \times 10^6$, which is within the range of variability measured using the mechanical ringdown technique, as this value together with the optimization technique to find α seemed to best reproduce the sideband asymmetry phonon number data.

One could argue that the empirically determined choice of Q is in essence a poor manual least squares optimization and that it would be sensible to simply do a multivariable least squares optimization on Q and α . However, attempts to do this showed that the global minimum of the error function occurs when $Q \approx 2 \times 10^6$ and $\alpha \approx 0$. This would imply that the membrane quality factor was smaller during the cooling measurement than it ever was during many different independent measurements by the ringdown technique, and also that the membrane was completely unaffected by the temperatures of the ^3He pot and the stage. These two implications are inconsistent with prior knowledge and physically unreasonable. Therefore, we decided to stick with our empirically determined $Q = 4.5 \times 10^6$ and optimize just on α .

In any case, upon minimizing the error function between the phonon numbers with $Q = 4.5 \times 10^6$, we find that $\alpha = 0.498$. The inferred bath temperature as a function of cooling power with this value of α is plotted in Figure 64.

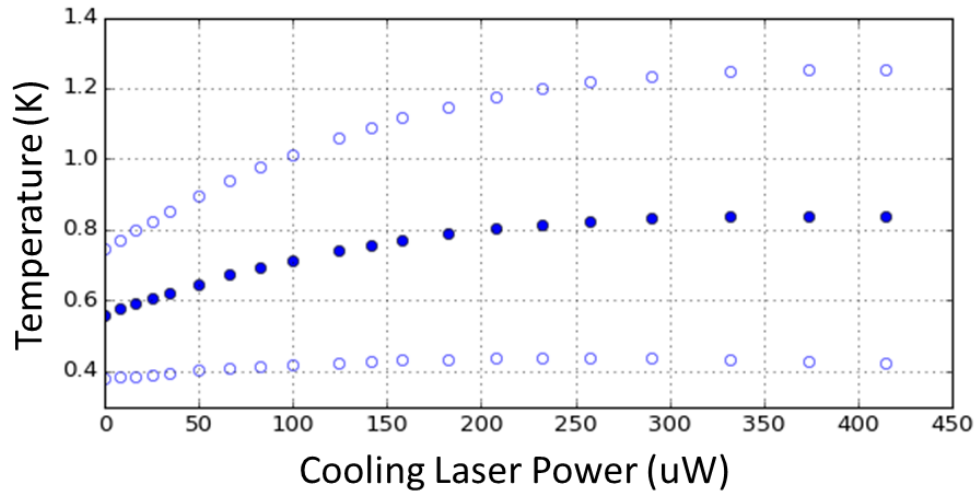


Figure 64: Plot of the membrane temperature computed using the linear model described in the text with $\alpha = .498$ (solid blue dots). The ^3He pot temperature is shown by the hollow dots on the bottom, and the stage temperature by the hollow dots on the top.

With this bath temperature data it is now possible to plot the phonon number calculated from linewidth along with the phonon number from asymmetry. This plot is shown in Figure 65. Qualitatively, these two estimates of phonon number agree quite well.

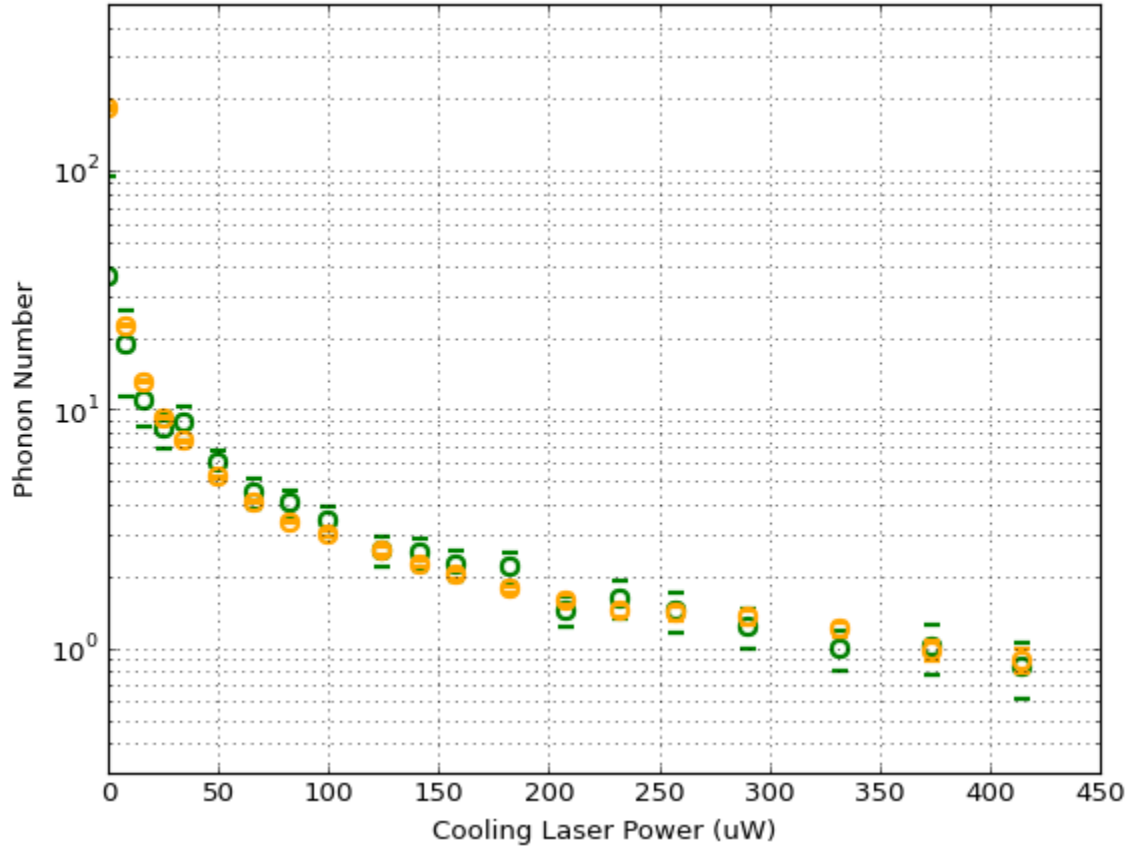


Figure 65: Plot of the phonon number of the membrane's mechanical mode calculated from sideband asymmetry (green), and linewidth (yellow).

There is one final way to cross-check these phonon number results which arises from the fact that the area under the power spectral density of the membrane's Brownian motion is proportional to the phonon number (see Equations (74) and (79) for L_{rr} and L_{bb}). The area can be easily computed from the results of the fits to generic Lorentzian lineshapes, and the theory expressions for the areas can then be used to solve for the phonon number. In terms of the results of the fits to generic Lorentzian lineshapes (recalling that f_h has units of kHz):

$$G_r^2 \sigma r K^2 L_{rr} = 1000 f_h s_{rr} \pi \quad (101)$$

$$G_b^2 \sigma r K^2 L_{bb} = 1000 f_h s_{bb} \pi \quad (102)$$

Making these substitutions into Equations (74) and (79), the phonon numbers extracted from each sideband are shown below:

$$n_m^{(rr)} = \frac{1000 f_h s_{rr} \pi}{G_r^2 \sigma^2 \beta^2 \left(\frac{P_{LO}}{\hbar \omega_L} \right) \kappa_{in} |\alpha_s^2| |\chi_c(-\omega_m)|^2} - 1 \quad (103)$$

$$n_m^{(bb)} = \frac{1000 f_h s_{bb} \pi}{G_b^2 \sigma^2 \beta^2 \left(\frac{P_{LO}}{\hbar \omega_L} \right) \kappa_{in} |\alpha_s^2| |\chi_c(\omega_m)|^2} \quad (104)$$

where the -1 in Equation (103) stems from the fact that $L_{rr} \propto (n_m + 1)$, while $L_{bb} \propto n_m$. Since the values of each of the parameters in these two equations are already known from independent characterization techniques, it is simple to compute a phonon number based on each sideband at every cooling power. Plotting these phonon numbers on top of those from sideband asymmetry and linewidth:

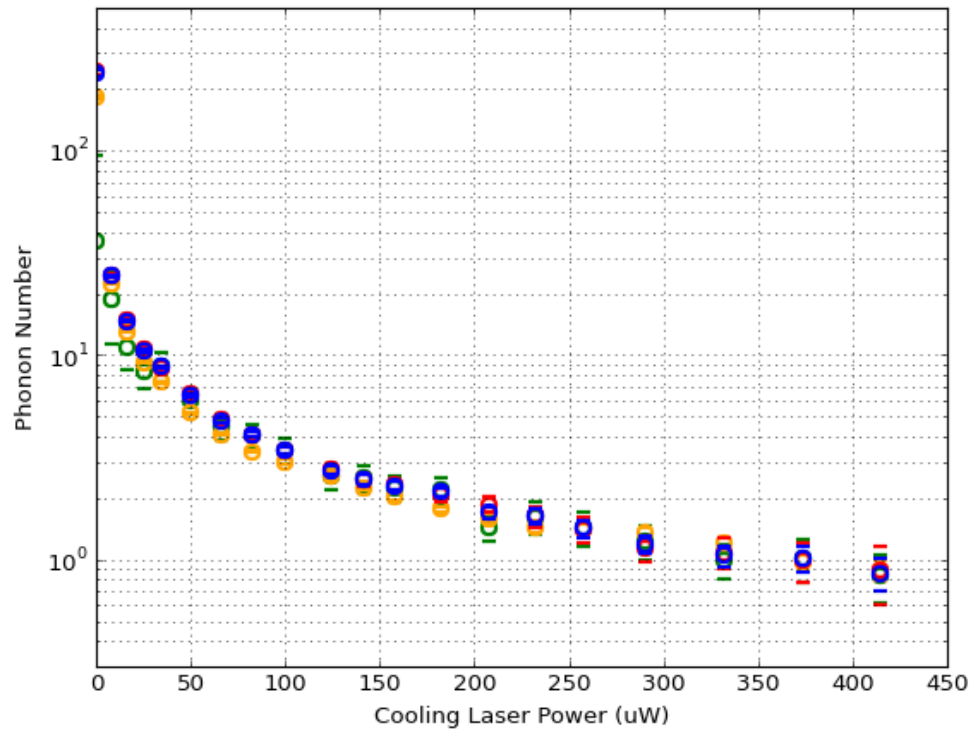


Figure 66: Plot of the phonon number of the mechanical mode as computed from sideband asymmetry (green), linewidth (yellow), and red and blue sideband areas (red and blue).

Lastly, since all of the parameters of the theory have been characterized, it is possible to directly compute the phonon number expected for each cooling power to compare with the data. This theory prediction is shown on top of the data in Figure 67.

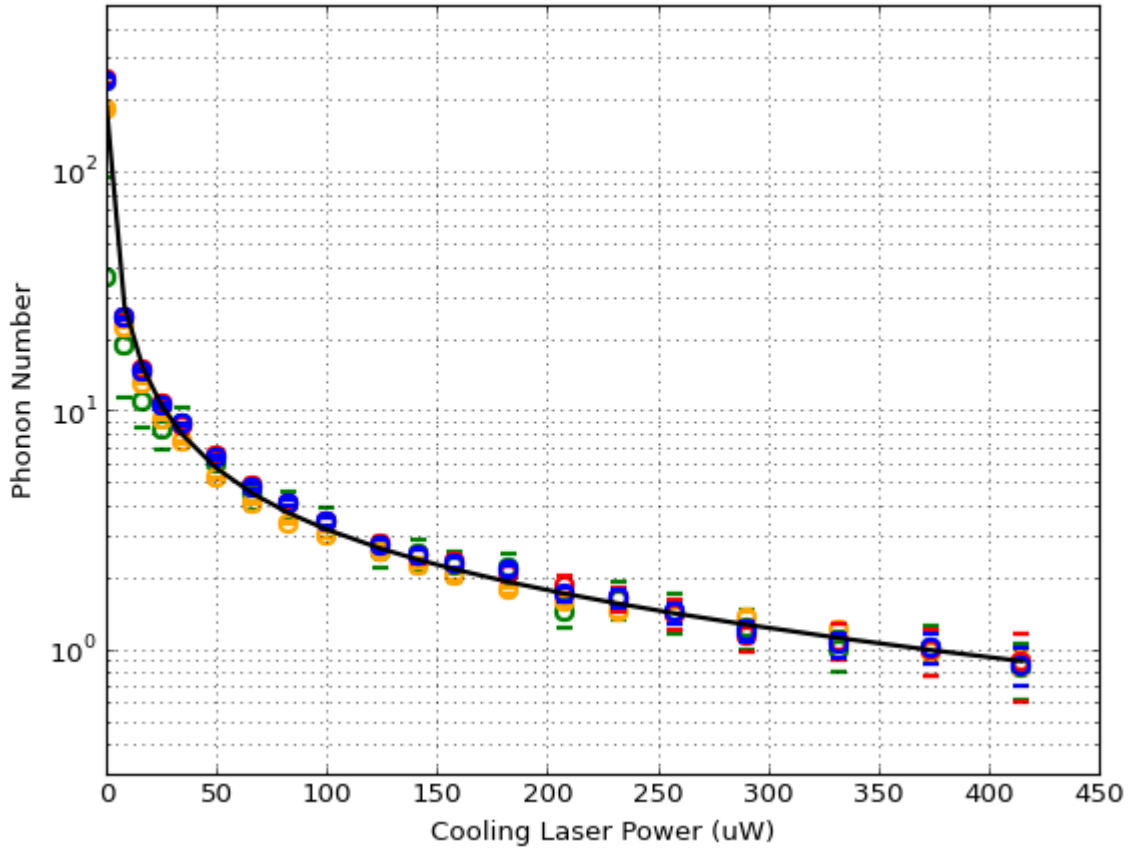


Figure 67: Final plot of the phonon number of the mechanical mode calculated by four techniques, along with a comparison to the theory prediction (black line).

The theory line, as expected, agrees best with the linewidth- and area-extracted phonon numbers at low cooling powers, confirming that sideband asymmetry is a poor estimate of phonon number when the sideband ratio is close to unity. Overall, the data agree quite well with each other as well as with the theory line at higher cooling powers. Since the data points in the plot are generally larger than the difference between them, it is useful to also look at a plot of the inverse phonon

number. This is shown in Figure 68, which confirms the good agreement between data and theory at all cooling powers.

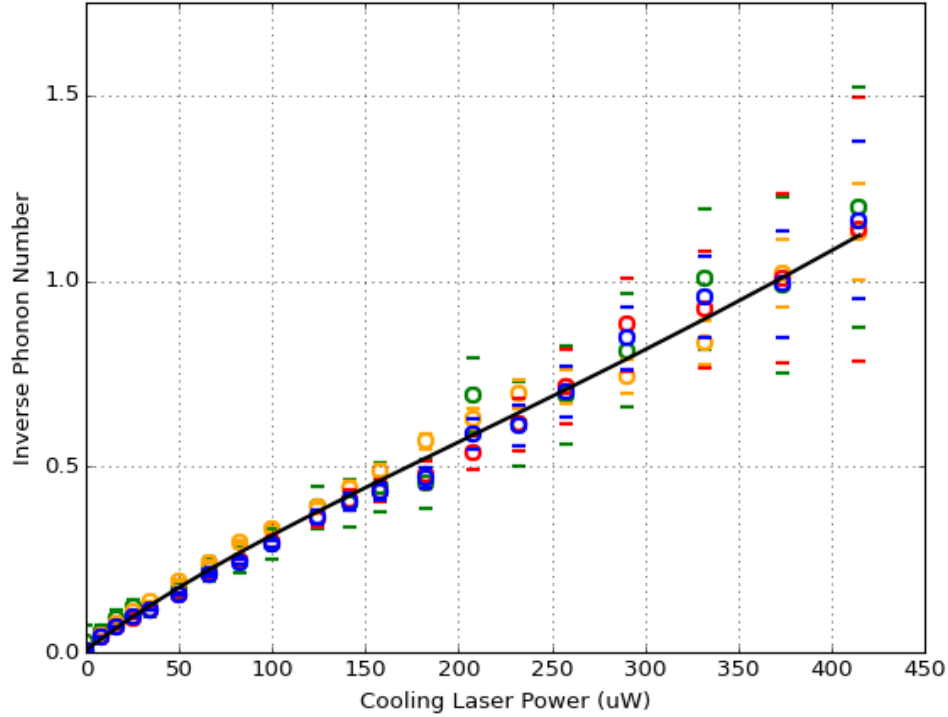


Figure 68: Plot of the inverse of the phonon number of the mechanical mode as a function of cooling power, with comparison to theory.

6. Future Directions

While we were successful in cooling the membrane to low phonon occupancy, there is still room for improvement. One improvement would be to reduce heating of the membrane due to absorption of light scattered in the cryostat. Since it is difficult to change the material properties of the cavity mirrors and the membrane to reduce absorption, the most practical method of reducing absorption is simply to reduce the amount of laser light that is allowed to enter the cryostat. While the measurement beam and the cooling beam are essential, it is not necessary to send the LO beam into the fridge. Since the LO beam is ~ 1.6 mW compared with the ~ 31 μ W measurement beam and the 400 μ W max cooling beam,

keeping the LO beam out of the fridge could result in decreased absorptive heating by a factor of a several. It is true that the large intracavity power from the measurement beam dwarfs the LO beam, but practice has shown that the LO beam is the dominant contributor to changes in the thermal bath temperature as indicated by the stage and ^3He pot thermometers. Since the LO beam does not enter the cavity, heating of the membrane due to the LO beam likely occurs indirectly through the membrane's thermal heatsinking wires to the ^3He pot and through its mechanical connections with the stage.

A membrane mode with higher quality factor would also aid in getting to lower phonon numbers. The optical damping produced by the cooling beam should be unchanged by membrane Q , but the cooling factor $\sim \gamma_p/\gamma$ will become larger if the starting linewidth is smaller. Other groups have reported measuring incredibly high quality factors ($Q > 1 \times 10^8$) in similar membranes cooled below 20 mK in a dilution refrigerator.⁵⁹ While the effects of optical absorption may prevent maintaining a temperature this low, it is nonetheless worth investigating whether a higher Q might be possible by further cryogenic cooling of the membrane.

The Regal group has studied the effect of patterning the membrane chip to form a phononic crystal around the membrane.⁶⁰ The acoustic band gap created by the phononic crystal provides isolation between membrane's mechanical modes and the environment. Though early results have not shown significant increases the mechanical quality factors of membrane modes due to the phononic crystal, future refinements to the phononic crystal's fabrication techniques may yield improvements. Additionally, the mechanical properties of membranes suspended within these phononic crystals have not been studied at cryogenic temperatures.

As I mentioned in the introduction, while an accomplishment in itself, ground state cooling is actually just a step along the road to future experiments and applications in quantum computing. Preparation of Fock or arbitrary quantum states, and the use of mechanical oscillators as quantum

memory elements both benefit from oscillators that have been cooled to reduce the influence of the thermal bath, as do proposed experiments to directly observe quantization of the mechanical oscillator's energy levels. Proposals for the latter of these, in fact, benefit both from ground state cooling, and the ability to couple the optical mode quadratically to the position of the membrane. In the second half of this dissertation, I will present our results studying the classical dynamics of our system with this type of quadratic coupling.

IV. Quadratic Optomechanics

The physics underlying the ground state cooling experiment just described was based on a Hamiltonian in which the resonant frequency of a single cavity mode scaled linearly with the displacement of the mechanical oscillator. As described briefly in the section General Treatment of a Membrane in a Cavity by Perturbation Theory, there are places within the cavity spectrum of a membrane-in-the-middle system where this description of the relationship between the cavity eigenmodes and the membrane's position is not correct. The presence of the membrane within the cavity induces coupling between optical modes that would normally be orthogonal, and this coupling is strongly affected by the position, tip, and tilt of the membrane. In regions where normally orthogonal cavity modes cross, coupling between the modes due to the membrane can cause the modes to avoid each other.^{37, 43, 49} At these avoided crossings, the perturbed optical modes can exhibit a frequency dependence which scales quadratically with membrane position rather than linearly. This quadratic coupling is predicted to be useful for producing squeezed optical and mechanical states⁶¹ and for studying the phonon statistics of a driven mechanical oscillator.^{62, 63}

There have been many theoretical and experimental studies of the static spectroscopy of avoided crossings in various optomechanical systems.^{37, 43, 49, 64, 65, 66, 67, 68} Fewer experiments have studied the dynamical effects that arise from the non-linear coupling between the cavity mode and mechanical oscillator.^{66, 67, 68} In this chapter I will present a theoretical description of, as well as an experimental study of, the dynamics arising from avoided crossings between cavity modes in a membrane-in-the-middle system. In particular, I will present results systematically characterizing the optical spring and optical damping in the vicinity of an avoided crossing, including their dependence on membrane position, laser detuning, and laser power. Using these results I will both show and explain the differences between linear and quadratic optomechanical coupling. Finally, I will present a

measurement exploiting quadratic optomechanical coupling to perform a measurement of classical fluctuations of the intracavity photon number, a first step toward using quadratic coupling to detect non-classical fluctuations in a driven oscillator.^{62, 63}

1. Introduction to Avoided Crossings

Using the theory described in the section General Treatment of a Membrane in a Cavity by Perturbation Theory, the effect of the membrane's position, tip, and tilt on the optical modes of the cavity can be predicted. As the perturbation theory integrals are easiest to compute numerically, I will use parameters appropriate for the membrane-in-the-middle system used in this experiment. Namely, these are: membrane thickness $L_d = 50$ nm, cavity length $L = 3.7$ cm, cavity mirror radii of curvature $R = 5$ cm, membrane index of refraction $n_{\text{SiN}} = 2.0$, and laser wavelength $\lambda = 1064$ nm. We can start by considering the $\text{TEM}_{0,0}$ singlet mode, and the $\text{TEM}_{2,0}$, $\text{TEM}_{1,1}$ and $\text{TEM}_{0,2}$ triplet modes. For a membrane located very close to the cavity waist with no tip or tilt relative to the cavity's optical axis, the three triplet modes are completely degenerate by symmetry and exhibit periodic crossings with the singlet mode of the next longitudinal mode number. This is shown in Figure 69 below, where the degenerate triplet modes are the lower of the two sinusoidal curves.

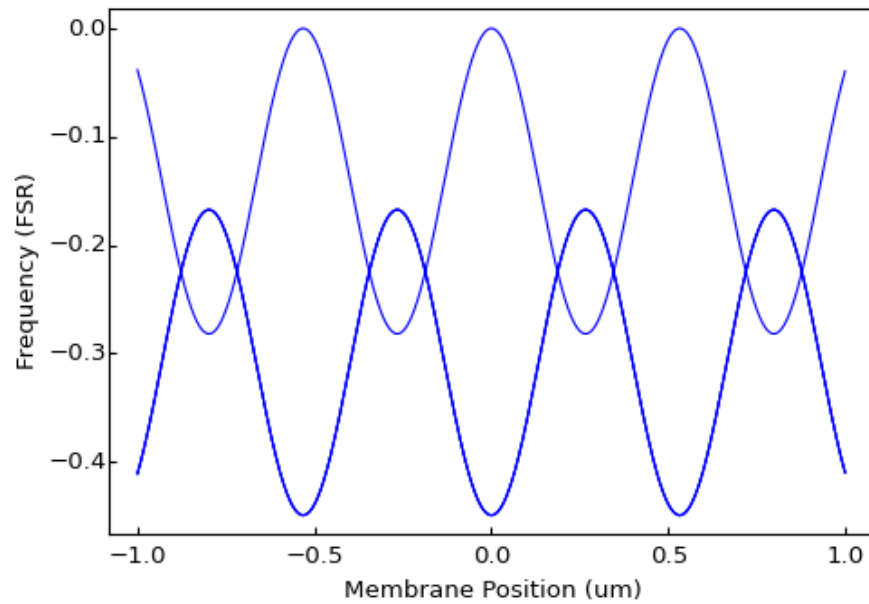


Figure 69: Theory plot of the resonant frequencies of the singlet and triplet modes as a function of a membrane position relative to the cavity waist.

Zooming in on one of the crossings between the triplet modes and the singlet mode, it is clear that the modes cross without interacting, i.e. the modes behave the same as they would if the other mode(s) were not present. This is expected, because despite the fact that the modes share a common eigenvalue at the crossing, they still have orthogonal transverse spatial distributions. Figure 70 shows this zoomed in view.

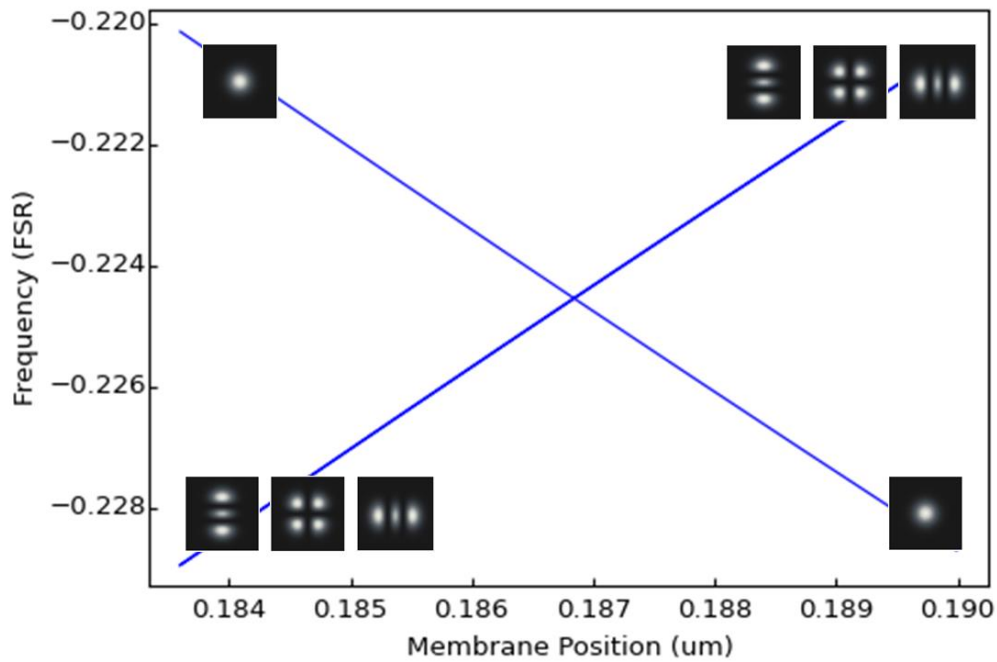


Figure 70: Zoomed in plot of the resonant frequencies of the singlet and triplet modes vs. membrane position relative to the cavity waist, focusing on the point at which the singlet and triplet mode frequencies cross.

Tilting the membrane has several effects on the modes at a crossing. We might consider, for example, the effect of tilting the membrane along an axis that intersects the optical modes as shown in Figure 71 below.

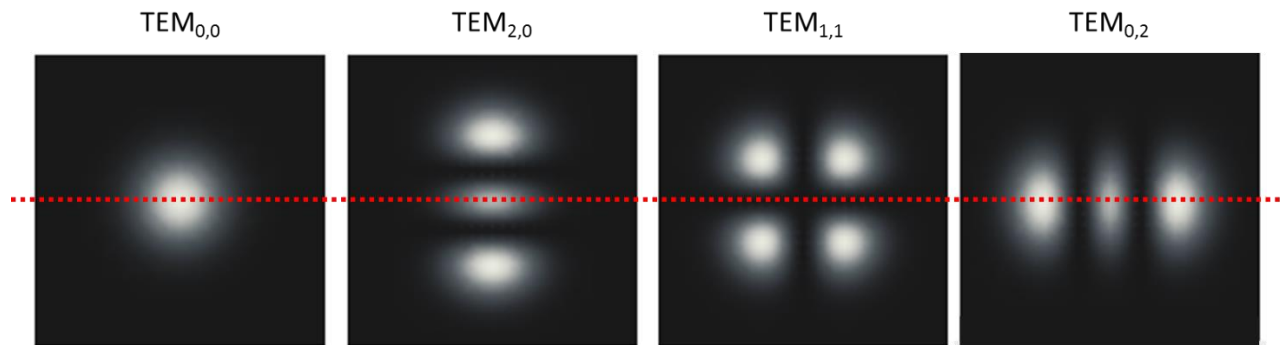


Figure 71: Intensity profiles of the singlet and triplet modes, overlaid with a dashed line indicating the orientation of the axis around which membrane is tilted relative to the modes.

While this tilt will affect all of the modes, it has a greater effect on modes whose spatial distributions extend further along the direction in which the membrane is tilted. Therefore, the $TEM_{2,0}$ mode is most

affected, followed by the $TEM_{1,1}$ and finally the $TEM_{0,2}$ and $TEM_{0,0}$. As a result of this, tilting the membrane will lift the degeneracy between the triplet modes, since they are unevenly affected by the membrane's orientation. Additionally, the tilt introduces coupling between the singlet mode and triplet modes, particularly between the $TEM_{0,0}$ mode and the $TEM_{2,0}$ mode. At the point in the cavity spectrum where these two modes cross, the coupling between them is strong enough that they are no longer eigenmodes of the system. The new eigenmodes consist of a superposition of the unperturbed $TEM_{0,0}$ and $TEM_{2,0}$ modes. The precise functional form and spatial profile of these new modes depends on the strength of the perturbation, i.e. the amount that the membrane was tilted. One possible combination of new eigenmodes is shown in Figure 72 below. The two modes shown (from left to right) correspond to the normalized combinations $\frac{1}{\sqrt{\pi}} \left(TEM_{0,0} \mp \frac{1}{2\sqrt{2}} TEM_{2,0} \right)$ respectively.

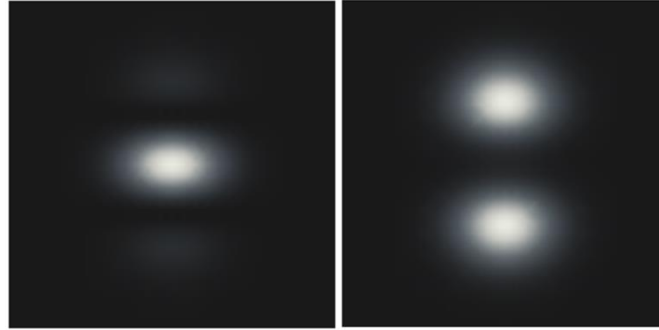


Figure 72: Intensity profiles of two possible eigenmodes of the cavity in the vicinity of the avoided crossing.

The eigenvalues of these new eigenmodes are slightly higher, and slightly lower than the original $TEM_{0,0}$ and $TEM_{2,0}$ mode eigenvalues would be at the point where the modes cross. Therefore, rather than crossing, the modes actually avoid each other in the cavity spectrum, as shown in Figure 73 below for a membrane tilt of approximately 0.1 milliradians.

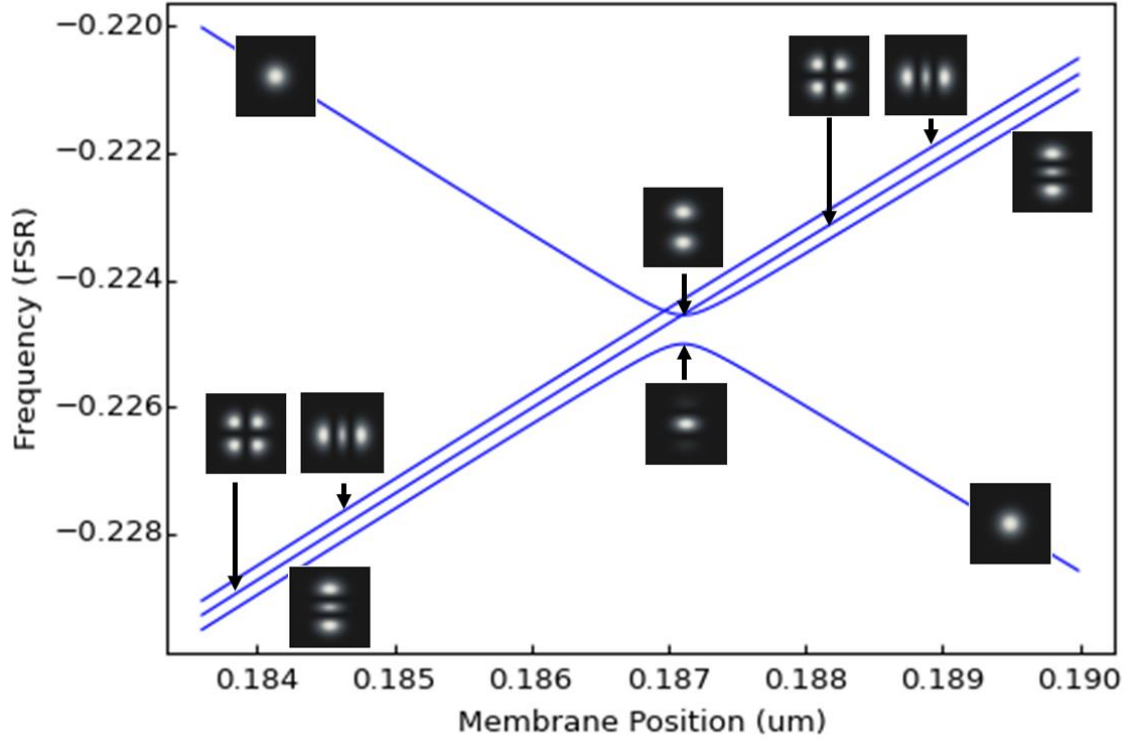


Figure 73: Plot of the resonant frequencies of the cavity eigenmodes as a function of membrane position relative to the cavity waist in the vicinity of a crossing between the singlet and triplet modes with a membrane tilt of 0.1 mrad.

The functional form of the eigenvalues of these modes near the avoided crossing is that of hyperbola, which in a Taylor expansion about the symmetric point has no linear term. Therefore, to lowest order, the avoided crossing resembles a pair of modes which couple quadratically to the position of the membrane. This allows for the study of non-linear optomechanical effects. The curvature of the modes at the avoided crossing (the second derivative of the mode's resonant frequency as a function of membrane position) determines the strength of the quadratic optomechanical interaction and can be adjusted by changing the tip/tilt of the membrane. For example, Figure 74 below shows a comparison of the avoided crossing between the $TEM_{0,0}$ mode and the $TEM_{2,0}$ mode for two different membrane tilts, illustrating the change in curvature of the avoided crossing.

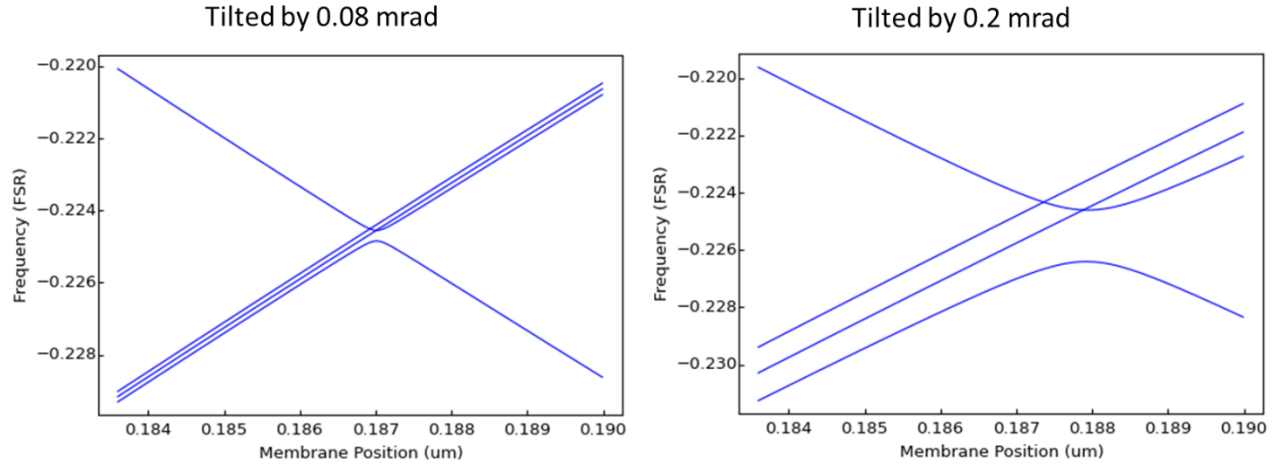


Figure 74: Theory plots of the cavity spectrum for membrane tilt of 0.08 mrad (left) and 0.2 mrad (right) illustrating how the curvature of the eigenfrequencies at the avoided crossing depends on the amount of membrane tilt.

Translating the membrane within the cavity can further break the symmetry of the cavity and can introduce coupling between the $TEM_{0,0}$ mode the other triplet modes, producing multiple avoided crossings. For example, below is a plot of the crossing between the singlet and triplet modes for a membrane that is tilted by 0.48 milliradians and translated by 500 microns from the cavity waist.

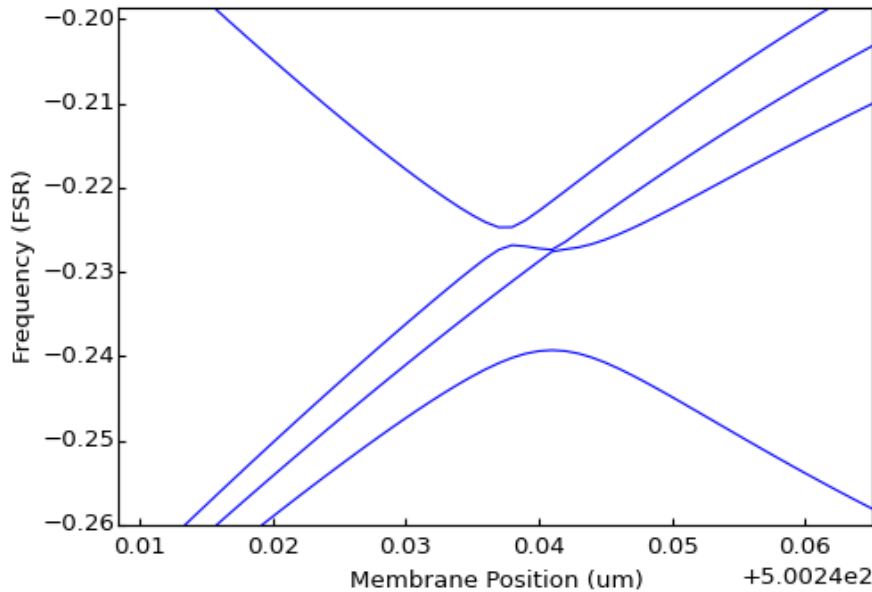


Figure 75: Theory plot of the cavity spectrum for membrane tilt of 0.48 mrad with a membrane displacement of 0.5 mm.

The membrane-in-the-middle system is extremely versatile when it comes to studying quadratic optomechanical effects due to the ease with which these avoided crossings can be tuned to produce points of quadratic coupling with various coupling strengths. In the next section, I will present a Hamiltonian describing the physics that can occur at these avoided crossings, and explain how these physics differ from the physics at a point of linear coupling.

2. Theory

Avoided crossings between optical cavity modes, due to the non-linear coupling of cavity frequency to membrane position, offer access to fundamentally different physics than what can be accessed at points of linear coupling. The physics that can be accessed at these points can be derived from a Hamiltonian which considers the presence of more than one optical mode. The simplest case is the Hamiltonian for an avoided crossing between two optical modes. It is conventional to refer to these modes as “left” and “right”, with corresponding subscripts L and R . The Hamiltonian for such a system with a stationary (non-oscillating) membrane is

$$\hat{H}_0 = \hbar(\omega_0 + g_{0,L}z_0)\hat{a}_L^\dagger\hat{a}_L + \hbar(\omega_0 + g_{0,R}z_0)\hat{a}_R^\dagger\hat{a}_R + \hbar(te^{i\phi}\hat{a}_L^\dagger\hat{a}_R + te^{-i\phi}\hat{a}_R^\dagger\hat{a}_L) + \hat{H}_{\text{env}} \quad (105)$$

The first two terms in this expression resemble the Hamiltonian for a single optical mode, e.g. Equation (41), and describe the basic optomechanical interaction between the membrane and each of the modes away from the avoided crossing. ω_0 is the frequency at which the modes cross, and $g_{0,L}$ and $g_{0,R}$ (which typically have opposite sign) are the optomechanical couplings of each mode determined by the slope of each mode in the cavity spectrum as a function of membrane position. z_0 is the membrane’s displacement from the center of the avoided crossing in units of its zero-point motion.

The third term in Equation (105) describes the coupling between the two optical modes induced by the presence of the membrane within the cavity. The coefficient t describes the strength of the coupling between the modes, and varies with the membrane’s orientation. ϕ describes the phase that

would be acquired by a photon tunneling from one mode to another, which is analogous to saying that each mode couples to the input drive with a unique phase, where the difference in phase between the two modes is ϕ . This Hamiltonian is conveniently expressed using vector notation to accommodate the multiple optical modes:

$$\hat{H}_0 = \hbar \bar{\hat{a}}^\dagger (\bar{\bar{\omega}}_c(z_0)) \bar{\hat{a}} + \hat{H}_{\text{env}} \quad (106)$$

In Equation (106) $\bar{\hat{a}} = \begin{pmatrix} \hat{a}_L \\ \hat{a}_R \end{pmatrix}$, and $\bar{\hat{a}}^\dagger$ is its conjugate transpose. The matrix $\bar{\bar{\omega}}_c(z_0)$ describes the optomechanical coupling between the modes as well as the tunneling rate between modes, and is defined as

$$\bar{\bar{\omega}}_c(z_0) = \begin{pmatrix} g_{0,L} z_0 & t e^{i\phi} \\ t e^{-i\phi} & g_{0,R} z_0 \end{pmatrix} \quad (107)$$

The equations of motion of the system can now be written as

$$\dot{\bar{\hat{a}}} = - \left(\frac{\bar{\bar{\kappa}}}{2} + i \bar{\bar{\omega}}_c(z_0) \right) \bar{\hat{a}} + \sqrt{\bar{\kappa}_{\text{in}}} a_{\text{in}} \quad (108)$$

$\bar{\bar{\kappa}}$ and $\sqrt{\bar{\kappa}_{\text{in}}}$ describe the bare linewidths of the modes and their input coupling rates, which need not be equal:

$$\bar{\bar{\kappa}} = \begin{pmatrix} \kappa_L & 0 \\ 0 & \kappa_R \end{pmatrix} \quad (109)$$

$$\sqrt{\bar{\kappa}_{\text{in}}} = \begin{pmatrix} \sqrt{\kappa_{L,\text{in}}} \\ \sqrt{\kappa_{R,\text{in}}} \end{pmatrix} \quad (110)$$

a_{in} represents the external laser drive, which in a frame rotating at the laser frequency, can be written as $a_{\text{in}}(t) = a_{\text{in}} e^{-i\Delta t}$. The steady state solution to Equation (108) is therefore

$$\bar{a}(t) = \bar{a}_0 e^{-i\Delta t}$$

$$\begin{aligned}\bar{a}_0 &= \left(\frac{\bar{\kappa}}{2} + i(\bar{\omega}_c(z_0) - \Delta) \right)^{-1} \sqrt{\kappa_{\text{in}}} a_{\text{in}} \\ &= \bar{\chi}_c[0] \sqrt{\kappa_{\text{in}}} a_{\text{in}}\end{aligned}\tag{111}$$

The cavity susceptibility matrix $\bar{\chi}_c[\omega] \equiv \left(\frac{\bar{\kappa}}{2} + i(\bar{\omega}_c(z_0) - \Delta - \omega) \right)^{-1}$ describes the response of each cavity mode to optical drives of detuning Δ . (Scalars such as Δ are assumed to be multiplied by the identity matrix, e.g. $\Delta \equiv \Delta \bar{I} = \begin{pmatrix} \Delta & 0 \\ 0 & \Delta \end{pmatrix}$.) With this, it is possible to express the reflected field amplitude as a sum of the laser drive with the light that has leaked out of the cavity from each mode:

$$a_{\text{refl}} = a_{\text{in}} - \sqrt{\kappa_{\text{in}}}^\dagger \bar{a}_0 = a_{\text{in}} \left(1 - \sqrt{\kappa_{\text{in}}}^\dagger \bar{\chi}_c[0] \sqrt{\kappa_{\text{in}}} \right)\tag{112}$$

Recall that so far, this formalism has dealt only with the case of a stationary membrane. To consider the case of an oscillating membrane, additional mechanical terms must be added to yield the complete the Hamiltonian:

$$\hat{H} = \hbar \bar{a}^\dagger \bar{g}_m \hat{a} \hat{z} + \hbar \omega_m \hat{c}^\dagger \hat{c} + \hat{H}_0\tag{113}$$

The first term describes the interaction between the optical mode and the oscillatory motion of the membrane. This interaction is not captured in the $\hbar \bar{a}^\dagger (\bar{\omega}_c(z_0)) \bar{a}$ term of Equation (106) because the coupling of the laser to an arbitrary mechanical mode of the membrane is unlikely be as large as the optomechanical coupling to static membrane displacements due to the curvature of the membrane during oscillation. In fact, for certain modes, if the laser is aligned symmetrically about a nodal line of the mechanical mode, the coupling could be as low as zero. Therefore, \bar{g}_m represents the mode specific optomechanical coupling. $\hat{z} = \hat{c}^\dagger + \hat{c}$ represents the instantaneous displacement of the mode from equilibrium. The second term in Equation (113) represents the energy of the mechanical mode, just as in

the single mode Hamiltonian (Equation (41)). The final term, \hat{H}_0 is the remainder of the Hamiltonian from Equation (106).

The equations of motion stemming from this new Hamiltonian are:

$$\dot{\bar{a}} = -\left(\frac{\bar{\kappa}}{2} + i\bar{\omega}_c(z_0)\right)\bar{a} - i\bar{g}_m\bar{a}z + \sqrt{\kappa_{\text{in}}}a_{\text{in}} \quad (114)$$

$$\dot{c} = -\left(\frac{\gamma_m}{2} + i\omega_m\right)c - i\bar{a}^\dagger\bar{g}_m\bar{a} + \sqrt{\gamma_m}\eta \quad (115)$$

These equations can be linearized by expressing $a(t) = (a_0 + d(t))e^{-i(\omega_c + \Delta)t}$ where a_0 represents the average field amplitude, and $d(t)$ represents small fluctuations about this amplitude. After defining the cavity enhanced optomechanical coupling as $\bar{\alpha} \equiv \bar{g}_m \bar{a}_0$, the equations of motion become

$$\dot{\bar{d}} = -\left(\frac{\bar{\kappa}}{2} + i\bar{\omega}_c(z_0) - i\Delta\right)\bar{d} - i\bar{\alpha}z \quad (116)$$

$$\dot{c} = -\left(\frac{\gamma_m}{2} + i\omega_m\right)c - i(\bar{\alpha}^\dagger\bar{d} + \bar{d}^\dagger\bar{\alpha}) + \sqrt{\gamma_m}\eta \quad (117)$$

Taking the Fourier transform of these equations and defining the mechanical susceptibility

$$\chi_m \equiv \left(\frac{\gamma_m}{2} + i(\omega_m - \omega)\right)^{-1} :$$

$$\bar{d}[\omega] = -i\bar{\chi}_c[\omega]\bar{\alpha}z[\omega] \quad (118)$$

$$\bar{d}^\dagger[\omega] = i\bar{\chi}_c[-\omega]\bar{\alpha}z[\omega] \quad (119)$$

$$c[\omega] = \chi_m[\omega](-i(\bar{\alpha}^\dagger\bar{d}[\omega] + \bar{d}^\dagger[\omega]\bar{\alpha}) + \sqrt{\gamma_m}\eta[\omega]) \quad (120)$$

$$c^\dagger[\omega] = \chi_m^*[-\omega](i(\bar{\alpha}^\dagger\bar{d}[\omega] + \bar{d}^\dagger[\omega]\bar{\alpha}) + \sqrt{\gamma_m}\eta^*[\omega]) \quad (121)$$

Equations (120) and (121) can be summed and the substitution $z[\omega] = c[\omega] + c^\dagger[\omega]$ made:

$$\begin{aligned} z[\omega] &= \chi_m[\omega](-i(\bar{\alpha}^\dagger\bar{d}[\omega] + \bar{d}^\dagger[\omega]\bar{\alpha}) + \sqrt{\gamma_m}\eta[\omega]) \\ &\quad + \chi_m^*[-\omega](i(\bar{\alpha}^\dagger\bar{d}[\omega] + \bar{d}^\dagger[\omega]\bar{\alpha}) + \sqrt{\gamma_m}\eta^*[\omega]) \end{aligned} \quad (122)$$

In the limit of high mechanical mode quality factor ($\omega_m/\gamma_m \gg 1$), the motion of the mechanical mode is tightly localized around $\omega \approx \omega_m$. Using these two criteria, the mechanical susceptibility terms from

Equation (122) can be approximated as $\chi_m[-\omega] = \left(\frac{\gamma}{2} + i(\omega + \omega_m)\right)^{-1} \approx (2i\omega_m)^{-1}$ and $\chi_m[\omega] = \left(\frac{\gamma}{2} + i(-\omega + \omega_m)\right)^{-1} \gg |\chi_m^*[-\omega]|$. Multiplying both sides of the equation by $\chi_m^{-1}[\omega]$, the $\chi_m^*[-\omega]$ is eliminated and the final result is achieved:

$$(\chi_m^{-1}[\omega] + \bar{\alpha}^\dagger(\bar{\chi}_c[\omega] - \bar{\chi}_c^\dagger[-\omega])\bar{\alpha})z[\omega] = \sqrt{\gamma_m}\eta[\omega] \quad (123)$$

In Equation (123), the bare mechanical susceptibility $\chi_m^{-1}[\omega]$ is modified by the second term, which is called the optomechanical self-energy, and is defined as follows:

$$\Sigma[\omega] = -i\bar{\alpha}^\dagger(\bar{\chi}_c[\omega] - \bar{\chi}_c^\dagger[-\omega])\bar{\alpha} \quad (124)$$

The change in mechanical frequency and linewidth due to optical spring and damping therefore can be easily computed from the self energy just as in the linear coupling case: $\delta\omega = \text{Re}(\Sigma[\omega_m])$ and $\delta\gamma = -2\text{Im}(\Sigma[\omega_m])$.

It is also possible to extend this theory to describe a system with more than two interacting optical modes. This is important because all Hermite Gauss optical modes other than the $\text{TEM}_{0,0}$ exhibit degeneracy. Therefore, any complete theory of the optomechanics at an avoided crossing must also consider the effects of the nearby nearly degenerate modes. The extension of the theory to account for additional modes is accomplished with the following substitutions.

$$\bar{\hat{a}} \rightarrow \begin{pmatrix} \hat{a}_L \\ \hat{a}_{R1} \\ \hat{a}_{R2} \\ \vdots \end{pmatrix} \quad (125)$$

$$\sqrt{\kappa_{in}} \rightarrow \begin{pmatrix} \sqrt{\kappa_{L,in}} \\ \sqrt{\kappa_{R1,in}} \\ \sqrt{\kappa_{R2,in}} \\ \vdots \end{pmatrix} \quad (126)$$

$$\bar{\kappa} \rightarrow \begin{pmatrix} \kappa_L & & & \\ & \kappa_{R1} & & \\ & & \kappa_{R2} & \\ & & & \ddots \end{pmatrix} \quad (127)$$

$$\bar{\bar{\omega}}_c(z_0) \rightarrow \begin{pmatrix} g_{0,L}z_0 & t_1 e^{i\phi_1} & t_2 e^{i\phi_2} & \dots \\ t_1 e^{-i\phi_1} & g_{0,R1}z_0 & & \\ t_2 e^{-i\phi_2} & & \sigma_2 + g_{0,R2}z_0 & \\ \vdots & & & \ddots \end{pmatrix} \quad (128)$$

In Equations (125)-(128), subscripts L and $R1$ are the modes from the original two-mode model. The additional nearly degenerate modes are represented by subscripts $\{R2, \dots\}$. Equation (128) accounts for the fact that these additional modes are not fully degenerate with the terms $\{\sigma_2, \dots\}$, which represent the frequency difference between modes $\{R2, \dots\}$ and the frequency of the avoided crossing between modes L and $R1$. The terms t_n and ϕ_n represent the coupling strength and phase acquired by photons tunneling from mode L to mode Rn .

The full algebraic expressions for the spring and damping are quite complicated to write out, even for the two-mode case, but numerical simulations can be easily used to predict the behavior of the optomechanical system for any number of modes. Figure 76 shows a plot of the optical spring and damping for a three mode system similar to the actual system we measured. (The parameters used to generate this theory plot are given in columns II and III of Table 2 later in this chapter.)

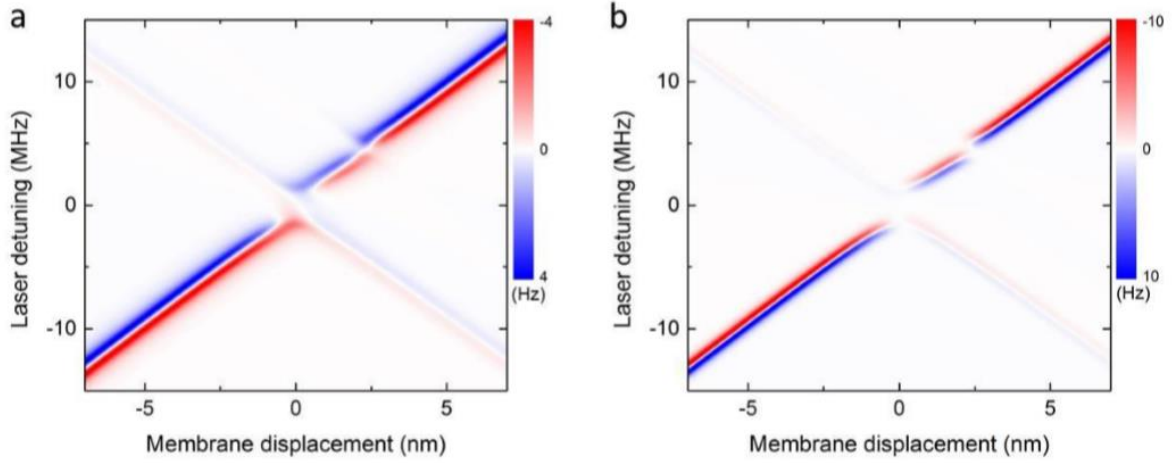


Figure 76: Theory plots of optical spring (left) and optical damping (right) for a system consisting of three optical modes forming two avoided crossings.

In Figure 76, it is clear that away from the avoided crossings, the system exhibits behavior typical of regular linear optomechanics. When red-detuned from the cavity resonance, the optical mode experiences positive damping and a negative spring shift. When blue detuned, the damping is negative and the spring shift is positive. At an avoided crossing, the spring shift occurs when the laser is on resonance rather than when it is detuned, and the damping becomes very small.

The reason why the mechanical mode responds like this near the avoided crossings can be understood by rewriting the Hamiltonian in the basis of the cavity eigenmodes at the avoided crossing. In this basis, at the avoided crossing, rather than two coupled modes that respond linearly to changes in position of the membrane, there are two modes that respond quadratically to the position of the membrane. A qualitative description of the system's dynamics can be extracted by looking at a Hamiltonian that considers only one of the two modes. Such a Hamiltonian can be written as:³⁷

$$\hat{H} \approx \hbar \left(\omega_c + \frac{1}{2} z_{\text{zpf}}^2 \omega_c'' (\hat{b}^\dagger + \hat{b})^2 \right) \hat{a}^\dagger \hat{a} + \hbar \omega_m \hat{b}^\dagger \hat{b} \quad (129)$$

In Equation (129), ω_c'' represents the second derivative of cavity mode's resonant frequency with respect to membrane position. In the rotating wave approximation, this Hamiltonian becomes:

$$\hat{H} \approx \hbar \left(\omega_c + z_{\text{zpf}}^2 \omega_c'' \left(\hat{b}^\dagger \hat{b} + \frac{1}{2} \right) \right) \hat{a}^\dagger \hat{a} + \hbar \omega_m \hat{b}^\dagger \hat{b} \quad (130)$$

Defining the quadratic optomechanical coupling strength $g_2 \equiv \omega_c'' z_{\text{zpf}}^2$ the terms in Equation (130) can be regrouped into a form that makes the origin of the spring shift at the avoided crossing in Figure 76 clear:

$$\hat{H} = \hbar \omega_c \hat{a}^\dagger \hat{a} + \frac{1}{2} g_2 \hat{a}^\dagger \hat{a} + \hbar (\omega_m + g_2 \hat{a}^\dagger \hat{a}) \hat{b}^\dagger \hat{b} \quad (131)$$

From Equation (131) it is clear that the quadratic coupling leads to a mechanical frequency shift per photon of light in the cavity. Therefore, at a point of quadratic coupling, the magnitude of the spring shift is maximized when the laser is on resonance with the cavity and the intracavity photon number $\hat{a}^\dagger \hat{a}$ is maximized. The lack of optical damping at a point of quadratic coupling is also evident from Equation (131). Since the Hamiltonian has no dependence on ω_c' , the net radiation pressure force acting on the membrane is zero. Thus, there can be no optical damping. It should be noted however that this simplified Hamiltonian does not capture the full behavior of the system. Damping can still occur through two phonon Raman processes that are not considered in the linearized Hamiltonian. Additionally, this Hamiltonian does not consider the presence of the second cavity eigenmode – as the gap between the two modes at the avoided crossing approaches the linewidth of the modes, it is no longer appropriate to approximate them as non-interacting quadratically coupled modes in the Hamiltonian, and the optical damping rate can become non-zero.⁴⁶

3. Experimental Setup

The experimental setup of the quadratic optomechanics experiment was overall quite similar to the setup used for ground state cooling, though there were significant differences in the optical setup outside of the cryostat, and a different membrane and cryogenic cavity were used.

A. Mechanical Mode

The membrane was a $1\text{ mm} \times 1\text{ mm} \times 50\text{ nm}$ high stress stoichiometric silicon nitride membrane just as in the ground state cooling experiment, though it was produced in an earlier batch by the manufacturer. The mode used was the $(1, 1)$ vibrational mode, which had a frequency $\omega_m/2\pi = 354.6\text{ kHz}$. A plot of this mode at the point of maximum deviation from equilibrium is shown in Figure 77.

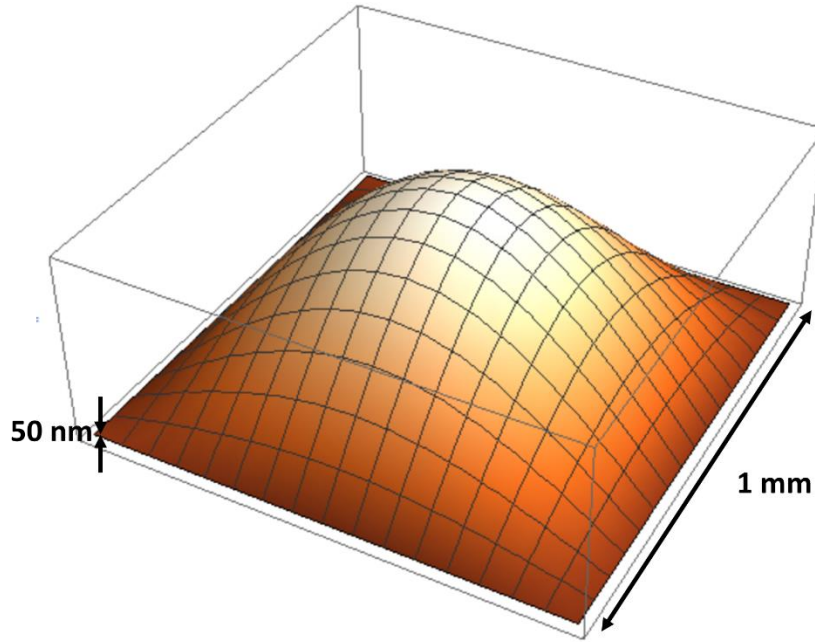


Figure 77: 3D plot of the $(1, 1)$ mechanical mode of a silicon nitride membrane.

The membrane was clamped onto the membrane plate using a stainless steel leaf spring as shown in Figure 25 rather than glued as in the ground state cooling experiment.

B. Optical Setup, Locking, and Measurement

The lasers used in the experiment were the same two Innolight Prometheus 1064 nm lasers as used for ground state cooling, though they served different purposes for this experiment. The optical setup outside of the cryostat is summarized in Figure 78. The setup for the quadratic optomechanics experiment was somewhat simpler than the setup for ground state cooling. As seen in Figure 78, filter

cavities were unnecessary for quadratic optomechanics, and therefore were not used. Additionally, since the filter cavities were unnecessary, the fiber AOM used to tune the lasers' frequencies after the filter cavities was also not necessary.

As with the ground state cooling experiment, a small amount of light from both lasers was picked off and combined on a fast photodiode (labelled FPD in Figure 78) so that the lasers could be frequency-locked approximately two free spectral ranges of the cryogenic cavity (~ 8 GHz) apart. The electrical circuit used for this lock is the same as the one shown in Figure 32.

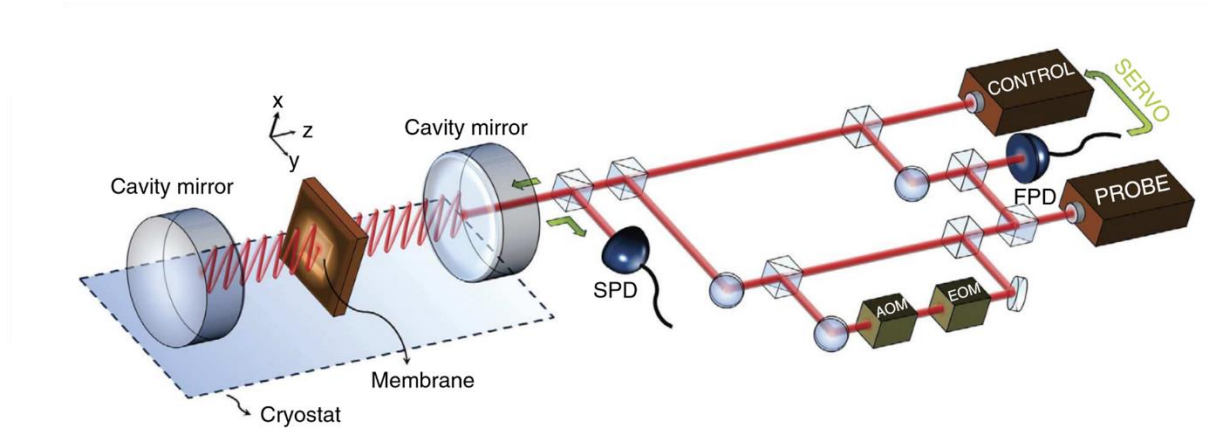


Figure 78: Schematic showing the optical setup for the quadratic optomechanics measurement.

Light from the probe laser (excluding the part picked off for frequency locking the control laser) passes through a beamsplitter. A portion of the light passes through an EOM which applies phase modulation sidebands for Pound-Drever-Hall locking to the cold cavity (we call this beam the “probe beam” or “measurement beam”). This beam then passes through an AOM which shifts it by 80 MHz for heterodyne detection. The shifted probe beam is then recombined with the unshifted, unmodulated beam (which we call the local oscillator, or LO). The combined probe beam and LO beam then recombine with the control beam and couple into a fiber which leads to the cryogenic cavity. The cryogenic optics setup is the same as shown in Figure 22. When locked on resonance, the probe beam enters the cavity and interacts with the membrane before it leaks back out, recombining with the

promptly reflected LO beam. The beams recouple into the optical fiber and return to the optical table where they are separated out using a beamsplitter and detected using a Thorlabs PDA10CF photodiode.

The scheme used to lock the measurement beam to the cryogenic cavity is summarized in Figure 79, and despite many similarities, is slightly different than the scheme used in the ground state cooling experiment. A Rigol DG1022 function generator applies ~ 15 MHz phase modulation sidebands onto the probe beam via an EOM. After interacting with the cavity, that light is detected by a photodiode. Part of the signal from the photodiode is low passed to remove the heterodyne signal, leaving only the Pound-Drever-Hall modulated signal. This is mixed with a reference tone from the function generator, producing signals at DC and ~ 30 MHz. The 30 MHz component is filtered out, and the DC component is then used as an error signal for a PI controller. The output of the PI controller goes through a high voltage amplifier, and finally to the laser piezo within the probe laser. To stabilize the system against slow long-term drifts that might exceed the tuning range of the probe laser piezo, the output of the PI controller is low passed and used as an error signal for a second PI controller. The output of this second PI controller is used to adjust the probe laser temperature so that the output to the piezo stays centered within its tuning range. This is summarized in Figure 79.

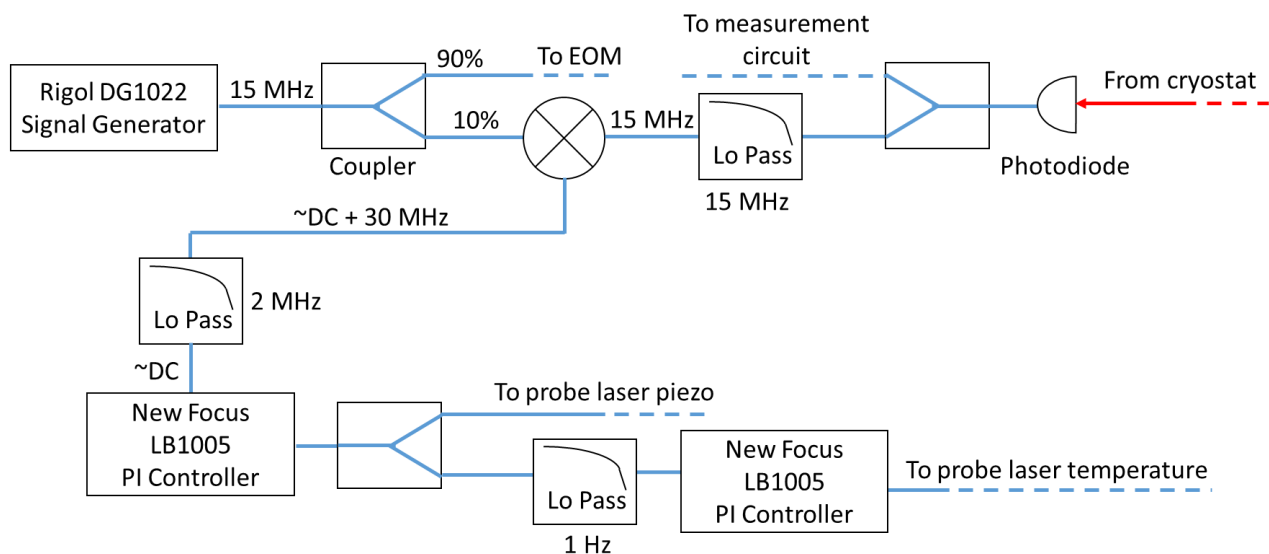


Figure 79: Schematic of the electrical circuit used to lock the probe laser to the cold cavity via a Pound-Drever-Hall technique.

The membrane's motion is measured using the same optical heterodyne technique as used for the ground state cooling experiment. The electronic circuit used to mix down the heterodyne signal for detection by the HF2-LI (shown in Figure 80) is largely the same, but the second HP signal generator that was used to drive the AOM that produces the optical heterodyne is not used in this configuration. Instead, power from the 101.3985 MHz signal generator is split off and mixed with a 21.3985 MHz tone from the HF2-LI. In this configuration, the frequency of the mixed down optical beat note is guaranteed to be precisely matched with the demodulation frequency in the HF2-LI, since they are both produced by a single oscillator within the HF2-LI.

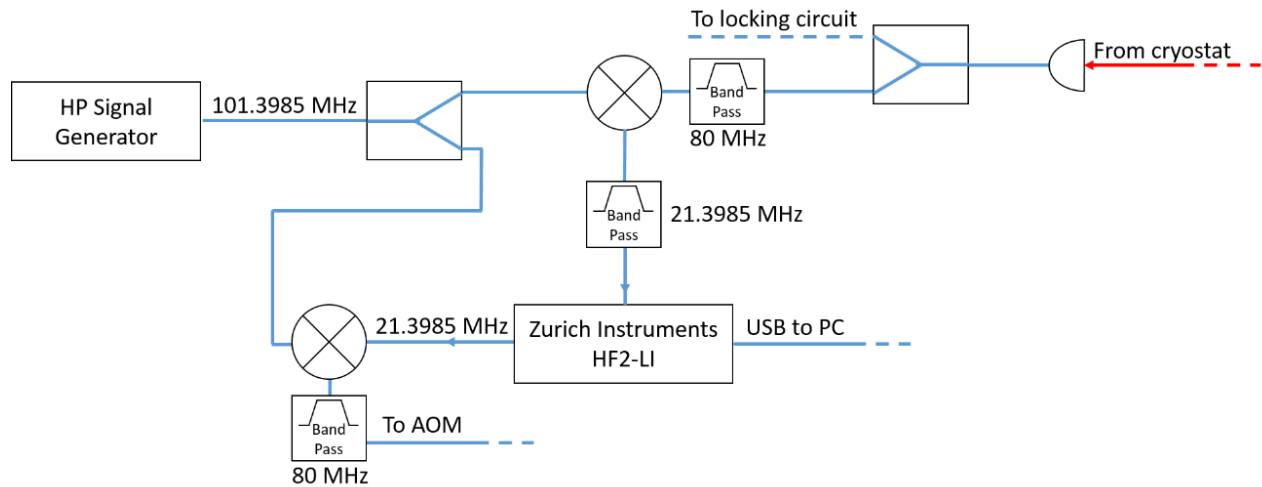


Figure 80: Schematic of the electrical circuit used to mix the optical heterodyne signal down to frequencies which can be detected by the Zurich Instruments HF2-LI.

Like in the ground state cooling experiment, locking the probe laser to the optical cavity involves more than just tuning the probe laser onto the cavity resonance. It is also necessary to move the membrane into a position such that both lasers are resonant with the right part of the cavity spectrum. For ground state cooling, the optimal spot in the cavity spectrum was the “sweet spot”, where both lasers were linearly coupled with identical slopes. For quadratic optomechanics, the optimal spot is in the vicinity of an avoided crossing.

We selected the avoided crossing between the $\text{TEM}_{0,0}$ mode and the three triplet modes: $\text{TEM}_{2,0}$, $\text{TEM}_{1,1}$, and $\text{TEM}_{0,2}$. We located the crossing by measuring the cavity spectrum as a function of control laser detuning and membrane position. After ensuring that the avoided crossing of interest was within the range of parameters accessible to the control laser, we manually detuned the probe laser by approximately two free spectral ranges of the cavity from the control laser (~ 8 GHz). We then measured the cavity spectrum using the probe laser. Since the probe laser, at this detuning, addresses different longitudinal modes of the optical cavity than the control laser, the avoided crossing occurs at a different membrane position than the crossing observed by the control laser. (This is a result of the fact that each longitudinal mode corresponds to a slightly different laser wavelength, and therefore a slightly shifted standing wave pattern inside the cavity.) This feature allows the probe laser to couple linearly to the membrane's position, while the control laser couples quadratically. Figure 81 shows the cavity spectra as measured by both lasers.

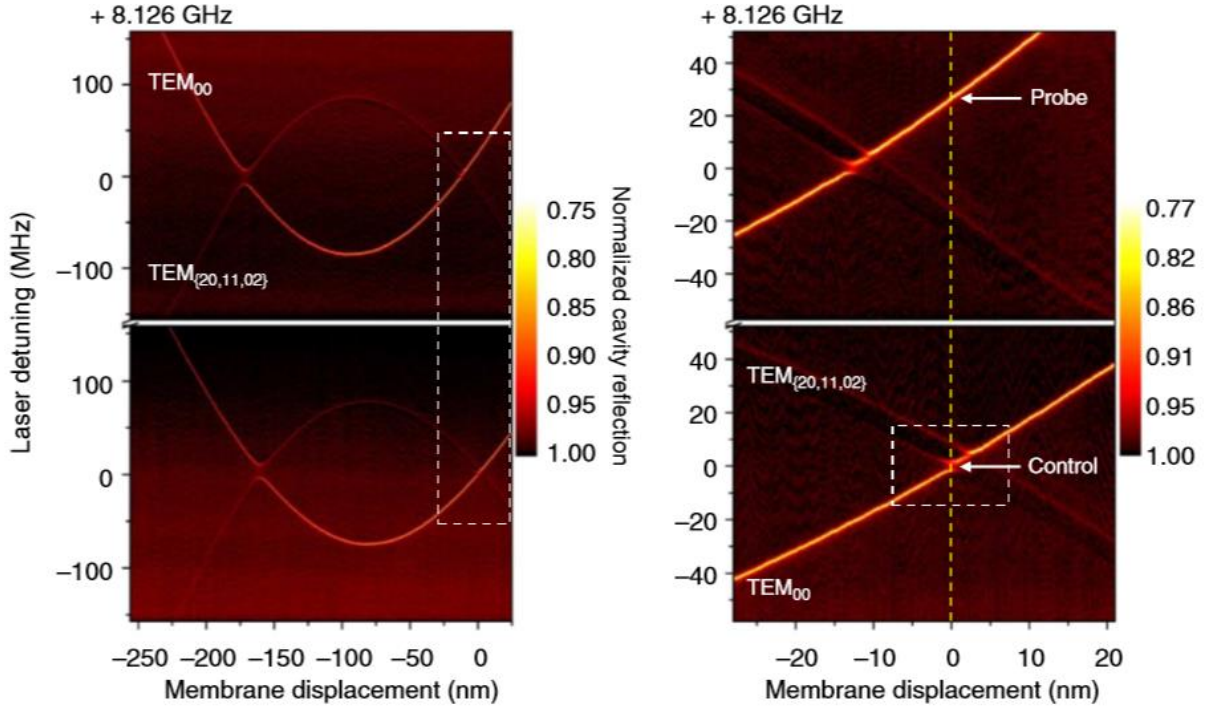


Figure 81: Left, cavity reflection spectra as measured by the probe laser (top) and control laser (bottom); right, zoom-in of the spectra around the avoided crossings

C. Characterizing Parameters

Calibrating the axes of the spectra in Figure 81 into units of frequency (for the y-axis) and displacement (for the x-axis) is the first step in characterizing experimental parameters to aid in analyzing results of measurements of the optomechanical dynamics of the system. The laser frequency and the membrane position are controlled by voltages to piezo actuators in the laser, and beneath the membrane, respectively. For the laser frequency, the piezo voltage-to-frequency conversion factor can be found by applying phase modulation sidebands at a known frequency using an EOM and measuring the piezo voltage difference necessary to shift the laser frequency such that first one, then the other sideband is on resonance with the cavity. This is the same procedure used to initially characterize the cavity linewidth in the ground state cooling measurement. See e.g., Figure 44.

For the membrane position, we calibrated the piezo voltage into units of displacement by measuring the difference in piezo voltage needed to move the membrane from a node of the cavity field to an antinode. Since nodes and antinodes should be separated from each other by a distance $\lambda/4$ and can be easily identified in the cavity spectrum as points of minimum and maximum resonance frequency (as shown in, e.g. Figure 9) this is a simple measurement to make by inspection of the cavity spectra.

The theory of multiple optical modes described at the beginning of this chapter should fully describe all aspects of the cavity spectrum in the vicinity of the avoided crossing. Therefore, once the measured cavity spectrum has been calibrated into units of laser frequency and membrane displacement, the spectrum itself can be used to characterize parameters in the theory. For example, the linewidth of each of the optical modes can be determined by fitting each vertical slice through the two-dimensional spectrum to a sum of Lorentzian lineshapes (with each representing one mode.) This process is shown in Figure 82, where the two modes that couple in the avoided crossing have been fit.

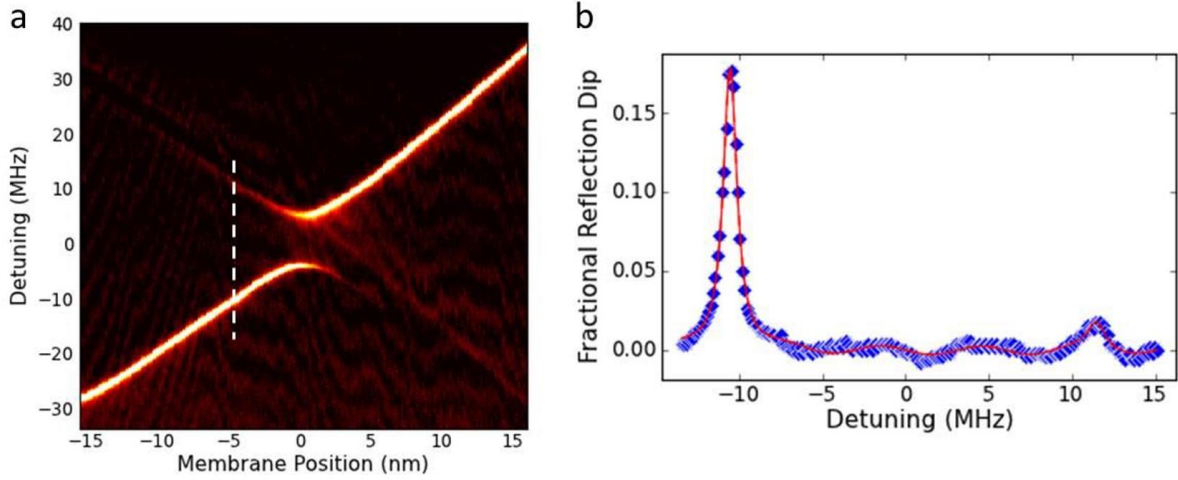


Figure 82: Left, zoom-in of the cavity spectrum around the avoided crossing. Right, a vertical slice of the two-dimensional spectrum, indicated on the left by the dashed white line. The red line is a fit to a sum of two Lorentzians plus a sinusoidal background, which comes from interference caused by reflections in the beam path.

From the fits to the slices of the two-dimensional cavity spectrum we extract the mode linewidths κ , and the coupling rate of each mode to the external field through the input mirror κ_{in} . The procedure for extracting κ_{in} here is the same as it was to extract κ_{in} in the ground state cooling measurement. Since the laser is primarily mode-matched to the $\text{TEM}_{0,0}$, the fractional reflection dip of the fainter mode in Figure 82 is small enough that it is difficult to fit above the sinusoidal background in the sweep. (The sinusoidal background is caused by back-reflections within the beam path causing periodic interference as the laser frequency is swept.) To improve the accuracy of fitting such faintly coupled modes, a series of fits is performed to slices at different membrane positions, and the fit results of parameters such as κ and κ_{in} are averaged.

For the particular avoided crossing shown in Figure 82, one of the triplet modes couples to the $\text{TEM}_{0,0}$ mode. However, depending on the membrane's tip and tilt in the cavity, there are configurations in which more than one mode can couple. In these cases, additional modes can be included in the fits to vertical slices of the cavity spectrum.

The fits to the slices of the two-dimensional cavity spectrum also yield the center frequency of each mode as a function of membrane position. Figure 83 shows a plot of the extracted center frequencies of the two modes shown in Figure 82.

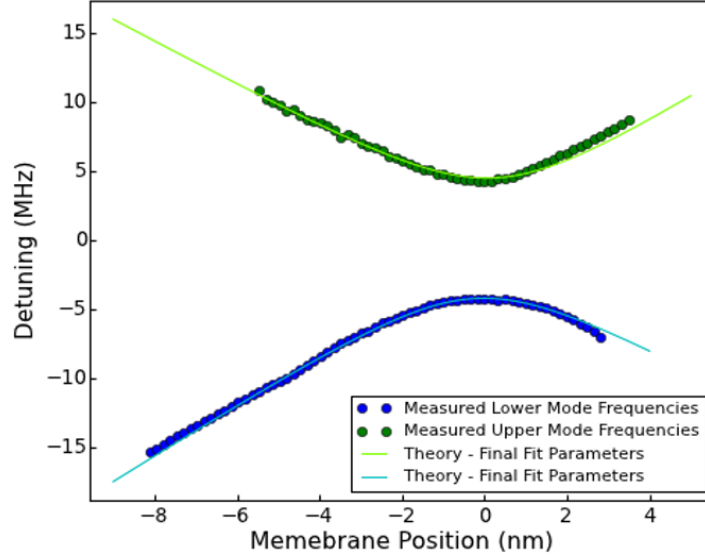


Figure 83: Plot of the center frequencies of two optical modes near the avoided crossing between the $TEM_{0,0}$ mode and one of the triplet modes. The center frequencies were extracted from fits to two-dimensional cavity spectra.

The behavior of the center frequencies of the modes near the avoided crossing is described by the eigenvalues of the matrix within the combined $\bar{a}^\dagger \bar{a}$ terms of the Hamiltonian. Writing out this term explicitly:

$$M = \hbar \bar{a}^\dagger \begin{pmatrix} \omega_c + g_{0,L} \hat{z}_0 + g_{m,L} \hat{z} & t e^{i\phi} \\ t e^{-i\phi} & \omega_c + g_{0,R} \hat{z}_0 + g_{m,R} \hat{z} \end{pmatrix} \bar{a} \quad (132)$$

The cavity spectra are produced by sweeping the static displacement of the membrane, \hat{z}_0 , and therefore they do not include contributions from \hat{z} . Setting $\hat{z} = 0$, the eigenvalues become:

$$\omega = \frac{1}{2} \left((g_{0,L} + g_{0,R}) \hat{z}_0 \pm \sqrt{4t^2 + (g_{0,L} - g_{0,R})^2 \hat{z}_0^2 + 2\omega_c} \right) \quad (133)$$

From Equation (133) it is clear that for $z_0 \gg t$, the eigenvalues are roughly the same as they would be if the modes were noninteracting. Near the avoided crossing, however, $z_0 \approx 0$, and each mode is shifted by an amount equal to t . The values of $g_{0,L}$ and $g_{0,R}$ can therefore be extracted from the slope of the modes' resonant frequencies away from the avoided crossing, while the value of t can be extracted from the size of the gap between the two modes at the crossing. For better precision in measuring t , it is useful to expand the eigenvalues in a Taylor series to second order in \hat{z}_0 and fit the modes around the avoided crossing with a quadratic function to determine the second-order coefficient in the Taylor series. It is then possible to extract t from this coefficient. We used this more precise method whenever possible to extract the value of t . This method also has the advantage of revealing the quadratic coupling strength ω_c'' , which is simply the coefficient of the quadratic term in the Taylor expansion.

The value of ϕ , the phase difference between the two modes at the avoided crossing, affects the interference between the modes when $z_0 \approx 0$. It can therefore be extracted by comparing the relative laser coupling strengths of the hybridized modes (indicated by their reflection dips) at the avoided crossing. Figure 84 shows theory plots of the cavity spectrum, zoomed in on the avoided crossing for various values of ϕ .

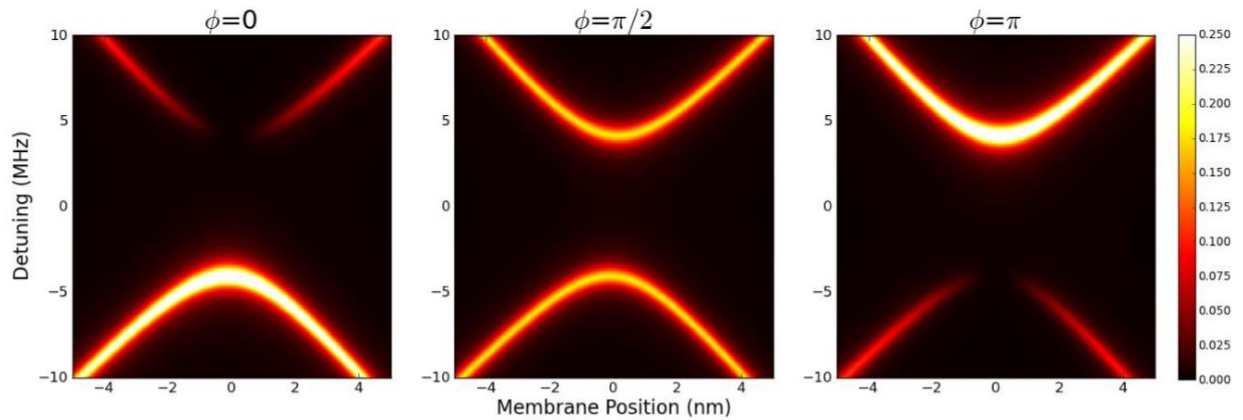


Figure 84: Theory plots of the cavity spectrum near the avoided crossing as a function of the tunneling phase ϕ .

The values of g_m cannot be determined by fitting cavity spectra alone, since they depend on the overlap of the intracavity field with a particular mechanical mode. Therefore, g_m for each mode is left as a fit parameter which is determined by fitting the observed optical spring and damping as a function of membrane position, laser detuning, and optical power. More details of this fitting process are described in the Results section.

In this experiment, we studied a total of three avoided crossings between the singlet and triplet optical modes. I shall refer to these crossings as *I*, *II*, and *III*. Spectra of these crossings are shown in Figure 85 below. Spectrum (a) in Figure 85 was taken with the membrane nominally at the cavity waist. Spectrum (b) was taken after translating the membrane -15 microns along the cavity waist and tilting it by 0.3 milliradians relative to the position in spectrum (a).

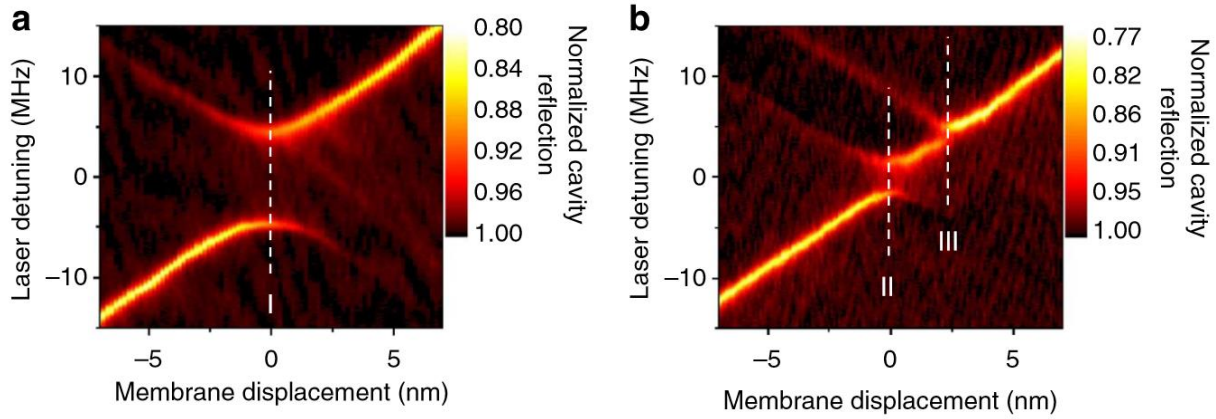


Figure 85: Cavity spectra of the avoided crossings studied in this experiment.

Table 2 below shows the system parameters measured at each of the avoided crossings shown in Figure 85. In each case, the subscript *L* refers to the $\text{TEM}_{0,0}$ mode, and the subscript *R* refers to the other mode at the avoided crossing.

Table 2: System parameters measured at each avoided crossing.

Parameter	Value		
	<i>I</i>	<i>II</i>	<i>III</i>
$\omega_c''/(2\pi)$	1.7 MHz/nm ²	4.2 MHz/nm ²	8.7 MHz/nm ²
$g_{0,L}/(2\pi z_{\text{zpf}})$	2.13 MHz/nm	1.87 MHz/nm	1.87 MHz/nm
$g_{0,R}/(2\pi z_{\text{zpf}})$	−1.82 MHz/nm	−1.77 MHz/nm	−1.77 MHz/nm
$g_{m,L}/(2\pi z_{\text{zpf}})$	2.13 MHz/nm	1.40 MHz/nm	1.26 MHz/nm
$g_{m,R}/(2\pi z_{\text{zpf}})$	−1.82 MHz/nm	−1.46 MHz/nm	−0.62 MHz/nm
$t/2\pi$	4.57 MHz	1.57 MHz	0.76 MHz
$\kappa_L/2\pi$	1.0 MHz	1.0 MHz	1.0 MHz
$\kappa_{L,\text{in}}/2\pi$	46.8 kHz	74 kHz	74 kHz
$\kappa_R/2\pi$	1.3 MHz	1.3 MHz	1.3 MHz
$\kappa_{R,\text{in}}/2\pi$	4.7 kHz	7.0 kHz	4.0 kHz
ϕ	1.6	1.9	1.1

4. Results

The first step^c in characterizing the optomechanical dynamics of avoided crossings was to position the membrane such that it was centered on crossing *II*. We then gradually stepped the control beam frequency over the avoided crossing, recording the membrane’s Brownian motion as measured by the optical heterodyne. Figure 86 shows the avoided crossing used for this portion of the experiment, along with a theory plot of the same crossing, generated using the parameters in Table 2. The qualitative agreement between the theory plot and the measured crossing suggests that the extracted of parameters from the cavity spectrum are quite accurate at reproducing the measured spectrum.

^c Chronologically, the membrane was initially positioned at crossing *I*, and later moved to crossings *II* and *III*. Though I have called measurements of crossing *II* the “first step”, I mean so only in the sense of that this order is amenable to discussion of the results.

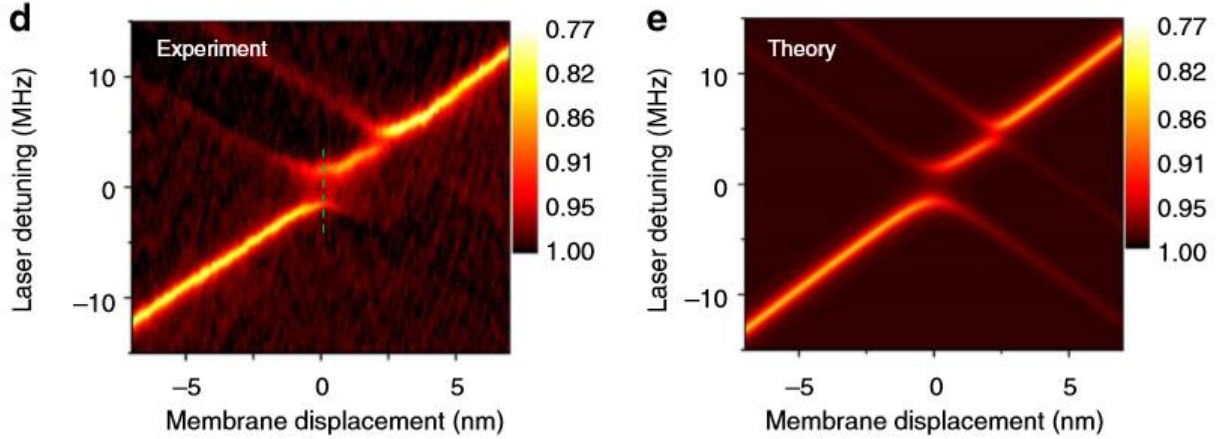


Figure 86: Left, zoom-in of the cavity spectrum around the avoided crossing with green dashed line showing the region over which the control laser was stepped. Right, a theory plot of the avoided crossing generated from the parameters extracted as described above.

The Brownian motion power spectral densities measured during the sweep over crossing *II* are plotted as a color plot in Figure 87. As expected from the theory, the mechanical mode's resonant frequency experiences a spring shift when the control beam is on resonance with either mode. The direction of the spring shift corresponds to the sign of the curvature of the mode in the two-dimensional cavity spectrum. Figure 87 also shows that a small amount of optomechanical damping and heating occur corresponding respectively to red and blue detuning of the control laser from either mode. While the color scale in Figure 87 makes this effect look rather pronounced, it really is quite small. The amplitude of the Lorentzian corresponding to the membrane's motion is only a factor of several larger at the point of maximum heating than it is when the control laser is at a point of maximum damping. This is in contrast to a laser that is red and blue detuned at a point of linear coupling, where, when blue detuned the membrane can be driven into self-sustained oscillations large enough to overwhelm the Pound-Drever-Hall lock, and when red detuned the membrane can experience positive damping by a factor of 10^4 or more.

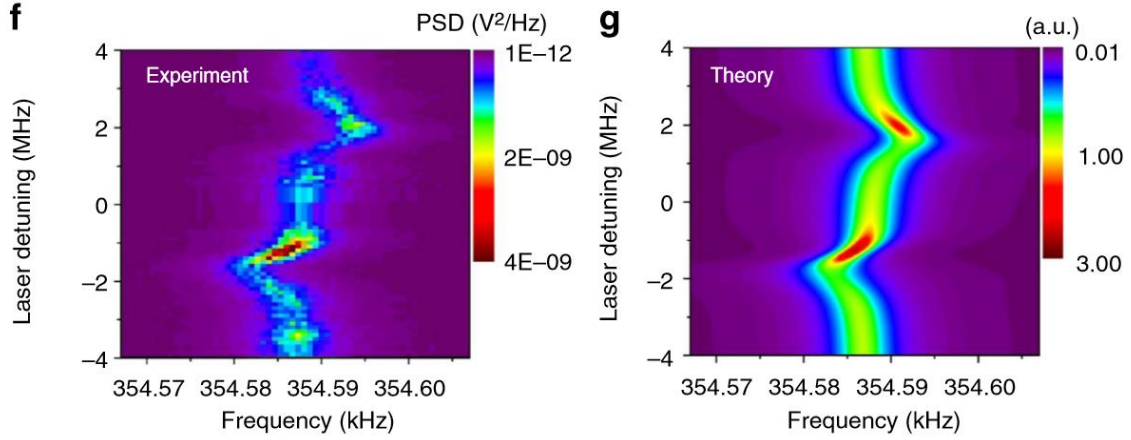


Figure 87: Left, color plot of Brownian motion power spectral densities of the membrane measured as the control laser was stepped across the avoided crossing. Right, color plot of the Brownian motion as predicted by theory.

After confirming that the Brownian motion behaves as expected right at the avoided crossing, we repeated the measurement at a series of membrane positions. This allowed us to observe the transition from traditional single-mode optomechanics to the multimode effects that occur at the avoided crossing. We picked ten different membrane positions around crossing *II* and stepped the control laser frequency over both of the optical modes at each membrane position. The membrane positions selected ranged from 0.96 nm to the left of the crossing to 1.38 nm to the right of the crossing. The laser power was fixed at 80 μ W incident on the cavity for the entire measurement. At each membrane position, the laser was swept over a frequency range of about 15 MHz, centered right in the middle of the two modes. Figure 88 shows the membrane positions and laser frequency ranges probed.

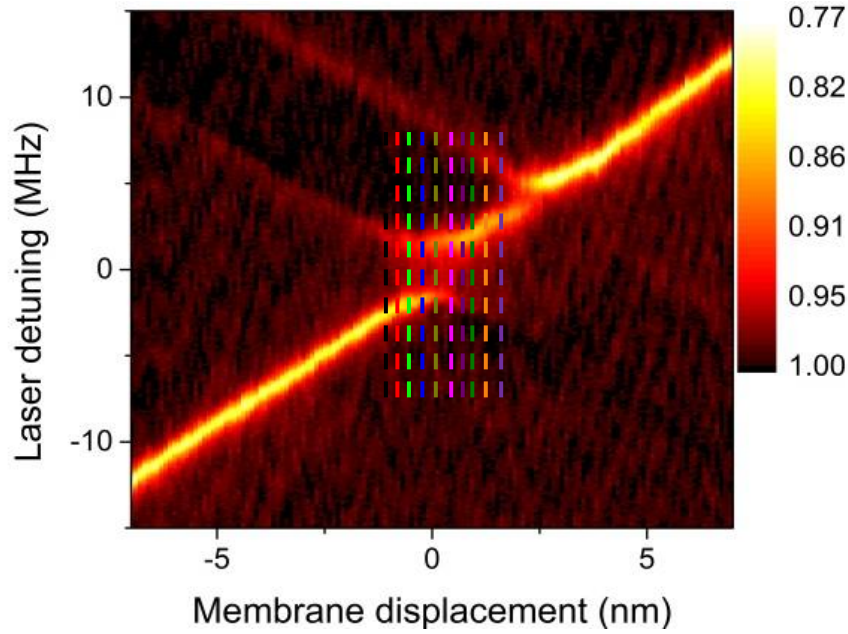


Figure 88: A zoomed-in plot of the cavity spectrum with dashed lines representing the range over which the control laser was stepped at various membrane positions around the avoided crossing.

The optical spring and damping were then extracted by fitting the recorded Brownian motion at each control laser step at each membrane position. The optical spring and damping are plotted below in Figure 89. The colors of the curves in Figure 89 correspond to the colors of the dotted lines in Figure 88 to allow simple comparison of the data in Figure 89 to the membrane position in the cavity spectrum at which it originates. The curved dashed lines in Figure 89 correspond to the center frequency of the optical modes in the cavity spectrum.

There are several notable features in Figure 89. When the membrane is away from the avoided crossing such as in the purple or the black curves, both the optical spring and the damping are antisymmetric about the optical modes. When red detuned from the optical mode, the mechanical mode experiences a negative spring shift, and positive damping. When blue detuned, the spring shift becomes positive, and the damping becomes negative. Each optical mode in this case behaves exactly as would be expected if it were the only mode in the system and the coupling was linear. This behavior is expected because the frequency spacing between the modes is much larger than other frequency scales

in the system, i.e. $|\omega_{0,0} - \omega_{\text{triplet}}|$ is much greater than κ of either mode, and also much greater than ω_m . The fact that the optomechanical effects are much stronger in the positive detuning mode for the purple curve, and negative detuning mode for the black curve is due to the fact that the laser is better coupled to the $\text{TEM}_{0,0}$ mode than to the triplet mode.

As the membrane is moved closer to the avoided crossing, as seen in the orange \rightarrow pink and red \rightarrow blue curves, the strong linear optomechanical effects become weaker, as expected from the reduced slope of the cavity resonant frequency ω_c as a function of membrane position \hat{z}_0 . At the same time, the quadratic optomechanical effects become stronger as the quadratic optomechanical coupling grows. Right at the avoided crossing, (the gold curve), the antisymmetric optical spring typical of linear optomechanics is gone, and instead a spring that is symmetric about each mode remains. The optical damping is dramatically reduced as expected for quadratic optomechanics, but remains antisymmetric about each mode.

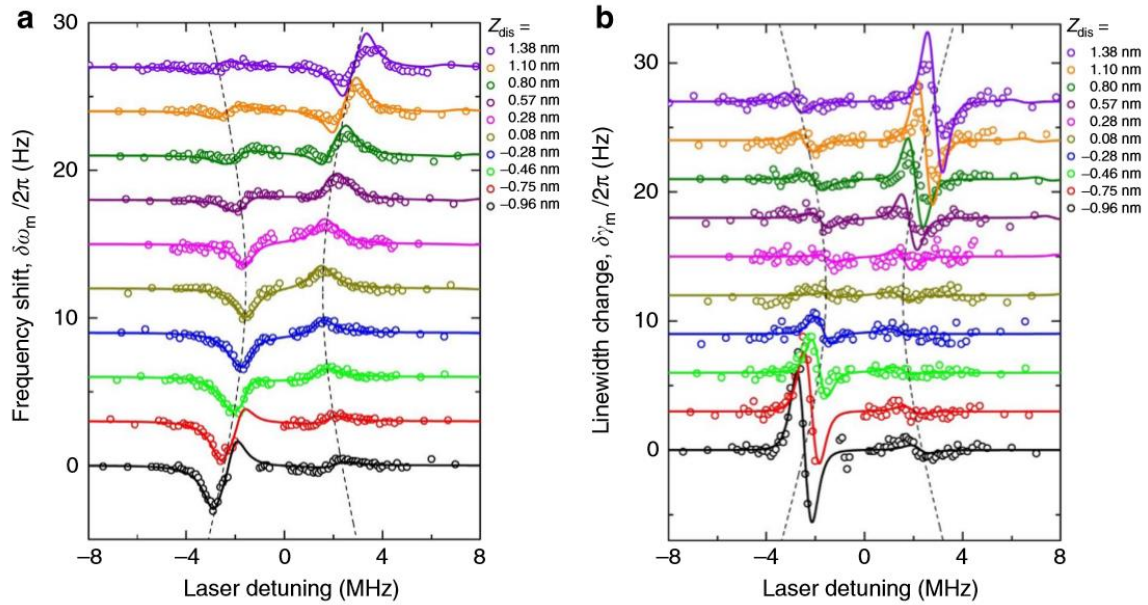


Figure 89: Left, plot of the optical spring of the mechanical mode as a function of laser detuning. Right, plot of optical damping of the mechanical mode as a function of laser detuning.

The solid lines in Figure 89 are theory plots with most parameters measured independently as previously described. The parameters $g_{m,L}$ and $g_{m,R}$, which cannot be extracted from the cavity spectrum, were determined here by first fitting the data at each membrane position with $g_{m,L}$ and $g_{m,R}$ as fit parameters. The results were then averaged, since $g_{m,L}$ and $g_{m,R}$ are not expected to vary with membrane position. The averaged values were used to make the theory plots, which show good agreement with the data.

After completing the measurement of optomechanics as a function of laser detuning and membrane position, we wanted to characterize the dependence of these effects on laser power. Turning up the laser power increases the number of photons in the cavity. Since the self-energy (Equation (124)) is proportional to $\alpha^\dagger \alpha$, it is reasonable to expect that the magnitude of both the spring and damping should scale linearly with laser power. To test this hypothesis, we positioned the membrane so that the control laser would be close to the center of avoided crossing *II*. We then stepped the control laser over the cavity resonances and recorded power spectral densities of the mechanical mode's Brownian motion. We fit these power spectral densities to extract the optical spring and damping at each laser detuning. The results of these fits are shown in Figure 90.

Each curve in Figure 90 represents a different control laser power. The solid lines are fits to theory based on the same parameters used for the theory curves in Figure 89, with laser power and membrane position as free parameters. The data show good agreement with theory, indicating that the optomechanical effects scale linearly as expected. Additionally, as seen in Figure 87 and Figure 89, since the membrane is centered on the avoided crossing, the optical spring is symmetric about each cavity mode, while the damping is antisymmetric.

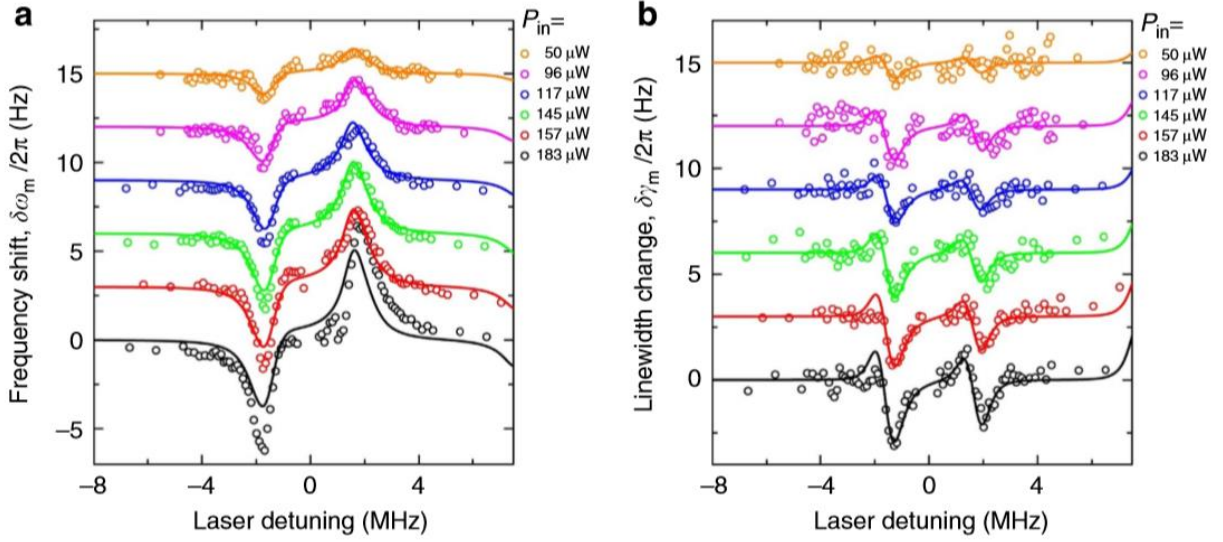


Figure 90: Left, plot of the optical spring of the mechanical mode as a function of laser detuning for various control beam powers. Right, plot of optical damping of the mechanical mode as a function of laser detuning for various control beam powers.

Toward the right side of both the spring and damping plots in Figure 90, the data do not appear flat as would be expected for a laser that is far detuned from the optical resonances, but rather show optomechanical effects due to the proximity of a third optical mode. As this data was taken roughly at the center of the avoided crossing, this third mode can be seen by looking at the top of the gold dashed line in Figure 88. While this third mode does influence the data seen in Figure 90, the laser couples to it linearly, and therefore we don't extend the stepping range of the control laser to include it.

As a sanity check on the extracted membrane position and control laser power for each of the curves in Figure 89 and Figure 90, we compared the values extracted from fits to values estimated by independent measurements. For the membrane position, the independent estimate arose from observation of the cavity spectrum as we stepped the membrane position, and for the control laser power, the independent estimate arose from measurements made using a power meter on the optical table, along with estimates of fiber coupling efficiencies and losses in the fiber and cryogenic system. In Figure 91a, the comparison of the fitted membrane displacement to the independently estimated membrane displacement shows good agreement between the two, indicating that the membrane

positions in the optomechanics vs. membrane position measurement (Figure 89) are accurate. Figure 91b shows that the fitted membrane displacement did not change as the control laser power was stepped during the optomechanics vs. control laser power measurement (Figure 90). Finally, Figure 91c indicates that the control laser powers during this measurement were close to what we intended them to be. Alternatively, Figure 91c can be interpreted as showing that the magnitude of the optomechanical effects in Figure 90 scaled linearly with control laser power as expected.

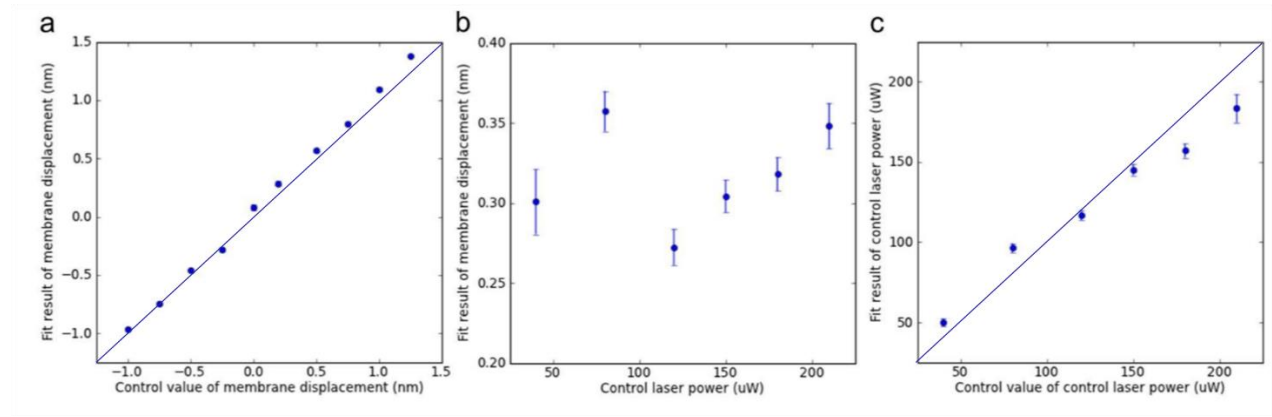


Figure 91: Plots comparing fitted membrane positions and control laser powers with their intended values.

Having demonstrated the predicted effects as a function of membrane position and laser power, we next sought to demonstrate how these effects change in response to changes in the geometry of the avoided crossing. To accomplish this, we positioned the membrane at the center of crossings *I* and *III* and stepped the laser frequency over the ranges indicated by dashed white lines in Figure 85. The control laser power was fixed at 80 μ W. The frequency shift of the mechanical mode as a function of laser detuning is shown in Figure 92. As the strength of quadratic coupling increases, the magnitude of the frequency shift increases as well. In *III*, the gap between the two modes is so small compared with the linewidth κ of each mode that it can no longer be resolved in the plot of the optical spring vs. laser detuning. In this case, qualitatively, the plot of the frequency shift at this avoided crossing largely resembles the antisymmetric shift that would be expected at a point of linear coupling, however, the

observed effects still arise from a multimode Hamiltonian and can be explained using quadratic coupling. If the membrane were stepped to various positions near this avoided crossing, it should still be possible to observe a transition from two separate linearly coupled modes with antisymmetric optical springs to the quadratic optical spring seen in curve *III* of Figure 92. From Figure 92 it is apparent that the magnitude of the optical spring effect seems to scale approximately linearly with quadratic coupling strength (ω_c''), as predicted by theory.

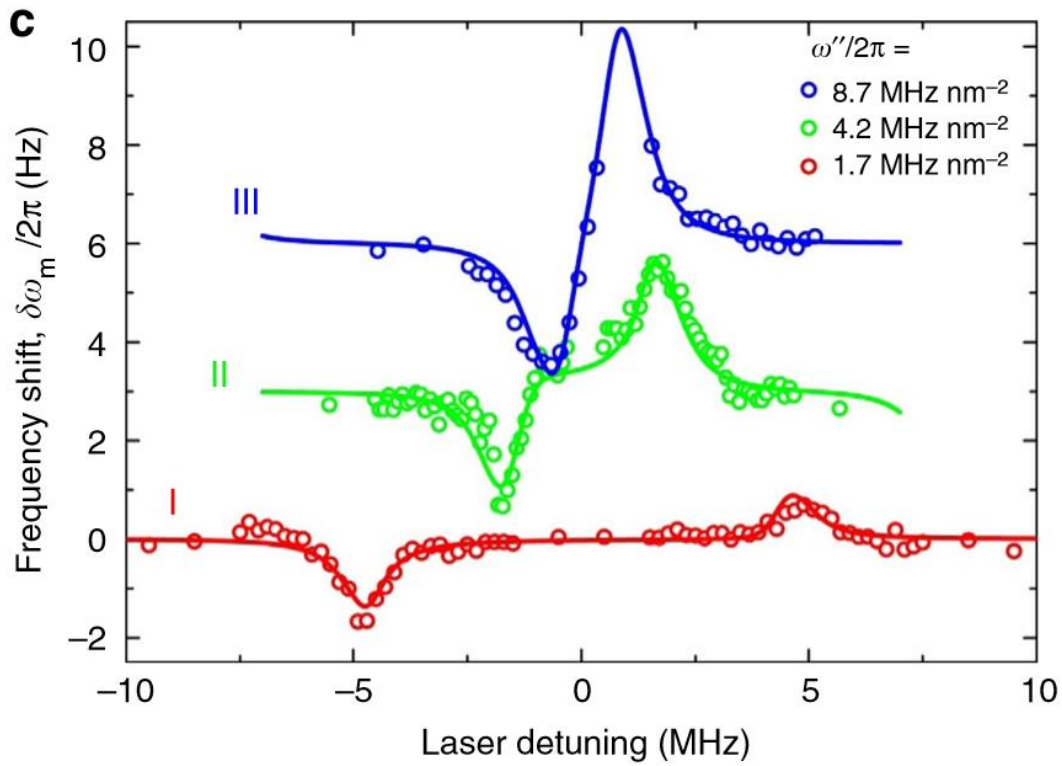


Figure 92: Plot of mechanical frequency shift as a function of laser detuning for three different avoided crossings.

While it is interesting to study quadratic optomechanical dynamics just to characterize them and compare them to theory, it is perhaps more useful to consider how these dynamics can be applied to make a practical measurement. Proposals for quantum non-demolition measurements of the optical mode's photon number or the mechanical mode's phonon number make use of a Hamiltonian similar to Equation (131), where the frequency of one oscillator is coupled to the photon/phonon occupancy of

the other. While much stronger quadratic coupling is required to make a measurement that can observe instantaneous quantum jumps between photon/phonon numbers,⁶⁹ weak measurements may still be able to observe non-classical fluctuations in the photon/phonon number of a driven oscillator.^{62, 63} As a first step in this direction, we used the membrane's mechanical frequency shift to detect classical fluctuations of the intracavity photon number.

To perform this demonstration, we used an AOM to modulate the intensity of the control laser with a modulation depth of 0.77 at 75 Hz. The periodic fluctuations of intracavity power caused by this modulation result in periodic modulation of the mechanical frequency of the membrane through the quadratic optical spring effect. In a power spectral density of the motion of the mechanical mode, this should manifest as 75 Hz frequency modulation sidebands around the membrane's mechanical resonance frequency. However, though the depth of the modulation of the control beam is large enough to produce large variations in intracavity power, the quadratic coupling is quite weak, so these frequency modulation sidebands are expected to be very small.

For example, the incident laser power used for the curves in Figure 92 was 80 μW . With a cavity finesse of $\mathcal{F} \approx 4,000$, this corresponds to a circulating power in the cavity of $P_{\text{circ}} = 320 \text{ mW}$. The intracavity photon number is therefore $n = \frac{P_{\text{circ}}}{(\text{FSR})(\hbar\omega_L)} = 4.3 \times 10^8$ photons. Looking at curve *I* in Figure 92, it is clear that these photons produce a frequency shift in the membrane via the quadratic optical spring effect of about 1.5 Hz. The coupling rate is therefore $3.5 \times 10^{-9} \text{ Hz/photon}$, which is quite small. For a modulation depth of 0.77, the amplitude of the modulation in terms of photon number is approximately 3.3×10^8 photons, which produces a maximum frequency shift of the mechanical mode of about 1 Hz. Therefore, though the intensity modulation index of the laser is a large 0.77, the resulting frequency modulation index (defined as $\Delta f / f_{\text{mod}}$ where Δf is the maximum frequency deviation and f_{mod} is the modulation frequency) of the membrane is a tiny 0.013. With such a modulation index, a

power spectral density of the membrane's motion will show that 99.992% of the power remains in the "carrier" frequency (the membrane's unperturbed resonant frequency), and only .004% appears in sidebands at 75 Hz. Therefore, to ensure that this tiny frequency modulation can be resolved above the noise of the measurement, it is useful to drive the membrane into a large amplitude oscillation.

To produce this large amplitude oscillation, we configured the HF2-LI to operate in a phase-locked loop mode to track the membrane's resonant frequency while simultaneously driving it. The input to the phase locked loop was adapted from the heterodyne detection circuit used for all of the preceding measurements. To serve as the input for the PLL, we took the mixed-down heterodyne signal with carrier frequency 21.3985 MHz and mixed it with a reference tone from the HF2-LI, also at 21.3985 MHz. This produced a signal near DC and a signal at 42.797 MHz. The higher frequency signal was removed with a low pass filter, and the low-frequency signal was used as the input to the phase locked loop. Since this signal was mixed down to DC, the portion of the signal containing information about the membrane's mechanical motion occurred at the actual frequency of the motion (354 kHz). Therefore, this signal was used as the input for the PLL.

The PLL controller within the HF2-LI locks onto the measured mechanical motion and then outputs a signal that tracks its phase and frequency exactly. This output signal was sent to the ring piezo under the membrane inside the cryostat to drive the membrane on resonance. The HF2-LI's phase locked loop controller is also able to output a frequency deviation signal corresponding to the frequency difference between an arbitrary setpoint and the frequency as tracked by the PLL. At DC this signal is affected by the arbitrary choice of setpoint, but at AC it contains useful information about the fluctuations in the mechanical frequency of the membrane. Therefore, we took this signal and recorded it by sending it from the auxiliary output port on the HF2-LI to to the HF2-LI's second input. A summary of the connections used to make this measurement is shown in Figure 93.

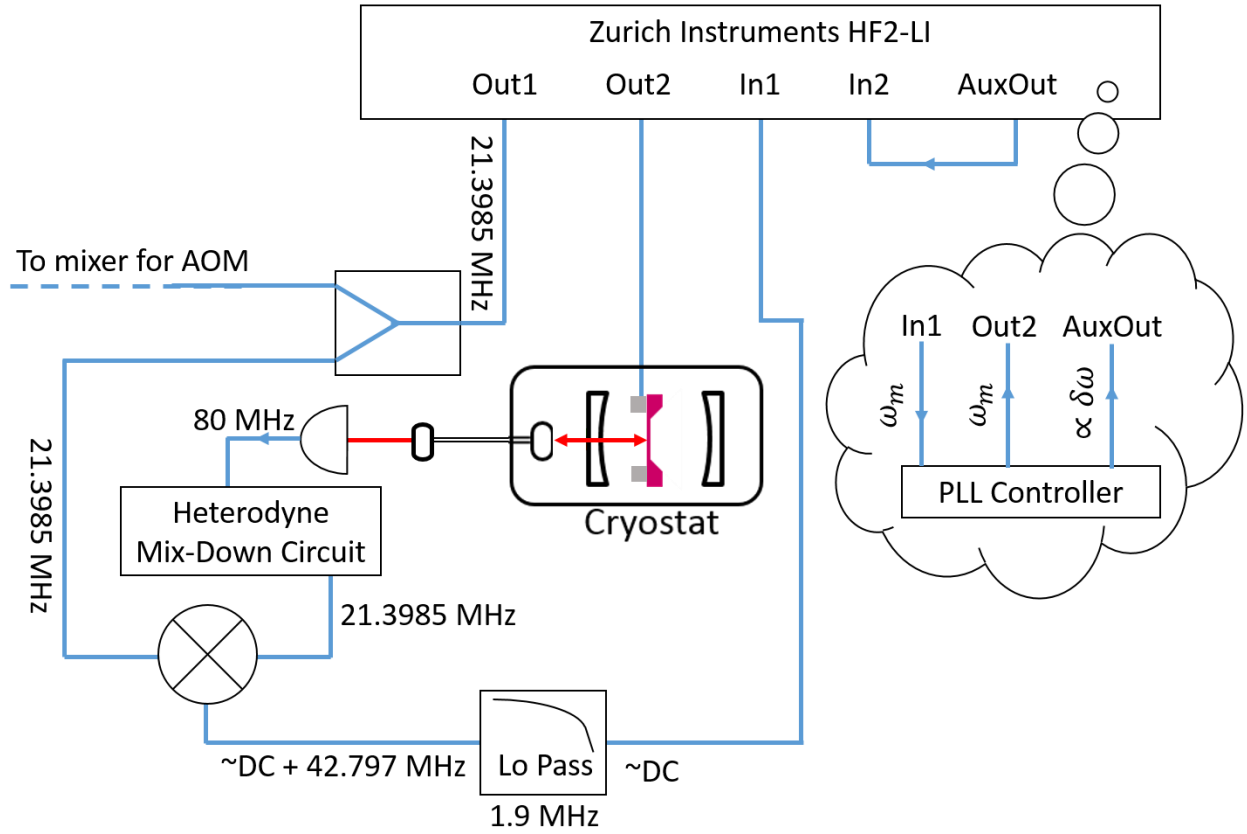


Figure 93: Schematic of the electrical circuit used for the phase-locked loop measurement.

By calibrating a power spectral density of the recorded frequency deviation signal into units of Hz^2/Hz , it is possible to recover the dynamic frequency deviation due to the optical spring. Such a power spectral density is shown in Figure 94. The peak at 75 Hz corresponds to the frequency modulation of the membrane's resonance due to the optical spring.

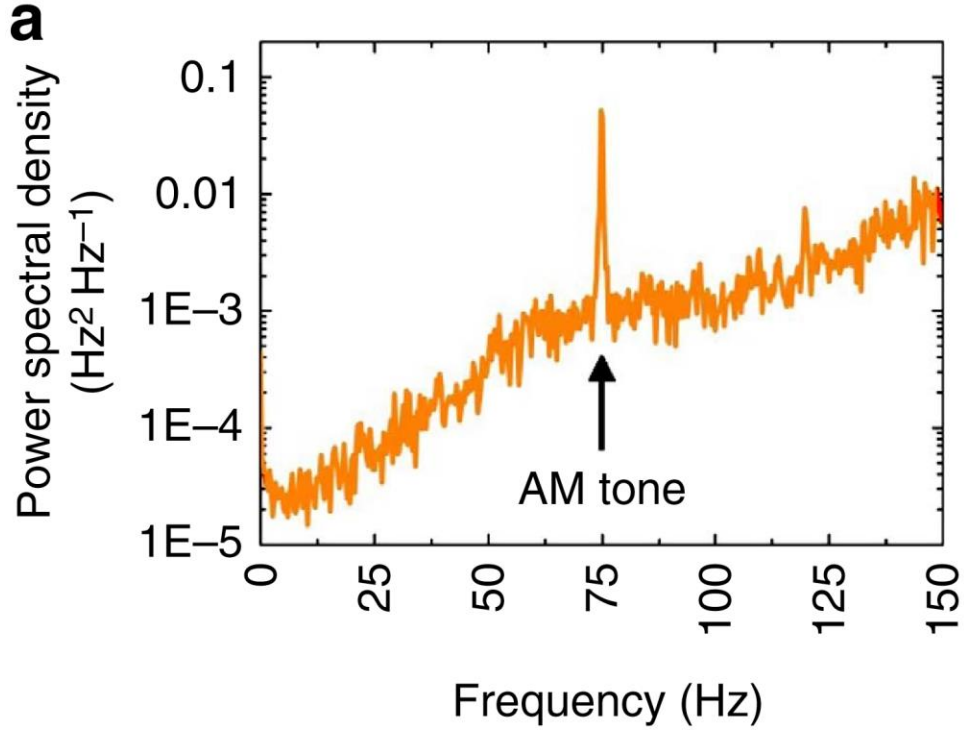


Figure 94: Power spectral density of the membrane's frequency deviation.

Taking the square root of the integral of the 75 Hz peak in Figure 94 gives the RMS frequency deviation of the of the membrane due to the optical spring. Figure 95 shows plots of this value for various laser detunings at both a point of quadratic coupling (left) and a point of linear coupling (right). With quadratic coupling, the maximum frequency deviation occurs when the control laser is directly on resonance. With linear coupling, the maximum frequency deviation occurs to both sides of resonance. These results are consistent with the observations of optical spring at quadratic and linear points in Figure 89a and the theory plots in Figure 76a. The magnitude of the optical spring induced frequency modulation (~ 0.2 Hz at a control laser detuning of 0) also roughly agrees with the ballpark prediction from above. The 1 Hz predicted maximum deviation above corresponds to an RMS deviation of 0.7 Hz, which matches the observed result within a factor of 3.5.

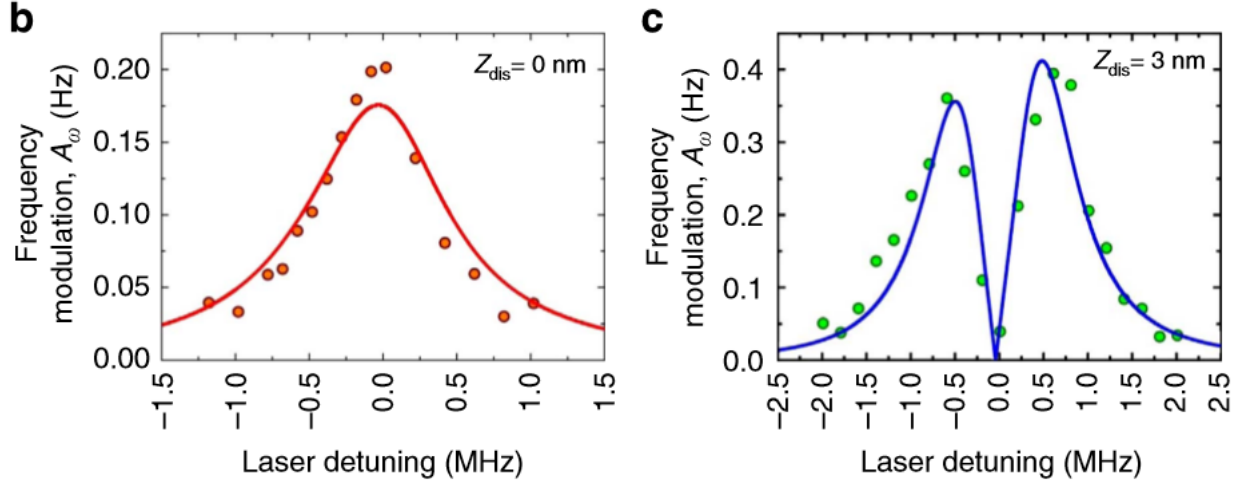


Figure 95: Plot of the maximum frequency deviation due to the optical spring effect caused by periodic modulation of the intracavity photon number. Left: with the membrane centered at an avoided crossing, giving quadratic coupling; right: with the membrane displaced by 3 nm from an avoided crossing, giving linear coupling.

Naively, other than their different behaviors as a function of laser detuning, nothing appears to be significantly different between the optical spring induced frequency shift at a linear vs. a quadratic point. However, their physical origin *is* different. At a linear point, the optical spring arises from the leakage of light into and out of the cavity with each oscillation of the membrane. At a quadratic point, the optical spring arises from the elastic energy stored in the intracavity field. Thus, in one case, the use of optical spring to characterize the intracavity photon number is based on the non-conservation of the intracavity the photon number, and in the other case it isn't. This difference is what creates the possibility of using quadratic coupling to perform quantum non-demolition measurements.

5. Future Directions

The quadratic optomechanics experiment described in this dissertation involved a first characterization of the optomechanical dynamics associated with avoided crossings between optical modes in the cavity spectrum. While the use of the quadratic coupling available at avoided crossings to directly observe quantum jumps between energy levels of the mechanical mode requires coupling strength not yet achieved in membrane-in-the-middle optomechanical systems, the quadratic coupling

at the avoided crossings in our system may be well suited to measurements of phonon shot noise. Phonon shot noise arises from non-classical fluctuations in the energy level of a mechanical oscillator in the same way that photon shot noise arises from vacuum fluctuations in an optical field. Careful measurements of the full counting statistics of such fluctuations in a mechanical oscillator are predicted to reveal evidence that these fluctuations have a quasiprobability distribution that is not positive-definite – a sign that these fluctuations cannot be described by any classical theory.⁶³ Proposals⁶² for measuring phonon shot noise call for a membrane cooled close to its ground state to reduce the influence of thermal noise on the spectrum of the oscillator’s motion, the ability to drive the membrane into large amplitude oscillations, and the ability to couple the cavity frequency to the phonon number of the oscillator. The membrane-in-the-middle system presented in this dissertation is uniquely suited to this type of measurement because of its ability to simultaneously achieve linear coupling for cooling using one longitudinal cavity mode and quadratic coupling at an avoided crossing in another longitudinal cavity mode.

Another step forward from the quadratic optomechanics experiment presented in this dissertation would be to explore the dynamics of other types of avoided crossings. The experiment presented here examined only the dynamics of what happens near avoided crossings between the frequencies of optical cavity modes. Under the right circumstances, such avoided crossings can also be observed between the frequencies of mechanical modes. Additionally, it is possible to generalize the concept of avoided crossings by considering eigenvalues in the complex plane, where the imaginary part of the eigenvalue represents the mode’s damping. When system control parameters are set such that the complex eigenvalues of two modes become degenerate, there is said to be an exceptional point in the space of the control parameters. For the mechanical eigenmodes of the membrane-in-the-middle system, suitable control parameters are laser power and cavity detuning. Theory predicts that the dependence of the complex eigenvalues upon these parameters has a non-trivial topological structure,

which can enable transfer of energy between eigenmodes by way of adiabatic topological operations. For example, one such operation would be to vary the laser power and detuning in such a way as to execute a closed loop around an exceptional point in the power-detuning plane. The presence of different damping rates of the mechanical eigenmodes during this operation is predicted to make the transfer of energy between modes non-reciprocal. A paper exploring these concepts experimentally was recently published by the Harris group at Yale using the same membrane-in-the-middle setup described in the ground state cooling portion of this dissertation.⁵⁶

V. Conclusion

The field of optomechanics has come a long way since the invention of the photophone. Modern systems that exploit radiation pressure to interact with macroscopic mechanical oscillators have demonstrated exceptional sensitivity to small mechanical displacements, and have started to explore the realm of non-linear coupling between mechanical oscillators and optical cavities. Using optomechanical systems, ground state cooling of macroscopic oscillators has been demonstrated, radiation pressure shot noise has been detected, and mechanical quantum states have been prepared. The future is indeed exciting, as systems begin to demonstrate processes useful for quantum information processing such as coherent state transfer from the optical field to a mechanical oscillator and back again.

The work I have described in this dissertation has contributed to the field of optomechanics in several important ways. First, I have described the cooling of one mechanical mode of a nanogram-scale silicon nitride membrane to a phonon occupancy of less than one. The large mass and low frequency of this oscillator compared to many others described in the literature made the cooling of the oscillator particularly difficult, and required particular attention to the influence of laser noise on the oscillator. This dissertation therefore describes the many steps we took to minimize the amplitude and phase noise on our lasers, and to verify that the noise was not significantly affecting the cooling or measurement of the membrane's motion. Though the methods used to do this may not in themselves be novel, the summary presented in this dissertation will most certainly be useful as a reference for future experiments.

Also in this dissertation, I have described the fundamentals of multimode optomechanics involving the coupling of one mechanical mode to two (or more) optical modes in the vicinity of avoided crossings. The results of this experiment demonstrated qualitatively different optomechanical effects

than would be expected for coupling to a single optical mode, and the results are consistent with a simple theory. I have shown that avoided crossings between cavity modes can be locally approximated as points of quadratic coupling, and described how quadratic coupling can provide access to novel physical effects such as quantum non-demolition measurements. The demonstration I presented of the readout of the intracavity photon number by way of the quadratic optical spring effect is a first step toward such a measurement.

Concluding this dissertation, I hope that I have fairly presented both the history and the state of the art of the modern field of optomechanics. There are certainly many exciting discoveries to come as groups across the world push mechanical oscillators further into the quantum regime, and as theory proposals for optomechanical systems turn into experiments, and then perhaps even into commercial products. The membrane-in-the-middle system is just one of many systems that will be studied in the years to come, and it is my hope that this work will be useful in future study.

VI. Works Cited

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